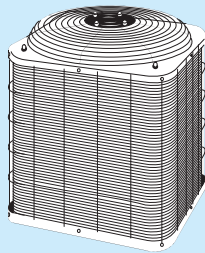


HVAC Basics and HVAC System Efficiency Improvement



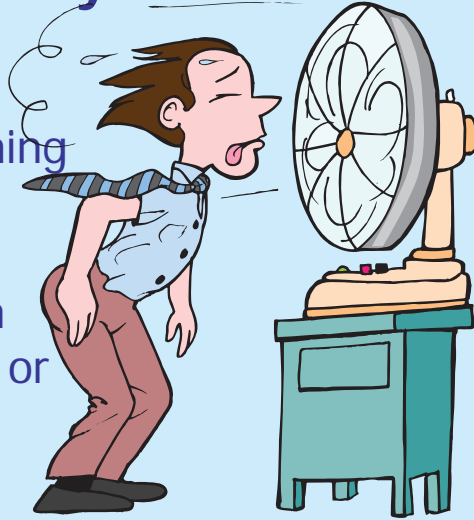
Introduction

- HVAC systems – or Heating, Ventilating and Air-Conditioning systems – control the environment for people and equipment in our facilities.
- They account for about 13% of the electrical energy use in a typical residential building; and about 20% in a typical commercial or institutional building. HVAC use in an office building might be as high as 30 – 50%.
- HVAC systems and chillers are also significant energy consumers in many manufacturing facilities.



Functions of HVAC Systems

- The purpose of a Heating, Ventilating and Air Conditioning (HVAC) system is to provide and maintain a comfortable and safe environment within a building for the occupants or for the process being conducted.
- Many HVAC systems were not designed with energy efficiency as one of the design factors



Session 8.3



Environmental Control Factors

- An HVAC system functions to provide an environment in which these three factors are maintained within desired ranges. Typical design conditions are:
 - 75 degrees F temperature (dry bulb)
 - 40 - 60% relative humidity
 - ASHRAE 62.1 – 2007/2010 Ventilation Standard
 - 17 CFM outside air per person, or
 - CO₂ less than 1000 PPM

Session 8.4



Temperature Control Strategies



- Vary the temperature of the supply air to the space while keeping the air flow rate constant. This is the *constant volume, variable temperature* approach.
- Vary the air flow rate while keeping the supply air temperature constant. This is the *variable volume, constant temperature* approach. VAV – variable air volume system.
- Vary the supply air temperature and the flow rate, as in a *variable volume reheat* system.



Session 8.5

Relative Humidity Control

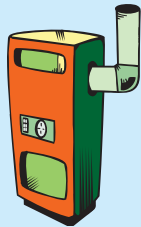
- Humidification - The air is too dry and water vapor must be added for comfort.
- Dehumidification - The air is too wet and water vapor must be removed for comfort.
- AC systems typically over-cool the air to remove water vapor, and then may have to heat the air back up - this is called reheat, and requires additional energy.
- Either way, energy required is around 1000 Btu per pound of water.



Session 8.6

Primary Equipment

- Chillers
- Direct expansion (DX) systems
- Boilers
- Furnaces



Session 8.7



Secondary Systems

- Single duct, single zone system
- Single duct, terminal reheat system
- Multizone system
- Dual duct system
- Single duct, variable air volume system
 - This is the most common system going in to large commercial buildings
- Fan coil system

Session 8.8

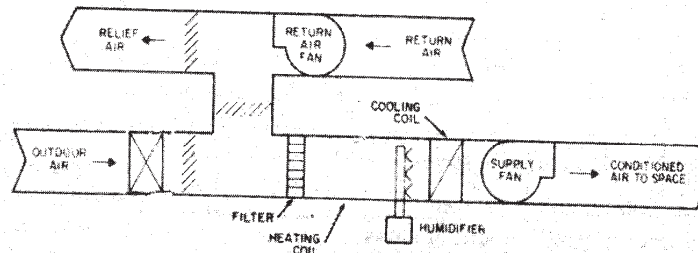


HVAC Systems

The summary below illustrates the types of systems frequently encountered in heating and air-conditioning systems.

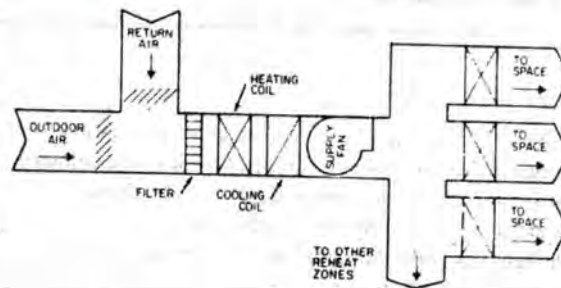
Single Zone System

Single zone systems consist of a mixing, conditioning and fan section. The conditioning section may have heating, cooling, humidifying or a combination of capabilities. Single zone systems can be factory assembled roof-top units or built up from individual components and may or may not have distributing duct work.



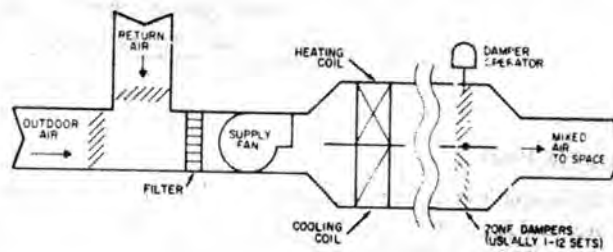
Terminal Reheat System

Reheat systems are modifications of single zone systems. Fixed cold temperature air is supplied by the central conditioning system and reheated in the terminal units when the space cooling load is less than maximum. The reheat is controlled by thermostats located in each conditioned space.

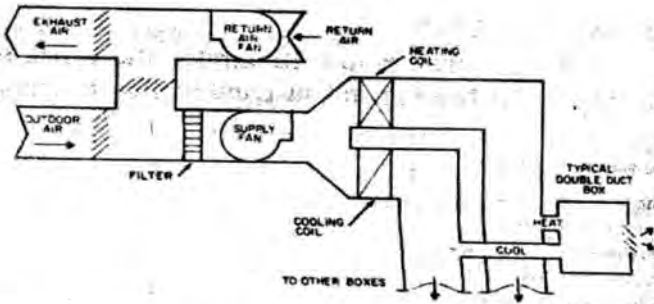


Multizone Systems

Multizone systems condition all air at the central system and mix heated and cooled air at the unit to satisfy various zone loads as sensed by zone thermostats. These systems may be packaged roof-top units or field-fabricated systems.

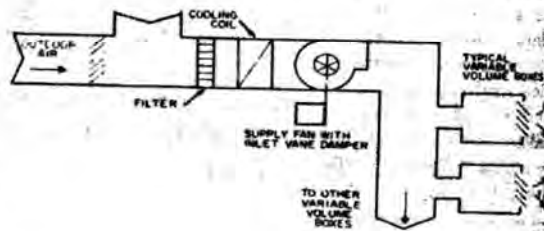


Dual duct systems are similar to multizone systems except heated and cooled air is ducted to the conditioned spaces and mixed as required in terminal mixing boxes.



Variable Air Volume Systems

A variable air volume system delivers a varying amount of air as required by the conditioned spaces. The volume control may be by fan inlet (vortex) damper, discharge damper or fan speed control. Terminal sections may be single duct variable volume units with or without reheat, controlled by space thermostats.



VAV HVAC Systems

- Most common new system going in to large commercial buildings.
- High efficiency because of use of variable speed drive on the supply air fan. For example, cutting air flow from full rate to 80% of full rate, cuts the fan power almost in half!
- Fan is controlled most often to keep a constant static pressure in the supply duct. Temperature is controlled by a local terminal box in the zone.



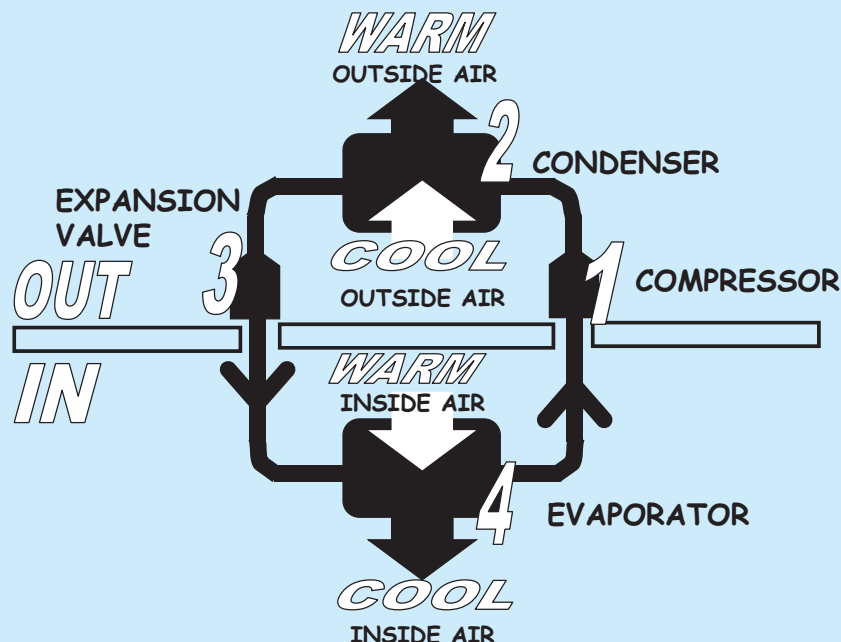
Power and Energy in Air Conditioning

- One ton of A/C = 12,000 Btu/hour
 - A ton is a measure of A/C power, and is used when sizing systems, or when determining electrical demand.
- One ton-hour of A/C = 12,000 Btu
 - A ton-hour is a measure of A/C energy, and is used when sizing storage tanks for thermal energy storage (TES) systems, or when determining electrical energy consumption.



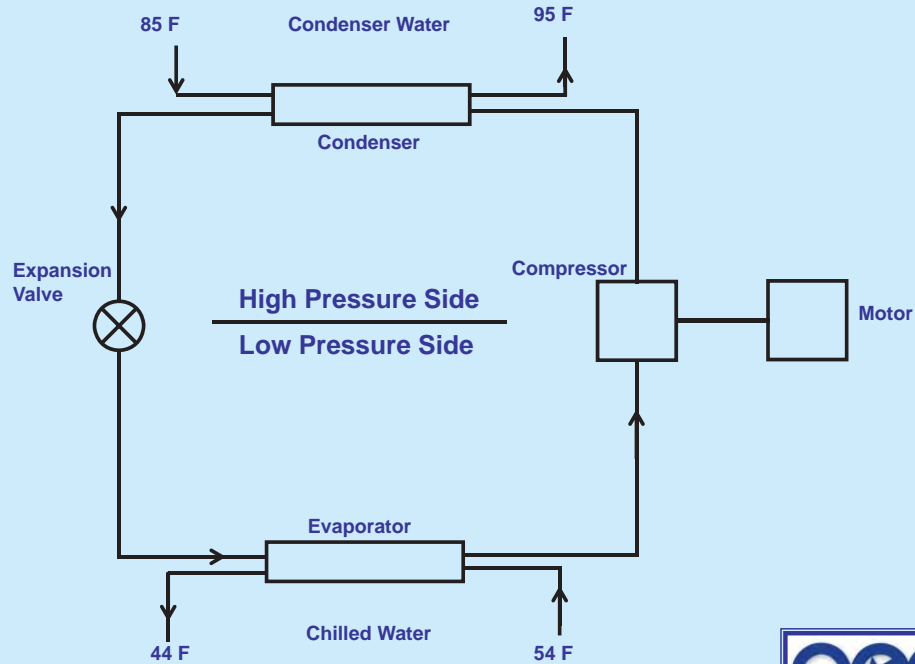
Session 8.13

Vapor Compression Cycle



Session 8.14

Chiller Diagram



Session 8.15



HVAC System Performance Measures

Energy Efficiency Ratio (EER)

$$\begin{aligned} \text{EER} &= \frac{\text{Btu of cooling output}}{\text{Wh of electric input}} \\ &= \frac{\text{Btu/h of cooling output}}{\text{W of electric power input}} \end{aligned}$$

Coefficient of Performance (COP)

$$\begin{aligned} \text{COP} &= \frac{\text{Energy or heat output (total)}}{\text{Energy or heat input (external only)}} \\ &= \text{EER} / 3.412 \text{ Btu/Wh} \end{aligned}$$

Session 8.16



Some Magic Numbers for EER and COP

$$\text{EER} = \frac{\text{Btu/h of cooling output}}{\text{W of electric power input}}$$

$$\text{COP} = \text{EER} / 3.412 \text{ Btu/Wh}$$

$$\frac{\text{X kW}}{\text{Ton}} = \frac{12}{\text{EER}}$$

Session 8.17



Sensible and Latent Heat

- Sensible heat - The heat associated with a temperature change of a substance at a constant moisture level.
- Latent heat - The heat associated with the phase change of a substance.
- Enthalpy - Total heat content of a substance, including both sensible heat plus latent heat.

Session 8.18



System Improvement Options

- Make building envelope improvements to reduce HVAC load
 - insulation, high performance windows and roofs
- Replace old HVAC units and chillers with more efficient models
- Possibly downsize units and chiller
 - Energy Star says chillers are oversized 60%



Session 8.19

- Consider multiple chillers
- Consider a chiller with a variable speed drive
- Consider installing a small chiller or separate HVAC system for 24/7 loads
- Use VSDs on pumps, cooling towers
- Replace constant volume systems with VAV - variable air volume systems



Session 8.20

- Consider adding a gas engine driven chiller with heat recovery for hot water
- Retrofit to DDC controls
- Use cooling towers where possible



Session 8.21



Absorption Chillers

- Absorption chillers can produce large quantities of chilled water using very little electric power and energy. Their prime energy source is heat from hot water or steam.
- Absorption chillers have no CFCs. Most absorption cycles use either ammonia and water or lithium bromide and water.
- Absorption chillers are not very efficient.
 - Single stage -- COPs about 0.6 - 0.8
 - Two-stage -- COPs about 1.0 - 1.2

Session 8.22



CEM Review Problems

1. In a vapor compression cycle air conditioner, the refrigerant is always in the vapor state.
A) True B) False
2. A roof top air conditioner has an EER of 9.2. What is its COP?
3. Reheat may still be needed in an HVAC system even if the outside temperature is very high.
A) True B) False



Session 8.23

4. A roof top air conditioner has an EER of 13.5. What is its kW/ton rating?



Session 8.24

Controls and BAS

- **Controls**

- Basic Types
- Inputs & Outputs
- Technologies
 - Pneumatic
 - Electric
 - DDC
- Terminology
- PID Controls
- Review



- **Building Automation Systems for Energy Management**

- Basic Functions
- Programs
- Review

Session 9.1



Basic Types

- **Manual**

- Switches
- Dimmers

- **Open Loop Automatic**

- Timer

- **Closed Loop Automatic**

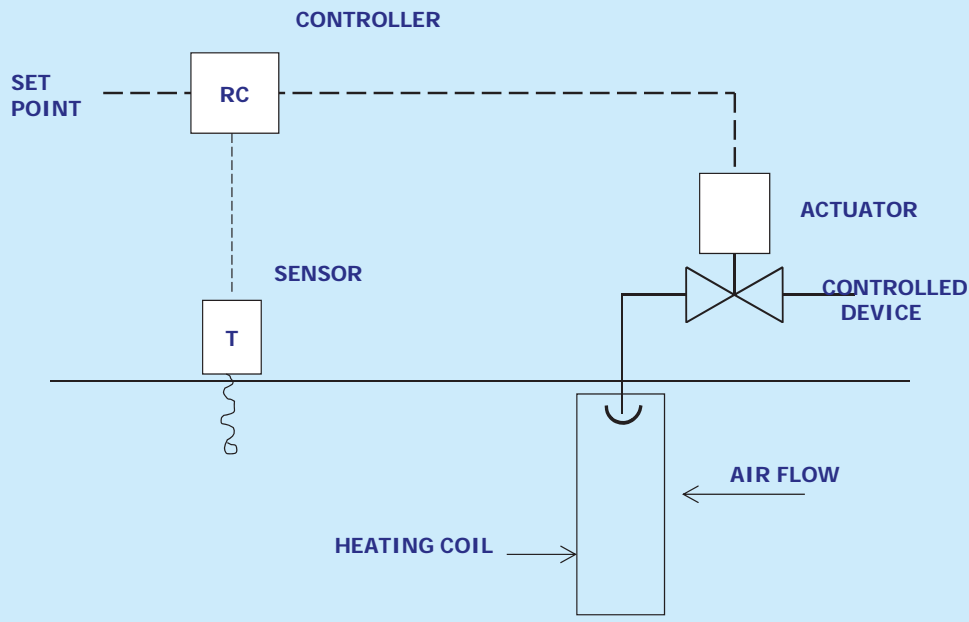
- Thermostat
- Humidistat
- Dimmable Ballast w/Photosensor



Session 9.2



Basic Feedback Control System



Session 9.3



Basic Types

- **Two Position (binary)**
 - The system is either OFF or ON (*gas furnace*).
 - Accomplished with a relay whose contacts are either open or closed, or a valve whose stem position is either open or closed.
- **Proportional**
 - A variation from the set point produces a proportional movement in the actuator.
 - Pneumatic controls vary the air pressure.
 - Electric controls use a potentiometer (a type of variable resistor).

Session 9.4



Basic Inputs & Outputs

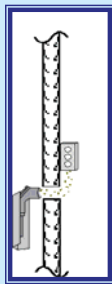
- **Digital (D) (~ binary)**
 - Signal with two states or positions which can be incremental (*on-off, day-night, open-closed, occupied-unoccupied, series of 1's & 0's*)
- **Analog (A)**
 - Signal can be monitored or controlled through a range of positions or values (*0 to 120°F, 3 to 5 PSI, 0 to 10 VDC, 4 to 20 milliamps*)

Session 9.5



Input Examples

Analog

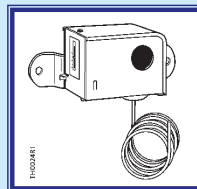


Outside Air Temp. Sensor
Analog Input

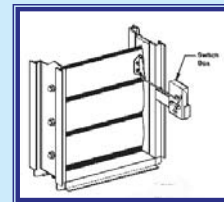


Room Temperature Sensor
Analog Input

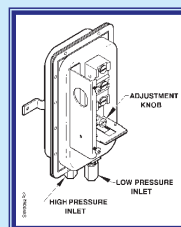
Digital



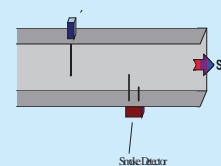
Low Temp. Detector
Digital Input



End Switch
Digital Input



High Pressure Detector
Digital Input



Duct Smoke Detector
Digital Input

Session 9.6



Inputs & Outputs

Input Points

Temperature **AI**

Flow **AI / DI**

Status or Proof **AI / DI**

Relative Humidity **AI**

Pressure **AI**

Air Quality **AI**

Output Points

Motors for Pumps / Fans **DO**

Lights **DO**

Variable Speed Drives **AO**
(or digital)

Valves **AO**

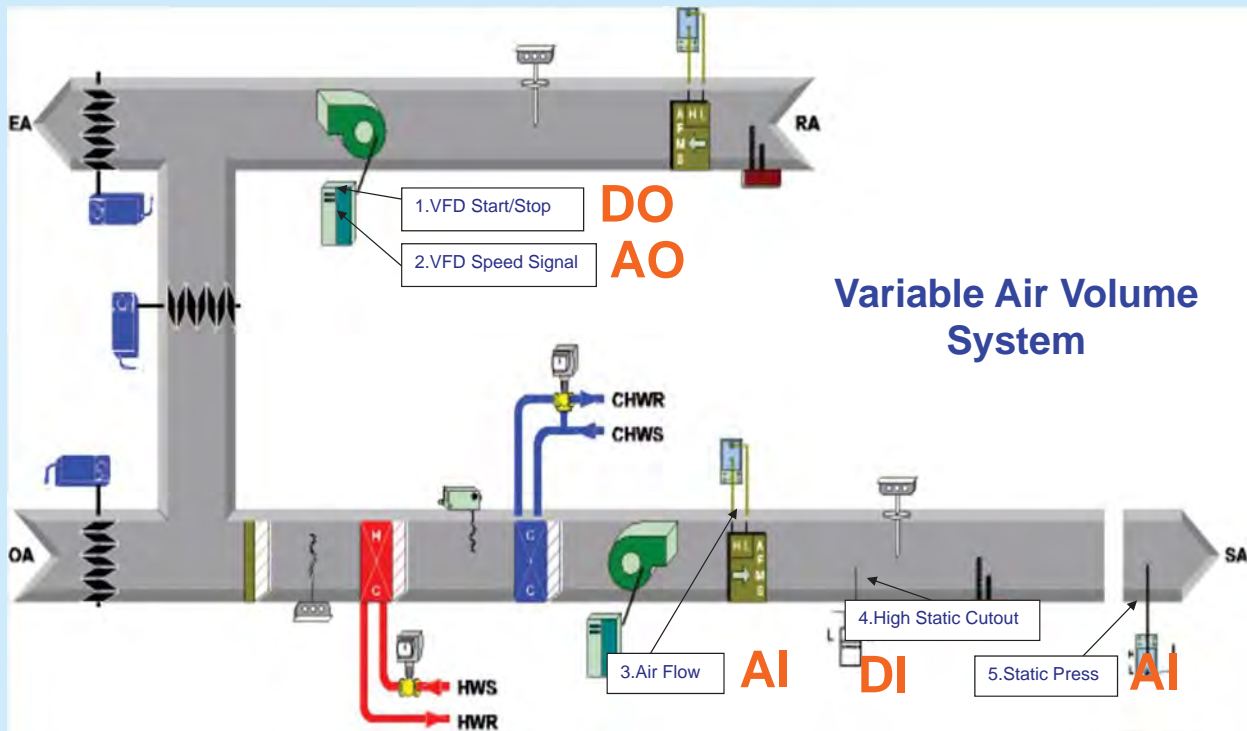
Dampers **AO**

Lighting Contactors **DO**

Session 9.7



Inputs & Outputs



Session 9.8



Basic Technologies

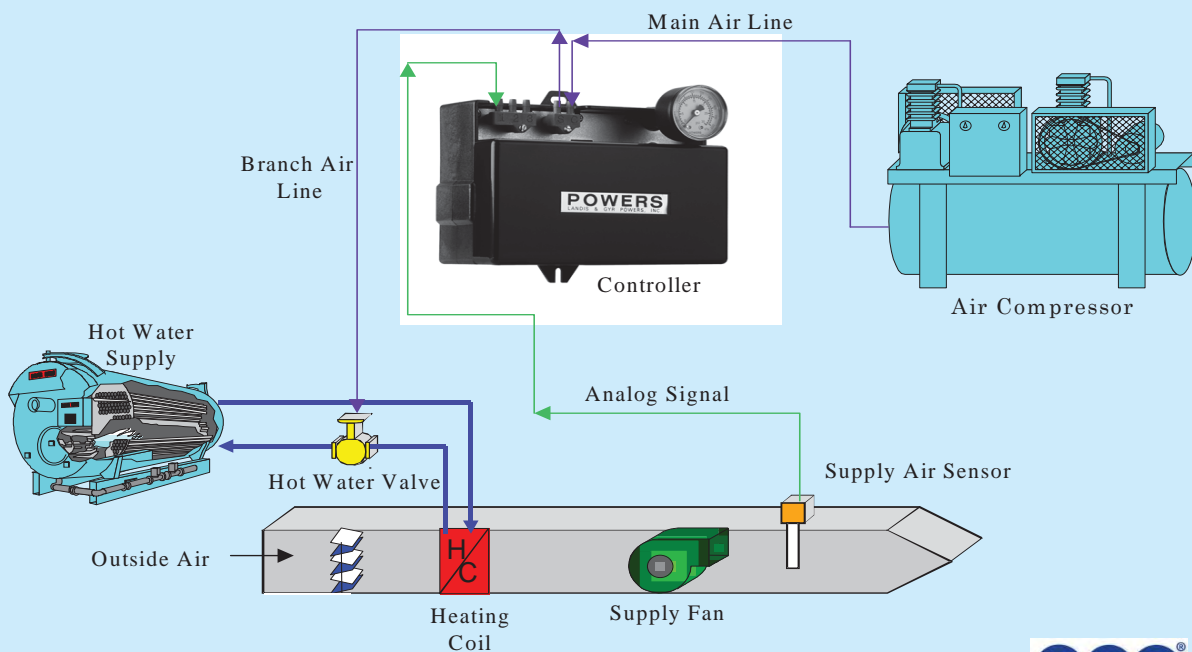
- Pneumatic
- Electric
- Direct Digital Control (DDC)

Session 9.9



Pneumatic Controls

- Use clean, dry, oil-free **compressed air** to operate the control system.
- Have been used in many HVAC applications.



Session 9.10



Pneumatic Controls

Advantages

- Are well understood by many designers and most maintenance people.
- Are inherently proportional and very reliable.
- Were relatively inexpensive in the past.

Disadvantages

- Not very precise.
- Typically required frequent calibration.
- Pneumatic control algorithms are hard to change.
(*typically pre-set by manufacturer*)

Session 9.11



Electric Controls

- Can be analog electric or electronic controls.
- Use a variable, but continuous, **electrical voltage or current** to operate the control system.
- Transmit signals quickly and accurately.



Session 9.12



Electric Controls

Advantages

- Can be very accurate and very stable.
- Do not require field calibration, and are drift-free, if good quality sensors are used.
- Relatively easy to implement proportional plus integral (PI) control electronically.

Disadvantages

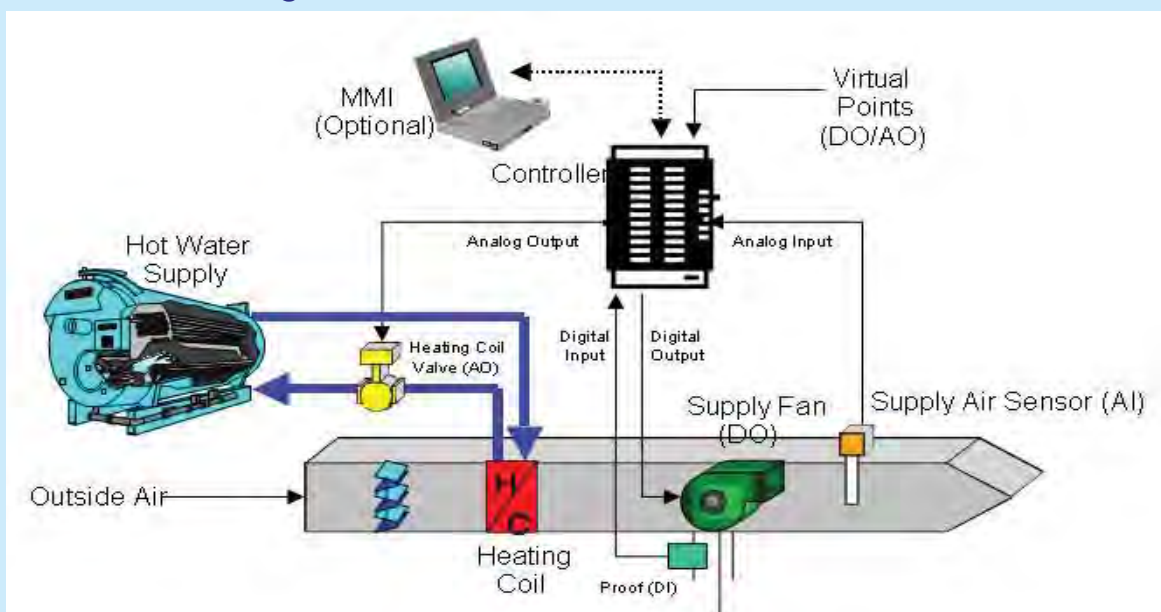
- Difficult to interchange parts easily because of the many different systems.
- Do not interface directly with our digital computers

Session 9.13



Digital Controls

- Use **electrical pulses** to send signals.
- Interface directly with microprocessors, directly &/or via the internet using TCP/IP.



Session 9.14



Digital Controls

Advantages

- Algorithms can be adjusted relatively easily after installation
- Precise
- No controller drift, recalibration is normally not necessary
- Cost effective (*similar to electronics market*)

Disadvantages

- Possibly not well understood by O&M staff
- Different communication protocols, interface standards, and internal logic are typically complex (*BACNET-ASHRAE 135-2004; and Lonworks – Lonmark Corporation; are both addressing these interface problems*)

Session 9.15



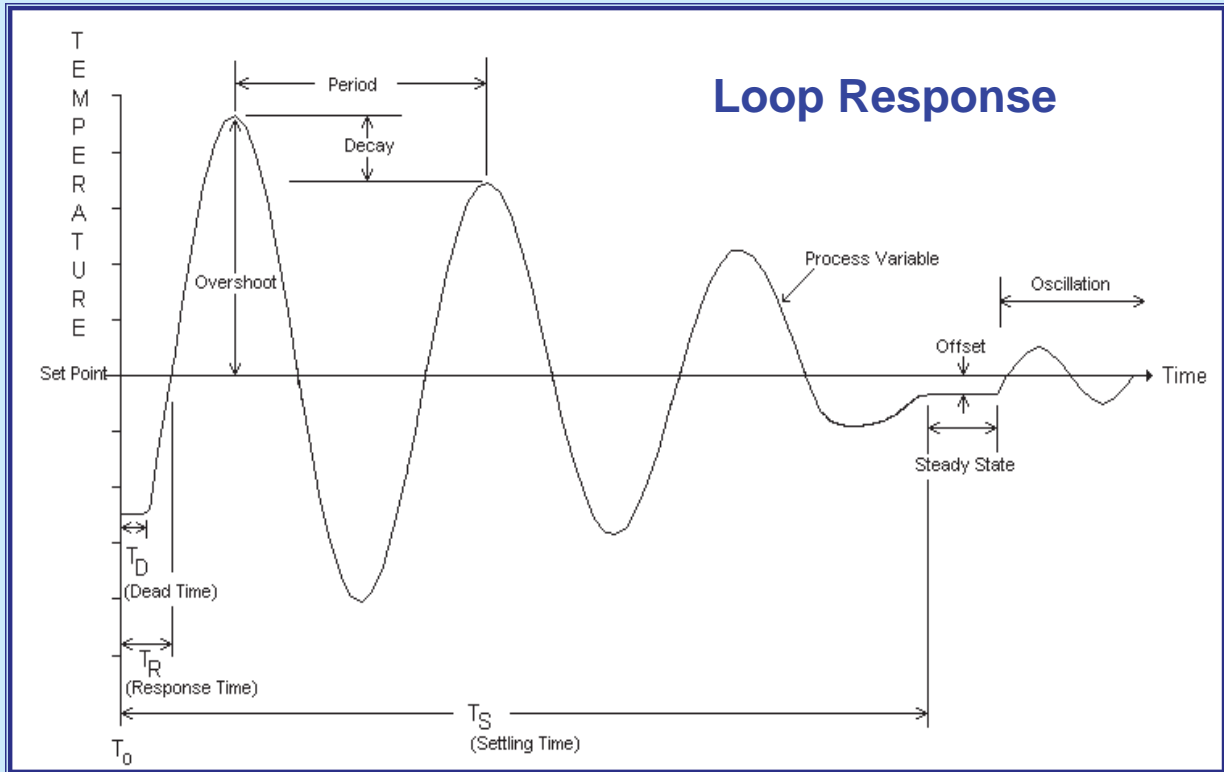
Control Algorithms

- PID Controls
 - P is proportional
 - I is integral
 - D is derivative
- Controls are usually P, PI or PID
- PID is considered the best of this group
- Newer control algorithms are:
 - Fuzzy logic
 - Learning systems
 - Self-optimizing systems

Session 9.16



PID Controls



Session 9.17



PID Controls

Summary Of Gains

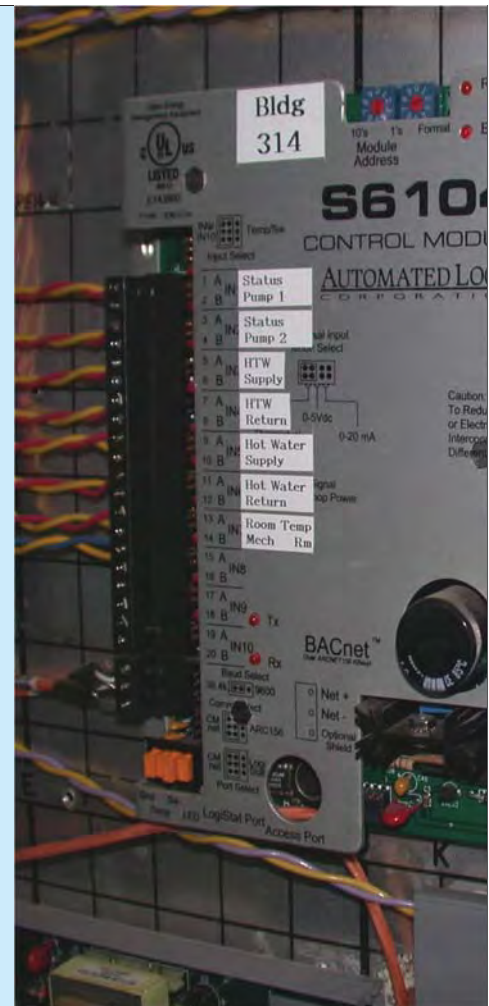
CONTROL MODE	PURPOSE	ERROR DETECTIONS
PG	Reacts to Change	Detects SIZE of Error
IG	Reduces/Eliminates Offset	Averages Error over TIME
DG	Senses High Rate Load	Detects RATE of change of error

$$CV \text{ (adjustable) } = \left\{ \left(\frac{PG}{1000} \times error * action \right) + \left(\frac{IG}{1000} \int error \, dt * action \right) + \left(\frac{DG}{1000} \times \frac{d(error)}{dt} * action \right) \right\} + bias$$

Session 9.18



Building Automation Systems (BAS)

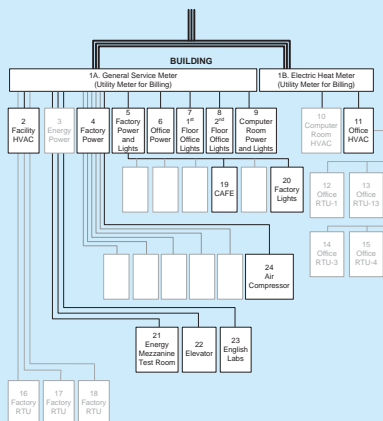


Session 9.19

Basic Functions

1. Monitoring/Surveillance

- Building Conditions
- Equipment Status
- Utility Submetering
- Climatic Data
- Fire & Security



Session 9.20

Basic Functions

2. Demand Limiting

- Load shedding
- Duty cycling

3. Maintenance

- Remote operation and control of equipment
- Generation of maintenance schedules
- Diagnosing breakdowns

4. Record Generation

- Trends and operation logs
- Utility demand profile (“baseline”)
- Modification/replacement analysis
- Energy conservation documentation

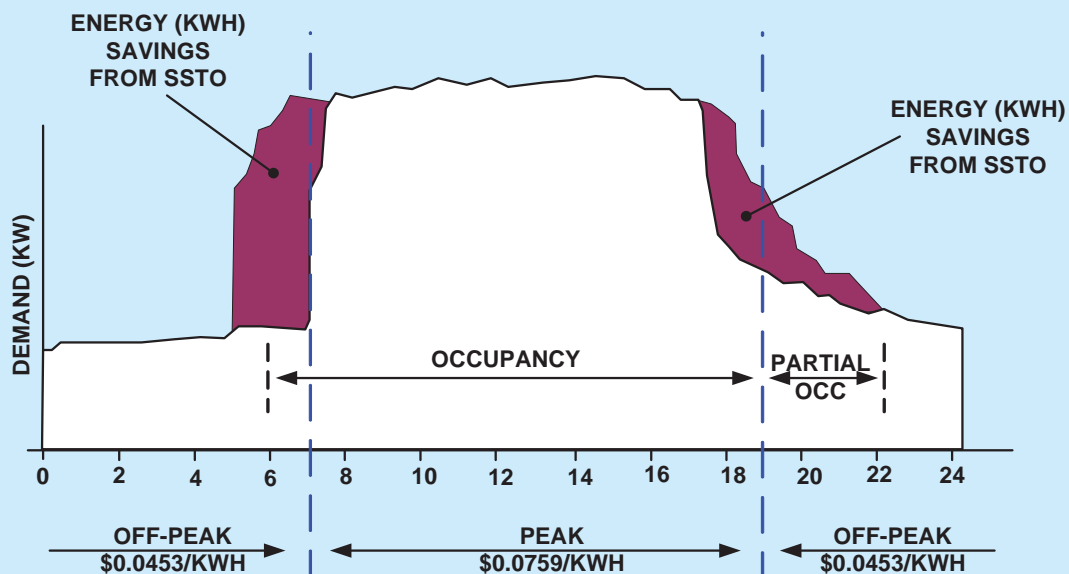
Session 9.21



Start-Stop Optimization

• How It Works

- Start the equipment at the latest possible time
- Stop at the earliest possible time



Session 9.22



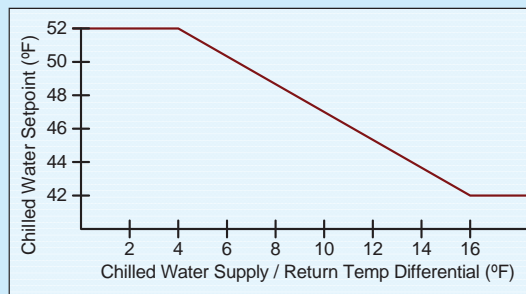
Chilled Water Reset

- Introduction

- Many buildings' chilled water setting is designed for the heaviest anticipated cooling load.
- Significant cost savings can result from resetting chilled water temperatures in anticipation of cooling load

- How It Works

- When the load, chilled water ΔT , or return chilled water temperature increases, the chilled water setpoint is lowered, and vice versa



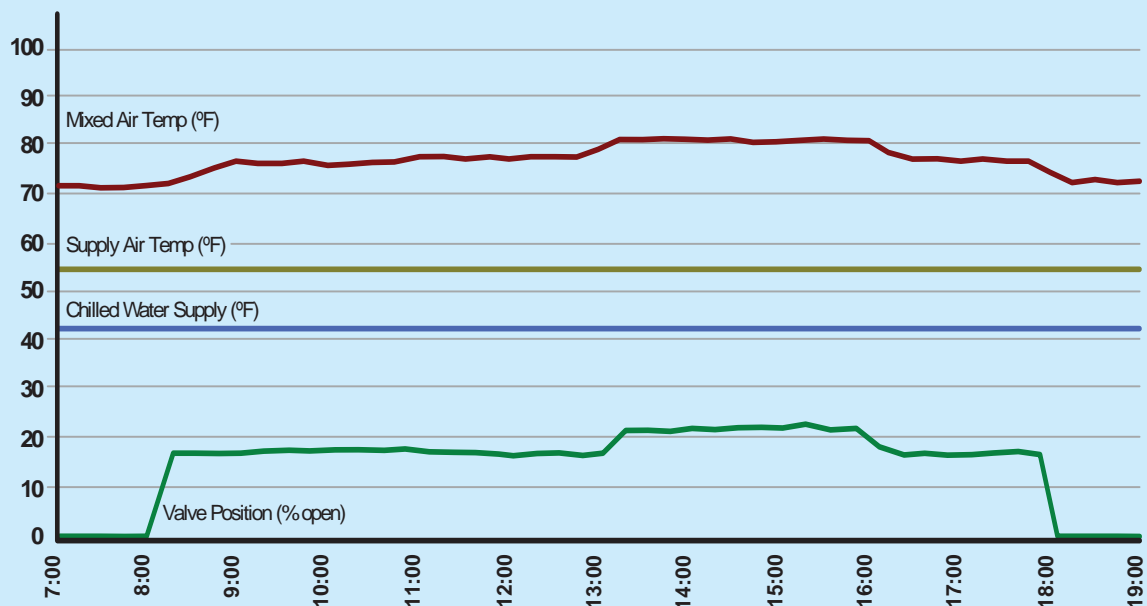
Session 9.23



Chilled Water Reset

- Proof of Performance

- Constant CHW Supply Temp Scenario:



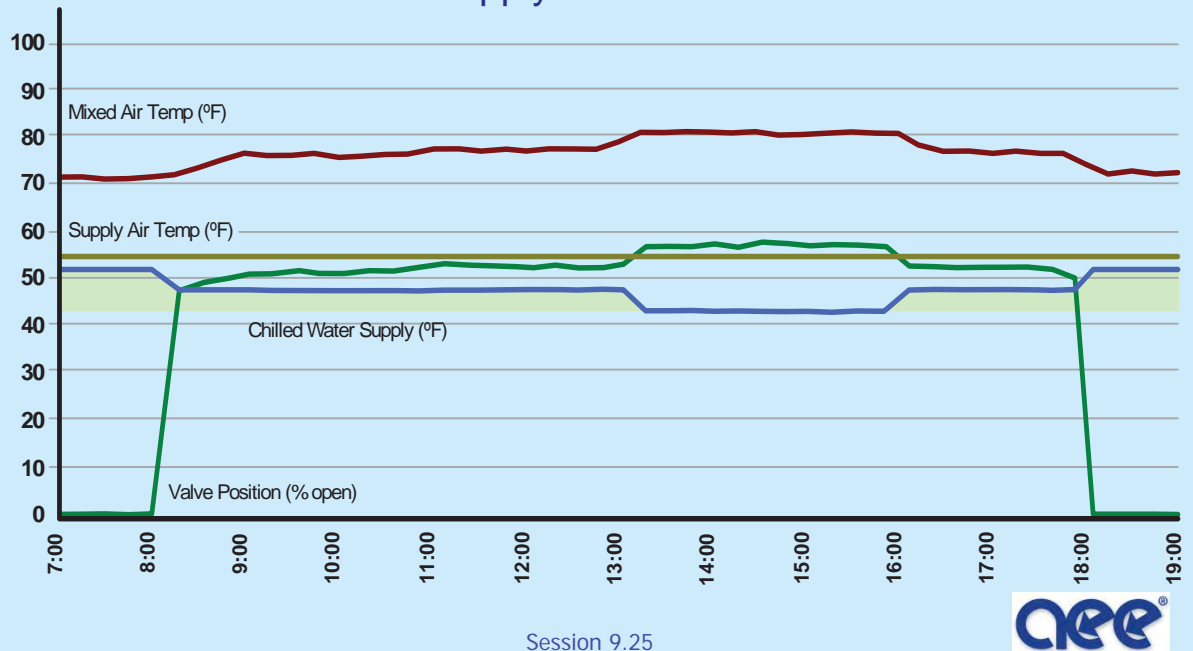
Session 9.24



Chilled Water Reset

- **Proof of Performance**

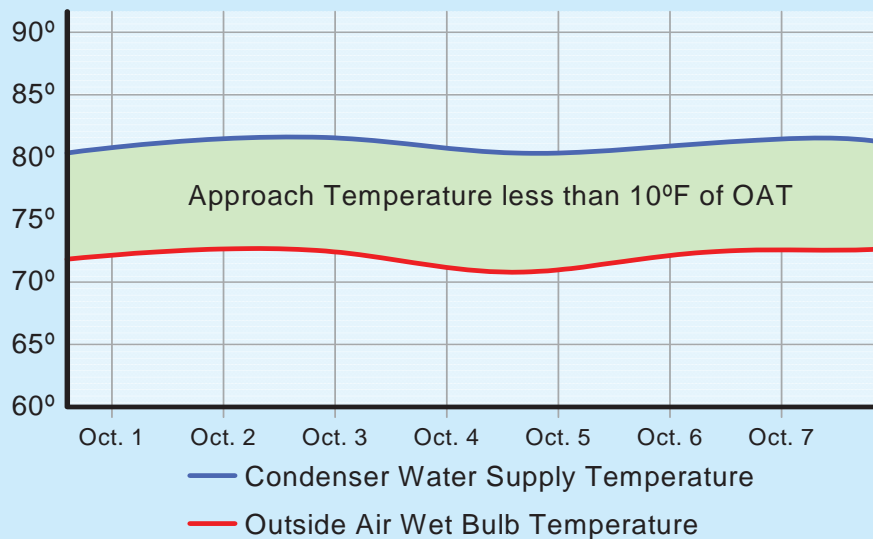
- As the outside air temperature decreases, the temperature of the chilled water supply can be allowed to increase



Condenser Water Reset

- **Proof of Performance**

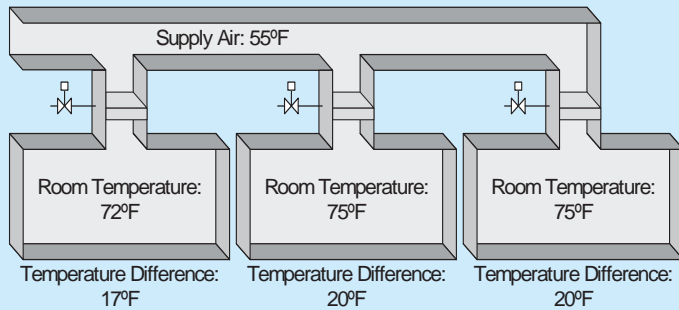
- This trend shows the cooling tower approach temperature
- Trend points at a fixed time interval to ensure operation
(Approach Temp, Cond Wtr Return Temp & Cond Wtr Supply Temp)



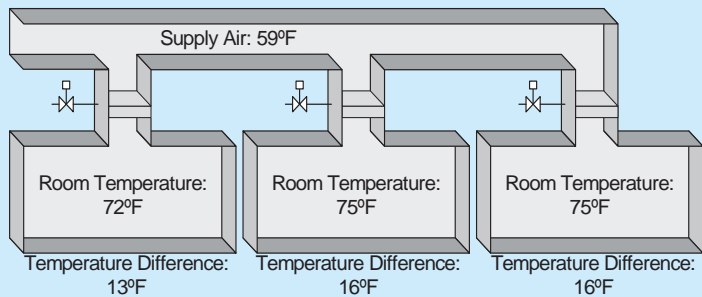
Supply Air Reset

- Introduction

- When supply air temperature is set to 55°F, rooms must mechanical reheat



- When supply air temperature is reset to 58°F, rooms require less mechanical reheat



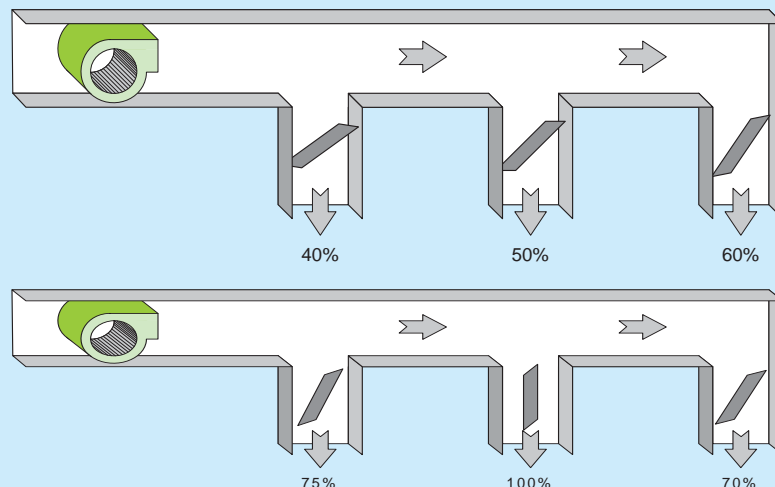
Session 9.27



Static Pressure Resets

- How It Works

- The DDC system monitors the VAV static pressure and lowers the pressure until only one damper is completely open
- The Static Pressure and VFD control sensors must be located on the same DDC control panel



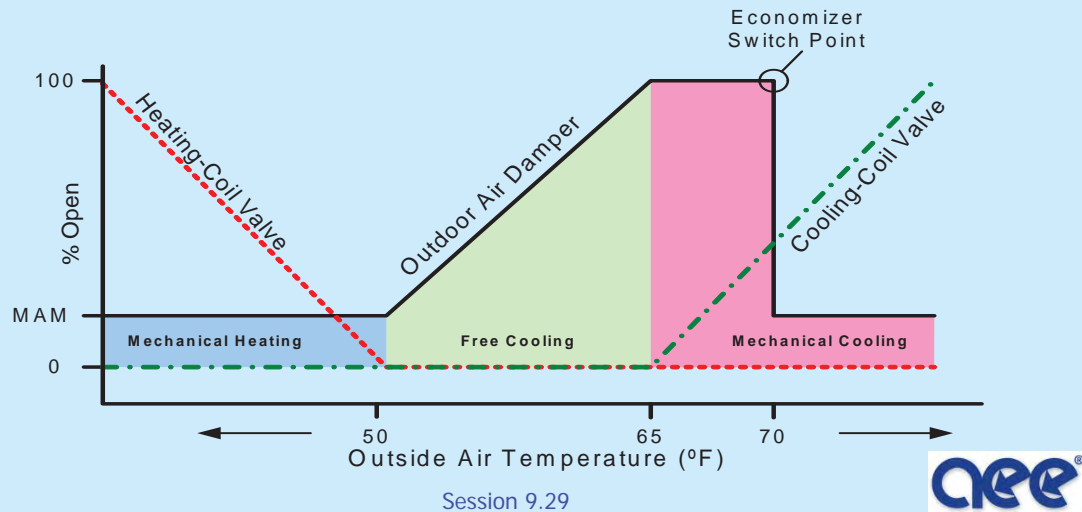
Session 9.28



Economizer

• How It Works

- Reduce cooling & heating energy by optimizing mixed air temp
- As the outside air allows, the outdoor air damper opens more
- Free heating/cooling occurs when occupant comfort is maintained without using mechanical heating or cooling

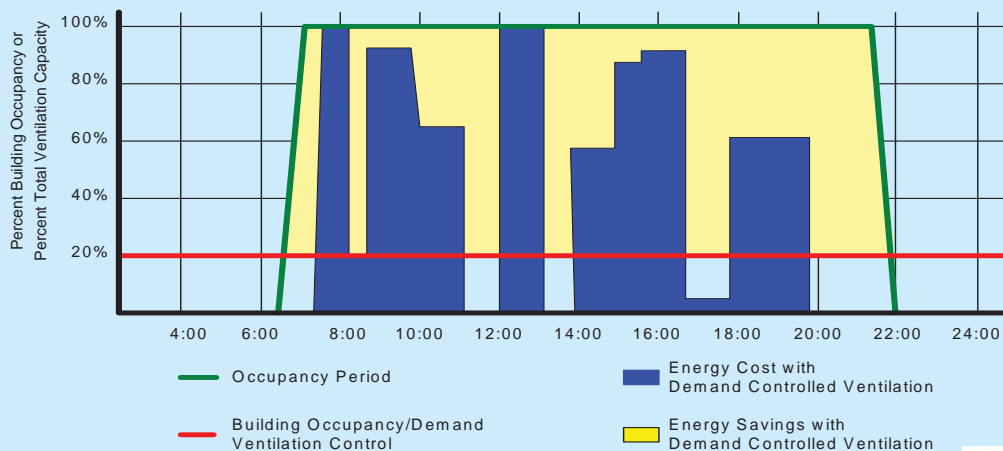


Session 9.29

Demand Control Ventilation

• How It Works

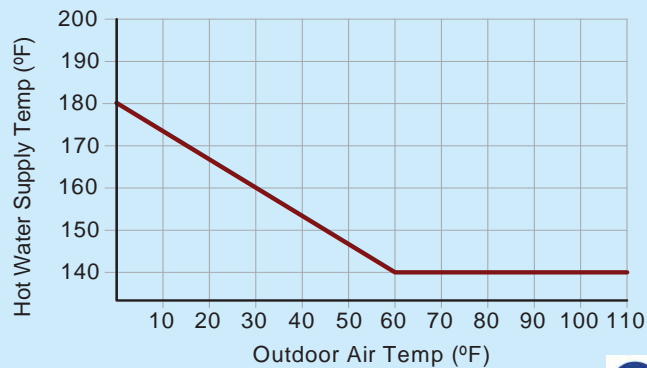
- DCV provides just the right amount of outside needed by occupants
- Modulates ventilation to main target cfm/person ventilation based on actual occupancy
 - Less than 700 ppm above outside CO₂ concentration



Session 9.30

Hot Water Reset - How It Works

- Hot water boilers are very efficient at partial load
 - Distribution losses are less when temperature is reduced
 - Hot water reset conserves energy by reducing the boiler's operating temperature
 - Hot water reset reduces thermal shock because it does not involve drastic temperature fluctuations
 - To minimize flue gas corrosion, do not reset lower than 140°F



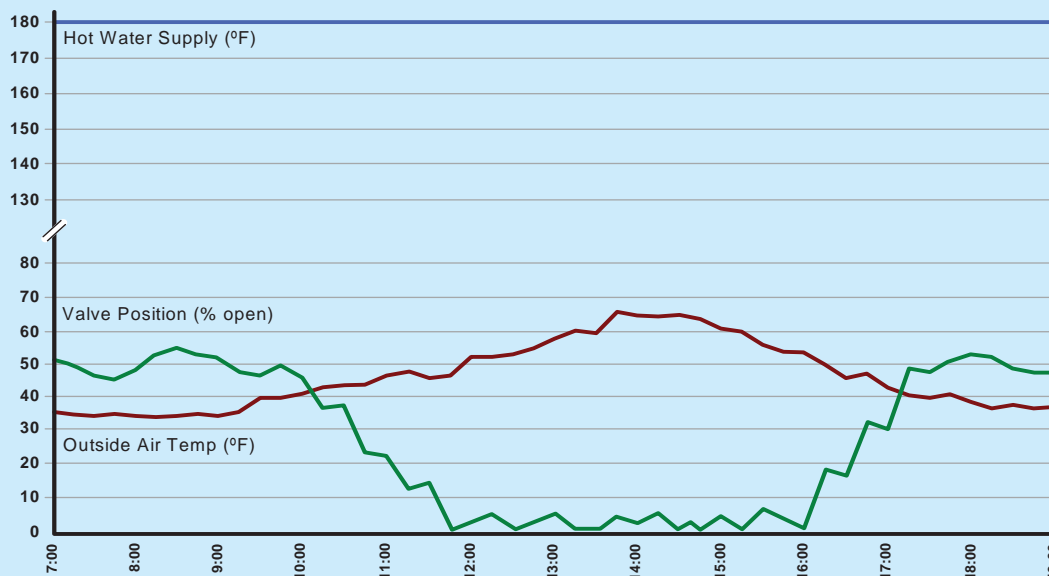
. Session 9.31



Hot Water Reset

• Proof of Performance

- Maintaining 180°F water temperature forces the heating valves to cycle open and closed



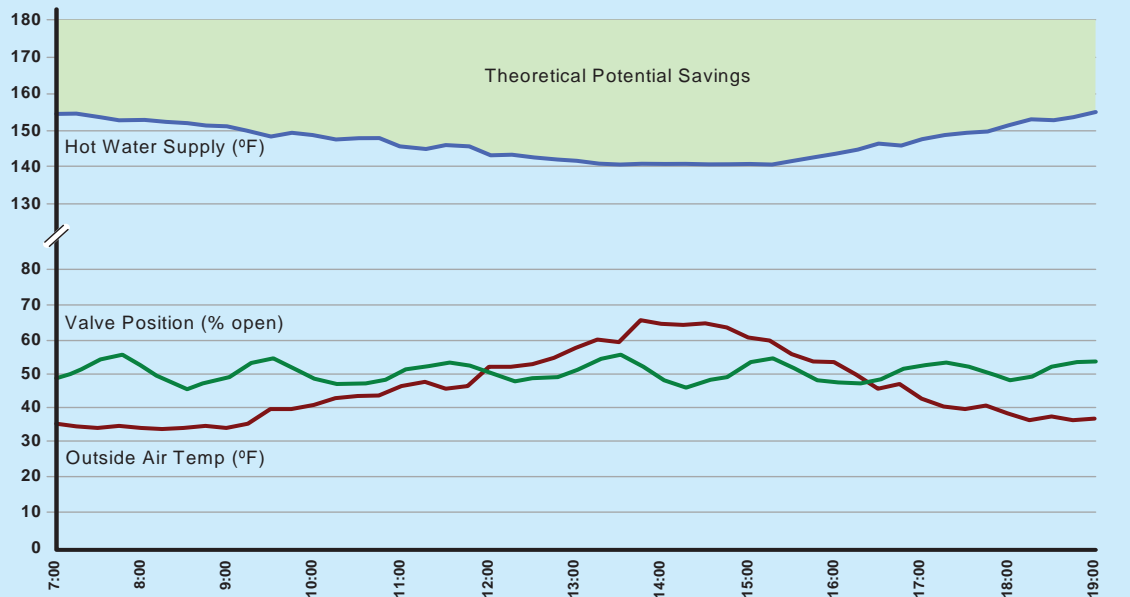
Session 9.32



Hot Water Reset

- **Proof of Performance**

- Resetting the hot water temperature allows heating valves to operate in more efficient mid-actuation positions



Session 9.33



Other EMS Programs

1. **Scheduled Start/Stop** – Starting and stopping equipment based upon the time of day, and the day of the week.
2. **Duty Cycling** – Shutting down equipment for predetermined short periods of time during normal operating hours.
3. **Demand Limiting** – Temporarily shedding electrical loads to prevent exceeding a peak value.
4. **Unoccupied Setback** – Lowering the space heating setpoint or raising the space cooling setpoint during unoccupied hours.

Session 9.34



- 5. Warm Up/Cool Down Ventilation & Recirculation** – Controls operation of the OA dampers when the introduction of OA would impose an additional thermal load during warm-up or cool-down cycles prior to occupancy of a building.
- 6. Hot Deck/Cold Deck Temperature Reset** - Selects the zone/area with the greater heating and cooling requirements, and establishes the minimum hot and cold deck temperature differential which will meet the requirements.
- 7. Steam Boiler Optimization** – Implemented in heating plants with multiple boilers. Boiler plant optimization is accomplished through the selection of the most efficient boiler to satisfy the space temperature requirements during the building occupied period.
- 8. Reheat Coil/Reset** – Selects the zone/area with the greatest need for reheat, and establishes the minimum temperature of the heating hot water so that it is just hot enough to meet the reheat needs for that time period.

Session 9.35



- 9. Chiller &/or Boiler Optimization** – For facilities with multiple chillers &/or boilers, the most efficient chiller(s) &/or boiler(s) are selected to meet the existing load with minimum demand and or energy.
- 10. Chiller Demand Limiting** – The chiller electrical load is reduced at certain times to meet a maximum pre-specified chiller kW load.
- 11. Lighting Control** – Turns lighting off and on according to a pre-set time schedule.
- 12. Remote Boiler Monitoring and Supervision** – Uses sensors at the boiler to provide inputs to the EMCS for automatic central reporting of alarms, critical operating parameters, and remote shutdown of boilers.
- 13. Maintenance Management** – Provides a maintenance schedule for utility plants, mechanical and electrical equipment based on run time, calendar time, or physical parameters.

Session 9.36



Review: BAS/EMS

1. Distinguish between analog and digital control.
2. Devices using 4-20 mA current loops are using digital data transmission.
(A) *True* (B) *False*

Session 9.37



3. A facility is heated and cooled by multizone units using hot water pumped from a central boiler system. List some EMS controls that could reduce the facility energy costs.
4. List some maintenance aids that could be provided by an EMS.

Session 9.38



5. An optimum start device is a control function that:
- (A) *Shuts off the outside ventilation air during start up of the building.*
 - (B) *Shuts off equipment for duty cycling purpose.*
 - (C) *Senses outdoor and indoor temperatures to determine the minimum time needed to heat up or cool down a building.*
 - (D) *Compares the enthalpy of outdoor and return air and determines the optimum mix of the two streams.*

Maintenance and Commissioning

Session 10.1



Importance of Maintenance Programs

Good maintenance saves energy costs

Major problems in maintenance

- Lack of adequate management attention
- Lack of adequate maintenance training
- Poor reporting of problems
- Lack of work order systems
- Poor analysis of problems
- Poor preventive maintenance efforts
- Poor control of maintenance activities



Session 10.2



Successful Maintenance Programs

- Require adequate funding and training
- Do not defer maintenance
- Use new Maintenance Management Systems

Session 10.3



Maintenance Management Systems

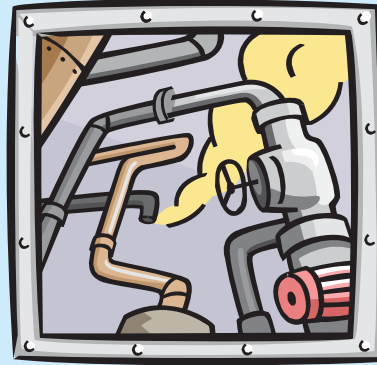
- MMS bring high tech to the maintenance area.
- MMS provide capability for inputs from sensors such as differential pressure across filters, equipment temperatures, and vibration.
- MMS provide data analysis, data summaries, generation of maintenance orders, and performance records on equipment.

Session 10.4



Costs of Poor Maintenance

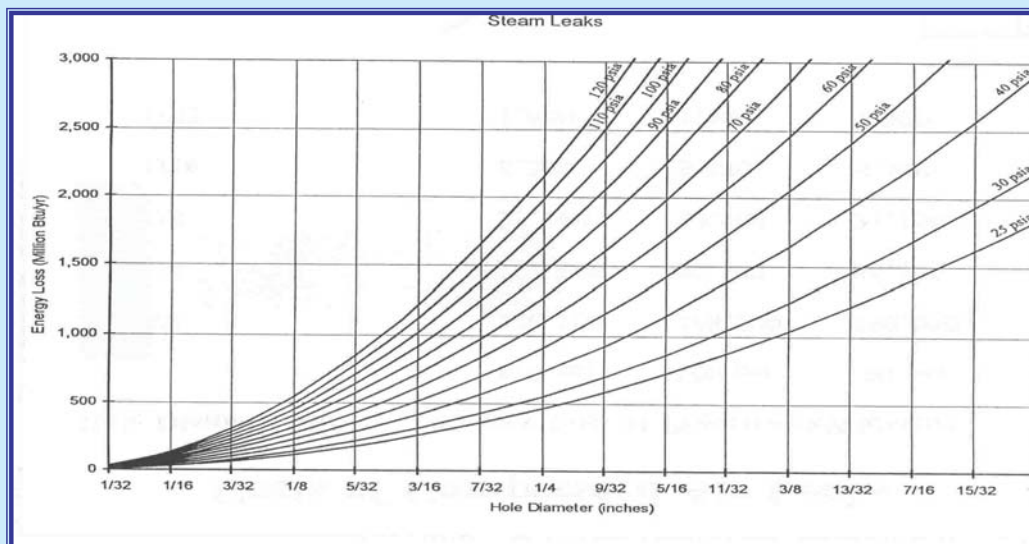
- Steam leaks
- Steam trap failures
- Compressed air leaks
- Uninsulated pipes
- Boiler scale



Session 10.5



Energy Loss of Steam Leak Chart in PSIA, and for 8760 h/yr operation



Session 10.6



Air Leaks

Table in PSIG, and for 8760 h/yr operation

Costs of Compressed Air Leaks

Hole Diameter (in)	Energy Loss at Pressure (kWh/year)		
	110 psi	100 psi	90 psi
3/8	226,100	208,100	190,000
1/4	100,500	92,500	86,300
1/8	25,100	23,100	21,100
1/16	6,300	5,800	5,300
1/32	1,600	1,400	1,300

Session 10.7



Compressed Air Leaks Recall from Section M:

- Quantifying air leaks
 - Apply the following formula to find standard cubic feet per minute (SCFM) lost

$$R = \frac{V (P_1 - P_2)}{(\Delta T) 14.7 \text{ lb/in}^2}$$

A standard cubic foot of air is one cubic foot of air at 14.7 lb/in².

R = average leakage rate (Scfm)

V = system volume (ft³)

P₁ = initial pressure (psig)

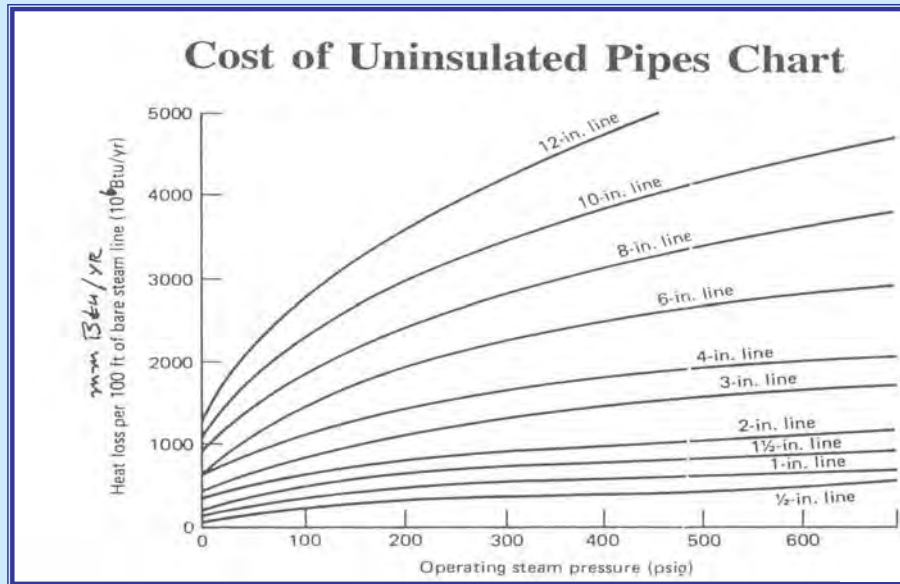
P₂ = final pressure (psig)

ΔT = time interval over which leaks are measured (minutes)

If V=200 ft³, P₁ - P₂ = 10 psig and ΔT = 12 min, compute R.



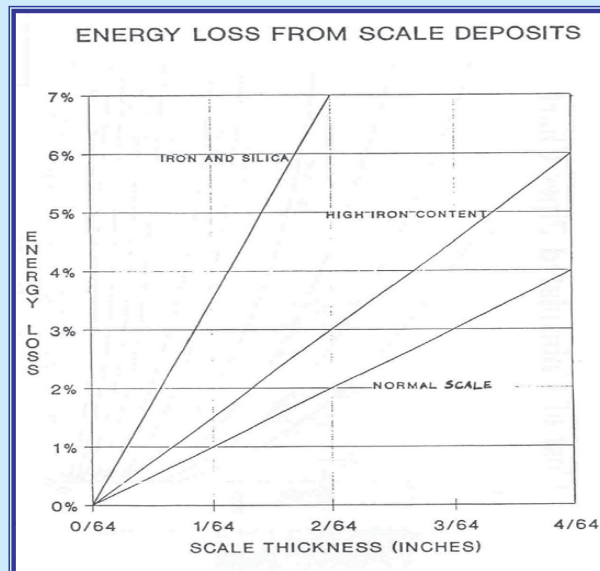
Cost of Uninsulated Steam Pipes in PSIG, and for 8760 h/yr operation, per 100 feet



Session 10.9



Cost of Boiler Scale Chart percent loss from factory clean



Session 10.10



Maintenance Help from Interesting Technologies

Infrared Photography And Vibration Analysis

Session 10.11



Infrared Photography

- Family of non-destructive testing techniques
- Produces images of heat usually invisible to the human eye
- Heat in some mechanical situations can be a sign of impending malfunctions
- Heat in some electrical wiring can be a sign of bad connections and/or phase problems
- Heat anomalies in steam systems (e.g. traps)
- Heat loss through building envelope



Session 10.12



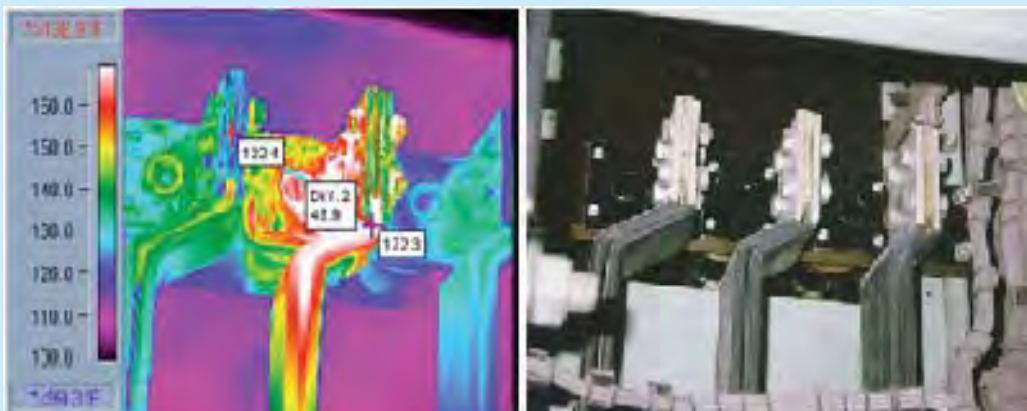
Infrared Photography Cont.

- Averted problems can be significant in nature (e.G. Primary transformer fires, etc.)
- IR operator needs some level of expertise
 - Attend training
 - Contract service
- One major malfunction discovered and infrared program just paid for itself

Session 10.13



Example IR Photos

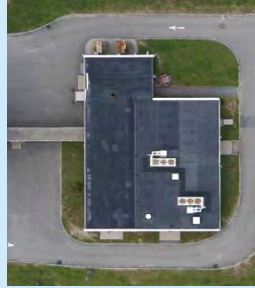


- Example IR photo of bad electrical connection – FLIR Systems Inc.

Session 10.14



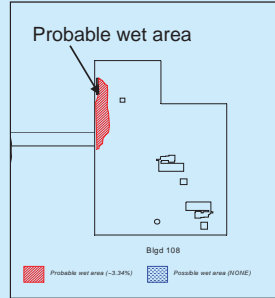
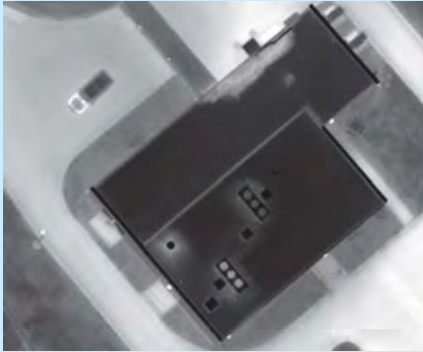
Aerial example of thermal image, visible image and CAD drawing



PHOTOGRAPH



THERMOGRAPH



CAD DRAWING

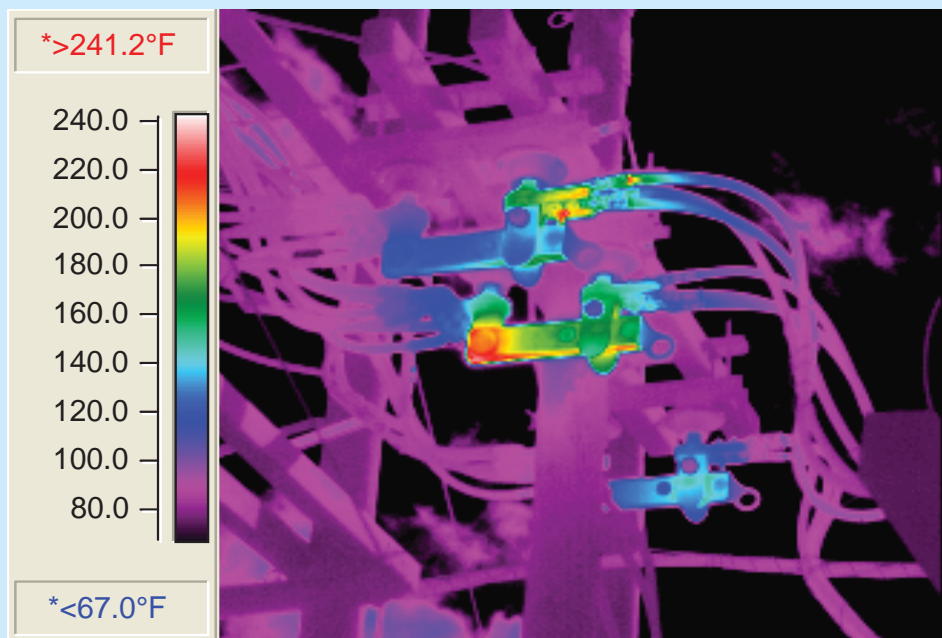


CAD OVERLAY

Session 10.15



Electrical P/PM



Session 10.16



Vibration Analysis

- Another powerful member of the family of non-destructive testing techniques
- Measures vibration signatures of machinery (usually rotating)
- Two Main Diagnostic Methods
 1. Vibration Signatures (Snap-Shot)
 2. Trending (Observation Over Time)

Session 10.17

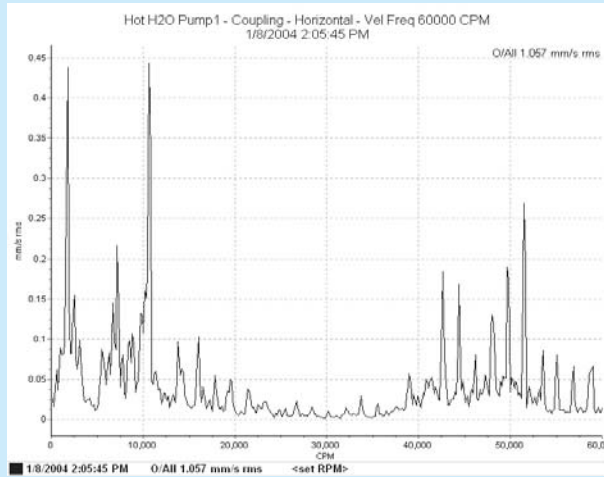
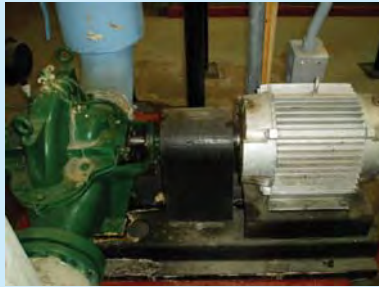


Vibration Analysis Cont.

- Even more so than infrared photography, training is mandatory (this is a science *and* an art)
- Trending vibration levels over time can remove some of the uncertainty of problems
- Technique used to spot:
 - Bad Bearings
 - Bad Gears
 - Loose Machine Mountings

Session 10.18



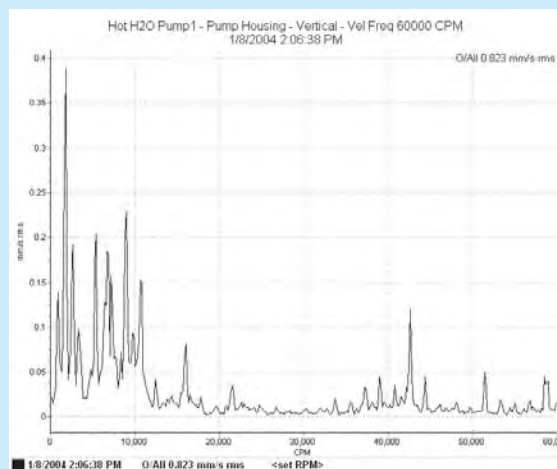


Problem was later localized to the motor coupling. Repair was initiated to avoid catastrophic failure.



Session 10.19

Vibration Analysis Examples



Water pump with suspicious (noise-floor) vibrations. Attempts to localize problem began. (Okla. State IAC)



Session 10.20

Maintenance Review Problems

1. Five bucket steam traps are stuck open in your facility. They all exhaust to the drain. If gas costs \$7.00/MCF and your steam system is 78% efficient, what is the cost of these malfunctioning steam traps per year? Each trap has a 1/8 inch orifice. The steam line pressure is 110 psia.
2. In problem 1, what else do you really need to explore?

Session 10.21



Maintenance Review Problems Cont.

3. You run a compressed air check in your facility and find the following leaks -- one 1/4 inch and one 1/8 inch. What is the annual cost to your facility if your system has 100 psig air and you are paying \$.05 per kWh for electricity?

Session 10.22



Commissioning



Session 10.23



CBECS DATA

	Energy Use (kBtu/sq.ft.yr)	Energy Cost (\$/sq.ft.yr.)
Energy Star (2000)	48.9	1.12
Average Buildings (1995 CBECCS*)	107.1	2.07
Efficient Buildings (Top 25% 1995 CBECS*)	51.4	1.25

* Commercial Buildings Energy Consumption Survey (US DOE)

Source: Energy Performance Pitfalls., Building Operations Management, p.43, March 2000

Session 10.24



CBECS DATA

	Energy Star 2000	1995 CBECS (Top 25%)	1995 CBECS (Bottom 25%)
Economizers	70%	30%	75%
VSDs	55%	20%	45%
VAV	70%	35%	65%
EVS	80%	25%	55%
Motion Sensors*	60%	10%	20%
* More Data Given - Not Presented Here			

Source: Energy Performance Pitfalls., Building Operations Management, p43, March 2000

Session 10.25



Lack of Commissioning Examples

- Air side economizers: 50% of the time do not function properly (ASHRAE says 70%)
- Chilled water pumps directly connected to CW return lines
- Parallel pumps with some connected backward
- Air handling units (AHU) with belts completely absent (motor still running)
- One study found 650 such discrepancies

Session 10.26



Do It Right The First Time And Keep It There

Commissioning

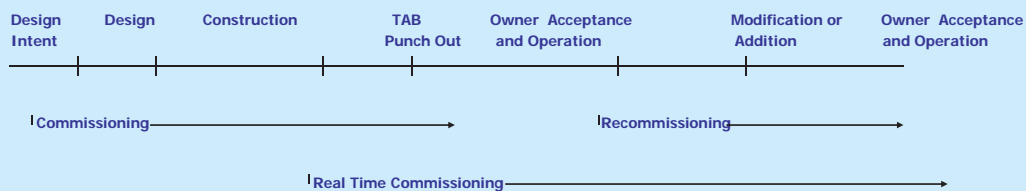
The process of ensuring that systems are designed, installed, functionally tested, and capable of being operated and maintained to perform in conformity with the design intent begins with planning and includes design, construction, start-up, acceptance and training, and can be applied throughout the life of the building (ASHRAE guideline 1-1996)

Session 10.27



Time Line

The chart depicted below is the author's concept of how these fit together. There is nothing accurate about the actual placement of these activities on the time line; the chart is intended to convey thought only.



Note: Modification also includes design intent, design, construction, etc. This has been simplified.

Session 10.28



Commissioning: General

1. Design intent is important (know what you want and express it).
2. Commissioning experience and training is important. Get someone with experience to help the first time(s) and obtain proper training. This process can be extremely cost effective; do it right.

Session 10.29



Commissioning: General

3. Facility staff should be involved in the process as soon as possible. This is especially critical in the formal commissioning activities (systems try out and tuning).
4. Commissioning is not a “quick fix” or a “one time activity”. It is a rather dramatic change in philosophy and an on-going activity. Management will be doing some phase of commissioning for the entire life of the facility. Remember, it is cost effective.

Session 10.30



Commissioning: General

5. Commissioning is not implicit in construction. Management must make it happen.
6. Commissioning almost guarantees a smooth transition.
7. Commissioning must include training, systems manuals, maintenance manuals, and other documentation as needed.

Session 10.31



Real Time Commissioning

A Texas A&M study started Continuous Commissioning™ on 80 campus buildings in 1995. By 1999, more than \$2,000,000 was spent for metering and another \$2,000,000 was spent for analysis and implementation of recommendations. Cumulative savings over the same time period were more than \$10,000,000.

Session 10.32



Commissioning: Where To From Here?

- Check Building Commissioning Association site and publications
- Check ASHRAE course and Guideline 1
- Check AEE course and certification

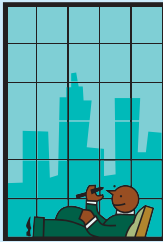
Session 10.33



Building Envelope

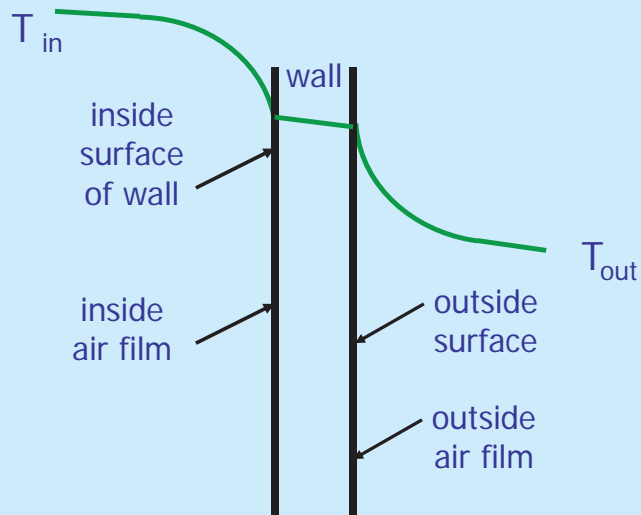


Fast Track CEM



Heat Loss and Gain

- Heat is lost and gained through walls and ceilings.
- Consider a three-quarter inch plywood wall.



Session 11.2



Basic Heat Flow Equations

- Conduction heat losses through walls and ceilings

$$q = \frac{A \times \Delta T}{\Sigma R} \left[\frac{\text{Btu}}{\text{hr}} \right]$$

- Heat loss is proportional to the area A and the temperature difference ΔT between inside and outside.
- ΣR is the sum of the resistances of everything that resists heat flow.

$$\Sigma R = R_{\text{inside air film}} + R_{\text{plywood}} + R_{\text{outside air film}} \left[\frac{\text{hr} \cdot \text{ft}^2 \cdot ^\circ \text{F}}{\text{Btu}} \right]$$



Session 11.3

Basic Heat Flow Equations

- The equation often is written

$$q = U \times A \times \Delta T \left[\frac{\text{Btu}}{\text{hr}} \right]$$

- where U is the overall thermal conductance.

$$U = \frac{1}{\Sigma R} = \frac{1}{R_{\text{Total}}} \left[\frac{\text{Btu}}{\text{hr} \cdot \text{ft}^2 \cdot ^\circ \text{F}} \right]$$



Session 11.4

Surface Air Film Resistance (Buildings) *

Wall or roof position	Direction of heat flow	R_s (hr·ft ² ·°F/Btu)
<u>Still air</u>		
horizontal	up	0.61
horizontal	down	0.92
vertical	horizontal	0.68
<u>Moving air</u>		
15 mph (winter)	All	0.17
7.5 mph (summer)	All	0.25

*Data from [2001 ASHRAE Fundamentals Handbook](#), American Society of Heating, Refrigerating and Air-Conditioning Engineers, Inc., Atlanta, Georgia. p. 25.2.



Session 11.5

Wall and Insulation Resistances

- R can be obtained from the conductance C, given for a specified thickness of material,

$$R = \frac{1}{C}$$

- If the conductivity k is given, R can be calculated knowing k and the material thickness t.

$$R = \frac{t}{k}$$



Session 11.6

Conductances and Conductivities

Table 11-2.				Table 11-2. (Continued)						
Material	Description	Conductivity K^a	Conductance C^b	Material	Description	Conductivity K^a	Conductance C^b			
Building boards	Asbestos-cement board	4.0	2.25	Masonry materials (cont'd)	Block, concrete, 3 oval core:	12.50	1.40			
	Gypsum or plaster board...1/2 in.	0.80			Sand & gravel aggregate 4 in.			0.90		
	Plywood	0.38			Sand & gravel aggregate 6 in.			0.90		
	Plywood...3/4 in.	0.49			Cinder aggregate 4 in.			0.58		
	Sheathing (impregnated or coated)	1.40			Cinder aggregate 6 in.					
Sheathing (impregnated or coated) 25/32 in.			Stone, lime or sand							
Insulating materials	Blanket and Batt:		0.18	Plastering materials	Cement plaster, sand aggregate	5.0	3.12			
	Mineral wool fibers (rock, slag, or glass)	0.27			Gypsum plaster:	1.5				
	Wood fiber	0.25			Lightweight aggregate...1/2 in.			5.6		
	Boards and slabs:								Lt. wt. agg. on metal lath...3/4 in.	1.7
		Cellular glass			0.39				Perlite aggregate	
Corkboard		0.27	Sand aggregate	7.70						
Glass fiber		0.25	Sand aggregate on metal lath 3/4 in.		7.70					
Insulating roof deck...2 in.		Vermiculite aggregate	7.70							
Masonry materials	Loose fill:					0.90	Roofing	Asphalt roll roofing		6.50
	Mineral wool (glass, slag, or rock)	0.27						Built-up roofing...3/8 in.	3.00	
	Vermiculite (expanded)	0.46		Siding materials			Asbestos-cement, 1/4 in. lapped		4.76	
	Concrete:						Asphalt insulating (1/2 in. board)		0.69	
		Cement mortar	5.0		Wood, bevel, 1/2 x 6, lapped			1.23		
		Lightweight aggregates, expanded shale, clay, slate, slags; cinder; pumice; perlite; vermiculite	1.7		Woods		Maple, oak, and similar hardwoods	1.10	1.02	
		Sand and gravel or stone aggregate	12.0				Fir, pine, and similar softwoods	0.80		
	Stucco	5.0	Fir, pine & sim. softwoods 25/32 in.							
	Brick, tile, block, and stone:									
		Brick, common	5.0							
Brick, face		9.0								
Tile, hollow clay, 1 cell deep, 4 in.		0.54								
Tile, hollow clay, 2 cells, 6 in.										

B. L. Capehart *et al.*, Guide to Energy Management, 4th ed., Fairmont Press, 2003, pp. 386-387.



Session 11.7

Basic Heat Flow Problem

- What is the rate of heat loss through a three-quarter inch plywood wall if the inside temperature is 60°F and the outside temperature is 30°F? The area of the wall is 100 square feet. Include surface films.



Session 11.8

Basic Heat Flow Equations

- How could we reduce the heat flow?
- Often the answer is insulation—a simple and inexpensive way of adding more resistance to the denominator of the equation

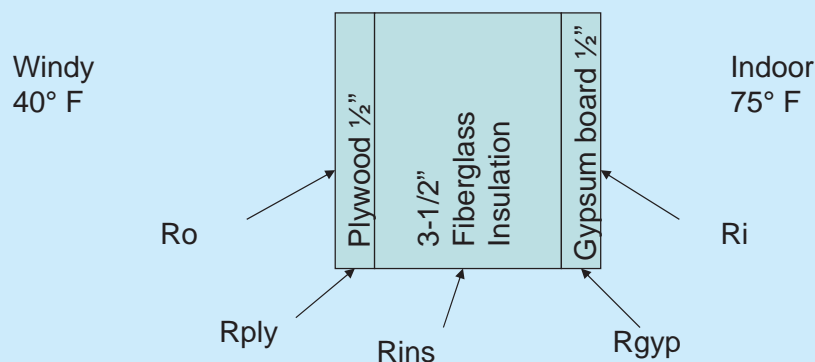
$$q = \frac{A \times \Delta T}{\Sigma R} \left[\frac{\text{Btu}}{\text{h}} \right]$$



Session 11.9

Composite Wall Example

A 160 ft² wall is exposed to an inside temperature of 75 °F and an outside winter temperature of 40 °F in windy weather. The wall consists of ½" plywood, 3 ½" of fiberglass insulation, and ½" gypsum board. How much heat is lost through the wall? Include surface films.



Session 11.10

Basic Heat Flow Equations

- Temperature varies with time of day and season, so we often resort to this heat conduction equation

$$Q = U \times A \times DD \times 24 \left[\frac{\text{Btu}}{\text{yr}} \right]$$

- where DD can be HDD or CDD. Annual units are $\frac{^{\circ}\text{F} \cdot \text{d}}{\text{yr}}$
- $U \times A$ is also known as the conduction part of the overall Building Load Coefficient BLC. Other parts include infiltration, ventilation, and slab-on-grade factor.



Session 11.11

Air and Water Heat Flow Problems

$$q = \dot{m} \Delta h \left[\frac{\text{Btu}}{\text{h}} \right] \text{ General}$$
$$= \dot{m} C_p \Delta T \left[\frac{\text{Btu}}{\text{h}} \right] \text{ Sensible heat only}$$

- \dot{m} = Mass Flow Rate (lb/h)
 Δh = Enthalpy Difference (Btu/lb)
 C_p = Heat Capacity (Btu/lb·°F)
 ΔT = Temperature Difference (°F)



Session 11.12

Air and Water Heat Flow Problems

Air: *Sensible Heat Only*

$$q = \left[\text{CFM} \times \frac{0.075 \text{ lb}}{\text{ft}^3} \times \frac{60 \text{ min}}{\text{h}} \right] \times \left[\frac{0.24 \text{ Btu}}{\text{lb} \cdot ^\circ\text{F}} \right] \times \Delta T$$

$$q = \text{CFM} \times 1.08 \times \Delta T \quad \left[\frac{\text{Btu}}{\text{h}} \right]$$

Sensible heat only



Session 11.13

Air and Water Heat Flow Problems

Air: *General*

$$q = \left[\text{CFM} \times \frac{0.075 \text{ lb}}{\text{ft}^3} \times \frac{60 \text{ min}}{\text{h}} \right] \times \Delta h$$

$$q = \text{CFM} \times 4.5 \times \Delta h \quad \left[\frac{\text{Btu}}{\text{h}} \right]$$



Session 11.14

Water: Sensible Heat only

$$q = \left(\text{GPM} \times \frac{8.34 \text{ lb}}{\text{gal}} \times \frac{60 \text{ min}}{\text{h}} \right) \times \left(\frac{1 \text{ Btu}}{\text{lb} \cdot ^\circ \text{F}} \right) \times \Delta T$$

$$q = \text{GPM} \times 500 \times \Delta T \quad \left[\frac{\text{Btu}}{\text{h}} \right]$$

Sensible heat only



Session 11.15

Fan Laws - or Affinity Laws - Review

- **Flow and Speed**

$$\frac{\text{CFM}_2}{\text{CFM}_1} = \frac{\text{RPM}_2}{\text{RPM}_1}$$

- **Pressure (Head) and Speed**

$$\frac{P_2}{P_1} = \left(\frac{\text{RPM}_2}{\text{RPM}_1} \right)^2 = \left(\frac{\text{CFM}_2}{\text{CFM}_1} \right)^2$$

- **Power and Speed**

$$\frac{\text{HP}_2}{\text{HP}_1} = \left(\frac{\text{CFM}_2}{\text{CFM}_1} \right)^3$$



Session 11.16

The Psychrometric Chart

The Psychrometric Chart graphically represents the steam tables for moisture in air at conditions we encounter in HVAC work.

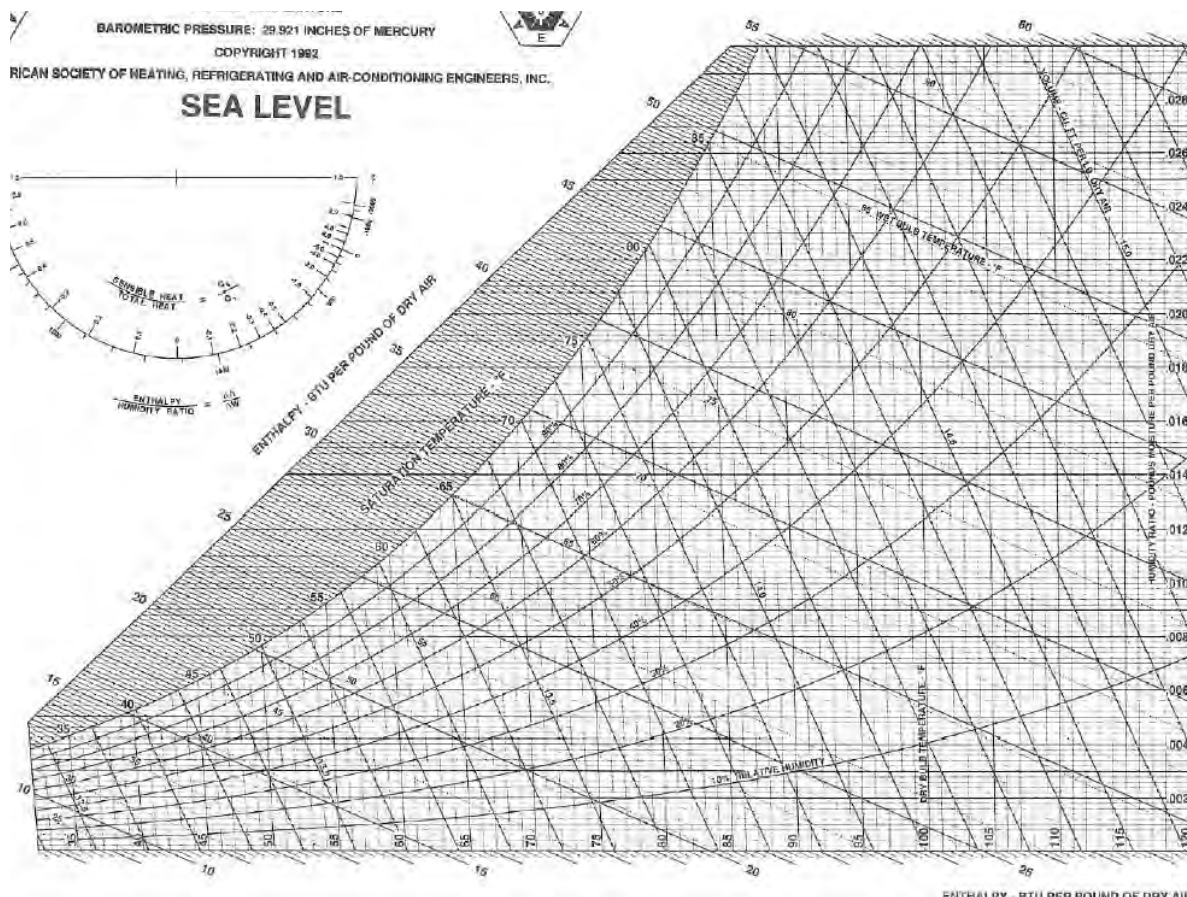
The Psych Chart allows complex problems to be worked out easily, and provides a feel for common HVAC processes that we are interested in.

The standard ASHRAE Psych Chart has a horizontal axis for dry bulb temperature, and a vertical axis for humidity ratio in pounds of moisture per pound of dry air.

Other parameters on the chart are: relative humidity, wet bulb temperature, enthalpy, specific volume, and saturation temperature.



Session 11.17



CEM Review Problems

An absorption chiller with a COP of 0.8 is powered by hot water that enters at 200°F and exits at 180°F at a rate of 25 gpm. The chilled water operates on a 10°F temperature difference. Calculate the chilled water flow rate. Solution of this problem does not require a knowledge of how absorption chillers work internally.

- (A) 10 gpm
- (B) 20 gpm
- (C) 40 gpm
- (D) 45 gpm
- (E) 30 gpm



Session 4.1.19

CEM Review Problems

The conduction part of the Building Load Coefficient ($U \cdot A$) for a building is 5000 Btu/h per degree F. Estimate the seasonal energy consumption for heating if the heating season has 3,500 degree days. The heating unit efficiency is 80%.

- (A) 625 MCF/yr
- (B) 350 MCF/yr
- (C) 420 MCF/yr
- (D) 656 MCF/yr
- (E) 525 MCF/yr



Session 4.1.20

Example

A wall has an area of 100 ft² and has a thermal conductance of 0.25 Btu/ft²·h·°F. If there are 3000 degree-days in the annual heating season, what is the total amount of heat that must be supplied by the heating system?

Session 4.1.21



Duct Loss Example

10,000 cfm of air leaves an air handler at 50 °F. It is delivered to a room at 65 °F. There was no air volume loss due to duct air leaks. No moisture was added, or taken away from the air in the duct. How many Btu/h heat gain occurred because of heat transfer by conduction?

- (A) 162,000 Btu/h
- (B) 75,000 Btu/h
- (C) 126,000 Btu/h
- (D) 256,000 Btu/h
- (E) 10,000 Btu/h

Session 4.1.22



Chilled Water Flow Example

A chiller supplies cold water with a ΔT of 10 °F.
How many GPM of this water is needed to provide
one ton of air conditioning?

Session 4.1.23



Practice Example

ACE Industries presently has a 5 hp ventilating fan that
draws warm air from a production area. The motor
recently failed, and they have determined the amount of
ventilation can be reduced by one-third.

What size motor is needed now?

Answer: 1.48 HP

Session 4.1.24



Psychrometric Heating Example

- For air, $q = \text{CFM} \times 4.5 \times \Delta h \left[\frac{\text{Btu}}{\text{h}} \right]$
- Air at 69°F dry bulb and 50% relative humidity flows at 6750 cubic feet per minute and is heated to 90°F dry bulb and humidified to 40% RH. How many Btu/h is required in this process?
 - (A) 50,000 Btu/hr
 - (B) 100,000 Btu/hr
 - (C) 150,000 Btu/hr
 - (D) 300,000 Btu/hr

Session 4.1.25



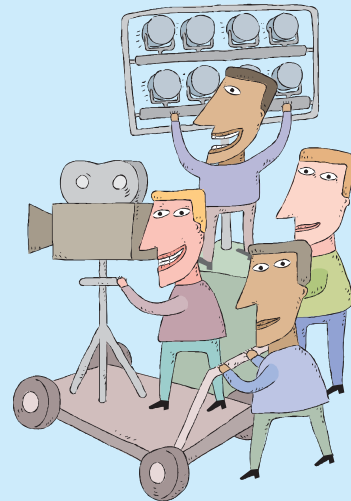
Psychrometric Cooling Example

- How many tons of air conditioning is required to cool 1000 CFM of air at 90 °F and 60 % relative humidity (RH) to 60 °F and 100 % RH?

Session 4.1.26



Lighting Basics and Lighting System Improvements



Session 12.1



Principles of Efficient Lighting Design

- Meet target light levels
- Efficiently produce light
- Efficiently deliver light
 - Balance efficiency with aesthetics, lighting quality, visual comfort
- Automatically control lighting operation

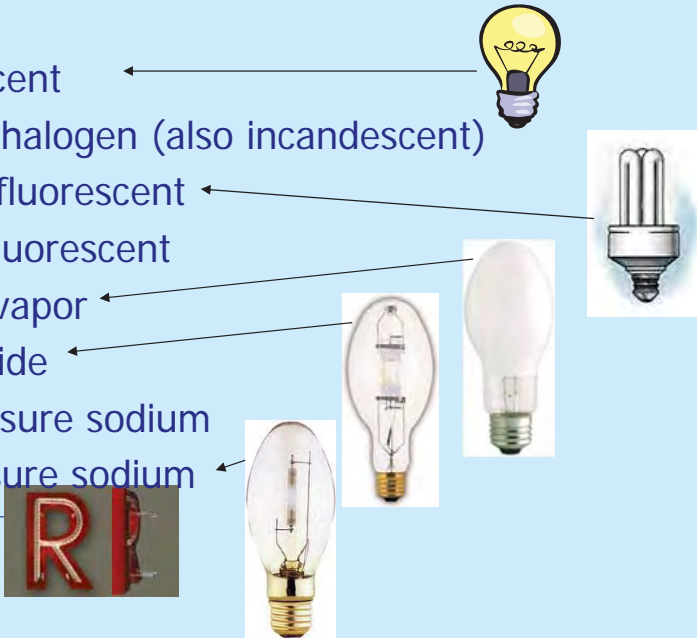


Session 12.2



- **Types of Lamps** - Common types of lamps include:

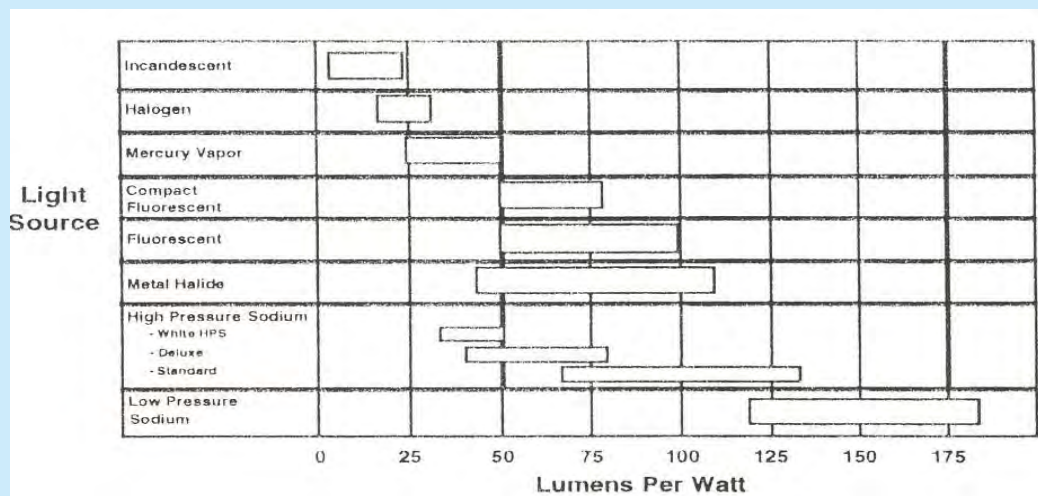
- incandescent
- tungsten halogen (also incandescent)
- compact fluorescent
- full-size fluorescent
- mercury vapor
- metal halide
- high-pressure sodium
- low-pressure sodium
- LED



Session 12.3



Light Source Efficacy



Note: Source efficacy values include ballast losses

Session 12.4



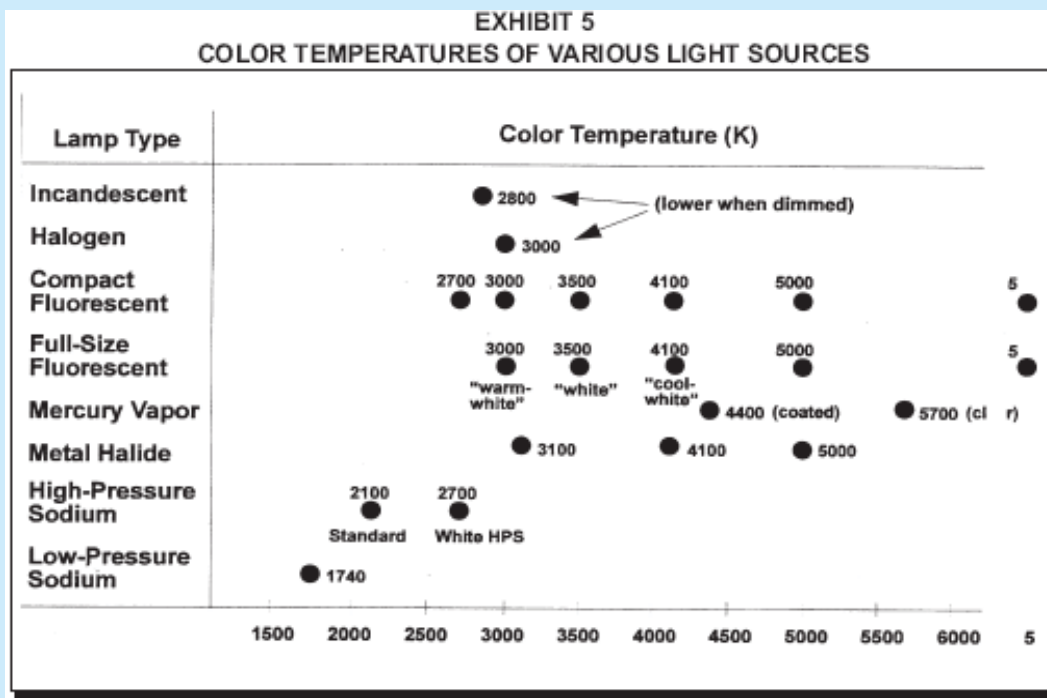
Color Rendering Index (CRI)

EXHIBIT 3
TYPICAL CRI VALUES FOR SELECTED LIGHT SOURCES

Source	Typical CRI Value
Incandescent/Halogen	100
Fluorescent	
Cool White T12	62
Warm White T12	53
High Lumen T12	73-85
T8	75-85
T10	80-85
Compact	80-85
Mercury Vapor (clear/coated)	15/50
Metal Halide (clear/coated)	65/70
High-Pressure Sodium	
Standard	22
Deluxe	65
White HPS	85
Low-Pressure Sodium	0



Session 12.5



Session 12.6

Amount of Light Required For Specific Applications

- We often use more light than is needed for many applications and tasks.
 - Light levels are measured in *footcandles (or lux, in SI units)* using an illuminance meter.

$$\text{FC} = \text{lumens} / \text{ft}^2$$

$$\text{Lux} = \text{lumens} / \text{m}^2$$

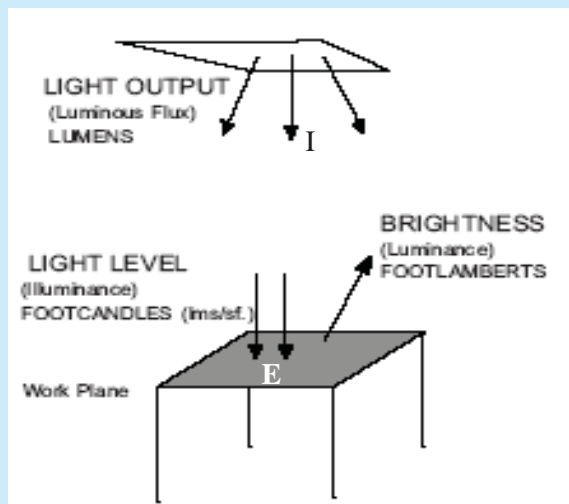


- Consensus standards for light levels are set by the Illuminating Engineering Society of North America (IESNA.org).



Session 12.7

Quantity of Illumination



Inverse Square Law

$$E = \frac{I}{d^2}$$

where d = distance from light source to surface of interest



Session 12.8

RECOMMENDED LIGHT LEVELS

Source: IESNA

TYPE OF ACTIVITY	RANGE OF ILLUMINANCE
Public spaces with dark surroundings	2-3-5 fc
Simple orientation for short temporary visits (typical hallway)	5-7½-10 fc
Working spaces where visual tasks are only occasionally performed	10-15-20 fc
Ambient lighting for computer use	20-25-30 fc
Performance of visual tasks	
High contrast or large size (typical office)	20-30-50 fc
Medium contrast or small size	50-75-100 fc
Low contrast or very small size	100-150-200 fc
Low contrast and very small size over a prolonged period	200-300-500 fc
Performance of very prolonged and exacting visual tasks	500-750-1000 fc
Performance of very special visual tasks of extremely low contrast and small size	1000-1500-2000 fc

Session 12.9



Some typical light levels needed are:

Parking lot	2 Footcandles
Hallways	10 Footcandles
Factory floor	30 Footcandles
Offices	50 Footcandles
Inspection	100 Footcandles
Operating room	1000 Footcandles

Session 12.10



Average Rated Life

- Average rated life of a lamp is median value of life expectancy of a group of lamps
 - Time at which 50% have failed, 50% are surviving
 - Fluorescent lamps rated at 3 hours on, 20 minutes off per operating cycle
 - HID lamps rated at 10 hours on, one hour off per operating cycle
- Increased frequency of switching will decrease lamp life in hours, but typically increase useful calendar life
 - Energy savings more significant than lamp costs

Session 12.11



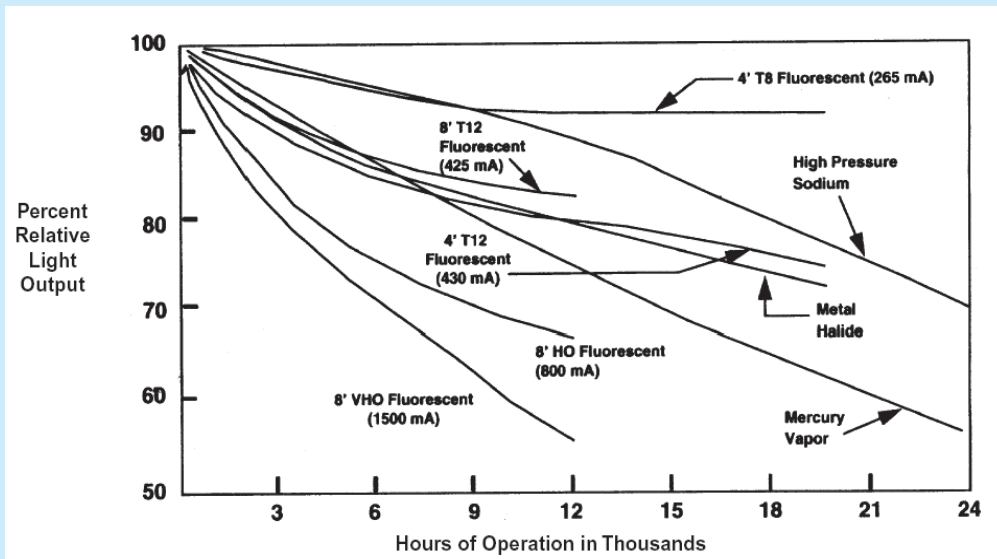
Lighting Maintenance Principles

- Light output of all lighting systems decreases over time
- Lighting systems are over-designed to compensate for future light loss
- Improving maintenance practices can reduce light loss (depreciation) and can either:
 - allow reductions in energy consumption (redesign), or
 - improve light levels
- Group maintenance practices save money

Session 12.12



Lamp Lumen Depreciation (LLD)



Source: US DOE



Session 12.13

Lighting System Design Methods

1. Lumen Method

- Assumes an equal footcandle level throughout the area.
- This method has been used frequently since it is simple.

2. Point by Point Method

- The current method of design based on the Fundamental Law of Illumination.
- Requires a computer program and extensive computation.



Session 12.14

Lumen Method Formula

$$N = \frac{F1 \times A}{Lu \times LLF \times Cu}$$

where

N = the number of lamps required

F1 = the required footcandle level at the task

A = area of the room in square feet

Lu = the lumen output per lamp

Cu = the coefficient of utilization

LLF = the combined light loss factor

Session 12.15



Example of Lumen Method

Find the number of lamps required to provide a uniform 50 footcandles on the working surface in a 40 x 30 room. Assume two 3000 lumen lamps each per fixture, and assume that LLF is 0.65 and CU is 70%.

$$N = \frac{50 \times 1200}{3000 \times 0.65 \times 0.7} = 44$$

The number of two-lamp fixtures needed is 22.

Session12.16



The Coefficient of Utilization (CU)

The coefficient of utilization is a measure of how well the light coming out of the lamps and the fixture contributes to the useful light level at the work surface.

It may be given, or you may need to find it:

- Use Room Cavity Ratio (RCR) to incorporate room geometry
- Use Photometric Chart for specific lamp and fixture

Session 12.17



Room Cavity Ratio (RCR)

$$\text{RCR} = 2.5 \times h \times (\text{Room Perimeter}) / (\text{Room Area})$$

Where

L = room length

W = room width

h = height from lamp to top of working surface

Session 12.18



Example

Find the RCR for a 30 by 40 rectangular room with lamps mounted on the ceiling at a height of 9.5 feet, and the work surface is a standard 30 inch desk.

$$h = 9.5 - 2.5$$

$$= 7 \text{ feet}$$

$$\begin{aligned} \text{RCR} &= 2.5 \times h \times (2L+2W)/(L \times W) \\ &= 2.5 \times 7 \times (60 + 80)/(30 \times 40) \\ &= 35 \times 70/1200 \\ &= 2.04 \end{aligned}$$



Session 12.19

Photometric Chart

Report LS1 1101 Lamp: 400 Watt Clear Lumens: 50,000 Mounting Surface/Pendant				COEFFICIENTS OF UTILIZATION Zonal Cavity Method Effective Floor Cavity Reflectance 0.20							
RCR	RC RW	80			70			50		30	
		70	50	30	70	50	30	50	30	50	30
1		94	91	89	91	88	86	83	81	78	77
2		88	84	80	86	81	78	77	74	73	71
3		83	77	72	80	75	70	71	67	67	65
4		78	71	65	75	69	64	66	61	63	59
5		73	65	59	70	63	58	60	56	58	54
6		68	58	52	66	58	52	55	51	53	49
7		63	54	48	61	53	47	51	46	49	44
8		59	49	43	57	48	43	46	41	45	40
9		55	45	39	53	44	38	42	37	41	36
10		51	41	35	49	40	35	39	34	37	33



Session 12.20

Example

Find the Coefficient of Utilization for a 30 by 40 rectangular room with a ceiling height of 9.5 feet, a ceiling reflectance of 70% and a wall reflectance of 50% using the photometric chart on the previous page.

The RCR from before was 2.04. Using $RC = 70\%$ and $RW = 50\%$, the CU is found as $CU = 0.81$, or 81%

Session 12.21



Fundamental Law of Illumination or Inverse Square Law

$$E = \frac{I}{d^2}$$

where

E = Illuminance in footcandles

I = Luminous intensity in lumens

d = Distance from light source to surface area of interest

One footcandle is equal to one lumen per square foot
(One lux is equal to one lumen per square meter)

Session 12.22



Example

In a high bay facility, the lights are mounted on the ceiling which is 40 feet above the floor. The lighting level on the floor is 50 footcandles. No use is made of the space between 20 feet and 40 feet above the floor.

In a theoretical sense – that is, using the fundamental law of illuminance – what would be the light level in footcandles directly below a lamp if the lights were dropped to 20 feet?

$$FC = 50(40^2/20^2) = 200 \text{ footcandles}$$

Session 12.23



What to Look for in Lighting Audit

- Lighting Equipment Inventory
- Lighting Loads
- Room Dimensions
- Illumination Levels
- Hours of Use
- Lighting Circuit Voltage



Session 12.24



Potential Lighting ECMs

- Fluorescent Upgrades
- Delamping
- Incandescent Upgrades
- HID Upgrades
- Controls Upgrades
- Daylight compensation



Session 12.25



What Does a Ballast Do?

- A ballast does three things:
 - Conditions the lamp to start
 - Applies a high voltage spike to start the gas discharge process
 - Applies a current limiter to reduce the lamp current to a safe operating level
- Ballast factor
 - Normal light output (0.85-0.95)
 - Can specify reduced or increased light output in electronic ballasts with proportional reduction or increase in power

Session 12.26



New Lighting Technology

- **Induction lamps**
 - Long life -- 100,000 hours for lamp & ballast
 - Phillips QL lamps in 55W, 85W and 165W
 - New application with reflector to replace metal halides as sign lights for road and commercial signs. Last four times as long



Session 12.27

- OSRAM/Sylvania is the other maker of long life induction lamps
- Ictron in 70W, 100W and 150W sizes
- Also 100,000 hours
- Properties about same as QL lamp
 - Efficacy around 80 L/W (150 W ICE)
 - CRI 80
 - Instant start, and re-start
 - Operate in hot and cold environments



Session 12.28

LED Lighting

- 80% of all new exit lights are LED lights
- But, there are some other interesting applications
 - **Traffic Signals**
 - Green 12" ball 140 W to 13 W LED
 - Red 12" ball 140 W to 11 W LED
 - Life 1 year to 7 years for LED
 - Cost \$3 to \$75 for LED
 - **Commercial Advertising Signs (Neon)**
 - Neon 15 mm tube 3 W/ft
 - LED 15 mm replacement 1.03 W/ft



Session 12.29

T5HO and High Bays



**High Bay & Fluorescent T5/HO
(it's not just for HID anymore!)**

<u>Lamp</u>	<u>Lamp and Ballast</u> *		<u>Wattage</u>
	<u>Maintained Light</u>		
4 - T5/HO Lamps	19,000 Lumens		242 watts
1 - MVR250/U	13,500		293
1 - MVR250/Pulse	17,000		288
6 - T5/HO Lamps	28,500 Lumens		363 watts
1 - MVR400/U	23,500		458
1 - MVR400/Pulse	33,000		456

* (Impact of Fixture Design on Performance NOT included)

**AND: Instant on, dimmable, choice of colors,
no color shift...**



Session 12.30

Compare Lighting Power Density to ASHRAE/IES 90.1 Values

- **Example Whole Building Lighting Power Densities (W/ft²)**

	<u>1989</u>	<u>1999</u>	<u>2003</u>	<u>2010</u>
• Offices	1.63	1.30	1.0	0.9
• Education	1.79	1.50	1.2	0.99
• Retail	2.36	1.90	1.5	1.4
• Warehouse	0.53	1.20	0.8	0.66



Session 12.31

Typical Lighting Operation

Building Type	Annual Hours of Operation
Assembly	2760
Avg. Non-Residential	3500
Education	2605
Food Sales	5200
Food Service	4580
Health Care	7630
Lodging	8025
Mercantile	3325
Office	2730
Warehouse	3295



Session 12.32

Lighting Control Technologies

- On/off snap switch
- Timers and control systems
- Solid-state dimmers
- Dimming electronic ballasts
- Occupancy sensors
- Daylighting level sensors

Session 12.33



Energy Savings Potential With Occupancy Sensors

<u>Application</u>	<u>Energy Savings</u>
Offices (Private)	25-50%
Offices (Open Spaces)	20-25%
Rest Rooms	30-75%
Corridors	30-40%
Storage Areas	45-65%
Meeting Rooms	45-65%
Conference Rooms	45-65%
Warehouses	50-75%

Session 12.34



Lighting Appendix

AEE:CEM

CEM Exam Review Questions

1. The efficacy of a light source refers to the color rendering index of the lamp.

A) True B) False
2. Increasing the coefficient of utilization of fixtures in a room will in many instances increase the number of lamps required.

A) True B) False
3. Which HID lamp has the highest efficacy – for the same wattage?

A) Mercury vapor
B) Metal halide
C) High pressure sodium



4. One disadvantage to metal halide lamps is a pronounced tendency to shift colors as the lamp ages.
- A) True B) False

Session 4.2.37



5. A 244,000 square foot high bay facility is presently lit with 800 twin 400 watt mercury vapor fixtures (455 watts per lamp including ballast). What are the annual savings of replacing the existing lighting system with 800 single 400-watt high-pressure sodium fixtures (465 watts per lamp including ballast)? Assume 8000 hours operation per year, an energy cost of \$0.05 per kWh, and a demand cost of \$6.00 per kW-month.

Solution

Session 4.2.38



Solution

$$\Delta kW = (800 \text{ fixtures})(.455 \text{ kW/lamp})(2 \text{ lamps}) - (800 \text{ fixtures})(.465 \text{ kW/fixture}) = 356 \text{ kW}$$

$$\text{Demand \$ savings} = (356 \text{ kW})(\$6/\text{kW-mo})(12 \text{ mo/yr}) = \$25,632/\text{yr}$$

$$\text{Energy \$ savings} = (356 \text{ kW})(8000 \text{ hrs/yr}) (\$0.05/\text{kWh}) = \$142,400/\text{yr}$$

$$\text{Total \$ savings} = (\$25,632 + \$142,400)/\text{yr} = \$168,032/\text{yr}$$

$$\text{Cost} = (800 \text{ fixtures})(\$400/\text{fixture}) = \$320,000 ??$$

Session 4.2.39



7. The Light Switch Problem (Just for Fun)

You must determine which switch on a three switch panel on the first floor of a building controls a light on the fifth floor of the building.

- The other two switches are not connected to anything and there is no way to see any light from the fifth floor without going up stairs.
- You have no tools and you cannot take the switch cover off.
- You can only make one trip up the stairs to the light. How can you determine which switch operates the light?

Session 4.2.40



Solution

Turn on the middle switch and the right-hand switch, wait 10 minutes.

Turn the middle one off and run up the stairs.

- If the light bulb is off and cold it is the left-hand switch.
- If the bulb is off and hot it is middle switch.
- If the bulb is on it is the right-hand switch.



Session 4.2.41

**EXHIBIT 4
LAMP CHARACTERISTICS**

	Standard Incandescent	Tungsten-Halogen	Fluorescent	Compact Fluorescent	Mercury Vapor	Metal Halide	High-Pressure Sodium	Low-Pressure Sodium
Wattage	3-1,500	10-1,500	4-215	4-85	40-1,250	30-2,000	35-1,000	18-180
Average System Efficacy (lm/W)	4-24	8-33	49-89	34-88	19-43	38-86	22-115	50-150
Average Rated Life (hrs)	750-2,000	2,000-4,000	7,500-34,000	7,000-20,000	24,000+	8,000-20,000	16,000-24,000	12,000-18,000
CRI	100	100	49-82	82-86	15-50	65-82	21-85	0
Life Cycle Cost	high	high	low	moderate	moderate	moderate	low	low
Fixture Size	compact	compact	extended	compact	compact	compact	compact	extended
Start to Full Brightness	immediate	immediate	0-5 seconds	0-1 min	3-9 min	3-5 min	3-4 min	7-9 min
Restrike Time	immediate	immediate	immediate	immediate	10-20 min	4-20 min	1 min	immediate
Lumen Maintenance	good/excellent	excellent	fair/excellent	good/excellent	poor/fair	good	good/excellent	excellent

Lighting Fundamentals • Lighting Upgrade Manual • EPA's Green Lights® Program • February 1997



Session 4.2.42


Lighting Quality Measures

- **Visual comfort probability (VCP)** indicates the percent of people who are comfortable with the glare (brightness) from a fixture
- **Spacing criteria (SC)** refers to the maximum recommended distance between fixtures to ensure uniformity
- **Color rendering index (CRI)** indicates the color appearance of an object under a source as compared to a reference source



Session 4.2.43

Compact Fluorescent Example



COMPACT FLUORESCENT LAMPS

For retrofit applications in task lights, wall sconces, downlights, wallwashers, and outdoor fixtures, a wide variety of lamp shapes and types are available. These include globes, capsules, floods, and twin/quad-tube units. The most common retrofit applications are in open fixtures with mounting heights less than 12 feet. In some applications, users may prefer to hardwire the ballast in the fixture to prevent replacement with incandescent lamps. Lamps with excellent color rendering capabilities (80-85 CRI) are recommended for most indoor applications.

Wattages: 5-26W
Replaces: 25-100W incandescents
Energy Savings: 60-75%
Efficacy: 26-58 lm/W
Lamp Cost: \$3-6
Ballast Cost: \$9-24
Rated Lamp Life: 10,000 hrs
Color Quality: 60-85 CRI
Dimming: Not in retrofit applications
Qualifications: These lamps can have starting limitations in freezing temperatures, and operation in enclosed fixtures may shorten lamp and ballast life and reduce light output. Because compact fluorescents are not point sources, other technologies may be more suitable in applications—such as retail display lighting—where good beam control is required. Some conventional fixture types may not be large enough to accept the larger size of the fluorescent lamp/ballast package. Check manufacturer data to determine impacts of ballast use on power factor and power quality.

Example: Compact Fluorescent Lamps

Old System: Incandescent hallway recessed downlighting in 12' ceiling; 100 fixtures @ 75 watts @ \$3.50 ; operating hours: 15 hrs/day, 6 days/week (4,680 hrs/yr)

New System: Screw-in fluorescent lamps (82 CRI) with integrated magnetic ballasts; downlighting reflectors; 100 fixtures @ 22 watts (including ballast operation)

Results: 71% energy savings; virtually unchanged light level and appearance; 10 times longer lamp life

Savings: Demand savings: 5.3 kW, energy savings: 24,800 kWh/yr, dollar savings @ \$4/kW/mo and \$0.08/kWh: \$2,238/yr; relamping savings: materials \$796/yr; labor \$2,106/yr
Net Savings: \$5,140/yr

Cost: Material: 100 lamp/ballast capsules @ \$16; 100 reflectors @ \$4; Labor: 100 installations @ \$3
Total Project Cost: \$2,500

Payback: \$2,500/\$5,140/yr = 0.48 yr



Session 4.2.44

Calculation for Compact Fluorescent Example

kW savings

$$100 \text{ fixtures } (.075 \text{ kW/fixture} - .022 \text{ kW/fixture}) = 5.3 \text{ kW}$$

kWh savings

$$(5.3 \text{ kW})(4680 \text{ hrs/yr}) = 24,804 \text{ kWh}$$

Demand \$ savings

$$(5.3 \text{ kW})(\$4/\text{kW-mo})(12 \text{ mo/yr}) = \$254.40/\text{yr}$$

Energy \$ savings

$$(24,804 \text{ kWh/yr})(\$0.08/\text{kWh}) = \$1,984.32/\text{yr}$$

Total dollar savings

$$(\$254.40 + \$1,984.32)/\text{yr} = \$2239/\text{yr}$$



Session 4.2.45

T-8 Example

Example: T-8 Fluorescent System Retrofit

Old System: Office lighting consisting of 360 fluorescent 2'x4' fixtures; operating hours: 14 hrs/day, 5 days/week (3640 hrs/yr); each fixture draws 188 watts with 4 standard cool white 40-watt fluorescent lamps (@ \$2) and 2 standard magnetic ballasts

New System: Each fixture now draws 112 watts with 4 tri-phosphor F40T8 32-watt fluorescent lamps and 1 electronic T-8 instant-start mode ballast*

Results: 40% energy savings; 2% reduction in light level; improved color rendering; 50% fewer ballasts to replace; 25% less lamp life using instant-start mode ballasts

Savings: Demand savings: 27.4 kW; energy savings: 99,600 kWh/yr; dollar savings @ \$4/kW/mo and \$0.08/kWh:

\$9,280/yr; relamping savings: materials <\$612/yr>; labor <\$437/yr>

Net Savings: \$8,231/yr

Cost: Material: 1440 F32T8 fluorescent lamps @ \$3.25; 360 T-8 instant-start mode electronic ballasts @ \$31; labor: 360 installations @ \$20

Total Project Cost: \$23,040

Payback: \$23,040/\$8,231/yr = 2.8 yrs (with 10% a/c factor, payback becomes 2.5 yrs)

*Building codes in some states do not allow all four lamps to be operated by a single ballast. In such cases, two adjacent fixtures can share two ballasts that are wired to leave the two-level switching intact. Assume an additional \$15 wiring cost per fixture. The payback period increases to 3.5 years.



Session 4.2.46

Calculation for T-8 Example

Demand savings

$$(360 \text{ fixtures})(.188 - .112) \text{ kW/fixture} = 27.4 \text{ kW}$$

Total \$ savings

$$(27.4 \text{ kW})[(\$4/\text{kW-mo})(12 \text{ mo/yr}) + (3640 \text{ hrs/yr} \times \$0.08/\text{kWh})] = \$9290/\text{yr}$$

Session 4.2.47



Occupancy Sensor Example

Uncontrolled System: Six conference rooms, each with four 4-lamp fluorescent fixtures (188 watts) operating 10 hrs/day, 5 days/week (2,600 hrs/yr); lamps @ \$2

Controlled System: When conference rooms are unoccupied for longer than five minutes, the fixtures are now automatically turned off; they now operate about 7 hrs/day, 5 days/week (1,820 hrs/yr)

Results: 30% energy savings; no change in fixture appearance or light level

Savings: Energy savings: 3,519 kWh/yr; dollar savings @ \$0.08/kWh: \$282/yr; relamping savings: materials \$7/yr; labor \$19/yr

Net Savings: \$308/yr

Cost: Material: 6 switch-mount occupancy sensors @ \$70; labor: 6 installations @ \$10

Total Project Cost: \$480

Payback: \$480/\$308/yr = 1.6 yrs (with 10% a/c factor, payback becomes 1.4 yrs)

Energy savings

$$(24 \text{ fixture})(.188 \text{ kW/fixture})(2600 \text{ hrs/yr})(0.30)(\$0.08/\text{kWh}) = \$282/\text{yr}$$

Session 4.2.48



Example Lighting / HVAC Interaction

Location	Cooling Loads	Heating Loads Large Building	Heating Loads Small Building
Tampa, FL	-33%	0%	0%
Phoenix, AZ	-30%	0%	0%
New Orleans, LA	-29%	1%	2%
Los Angeles, CA	-23%	0%	0%
Knoxville, TN	-21%	4%	11%
Philadelphia, PA	-17%	6%	18%
Denver, CO	-16%	7%	22%
San Francisco, CA	-16%	1%	2%
Detroit, MI	-14%	8%	23%
Providence, RI	-13%	7%	22%
Seattle, WA	-7%	4%	13%

Source: Advanced Lighting Guidelines 2003
(based on methodology of Rundquist, et.al. 1993)



Thermal Energy Storage



Session 13.1



Why is There Interest in Thermal Energy Storage?

- Reduced peak demand costs
- Some utilities offer rebates and rate incentives
- Reduced equipment size and cost (new)
- May be improved reliability due to production and storage
- Smaller fans and pumps (colder water with ice storage)

Session 13.2



Economic Payback Time

- Typical simple payback of 5 to 7 years (maybe 3 to 5 in some cases) for existing buildings and chillers.
- Recent examples from ASHRAE and others are showing the payback may be immediate to 1 – 2 years for good design in new construction.

Session 13.3



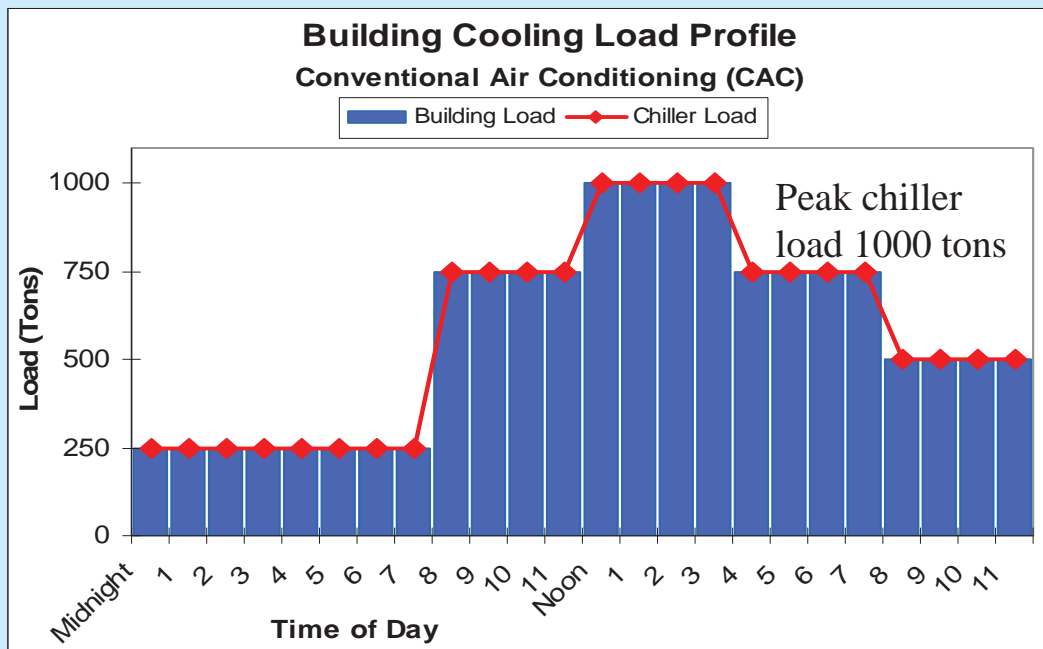
Conventional Air Conditioning Operation

- CAC system peaks at peak cooling time
- CAC system is sized to meet peak cooling load
- CAC system may have its lowest efficiency at the time it is needed the most

Session 13.4



CAC Operation



Session 13.5



Off-Peak Air Conditioning Operation

- CAC together with storage is used to meet peak cooling loads
- Chilled water or ice is used for storage medium
- Daytime peak load is reduced or eliminated
- OPAC system operates at night when efficiencies are usually higher due to lower outside temperatures

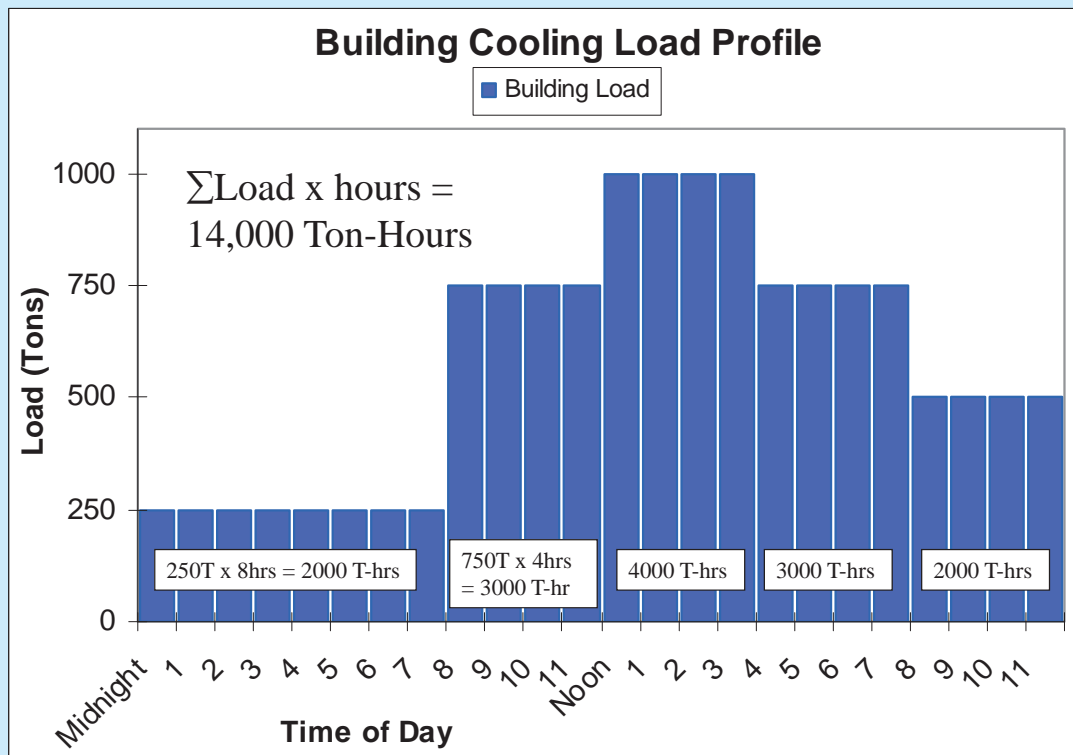
Session 13.6



Off-Peak Air Conditioning Operation

- The *Total Daily Cooling Load* (plus system losses) must be met
- The *Instantaneous Cooling Loads* must also be met when they occur, just not directly from the chillers.
- We are simply decoupling the Load (demand) from the Chiller (supply)
- If we take advantage of optimal chiller loading (sweet spot) and cooler condenser temperatures, we may gain significant efficiencies.

Session 13.7



Session 13.8



OPAC Operating Strategies

- **Load leveling**
 - Partial shifting of AC load to off-peak hours
 - Chiller runs at constant load or near constant load for 24 hours per day
 - Very cost effective for new construction
 - Less costly to purchase
 - Less space needed
 - But ~ less savings

Session 13.9

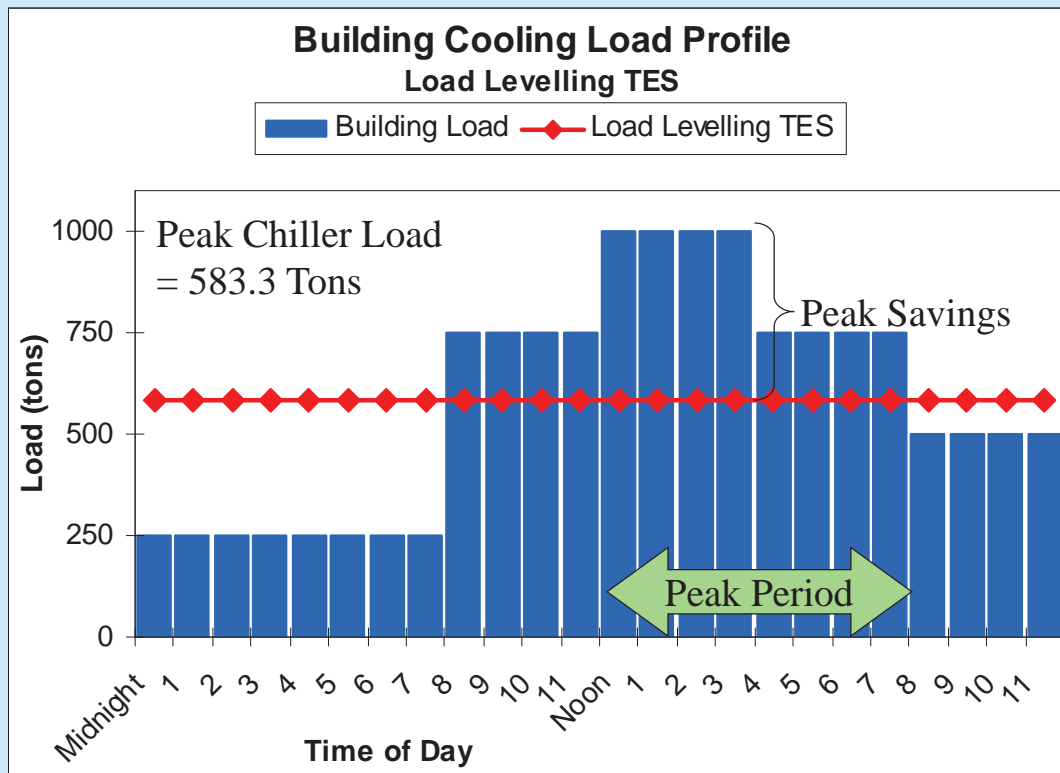


Load Leveling Chiller Load Calculations

- Where would we need to operate the chiller(s) in order to satisfy the building load? Peak period between 12:00 p.m. to 8 p.m.
- Total Ton Hours / Hours available to operate chillers
- For the Load Leveling Strategy, the chiller will operate 24 hours per day, at a load of:
 - $14,000 \text{ Ton-hours} / 24 \text{ hours} = 583.3 \text{ Tons}$

Session 13.10





Session 13.11



- **Load shifting**

- Complete shifting of peak hour AC load to off-peak hours
- OPAC system must be sized to meet peak cooling load in ton-h
- Usually more cost effective for retrofit situations because of large existing chiller load that can be moved mostly off peak
- More costly to purchase and install
- Requires more space for storage tanks
- But ~ more savings

Session 13.12



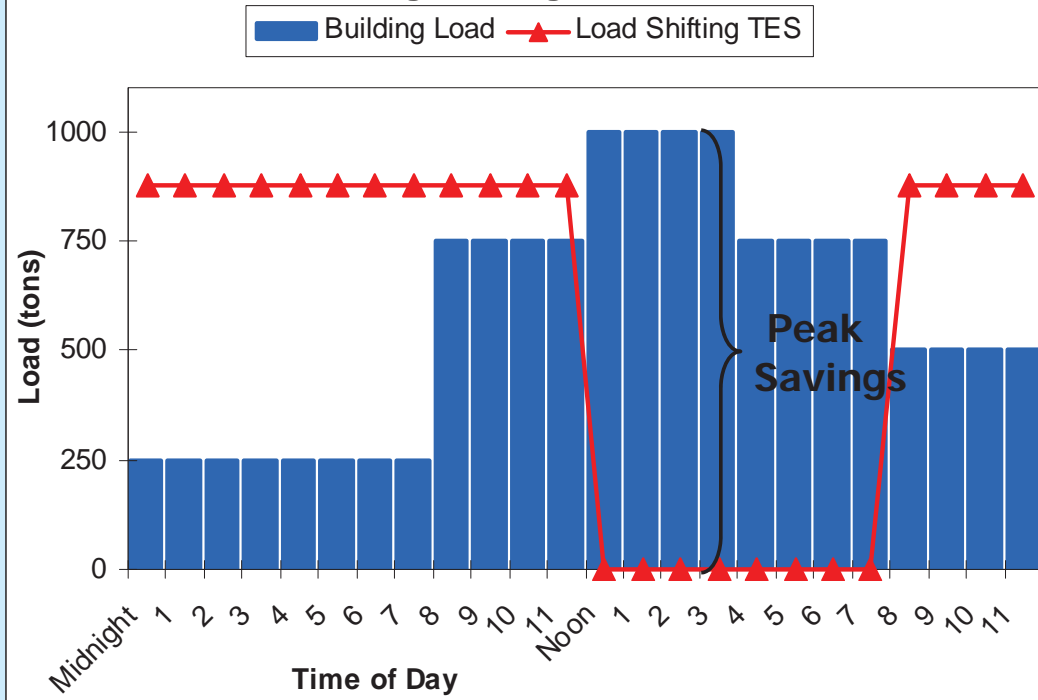
OPAC Load Calculations

- Total Ton Hours / Hours available to operate chillers
- A peak period from noon to 8 p.m. would only leave 16 hours to generate cooling capacity.
- For the Load Shifting Strategy, the chiller will operate at a load of:
 - 14,000 Ton-hours / 16 hours = 875 Tons

Session 13.13



Building Cooling Load Profile



Session 4.3.14



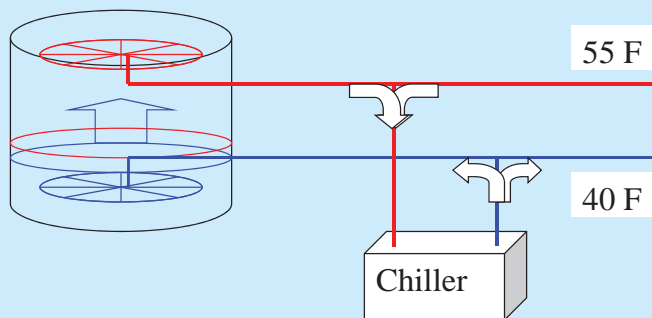
TES Storage Media

- Chilled water storage
 - Simple ~ but large tanks needed; lots of space. Requires 4 to 5 times the space of ice storage
 - Typical water temperatures of 39 to 40 deg F
 - Practical considerations for water storage tanks
 - Need to minimize mixing of warm return water with the cold water in storage
 - May need two tanks ~ if full capacity of a tank is needed. If temperature stratification of tank is used, the tank may need to be up to 20% bigger

Session 13.15



Stratified Water Tank

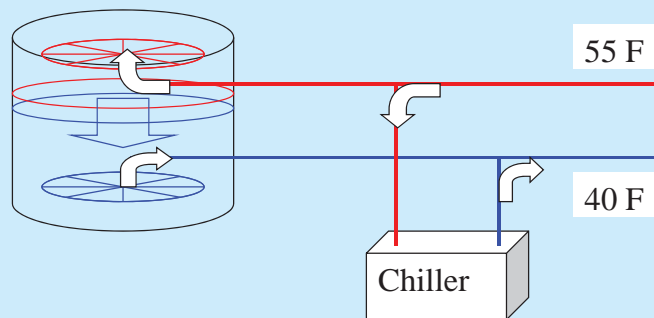


In Charge mode, the Chiller will generate cooling, and the valve and pump controls will supply building cooling with excess going to the tank.

Session 13.16



Stratified Water Tank



In Discharge mode, the tank will supplement the chiller (“load leveling” strategy) or Supply all the cooling required by the building as in the “full shift” strategy.

Session 13.17



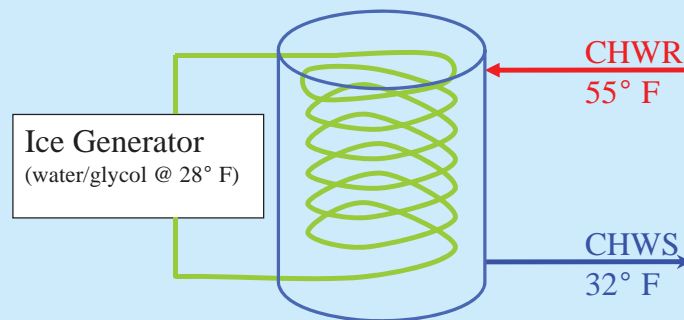
• Ice Storage

- More complex tanks and auxiliary equipment needed; more complex to maintain
- Ice/water requires around 20 to 30% of the space needed for chilled water tanks
- Solid ice requires around 10% of the space needed for chilled water tanks
- Very low temperature water can be used ~ around 34 degrees F
- Can use ice harvester, ice on coil, or ice/water (slush)

Session 13.18



Ice on Coil Tank



In Charge mode, the Ice Generator will circulate 28°F Water/Glycol mixture through Chilled Water Tank, freezing Chilled water on the coils. Warm CHWR melts the ice during Discharge, thus cooling the CHWS.

Session 13.19



Properties of Storage Media

- Chilled water systems are typically operated in a manner to use only sensible heat storage and thus stores one Btu per pound of water for each °F of temperature difference between the stored water and the returned water
- Ice systems are typically operated in a manner to use only latent heat associated with freezing and melting, and one pound of ice at 32°F absorbs 144 Btu to become 32°F water

Session 13.20



Sizing Chilled Water Storage Tanks

Assume that chilled water is stored at 39°F and is returned at the standard temperature of 54°F

- This is a 15°ΔT for the AC system.
- Thus, one pound of water stores 15 Btu.
- One ton-h of AC is 12,000 Btu

So, to store 1 ton-h you need:

$$(1 \text{ lb H}_2\text{O}/15 \text{ btu})(12,000 \text{ btu/ton-h})$$
$$800 \text{ lbs/ton-h}$$

Session 13.21



Sizing Chilled Water Storage Tanks

Assuming there are 8.34 lbs/gal. We have approximately:

$$(800 \text{ lbs H}_2\text{O/ton-hr})(1 \text{ gal H}_2\text{O}/8.34 \text{ lbs})$$
$$96 \sim 100 \text{ gals/ton hr @ 15 dT}$$

So our full storage system of 7000 ton-hr would be around 700,000 gallons

Or a tank 60' dia, and 30' high.

Session 13.22



Conditions That Favor TES

- High peak demand charges
- Low cost of energy used at night
- High on-peak loads
- Low AC loads at night
- Need for increased cooling system capacity



Session 13.23



CEM Review Problems

1. TES systems yield large energy savings.
A) True B) False
2. Distinguish between full and partial storage systems. Which would likely be better in new system design?
3. Why are utilities encouraging TES?

Session 4.3.24



4. Temperature stratification can occur in
- (A) Chilled water storage
 - (B) Hot water storage
 - (C) A & B
 - (D) None of the above
5. TES for heating uses some of the following storages: 1) building mass; 2) hot water; 3) ground couple; 4) compressed air tanks; 5) rocks; and 6) propane containers. Select the right combination:
- (A) 1,2,3,4
 - (B) 3,4,5,6
 - (C) 1,2,3,5
 - (D) 2,4,5,6

Session 4.3.25



6. With a load leveling TES strategy, a building manager will
- (A) Not operate the chiller during peak hours
 - (B) Essentially base load the chiller (i.e., operate at high load most of the time)
 - (C) Operate only during the peak times
 - (D) Operate in the "off" season
7. A large commercial building will be retrofitted with a closed loop water to air heat pump system. Individual department meters will meter costs to each department. Demand billing is a small part of the total electrical cost. Would you recommend a TES?
- (A) Yes
 - (B) No

Session 4.3.26



TES Appendix

Session 4.3.27



Another Storage Medium

- There is one more storage medium that is available, but it is almost never used. It is Eutectic Salt.
- Eutectic salt was used some in the 1970s and early 1980s for storage of heat, but its use for air conditioning is not common today. But, it could be used.

Session 4.3.28



- **Eutectic Salt Storage**

- Expensive, high tech solution
- Allows use of existing 44 degree F chillers
- Typical melt range is 41 to 47 degrees F
- Requires only 30 to 50% of the space needed for chilled water tanks
- Requires secondary heat exchanger
- May be considered hazardous
- The salt has a useful life of about five years, and must then be sent back and replaced

Session 4.3.29



Storage Capability of Eutectic Salts

- Eutectic salts use latent heat associated with freezing and melting, but one pound of solid eutectic salt absorbs only about 50 Btu to become liquid

Session 4.3.30



Summary of Storage Tank Sizing

These are real world, practical numbers – not for use on
CEM test

- Chilled water
15 to 18 cubic feet per ton-hr
- Eutectic salt
3.5 to 6 cubic feet per ton-h
- Ice
3 to 3.5 cubic feet per ton-h

Session 4.3.31

