

Using Water-Cooled Chillers to Exceed ASHRAE Minimum Efficiency Standard by > 50%

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At the end of this session you should be able to:

- Describe the difference in part load operation between centrifugal and screw chillers
- Explain how condenser water temperature impacts screw and centrifugal operation and efficiency
- Explain how a screw chiller in a seriescounterflow arrangement can provide savings over centrifugals in standard parallel arrangement



AGENDA WEATHER

Designing for 50% Energy Savings:

- Weather
 - Determinants of Chiller Energy Consumption
 - Variable Speed Screw Chillers
 - Chiller Plant Design

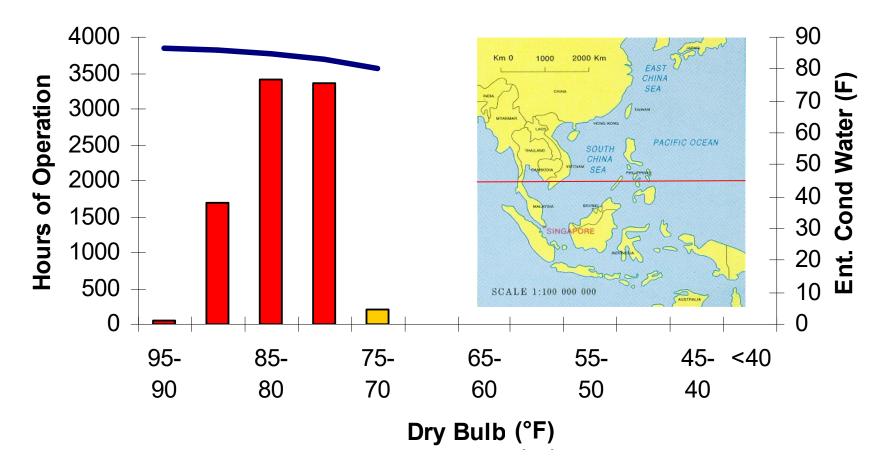
Results:

- Test Data
- Energy Analysis

Basic Building Blocks



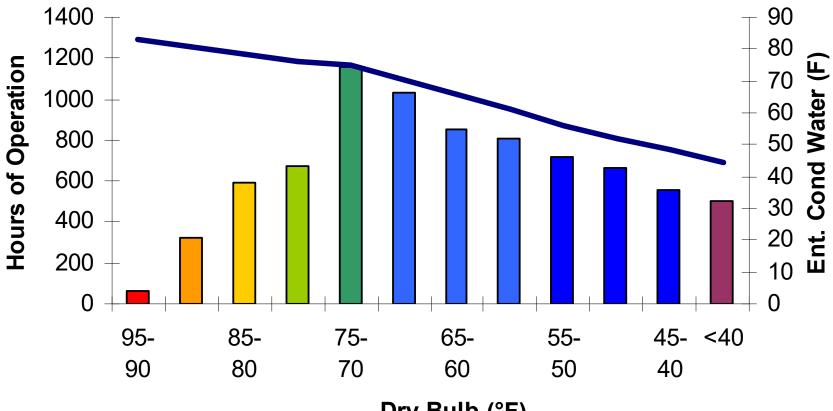
2% of hours have less than 80°F entering condenser water



Weather is a Key Design Factor



68% of hours have less than 70.4°F entering condenser water

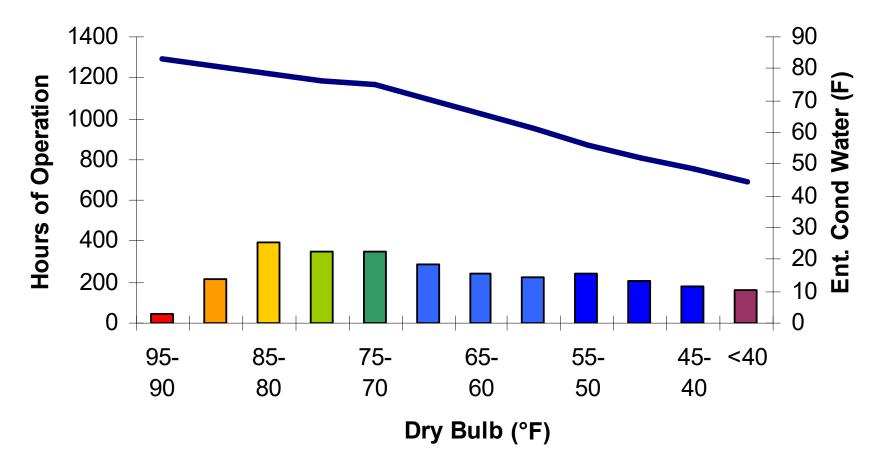


Dry Bulb (°F)

Weather Varies by Location



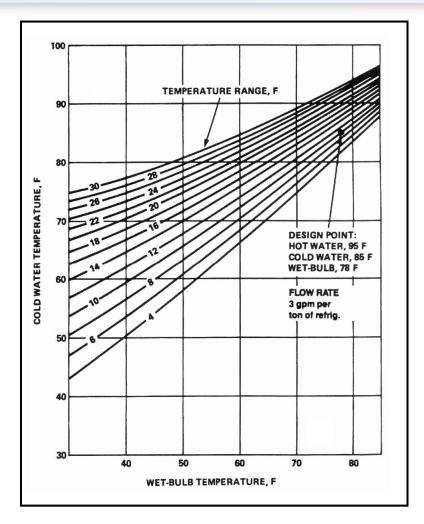
58% of hours have less than 70.4°F entering condenser water



Occupancy Schedule Relevant



COOLING TOWER



As outdoor wet bulb drops, approach rises slightly.

Approach = 7 + (78-WB)*0.08F

For constant flow: As chiller load falls, range (LCWT-ECWT) decreases causing temperature levels to fall to a closer approach.

At 78 WB, approach decreases by 0.5 to 1.0 F per 10% load.

Local Weather and Tower Dictate ECWT



AGENDA ENERGY

Designing for 50% Energy Savings:

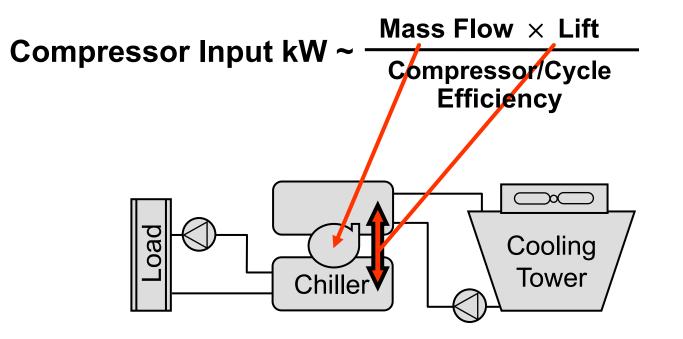
- Weather
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Like pumps, chiller energy consumption is a function of mass flow and differential pressure.

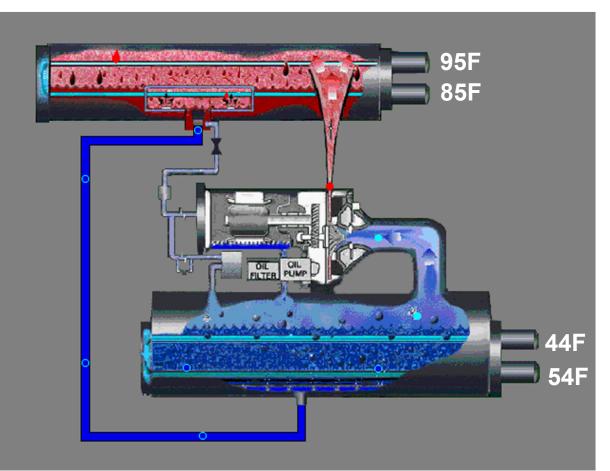


Chiller Energy (kW) ~ Tons x Lift



LIFT (PRESSURE DIFFERENCE)

Lift = SCT less SST



Saturated temperatures are surrogates for pressures

To condense, refrigerant must be warmer than the *leaving* condenser water.

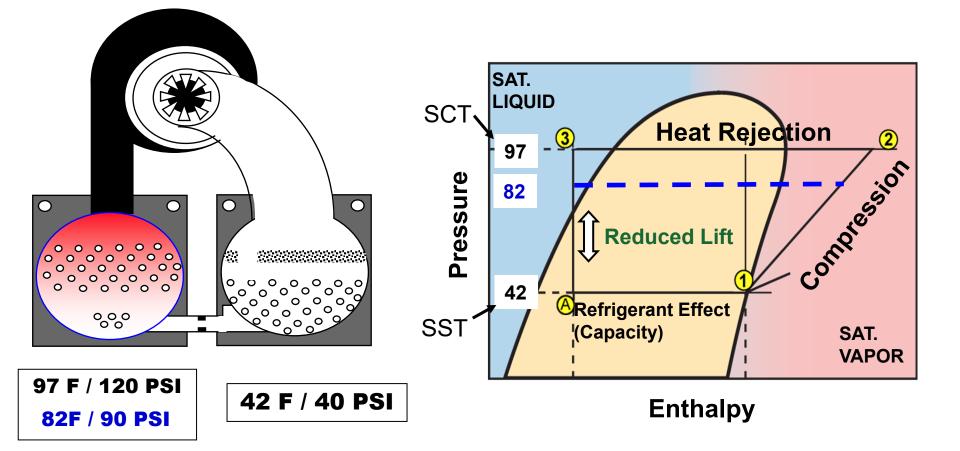
SCT = 95F + 2F approach = 97F

To boil, refrigerant must be colder than the *leaving* chilled water.

SST = 44F – 2F approach = 42F

Lift is Based on *Leaving* ...

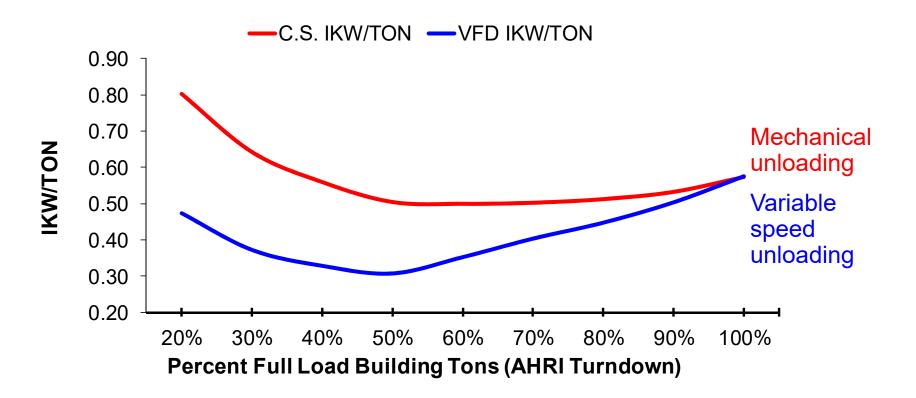
PRESSURE ENTHALPY CHART



Lower Lift = Less Work = Lower kW

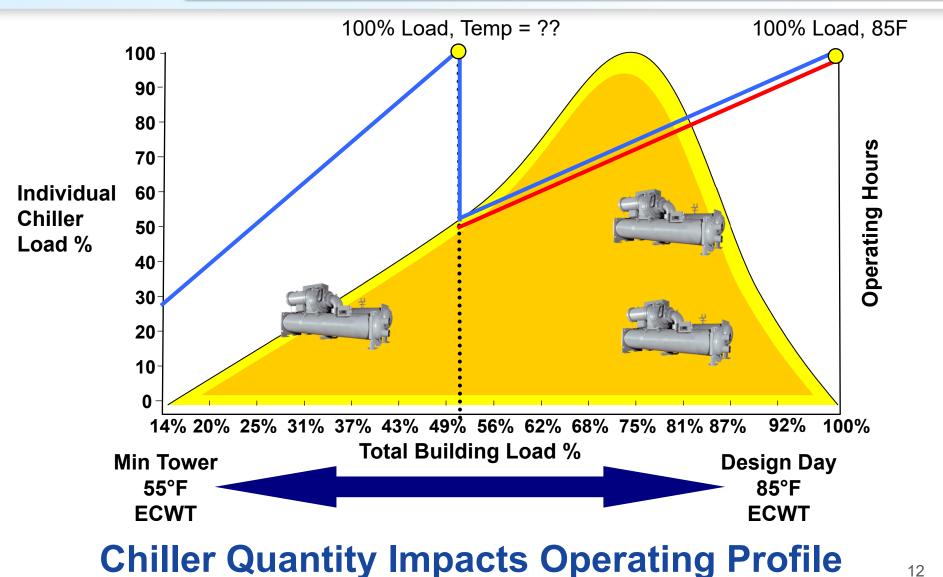


VFD Driven Chillers take advantage of lift and/or load reduction to reduce energy consumption.



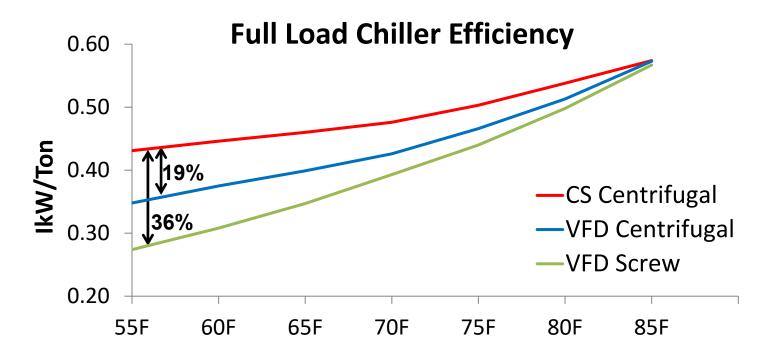
VFDs Do Less Lift (Work) More Efficiently

TWO CHILLER OPERATION / EFFICIENCY





VFD Driven Chillers take advantage of lift reduction to reduce energy consumption <u>even at full capacity</u>.



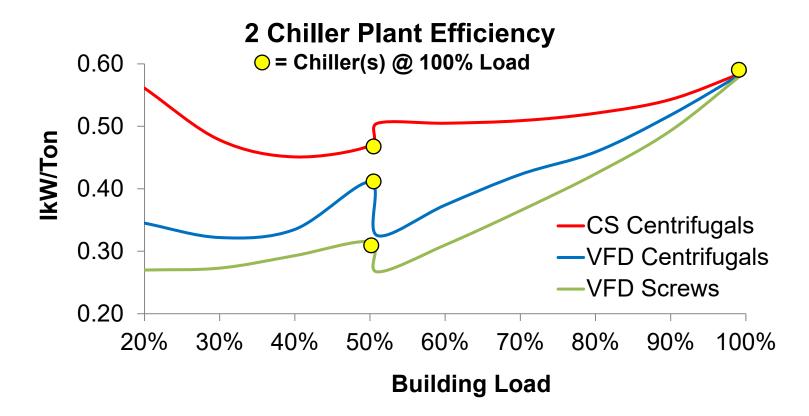
Entering Condenser Water Temperature (F)

Big VFDs Savings @ Full Capacity



EFFICIENCY

VFD Driven Chillers take advantage of lift and/or load reduction to reduce energy consumption.



Big FL VFD Savings in Real Plants



Designing for 50% Energy Savings:

- Weather
- Determinants of Chiller Energy Consumption
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TOWER FAN FAILURE

Actual Field Event – Cooling Tower Failure, Fayetteville, Arkansas, May 31, 2005





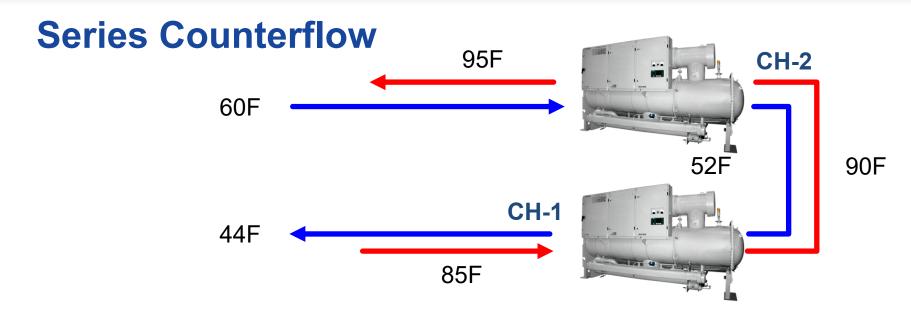
Designing for 50% Energy Savings:

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Upstream chiller (CH-2) cools 60F – 52F

Downstream chiller (CH-1) cools 52F - 44F

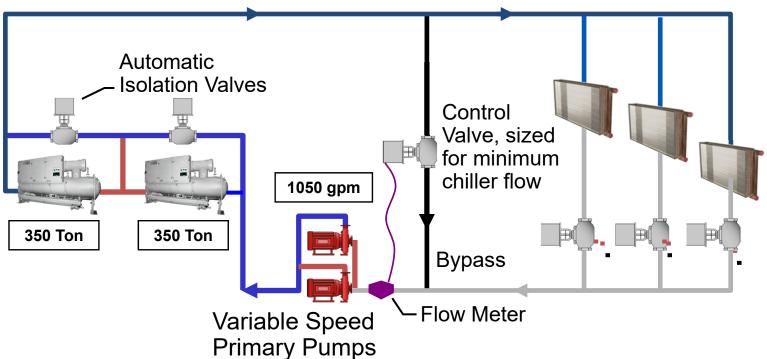
Evaporator water flows thru CH-2 and then CH-1

Condenser water flows thru CH-1 and then CH-2

One pass heat exchangers for water side pressure drop



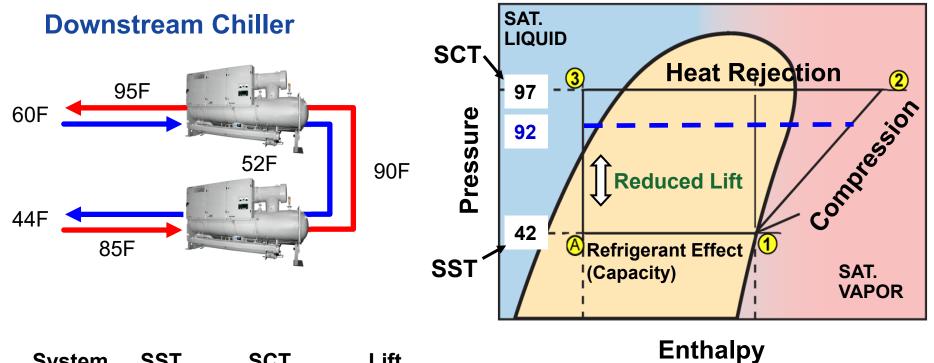




700 tons / 2 chillers = 350 tons per chiller

When building 100% loaded, entering condenser water = 85F

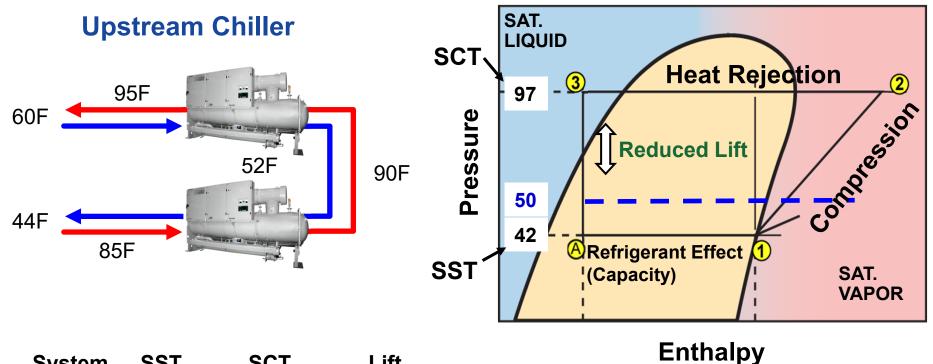
SERIES COUNTERFLOW REDUCES LIFT



<u>System</u>	221	361	LITT
Parallel	42	97	55
SCF	42	92	50

Lower Lift = Less Work = Lower kW

SERIES COUNTERFLOW REDUCES LIFT



<u>System</u>	551	SCI	Lift
Parallel	42	97	55
SCF	50	97	50
	00	51	00

Lower Lift = Less Work = Lower kW



Efficiency Benefit:

Series chiller efficiency improves with high return water temperatures because the upstream chiller produces warmer water.

System Design	Downstream	Upstream
44-54°F	44°F	49°F
44-56°F	44°F	50°F
44-58°F	44°F	51°F
44-60°F	44°F	52°F



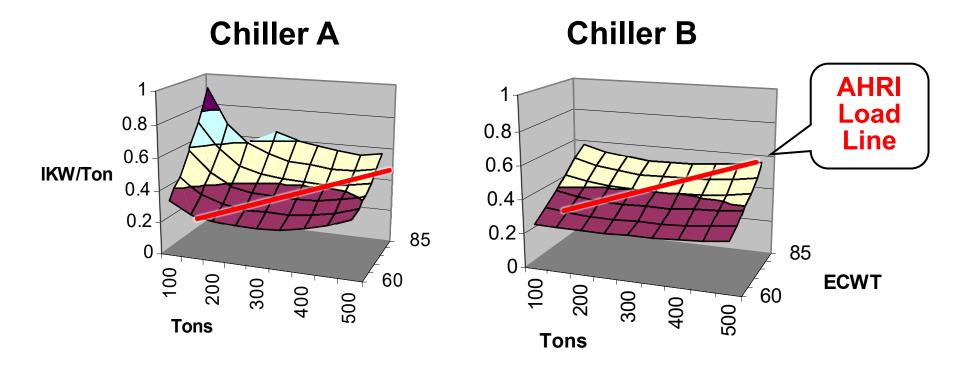
Efficiency Benefit:

Series chiller efficiency improves with variable flow because the upstream chiller leaving water temperature remains constant.

Var. Flow	100%	75%	50%	25%	
Return	60°F	60°F	60°F	60°F	
US Lvg	52°F	52°F	52°F	Off	
DS Lvg	44°F	44°F	44°F	44°F	
Const. Flow	100%	75%	50%	25%	
Return	60°F	56°F	52°F	48°F	
US Lvg	52°F	50°F	48°F	Off	
DS Lvg	44°F	44°F	44°F	44°F	



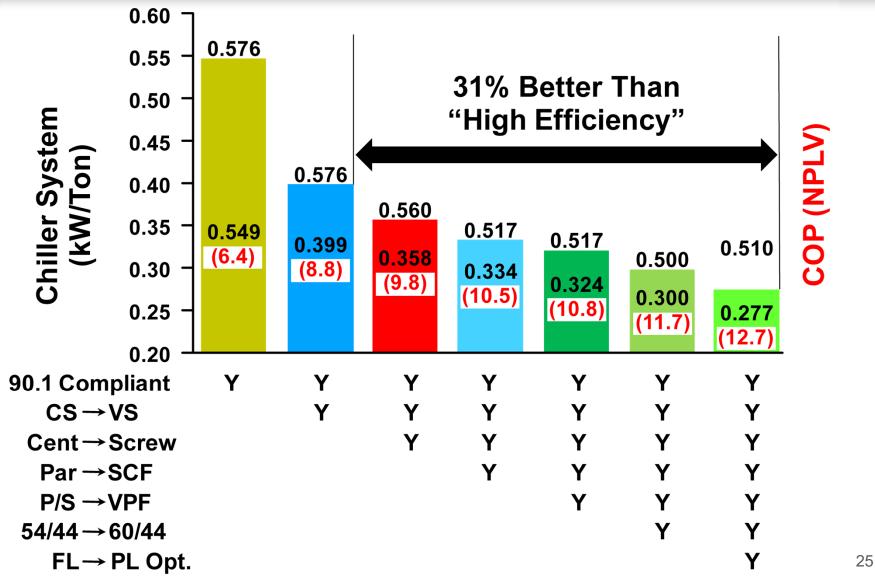
ARE ALL IPLV'S EQUAL?



If both machines were 0.35 IPLV (10.0 COP) would you consider them equal?

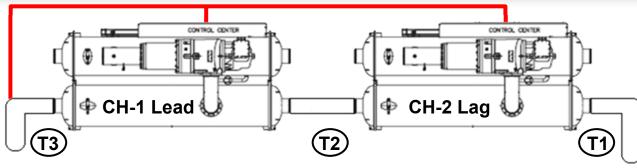


MOVING THE FRONTIER





CHILLER STAGING



Condenser piping not shown

Bypass piping, isolation valves not shown

CH-1 Programmed as lead chiller, CH-2 programmed as lag chiller.

CH-1 loads up until compressor speed indicates 2nd chiller appropriate

Based on compressor speed, CH-1 commands CH-2 to turn on. Since chillers are in series, pump flows are already established.

CH-1 and CH-2 operate together to regulate T3 to set point.

At appropriate compressor speed, CH-1 commands CH-2 to turn off.

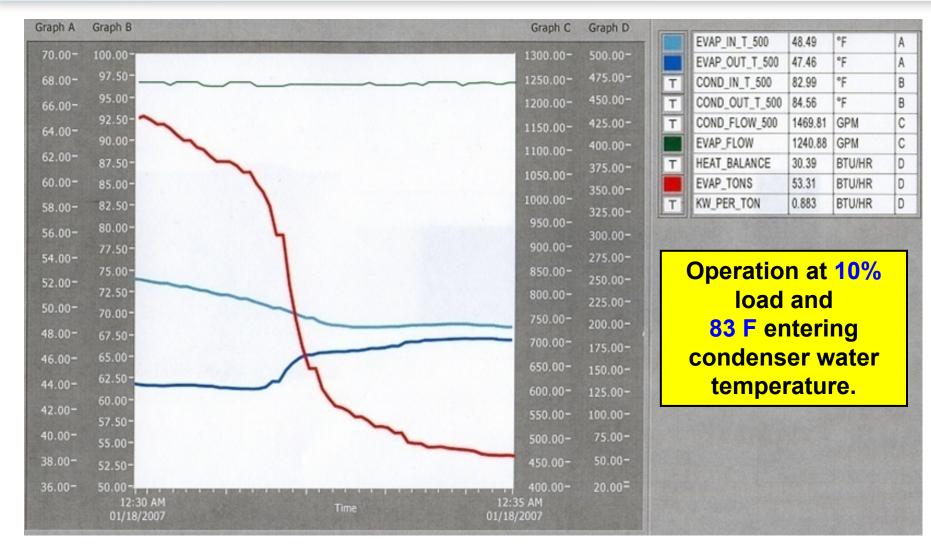
If CH-1 becomes disabled, the on board controls rotate CH-2 as the lead chiller.

CH-2 regulates to the leaving temperature T3 via the 32MP sensor.

No Human Intervention Required

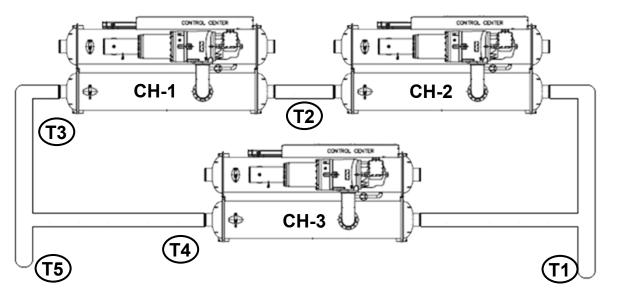


LOW LOAD OPERATION





SPARE CHILLER



Variable Flow Scenarios	T1	T 2	Т3	T4	T5
100% Load, CH-1 disabled	60	44	44	44	44
100% Load, CH-2 disabled	60	60	40	44	44

BSM enables LEAD/LAG Chillers (CH-1 and CH-2) as Required to meet T5.

LEAD chiller stages, optimizes both compressors for maximum operating efficiency.

If CH-1 disabled, CH-2 chiller becomes LEAD, CH-3 becomes LAG.

LEAD chiller stages, optimizes both compressors for maximum operating efficiency.

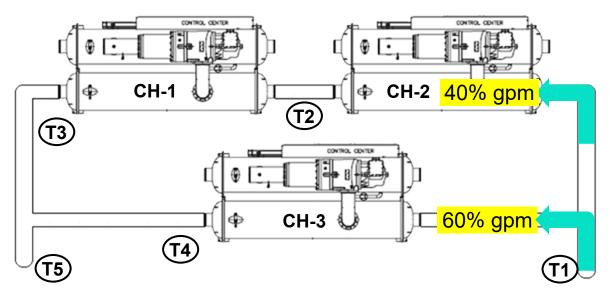
Condenser piping not shown.

Bypass piping, isolation valves not shown.

No Human Intervention Required



SPARE CHILLER



Variable Flow Scenarios	T1	T2	Т3	T4	T5
100% Load, CH-1 disabled	60	40	40	48	44
100% Load, CH-2 disabled	60	60	40	48	44

When standby chiller enabled, flow may not be balanced between branches.

LEAD chiller stages, optimizes both compressors for maximum operating efficiency.

If CH-1 disabled, CH-2 chiller becomes LEAD, CH-3 becomes LAG.

LEAD chiller stages, optimizes both compressors for maximum operating efficiency.

Leaving temps may vary, but T5 will be met.

Condenser piping not shown.

Bypass piping, isolation valves not shown.



SCHEDULE

			Entering	Leaving		Pressure				
	Capacity	Efficiency	Temp	Temp	Flow Rate	Drop				
Mode	(Tons)	(kW/Ton)	(F)	(F)	(gpm)	(Ft wg)	Passe	s Fluid	Туре	
Cooling	450	0.524	52	44	1350	6.2	1	Fresh	nwater	
Cooling	450	0.483	60	52	1350	6.1	1	Fresh	nwater	
Cooling	450	0.672	60	40	540	1.2	1	Fresh	nwater	
		Condens	er Data							
Entering	Leaving		Pressure							Operating
Temp	Temp	Flow Rate	Drop							Weight
(F)	(F)	(gpm)	(Ft wg)	Passes	Fluid Type	Voltage	MCA	MOCP	Manu.	(lbs)
85	89.6	2700	14.2	1	Freshwater	460-3-60	495	700		22849
89.6	94.2	2700	14.1	1	Freshwater	460-3-60	495	700		22849
85	94.5	1350	3.9	1	Freshwater	460-3-60	495	700		22849

Notes:

CH-1,CH-2, CH-3 shall utlize screw compressors

CH-1 and CH-2 shall operate in series counterflow.

CH-1 and CH-3 shall operate in parallel if CH-2 disabled.

CH-2 and CH-3 shall operate in parallel if CH-1 disabled.

CH-1, CH-2, CH-3 shall all be identical.

Chillers shall operate down to 40% of design evaporator flow (525 gpm).

VENDOR DATA SHEET

							Date:	20-Jul-11
						Chiller Model #		
	Load	LCHWT	Evap GPM	ECdWT	Cond GPM	lkW/Ton	Evap PD (ft)	Cond PD (ft)
(*)	100%	44F	1350	85F	2700			
(*)	100%	52F	1350	89.6F	2700			
	75%	44F	1013	75F	2700			
	75%	52F	1013	78.3F	2700			
(*)	50%	44F	675	65F	2700			
	50%	52F	675	67.2	2700			
	25%	44F	540	65F	1350			
(*)	100%	40F	540	85F	1350			
(*)	20%	40F	540	85F	1350			

Notes:

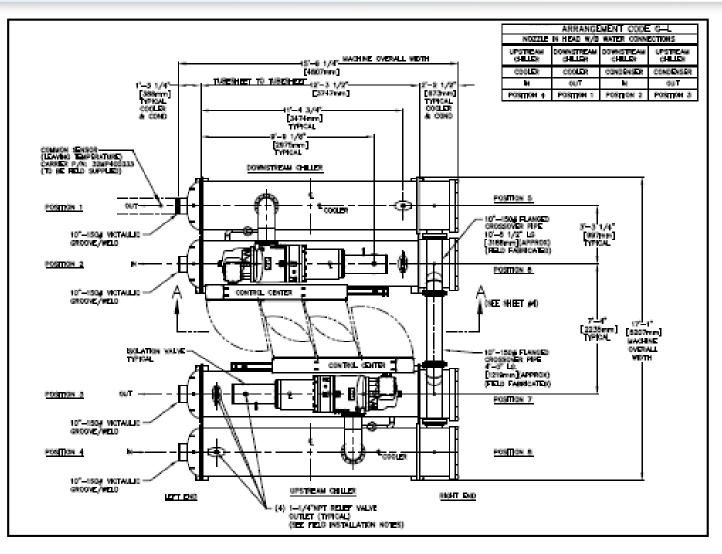
1.(*) Indicates factory test point per AHRI 550/590

2. Since all three chillers are the same, only one of the chillers will be tested

3.Chillers shall be operated for a period of two hour at each point during the factory test

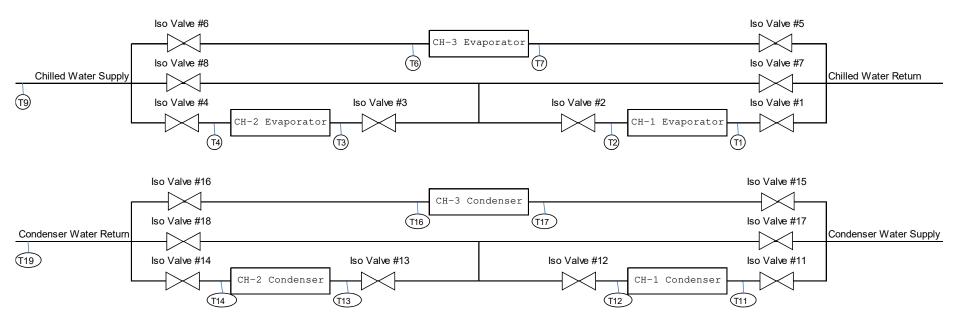


DRAWINGS





ISOLATION VALVES





ISOLATION VALVES

ls	Isolation Valve Table (Evaporator)						solation Va	lve Table	(Condense	ər)
CH-1	On	Off	On	Off		CH-1	On	Off	On	Off
CH-2	On	On	Off	Off		CH-2	On	On	Off	Off
CH-3	Off	On	On	Off		CH-3	Off	On	On	Off
#1	Open	Closed	Open	Closed		#11	Open	Closed	Open	Closed
#2	Open	Closed	Open	Closed		#12	Open	Closed	Open	Closed
#3	Open	Open	Closed	Closed		#13	Open	Open	Closed	Closed
#4	Open	Open	Closed	Closed		#14	Open	Open	Closed	Closed
#5	Closed	Open	Open	Closed		#15	Closed	Open	Open	Closed
#6	Closed	Open	Open	Closed		#16	Closed	Open	Open	Closed
#7	Closed	Open	Closed	Closed		#17	Closed	Open	Closed	Closed
#8	Closed	Closed	Open	Closed		#18	Closed	Closed	Open	Closed

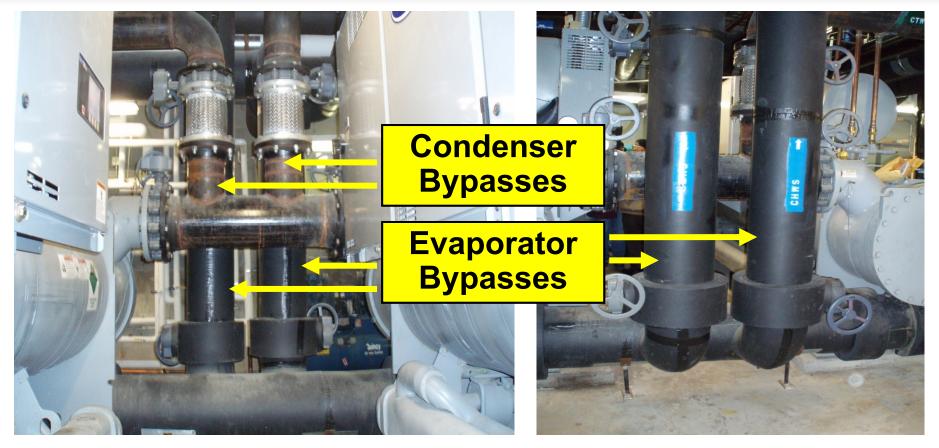


PHOTOS DRAWINGS





PHOTOS DRAWINGS



Inside View

Outside View

USING WATER-COOLED CHILLERS TO EXCEED ASHRAE MINIMUM EFFICIENCY STANDARD BY > 50%





