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# **Buckling Analysis of Simply-Supported Anisotropic Laminated Composite Plates**



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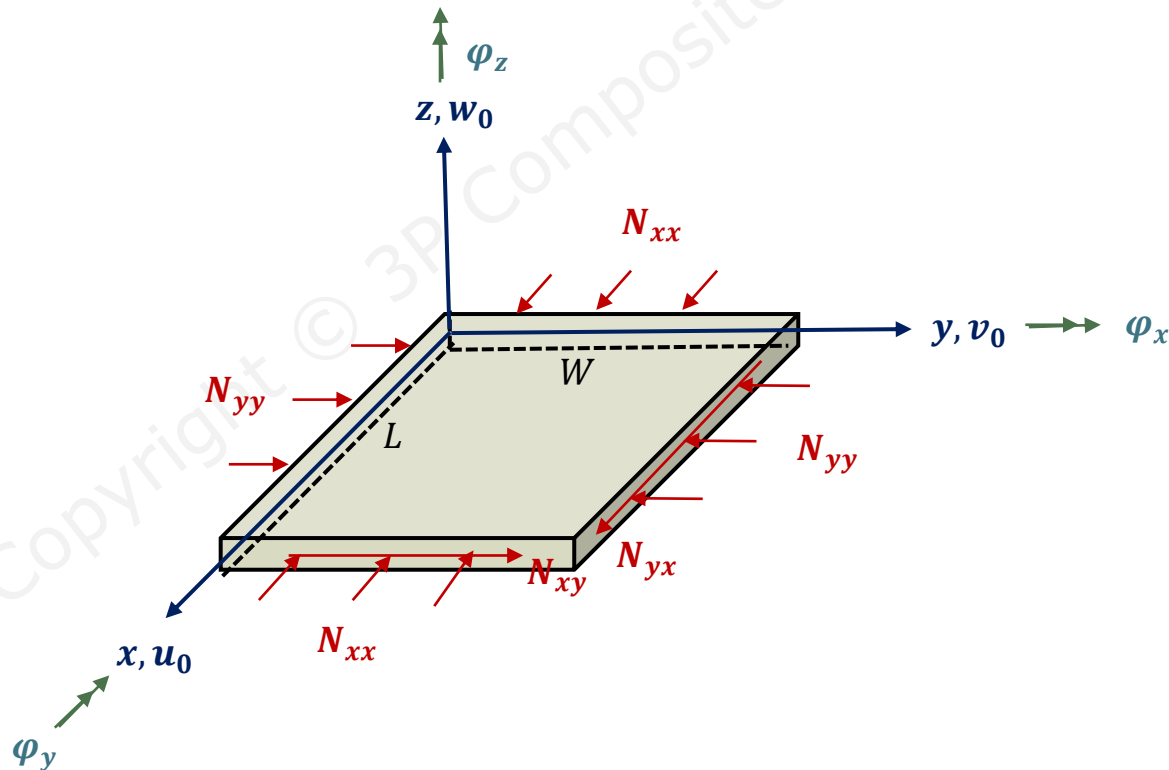


# Contents: 3pcsolver003

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# Overview

- ❖ **3pcsolver003** performs **linear** elastic buckling analysis of simply-supported **fully anisotropic** composite laminates subjected combined in-plane edge compression/tension and/or edge shear loads. The edge loads can also include the effects of uniform temperature and/or moisture expansion or contraction. Simply-supported boundary condition is most widely used in the analysis of plates and shells. Applied compressive bi-axial loads and positive shear load is shown below:



# Overview

❖ Following combinations of in-plane buckling loads can be analyzed using the solver **3pcsolver003**:

- ✓ Longitudinal Compression  $N_{xx}$
- ✓ Lateral Compression  $N_{yy}$
- ✓ In-plane Shear  $N_{xy}$
- ✓ Bi-axial Compression  $N_{xx}$  and  $N_{yy}$
- ✓ Longitudinal Compression  $N_{xx}$  and Lateral Tension  $N_{yy}$
- ✓ Longitudinal Tension  $N_{xx}$  and Lateral Compression  $N_{yy}$
- ✓ Longitudinal Compression  $N_{xx}$  and In-plane Shear  $N_{xy}$
- ✓ Longitudinal Tension  $N_{xx}$  and In-plane Shear  $N_{xy}$
- ✓ Lateral Compression  $N_{yy}$  and In-plane Shear  $N_{xy}$
- ✓ Lateral Tension  $N_{yy}$  and In-plane Shear  $N_{xy}$
- ✓ Longitudinal Compression  $N_{xx}$ , Lateral Tension  $N_{yy}$  and In-plane Shear  $N_{xy}$
- ✓ Longitudinal Tension  $N_{xx}$ , Lateral Compression  $N_{yy}$  and In-plane Shear  $N_{xy}$
- ✓ Longitudinal and Lateral Tensions  $N_{xx}$  &  $N_{yy}$  and In-plane Shear  $N_{xy}$
- ✓ Bi-axial Compression  $N_{xx}$  &  $N_{yy}$  and In-plane Shear  $N_{xy}$

# Applications

- ❖ Linear elastic buckling (or static stability) analysis performed by **3pcsolver003** solver is applicable to laminates built-up (or fabricated) from a LAMINA that
  - has any kind of FIBER such as boron, carbon, graphite, glass, Kevlar, Aramid, polyester, natural fibers, etc.,
  - is in any type of broad form such as short and long continuous, unidirectional, bi-directional 2D textile weaves like plain weave, twill and harness, biaxial and triaxial braids, chopped random fibers, non-crimp, nonwoven fabrics, etc.
  - Is impregnated with any RESIN/MATRIX, thermoset or thermoplastic systems such as epoxy, polyester, vinyl ester, polyurethane, phenolic, cyanate ester, bis-maleimide, polyimides, benzoxazine, Acrylic, ABS, Polylactic acid PLA, Polybenzimidazole PBI, Polyether sulfone PES, Polyoxymethylene POM, Polyether ether ketone PEEK, Polyetherimide PEI, Polyphenylene oxide PPO, Polyphenylene sulfide PPS, Polystyrene PS, Polypropylene PP, Polyvinyl chloride PVC, Teflon PTFE, etc.
  - is cured using any MANUFACTURING PROCESS such as Autoclave, Resin Transfer Molding like VARTM, SQRTM, RIM, SRIM, Filament Winding, Pultrusion, Compression Molding, Wet-lay up, etc.
- ❖ The analysis is equally applicable to Hybrid Laminates manufactured from a single or multiple types of lamina materials and/or ply broad forms or fiber types or single or multiple materials systems or their combinations

# Theoretical Background

- ❖ **3pcsolver003** solver is based on First-Order Shear Deformation Laminated Plate Theory (Mindlin Type). Spatial distributions of displacements  $u$ ,  $v$  and  $w$ , and rotations  $\varphi_x$  and  $\varphi_y$  of the plate's reference surface are assumed using double Fourier series satisfying the kinematic boundary conditions at all four simply-supported edges of the laminated plate exactly. Principle of virtual work and Ritz analysis procedure are used to obtain a **highly coupled** system of algebraic equations for linear elastic buckling analyses of **fully anisotropic laminated plate** (see below):

$$\begin{bmatrix} K_{11} & K_{12} & 0 & K_{14} & K_{15} \\ K_{21} & K_{22} & 0 & K_{24} & K_{25} \\ 0 & 0 & K_{33} + \lambda_{mn}F_{33} & K_{34} & K_{35} \\ K_{14} & K_{42} & K_{43} & K_{44} & K_{45} \\ K_{51} & K_{25} & K_{53} & K_{54} & K_{55} \end{bmatrix} \begin{Bmatrix} u_{mn} \\ v_{mn} \\ w_{mn} \\ \varphi_{xmn} \\ \varphi_{ymn} \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

- ❖ In the system of equations given above,  $K_{ij}$  are the stiffness terms containing the laminate  $A_{ij}$ ,  $B_{ij}$  and  $D_{ij}$ .  $u_{mn}$ ,  $v_{mn}$  and  $w_{mn}$  are the unknown coefficients of displacements, and  $\varphi_{xmn}$  and  $\varphi_{ymn}$  are the unknown coefficients of rotations of the laminated plate. It is assumed that the edge loads can be expressed as  $N_{xx}^0 = \lambda N_{xx}^0$ ,  $N_{yy}^0 = \lambda N_{yy}^0$  and  $N_{xy}^0 = \pm \lambda N_{xy}^0$  (where  $\lambda$  is the critical buckling factor).  $F_{33}$  term contain information about the total applied edge loads from mechanical and hydrothermal loadings



# Theoretical Background

- ❖ Hygrothermal effects can be accounted for the given difference in temperature  $\Delta T$  and difference in moisture content  $\Delta C$ . Laminated plate theory can be used to compute thermal force resultants  $N_{xx}^T$ ,  $N_{yy}^T$  &  $N_{xy}^T$  and/or moisture force resultants  $N_{xx}^M$ ,  $N_{yy}^M$  &  $N_{xy}^M$  as shown below

$$\begin{Bmatrix} N_{xx}^{T \text{ or } M} \\ N_{yy}^{T \text{ or } M} \\ N_{xy}^{T \text{ or } M} \end{Bmatrix} = [A'] \begin{Bmatrix} \varepsilon_{xx}^{T \text{ or } M} \\ \varepsilon_{yy}^{T \text{ or } M} \\ \gamma_{xy}^{T \text{ or } M} \end{Bmatrix} \quad A' = -BD^{-1}B^T + A$$

- ❖ The total applied in-plane edge (or buckling) loads are then obtained as summation of mechanical and hydrothermal force resultants as

$$\begin{Bmatrix} N_{xx}^{Total} \\ N_{yy}^{Total} \\ N_{xy}^{Total} \end{Bmatrix} = \begin{Bmatrix} N_{xx}^T \\ N_{yy}^T \\ N_{xy}^T \end{Bmatrix} + \begin{Bmatrix} N_{xx}^M \\ N_{yy}^M \\ N_{xy}^M \end{Bmatrix} + \begin{Bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{Bmatrix}$$

# Theoretical Background

- ❖ The determinant of the system of  $5M \times 5N$  equations for the Eigen-value problem derived above is set to zero to obtain buckling load factors  $\lambda_{mn}$  for a simply-supported fully anisotropic plate subjected to various combinations of applied compression/tension and/or edge shear loads

$$\begin{vmatrix} K_{11} & K_{12} & 0 & K_{14} & K_{15} \\ K_{21} & K_{22} & 0 & K_{24} & K_{25} \\ 0 & 0 & K_{33} + \lambda_{mn}F_{33} & K_{34} & K_{35} \\ K_{14} & K_{42} & K_{43} & K_{44} & K_{45} \\ K_{51} & K_{25} & K_{53} & K_{54} & K_{55} \end{vmatrix} = \{0\}$$

- ❖ Solution to the Eigen-value problem is obtained for truncated Fourier series using  $m = 1, 2, \dots, M$  terms in the  $x$ -direction and  $n = 1, 2, \dots, N$  terms in the  $y$ -direction. Without loss of generality,  $M = N$  is assumed for the solution. The buckling mode shapes for each buckling factor  $\lambda_{mn}$  can be obtained by substituting the  $\lambda_{mn}$  in the system of equations given on the previous slide
- ❖ Given the lamina/ply material properties, laminate stack-up and its length and width dimensions, **3pcsolver003** solver calculates buckling load factors for an anisotropic laminated plate subjected to any combination of the load described on Slide 5

# Theoretical Background

- ❖ The **3pcsolver003** is a unique solver which is based on FSDT of laminated plates, employs a closed-form Ritz solution procedure, considers the fully anisotropic laminate effects, and obtain buckling load factors for combined edge compression/tension and shear loads. All types of laminate coupling terms represented by the non-zero  $A_{i6}$ ,  $B_{ij}$  and  $D_{i6}$  ( $i = 1,2$ , and  $j = 1,2,6$ ) are included in the buckling analysis of laminated composite plates. Most closed-form analyses neglect these coupling effects due to the complexity in deriving the system of equations, and hence, assume the laminated plates as being specially orthotropic (i.e.  $A_{i6} = B_{ij} = D_{i6} = 0, i = 1,2, j = 1,2,6$ )
- ❖ Numerous examples are solved using **3pcsolver003**, and results are compared with those (i) obtained from standard commercially available finite element analysis software, and (ii) available in open literature. Many types of material systems, ply orientations, laminate stack ups, laminate dimensions, and types of buckling loads are considered to check the accuracy of the solver. Excellent correlations are obtained in all cases. Numerical examples highlight the adverse effects of various types of laminate stiffness couplings on buckling of anisotropic laminated composite plates
- ❖ Details of the theoretical approach along with numerous verification and application examples are available in the training module **3pcmodule003**

# Inputs





- ❖ All inputs should be in consistent units. Use either (N, m, Pa) OR (N, mm, MPa) or (lbs, in, Psi) consistently. Inputs in scientific notation (0.0+e) are acceptable
  
- ❖ Input process is intuitive and uses the following logical order:
  - Materials
  - Plies / Laminae
  - Laminates
  - Panels
  - Loads
  - Analysis Options

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# Inputs: Materials

## ❖ Material Properties:

In the SI system, MPa and mm or Pa and m, and in the US system Msi and in are used to input the orthotropic lamina Moduli  $E_1$ ,  $E_2$ ,  $G_{12}$ ,  $G_{13}$  and  $G_{23}$ .  $\nu_{12}$  is major Poison's ratio. Multiple lamina types and lamina materials can be input by simply clicking the '+' sign on the extreme right. Based on the type of analyses selected, the required material inputs for an orthotropic Lamina can vary as shown below:

Materials    

ID	$E_1$	$E_2$	$G_{12}$	$G_{23}$	$G_{13}$	$\nu_{12}$			
1	0	0	0	0	0	0	+	-	
<b>MUST BE +VE REAL MUST BE +VE REAL MUST BE +VE REAL MUST BE +VE REAL MUST BE +VE REAL MUST BE +VE REAL</b>									

Copyright ©

# Inputs: Plies

## ❖ Plies/Laminae:

Types of plies in a laminate are required as input. Each ply type is defined by its angle (or orientation) in degrees, material type and the thickness. Material of a ply/lamina can be selected from a predefined list of materials that are input in the Material Properties Section above. The thickness of the ply or lamina is in mm or m in the SI system or inch in the US system. Multiple ply or lamina types can be input by simply clicking the '+' sign on the extreme right. The required ply/lamina type inputs with few examples are shown below:

Plies    

ID	Angle (deg)	Material	Thickness		
1	0	Uni ▾	0.005	+	-
2	45	PW ▾	0.010	+	-
3	90	Uni ▾	0.005	+	-

Plies    

ID	Angle (deg)	Material	Thickness		
1	0	CEP ▾	0.005	+	-
2	30	Flax ▾	0.010	+	-
3	60	CEP ▾	0.005	+	-





Plies    

ID	Angle (deg)	Material	Thickness		
1	0	CEP ▾	0.005	+	-
2	45	CEP ▾	0.005	+	-
3	-45	CEP ▾	0.005	+	-
4	90	CEP ▾	0.005	+	-
5	0	Flax ▾	0.01	+	-

# Inputs: Laminates

## ❖ Laminates:

Multiple laminates can be quickly created by defining their stacking sequences using the plies defined in the previous step. Laminate Offset is fixed at middle (default). Hybrid laminates can be defined using different ply and material combinations established in the previous steps. Additional laminates can be added by simply clicking the '+' sign on the extreme right. A few examples of laminates and their inputs are shown below:

Laminates    

ID	Stacking Sequence	Stacking Sequence (Angle)	Offset		
CEP-QI	2,3,1,4,4,1,3,2	45, -45, 0, 90, 90, 0, -45	Middle ▾	+	-
CEP-Cross Ply	1,4,1,4,1,4,1,4	0, 90, 0, 90, 0, 90, 0, 90	Bottom ▾	+	-
CEP-Angle Ply	2,3,2,3,2,3,2,3	45, -45, 45, -45, 45, -45	Top ▾	+	-
CEP-Flax Hybrid	1,2,3,4,5	0, 45, -45, 90, 0	Middle ▾	+	-

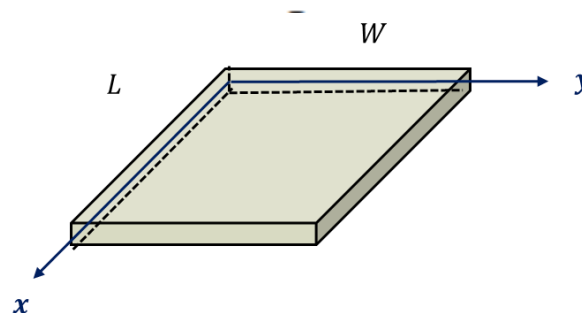
# Inputs: Panels

## ❖ Panels:

Panels are easily created by using the predefined laminates, and by providing the length  $L$  and width  $W$  of the plate as shown below. Additional panels can be added by simply clicking the '+' sign on the extreme right (see below):

Panels ⓘ ⬆ ⬇

ID	Length	Width	Laminate		
1	10	10	1 ▾	+	-
2	10	10	2 ▾	+	-
3	10	10	3 ▾	+	-
4	10	10	4 ▾	+	-

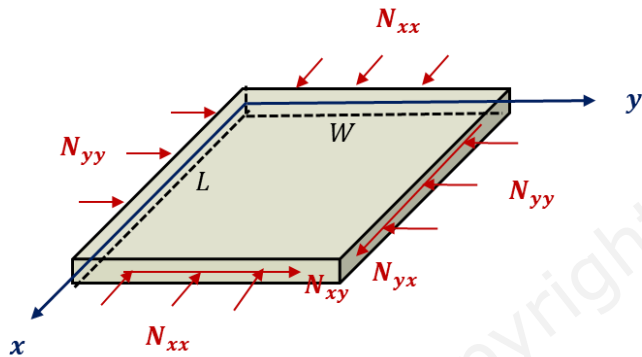




# Inputs: Loads

## ❖ Loads:

As mentioned earlier (see overview Section), various combinations of compressive/tensile and/or shear edge loads  $N_{xx}$ ,  $N_{yy}$  and  $N_{xy}$  (force per unit length) can be applied to the panels (see figures below).



Loads ⓘ ⬆ ⬇

ID	Panel	$N_{xx}$	$N_{yy}$	$N_{xy}$	+	-
1	1	-1	0	0	+	-
2	2	-1	0	0	+	-
3	1	-1	-0.473	0	+	-
4	2	-1	-0.473	0	+	-
5	1	-1	15.6563	0	+	-
6	2	-1	15.6563	0	+	-
7	1	0	-1	0	+	-
8	2	0	-1	0	+	-

# Inputs: Analysis Options

## ❖ Analysis Options:

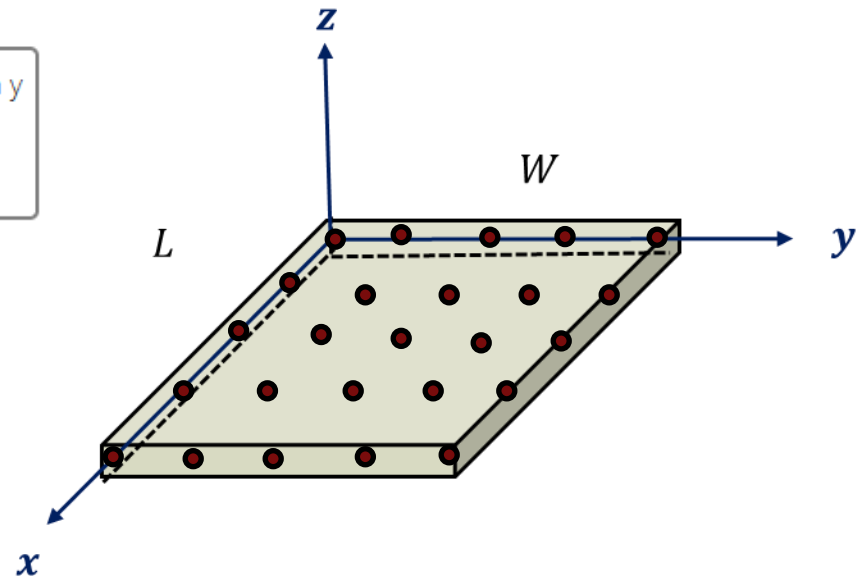
User has the option to define the number of terms in Fourier series solution of the solver. By default,  $M = N = 8$  is assumed.  $M = N$  can be varied from 2 to 21.

Output quantities from the analysis can be requested at select number of points (a.k.a grid points) in the plate domain. By default, a  $5 \times 5$  grid is assumed within the domain of the plate bounded by  $0 \leq x \leq L$  and  $0 \leq y \leq W$  to output the analysis solution at 25 equally divided grid points (see below)

Default analysis options are also shown below:

Analysis Options

Number of Terms	Number of Points in x	Number of Points in y
<input type="text" value="8"/>	<input type="text" value="5"/>	<input type="text" value="5"/>





# Outputs

## ❖ Analysis Outputs:

Following information is output for each Load Case:

- Panel Geometry
- Terms in Fourier Series solution
- Number of Grid Points selected to get output information
- Material Properties and Laminate Information
- Laminate [A], [B], [D] stiffness matrices
- First Five (or lowest five) critical buckling loads
- Grid Points coordinates  $x$  and  $y$ , and transverse displacements  $w$  for first five critical buckling loads

Note that all output is consistent with the unit system used during the material, lamina, laminate, and loads Inputs.

- Laminate [A] stiffness matrices N/m or N/mm or lb/in
- Laminate [B] stiffness matrices N-m/m or N-mm/mm or lb-in/in
- Laminate [D] stiffness matrices N-m or N-mm or lb-in
- Displacements in mm, m or in and Rotations in 1/mm, 1/m or 1/in
- Critical Plate Force resultants in N/m, N/mm or lb/in

A typical output is shown below:

# Output Text

3pc-solver003, v1.1b1

LOADS ID PANEL ID  
1 1

PANEL GEOMETRY  
LENGTH: 10.00  
WIDTH : 10.00

ANALYSIS OPTIONS  
m = 8  
n = 8

OUTPUT OPTIONS  
NUMBER OF POINTS IN X DIR: 5  
NUMBER OF POINTS IN Y DIR: 5

MATERIAL PROPERTIES

ID	E1	E2	G12	G23	G13	v12
aiaa-2009	1.80e+07	1.60e+06	8.70e+05	6.40e+05	8.70e+05	0.3000

LAMINATE GEOMETRY  
STACKING SEQUENCE (PLY ANG): [+45.0 , -45.0 , +45.0 , -45.0 ]  
STACKING SEQUENCE (PLY MAT): [aiaa-2009 , aiaa-2009 , aiaa-2009 , aiaa-2009 ]  
TOTAL THICKNESS: 0.0210  
OFFSET: 0.0000

LAMINATE PROPERTIES  
A MATRIX

+127080.48	+90540.48	+0.00
+90540.48	+127080.48	+0.00
+0.00	+0.00	+98649.19

A MATRIX - TRANSVERSE SHEAR

+15855.00	+0.00
+0.00	+15855.00



# Output Text

## B MATRIX

+0.00	+0.00	-227.84
+0.00	+0.00	-227.84
-227.84	-227.84	+0.00

## D MATRIX

+4.67	+3.33	+0.00
+3.33	+4.67	+0.00
+0.00	+0.00	+3.63

## LAMINATE INPLANE AND FLEXURAL ENGINEERING CONSTANTS

Ex	Ey	Gxy	vxy	vyx	Efx	Efy	Gfxy	vfyx	vfyx	
+2.98e+06	+2.98e+06	+4.70e+06	+0.7125	+0.7125	+0.7125	+0.7125	+2.98e+06	+2.98e+06	+4.70e+06	+0.7125
+0.7125										

## APPLIED LOADS

NUMBER	NXX	NYY	NXY	NXXCR	NYYCR	NXYCR
1	-1.0000	0.0000	0.0000	-22.1208	0.0000	0.0000
2	-1.0000	0.0000	0.0000	-22.5359	0.0000	0.0000
3	-1.0000	0.0000	0.0000	-26.2087	0.0000	0.0000
4	-1.0000	0.0000	0.0000	-31.7513	0.0000	0.0000
5	-1.0000	0.0000	0.0000	-36.0108	0.0000	0.0000

## CRITICAL LOADS





# Inputs and Outputs: Consistent Units

Quantity	SI System 1	SI system 2	US System
$E_1, E_2, G_{12}, G_{13}, G_{23}$ $E_x, E_y, G_{xy}, E_{fx}, E_{fy}, G_{fxy}$	MPa (N/mm <sup>2</sup> )	Pa (N/m <sup>2</sup> )	Psi (lb/in <sup>2</sup> )
$\alpha_1, \alpha_2, \alpha_x, \alpha_y, \alpha_{xy}$	mm/mm/°C	m/m/°C	in/in/°F
$\beta_1, \beta_2, \beta_x, \beta_y, \beta_{xy}$	mm/mm/Kg/Kg	m/m/Kg/Kg	in/in/lb/lb
$\sigma_{11}^T, \sigma_{11}^C, \sigma_{22}^T, \sigma_{22}^C, \tau_{12}^S, \sigma_1, \sigma_2, \tau_{12}, \tau_{23}, \tau_{13},$ $\sigma_x, \sigma_y, \tau_{xy}, \tau_{yz}, \tau_{xz}$	MPa (N/mm <sup>2</sup> )	Pa (N/m <sup>2</sup> )	Psi (lb/in <sup>2</sup> )
$\varepsilon_{11}^T, \varepsilon_{11}^C, \varepsilon_{22}^T, \varepsilon_{22}^C, \gamma_{12}, \varepsilon_1, \varepsilon_2, \gamma_{12}, \gamma_{13}, \gamma_{23}, \varepsilon_{x0},$ $\varepsilon_{y0}, \gamma_{xy0}, \gamma_{yz0}, \gamma_{xz0}, \varepsilon_x, \varepsilon_y, \gamma_{xy}, \gamma_{yz}, \gamma_{xz}$	mm/mm	m/m	in/in
$K_{x0}, K_{y0}, K_{xy0}$	1/mm	1/m	1/in
$N_{xx}, N_{yy}, N_{xy}, N_{xx}^T, N_{yy}^T, N_{xy}^T,$ $N_{xx}^H, N_{yy}^H, N_{xy}^H, [A]$	N/mm	N/m	lb/in
[B]	N – mm/mm	N – m/m	lb – in/in
[D]	N – mm	N – m	lb – in
$\Delta T$	°C	°C	°F
$\Delta C$	Kg/Kg	Kg/Kg	lb/lb
Ply Angle, $\theta$	Degree	Degree	Degree
Ply or Laminate thickness or Offset or $w$	mm	m	in








# Other Features

## ❖ Upload/Download:

Users can upload and download Material properties, Plies, Laminates, Panels and Loads data files (\*.json) using the upload  and download  buttons next to these inputs.

## ❖ Additional Output:

Users can review a few intermediate calculations such as minor Poison's ratios  $\nu_{21}$ ,  $Q_{ij}$  for each ply type and laminate ABD by using the calculation button . Few such examples are shown below:

		Materials    							
	$E_1$	$E_2$	$G_{12}$	$G_{23}$	$G_{13}$	$\nu_{12}$	$\alpha_1$	$\alpha_2$	$\beta_1$
ID		$\nu_{21}$							
GMS4020 PW		0.05							
GMS4020 Tape		0.0254							
2024-T3		0.3							
Rastogi_Fiberglass		0.02667							
Tuttle		0.01662							

# Other Features

## Plies

ID	Angle (deg)	Material	Thickness		
1	0	Tuttle	0.0075	+	-
2	90	Tuttle	0.0075	+	-

ID	Q	Q44	Q55	Qbar	Q44bar	Q45bar	Q55bar
1	[[22627882.74, 376125.7, 0.0], [376125.7, 1106252.04, 0.0], [0.0, 0.0, 640000.0]]	640000	640000	[[22627882.74, 376125.7, 0.0], [376125.7, 1106252.04, 0.0], [0.0, 0.0, 640000.0]]	640000	0	640000
2	[[22627882.74, 376125.7, 0.0], [376125.7, 1106252.04, 0.0], [0.0, 0.0, 640000.0]]	640000	640000	[[1106252.04, 376125.7, 0.0], [376125.7, 22627882.74, 0.0], [0.0, 0.0, 640000.0]]	640000	0	640000
3	[[22627882.74, 376125.7, 0.0], [376125.7, 1106252.04, 0.0], [0.0, 0.0, 640000.0]]	640000	640000	[[6761596.54, 5481596.54, 5380407.67], [5481596.54, 6761596.54, 5380407.67], [5380407.67, 5380407.67, 5745470.85]]	640000	0	640000
4	[[22627882.74, 376125.7, 0.0], [376125.7, 1106252.04, 0.0], [0.0, 0.0, 640000.0]]	640000	640000	[[6761596.54, 5481596.54, -5380407.67], [5481596.54, 6761596.54, -5380407.67], [-5380407.67, -5380407.67, 5745470.85]]	640000	0	640000

## Laminates

ID	Stacking Sequence	Stacking Sequence (Angle)	Offset		
1	1,1,1,1,1,1,1	0,0,0,0,0,0,0	Middle	+	-
2	1,2,1,2,1,2,1	0,90,0,90,90,0,90,0	Middle	+	-
3	3,4,3,4,4,3,4,3	45,-45,45,-45,-45,45	Middle	+	-

ID	Thickness	A	B	D	A44	A45	A55
1	0.06	[[1357672.96, 22567.54, 0.0], [22567.54, 66375.12, 0.0], [0.0, 0.0, 38400.0]]	[[0.0, 0.0, 0.0], [0.0, -0.0, 0.0], [0.0, 0.0, 0.0]]	[[407.3, 6.77, 0.0], [6.77, 19.91, 0.0], [0.0, 0.0, 11.52]]	38400	0	38400
2	0.06	[[712024.04, 22567.54, 0.0], [22567.54, 712024.04, 0.0], [0.0, 0.0, 38400.0]]	[[0.0, 0.0, 0.0], [0.0, -0.0, 0.0], [0.0, 0.0, 0.0]]	[[286.24, 6.77, 0.0], [6.77, 140.97, 0.0], [0.0, 0.0, 11.52]]	38400	0	38400
3	0.06	[[405695.79, 328895.79, 0.0], [328895.79, 405695.79, 0.0], [0.0, 0.0, 344728.25]]	[[0.0, -0.0, 0.0], [-0.0, -0.0, 0.0], [0.0, 0.0, -0.0]]	[[121.71, 98.67, 36.32], [98.67, 121.71, 36.32], [36.32, 36.32, 103.42]]	38400	0	38400



# General Information

- ❖ Subscription fee to access **3pcsolver003** is \$39/year per for a single-login license
- ❖ Training module **3pcmodule003** supports the solver **3pcsolver003**. Users' can buy the training module **3pcmodule001** online at  
  
<https://www.3pcomposites.com/>
- ❖ 3P Composites, LLC can conduct online or in-class trainings for the **3pcsolver003** and **3pcmodule003**. The training can be adapted to meet the requirements of individual needs and/or industrial applications
- ❖ For questions, issues, comments, suggestions, trainings, please contact us at [3pcomps@gmail.com](mailto:3pcomps@gmail.com). Your feedback is appreciated in helping us continuously improve the product



# Examples: Buckling of Laminated Plates

- ❖ Lamina Properties:  $E_1 = 1.8e7 \text{ psi}$ ,  $E_2 = 1.6e6 \text{ psi}$ ;  $G_{12} = G_{13} = 8.7e5 \text{ psi}$ ,  $G_{23} = 6.4e5 \text{ psi}$ ;  $\nu_{21} = 0.3$ ,  $t_{ply} = 0.00525 \text{ inch}$ ,  $\rho = 1.49 \times 10^{-4} \text{ lb/in}^3$
- ❖ Plate Dimensions:  $L = 15 \text{ in.}$ ,  $W = 10 \text{ in.}$ , Aspect Ratio  $\frac{L}{W} = 1.5$
- ❖ Laminate I:  $[\pm 45]_{2S}$ ,  $D_{16} \neq D_{26} \neq 0$ 
  - I: Applied Axial Edge Compression  $N_{xx} = -10 \text{ lb/in}$
  - II: Applied Bi-Axial Edge Compression  $N_{xx} = -10 \text{ lb/in}$  and  $N_{yy} = -5 \text{ lb/in}$
  - III: Applied Edge Shear  $N_{xy} = 10 \text{ lb/in}$
  - IV: Applied Edge Shear  $N_{xy} = -10 \text{ lb/in}$
  - V: Applied Edge Compression, Tension and Shear  $N_{xx} = -10 \text{ lb/in}$  and  $N_{yy} = +5 \text{ lb/in}$ ,  $N_{xy} = -10 \text{ lb/in}$

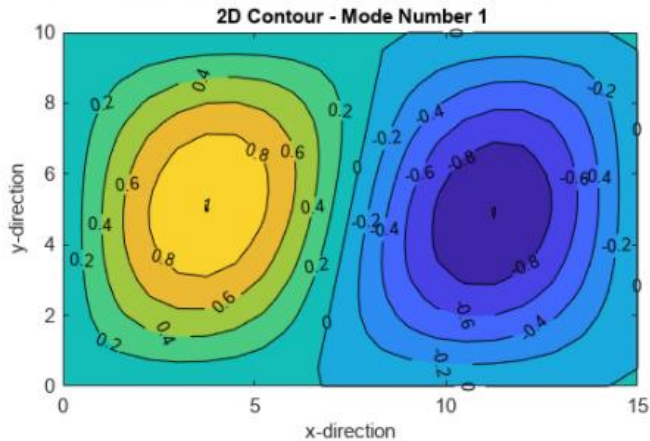
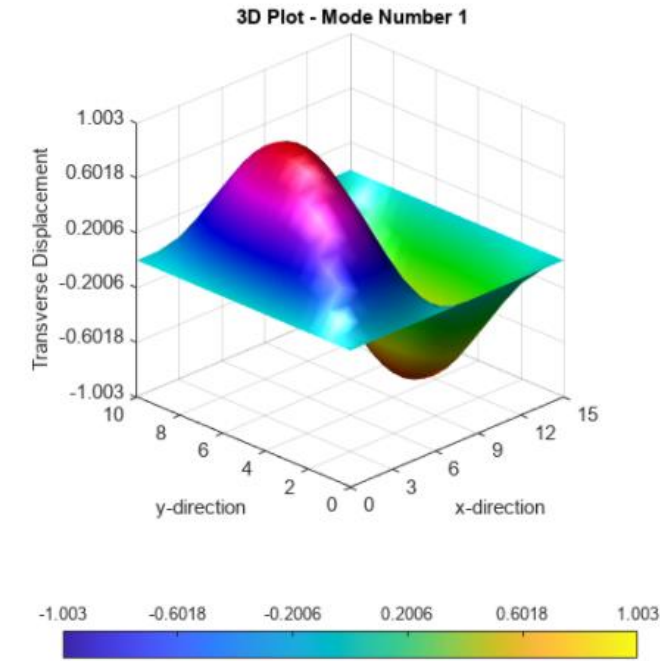
Buckling Load Factor	CASE				
	I	II	III	IV	V
1	2.41	1.208	4.835	2.967	1.839
2	2.564	1.856	5.363	3.246	1.999
3	3.062	2.683	9.839	5.767	2.649
4	4.085	3.538	10.74	6.292	2.978
5	5.428	3.816	16.63	9.991	4.099

# Buckling of Laminated Plates

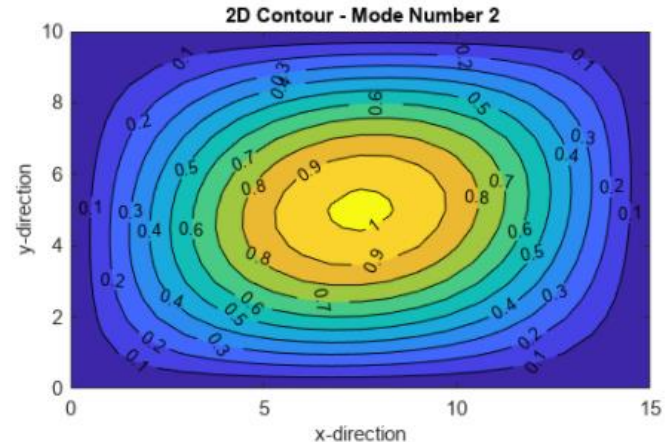
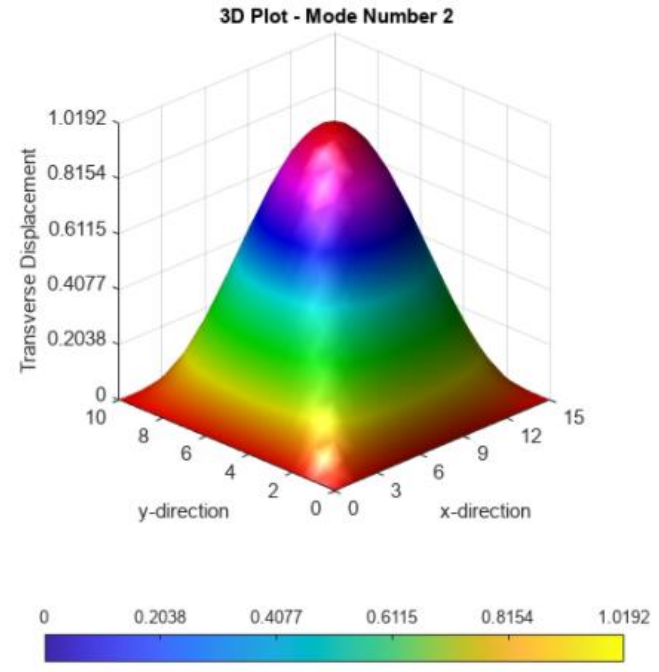
- ❖ Laminate II:  $[\pm 45]_T, B_{16} \neq B_{26} \neq 0$ 
  - I: Applied Axial Edge Compression  $N_{xx} = -1$  lb/in
  - II: Applied Edge Tension, Compression and Shear  $N_{xx} = +1$  lb/in and  $N_{yy} = -1$  lb/in,  $N_{xy} = 1$  lb/in
  
- ❖ Laminate III:  $[0/90]_T, B_{11} \neq B_{22} \neq 0$ 
  - I: Applied Bi-Axial Edge Compression  $N_{xx} = -1$  lb/in and  $N_{yy} = -1$  lb/in
  - II: Applied Bi-Axial Edge Compression and Tension  $N_{xx} = -1$  lb/in and  $N_{yy} = +1$  lb/in
  
- ❖ MATLAB scripts are used to create both 3D and 2D contour plots of the transverse displacement  $w$  depicting the mode shapes of laminated composite plates subjected to edge buckling loads

# Case I: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

First Buckling Factor: 2.41



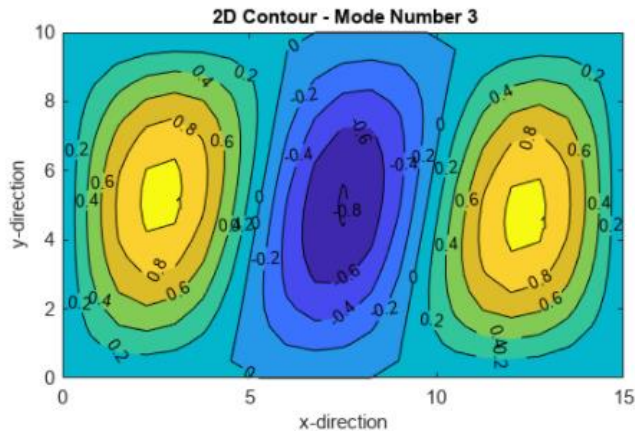
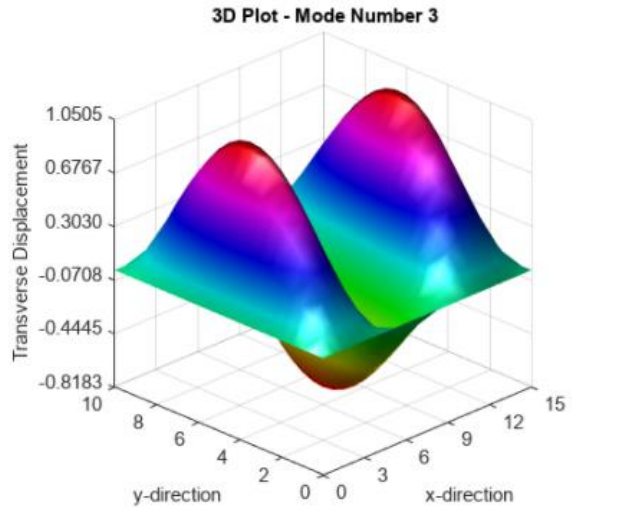
Second Buckling Factor: 2.564



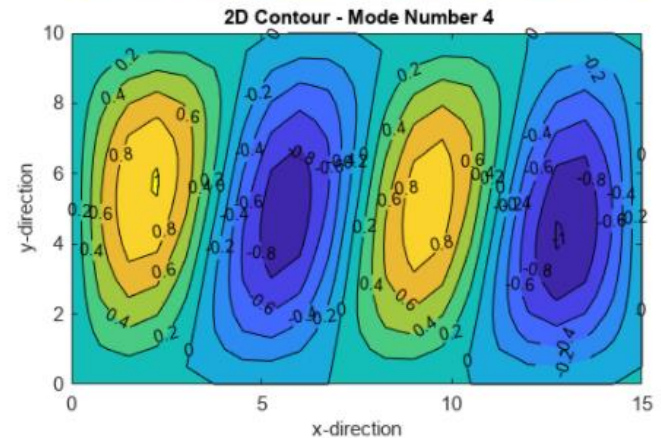
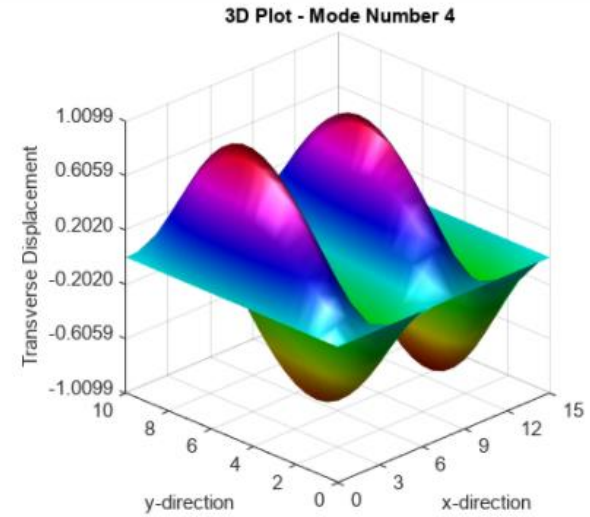


# Case I: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

Third Buckling Factor: 3.062

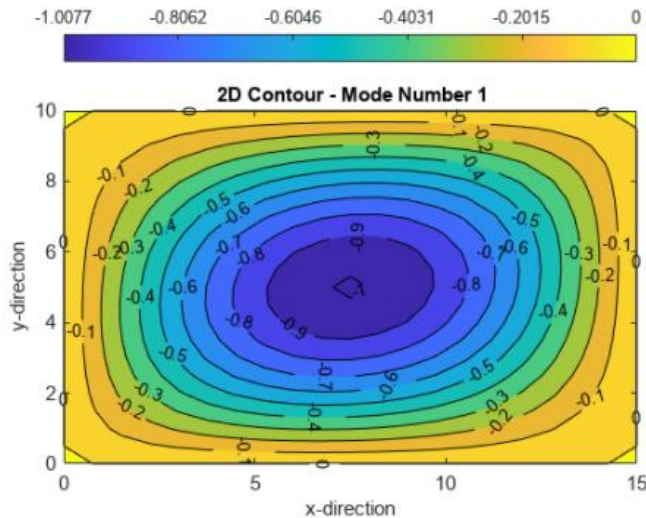
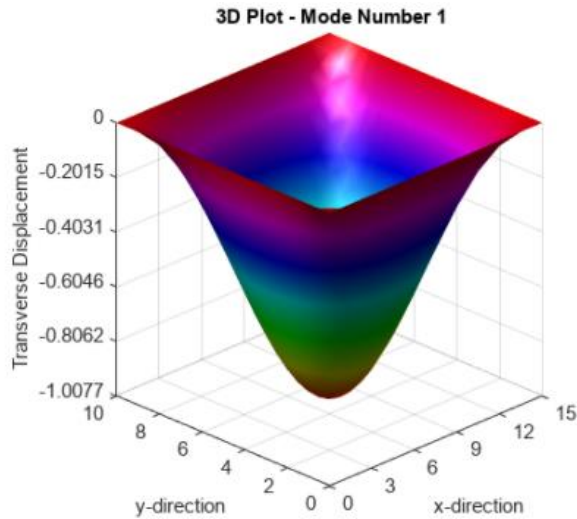


Fourth Buckling Factor: 4.085

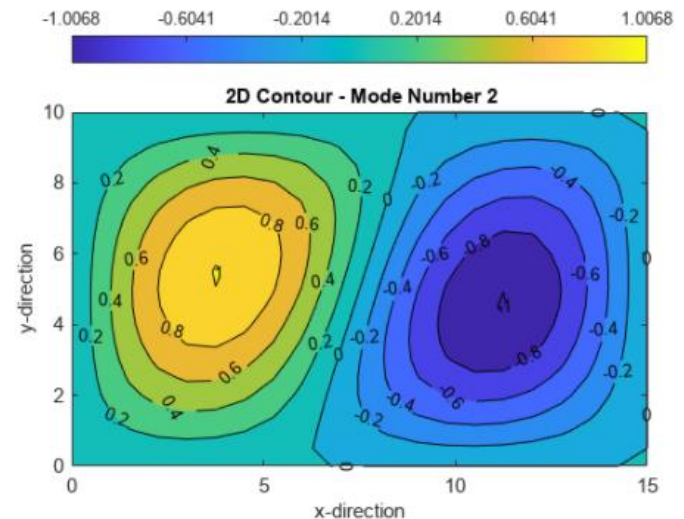
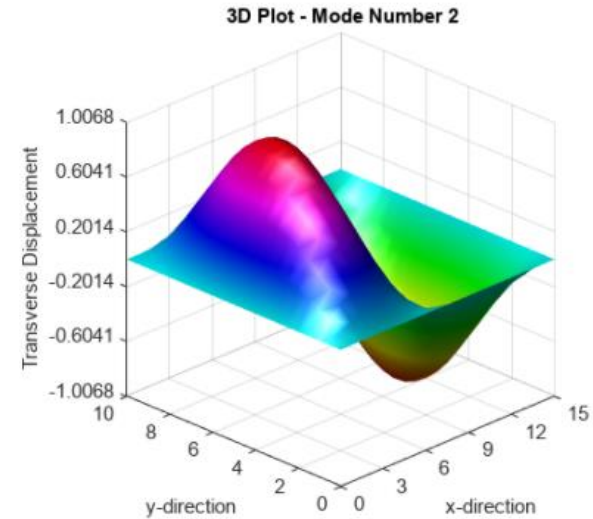


# Case II: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

First Buckling Factor: 1.21



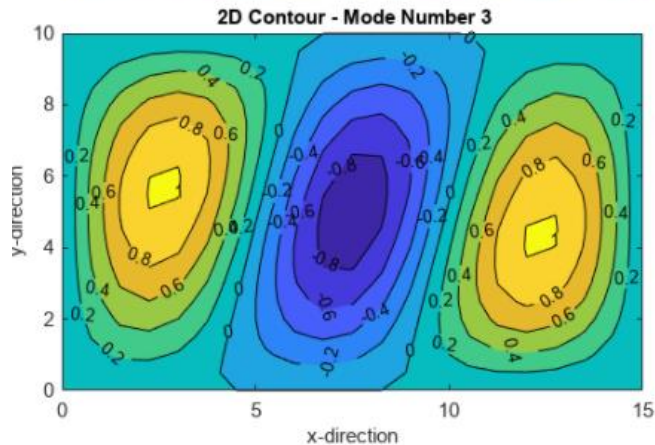
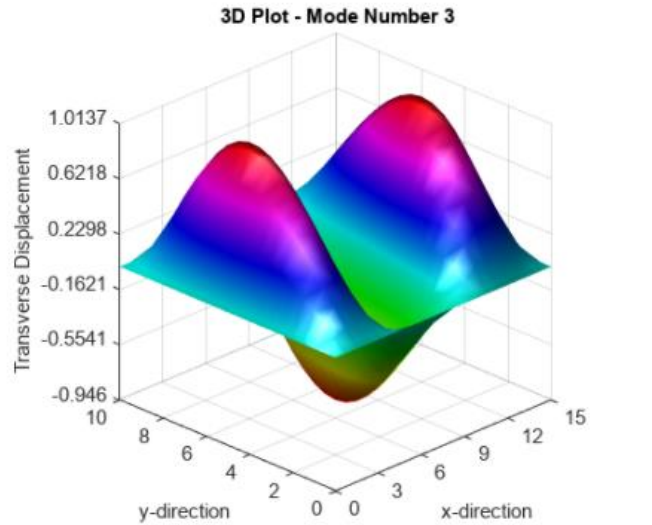
Second Buckling Factor: 1.856



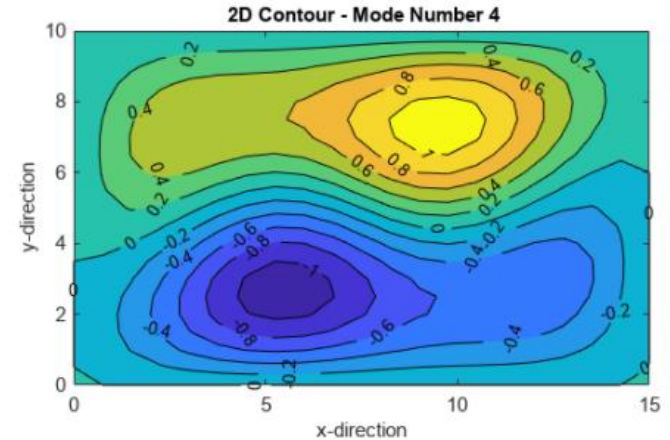
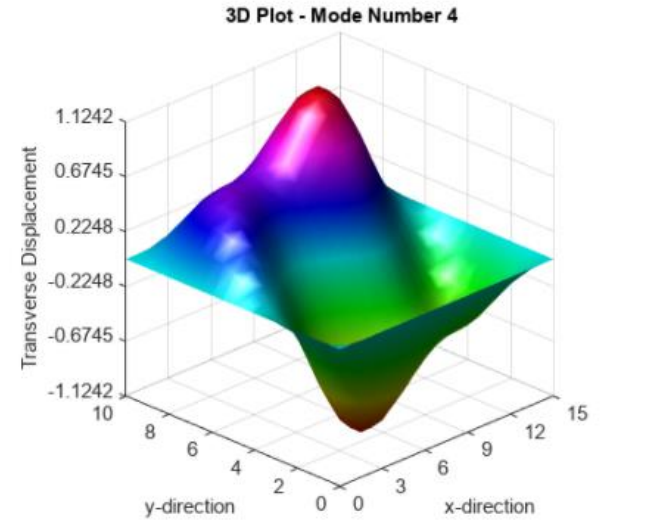


# Case II: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

Third Buckling Factor: 2.683

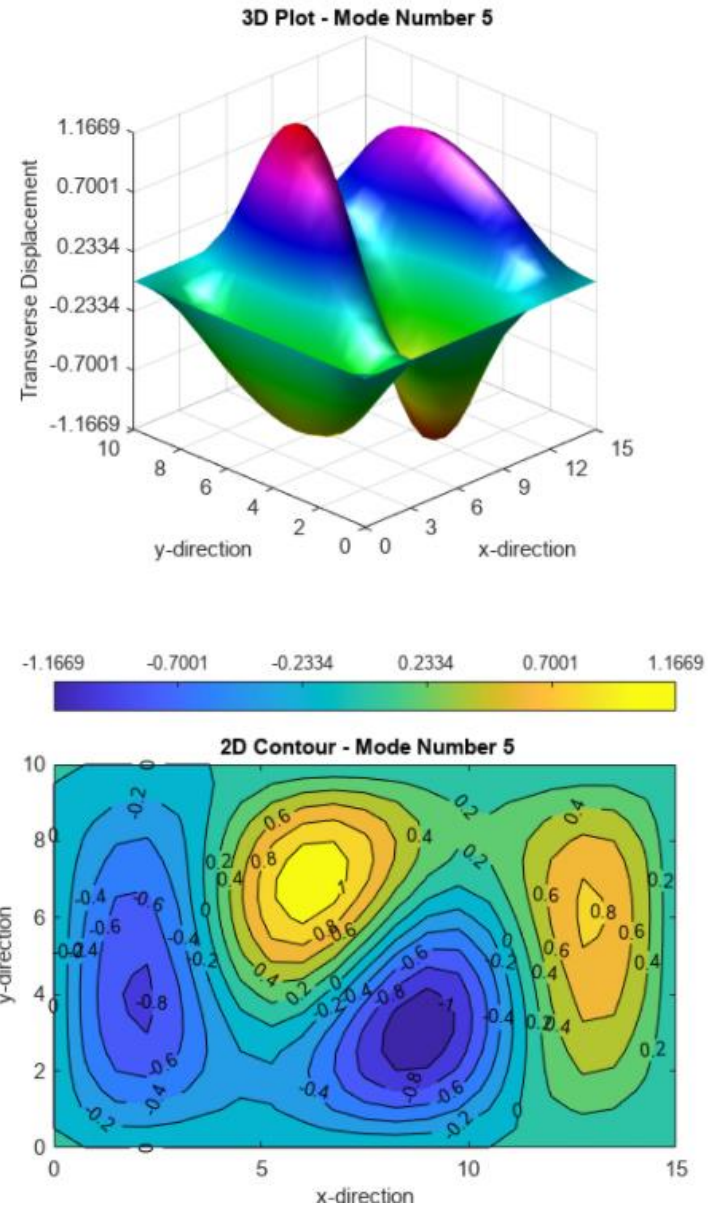


Fourth Buckling Factor: 3.538



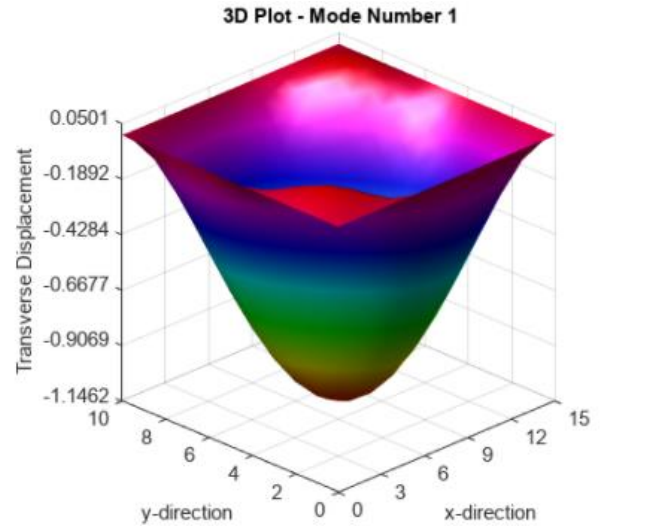
# Case II: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

Fifth Buckling Factor: 3.816

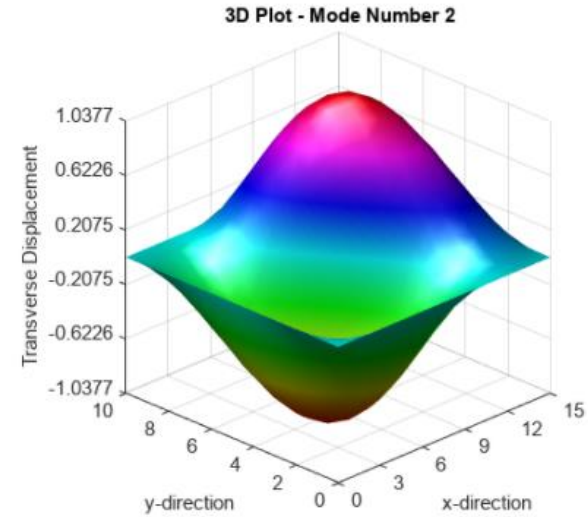


# Case III: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

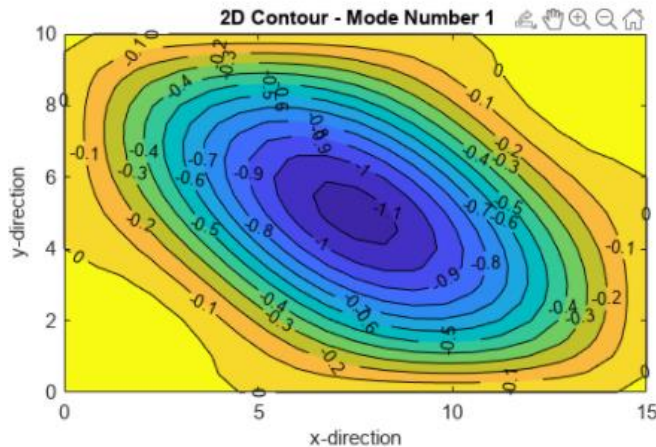
First Buckling Factor: 4.835



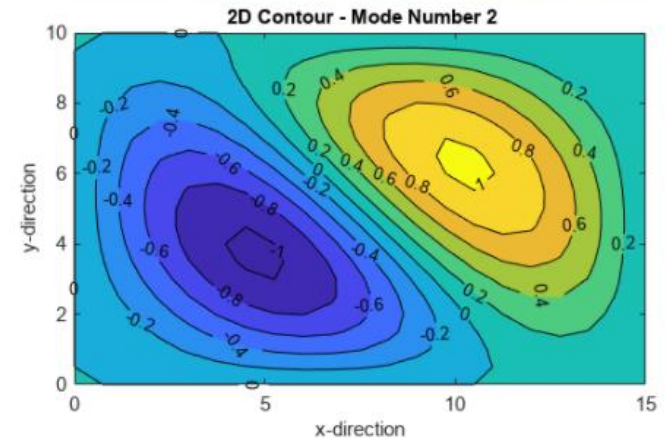
Second Buckling Factor: 5.363



-1.1462 -0.9069 -0.6677 -0.4284 -0.1892 0.0501



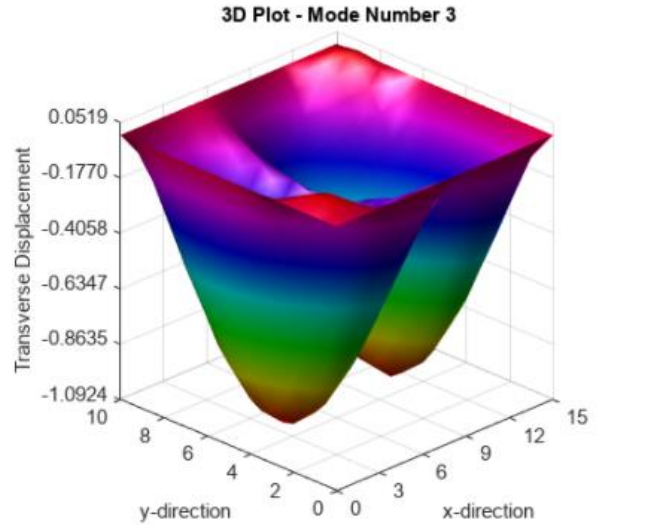
-1.0377 -0.6226 -0.2075 0.2075 0.6226 1.0377



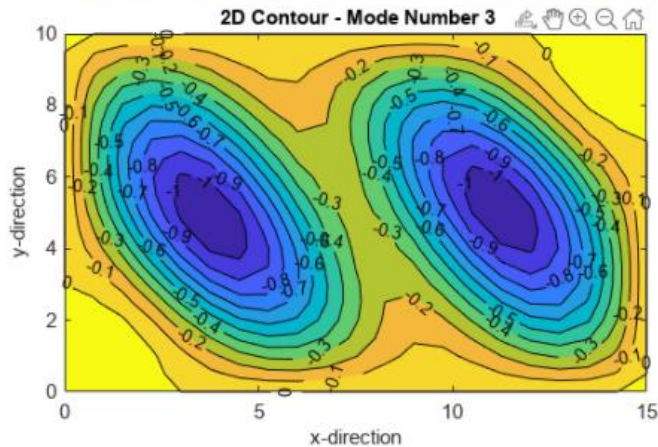


# Case III: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

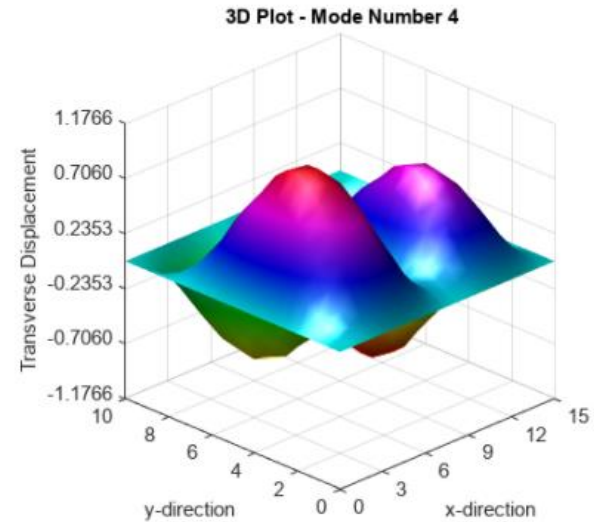
Third Buckling Factor: 9.84



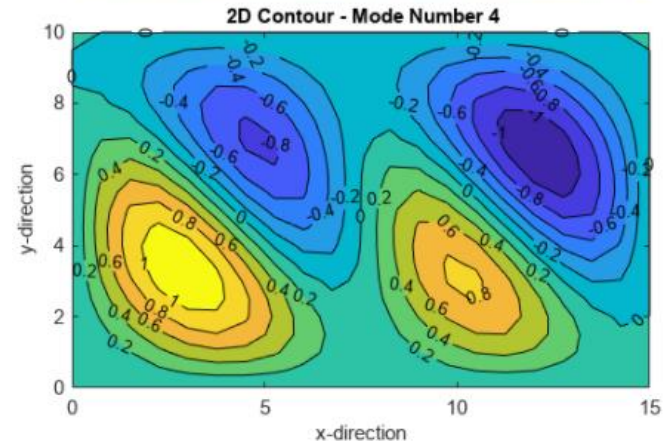
-1.0924 -0.8635 -0.6347 -0.4058 -0.1770 0.0519



Fourth Buckling Factor: 10.74

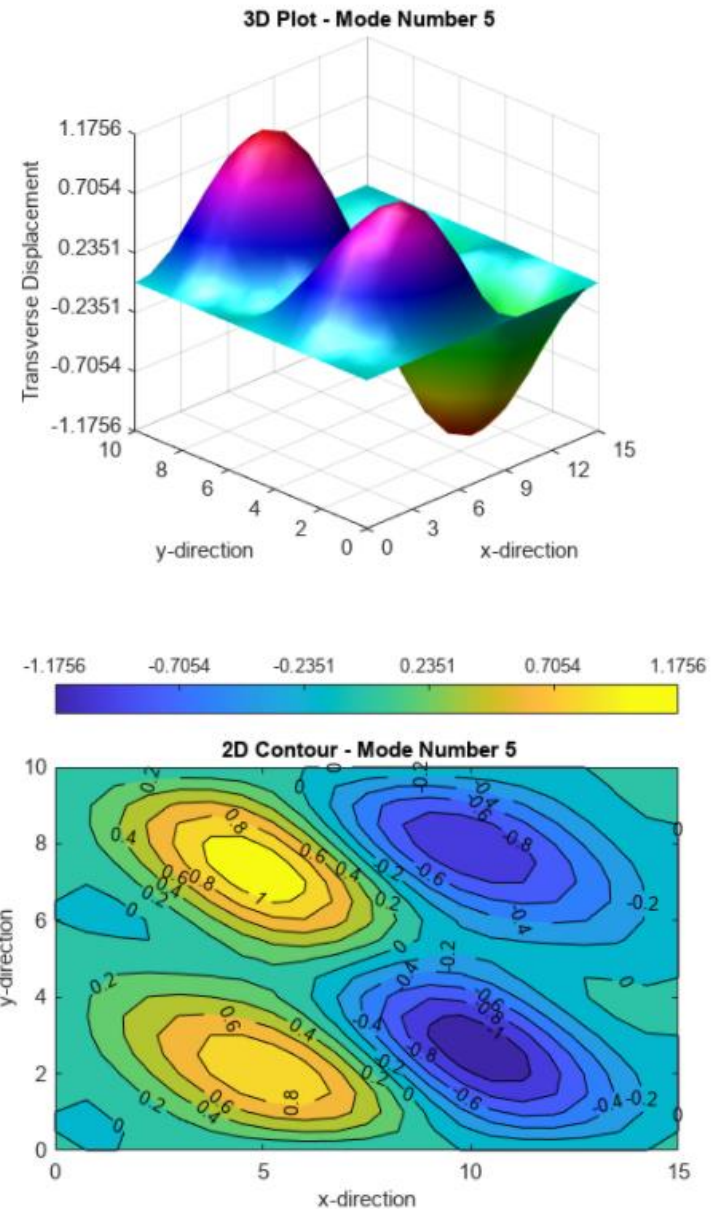


-1.1766 -0.7060 -0.2353 0.2353 0.7060 1.1766



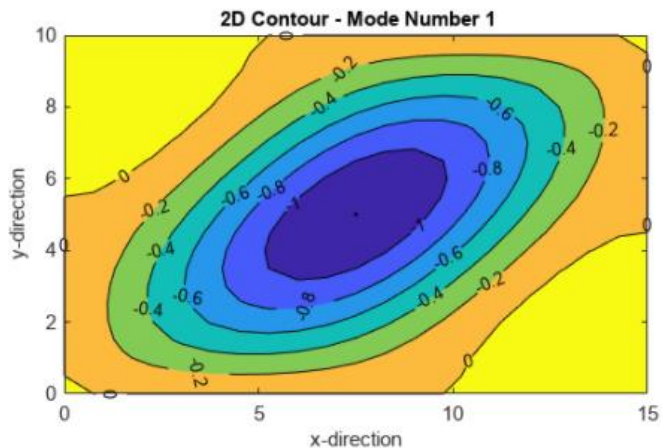
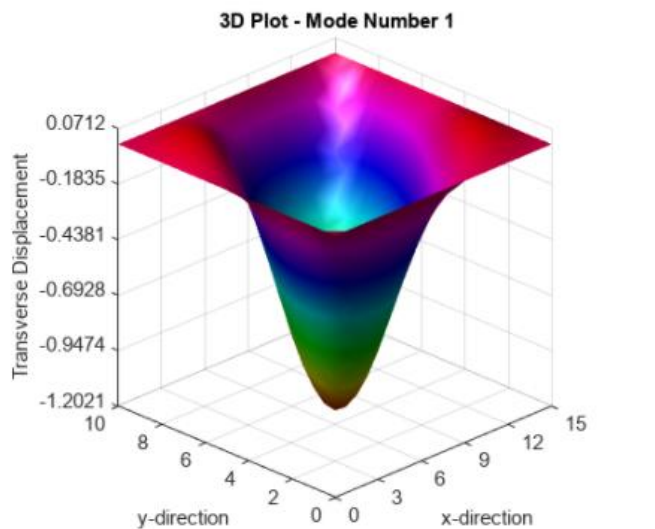
# Case III: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

Fifth Buckling Factor: 16.63

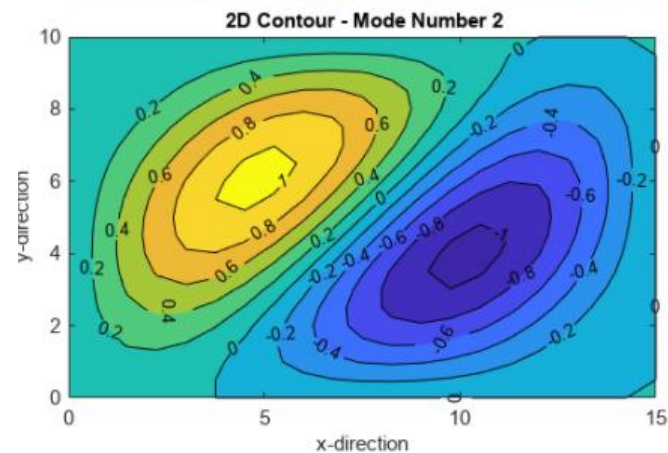
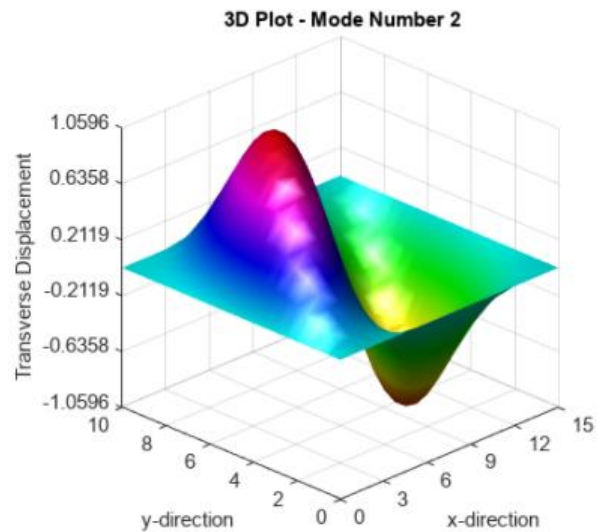


# Case IV: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

First Buckling Factor: 2.967



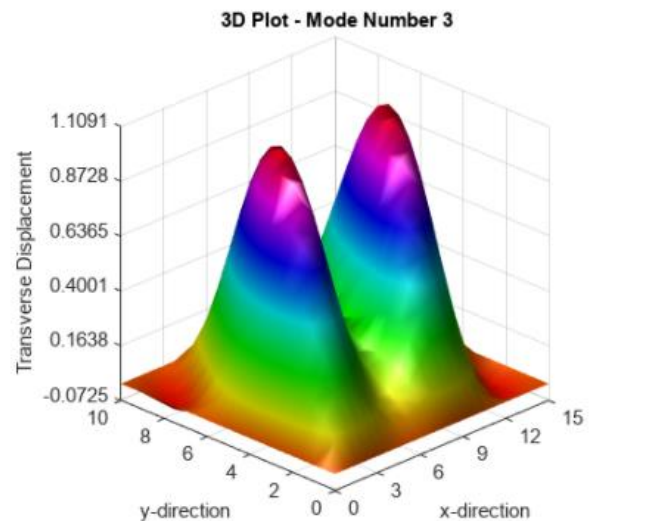
Second Buckling Factor: 3.246



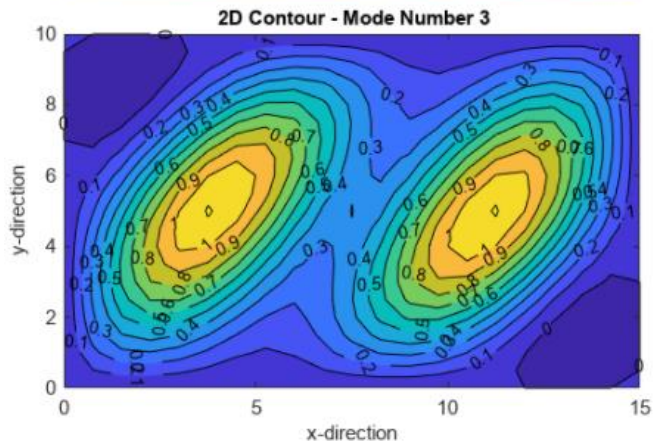


# Case IV: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

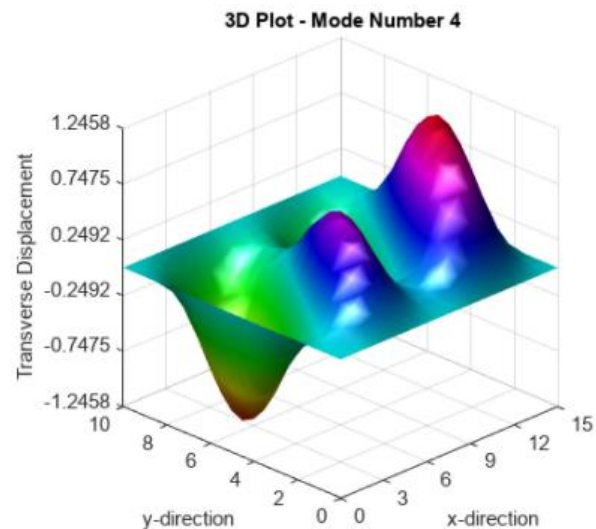
Third Buckling Factor: 5.767



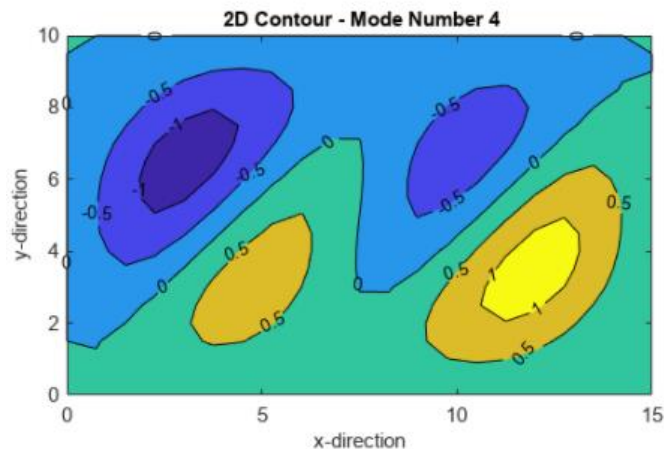
-0.0725 0.1638 0.4001 0.6365 0.8728 1.1091



Fourth Buckling Factor: 6.292

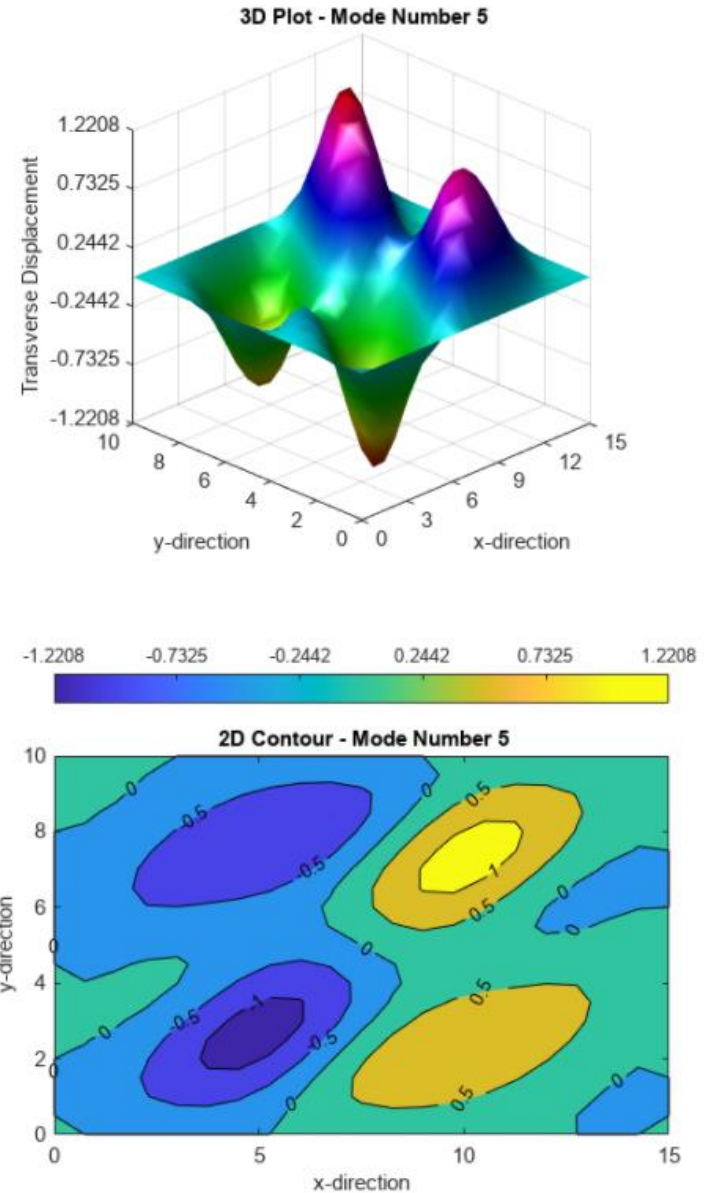


-1.2458 -0.7475 -0.2492 0.2492 0.7475 1.2458



# Case IV: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

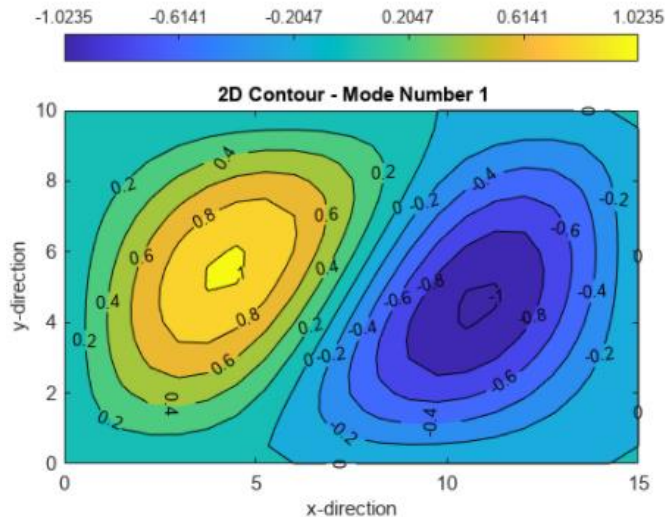
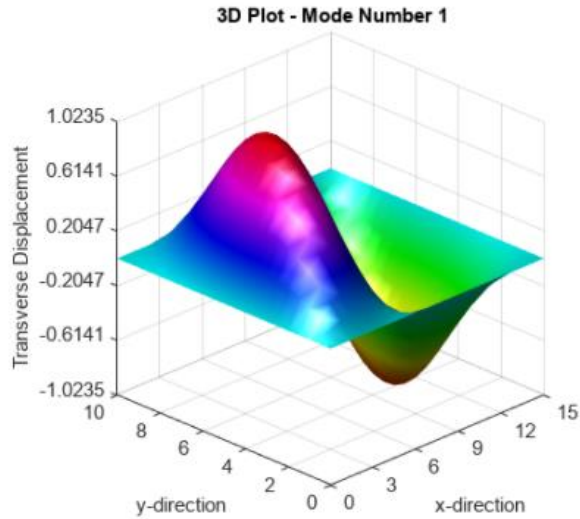
Fifth Buckling Factor: 9.991



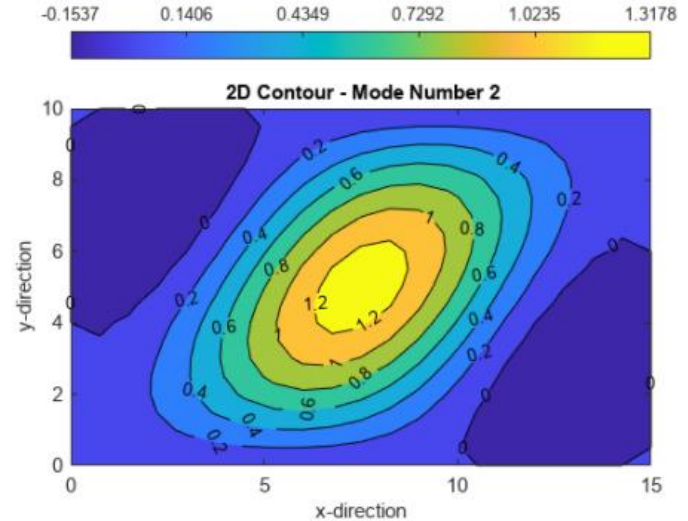
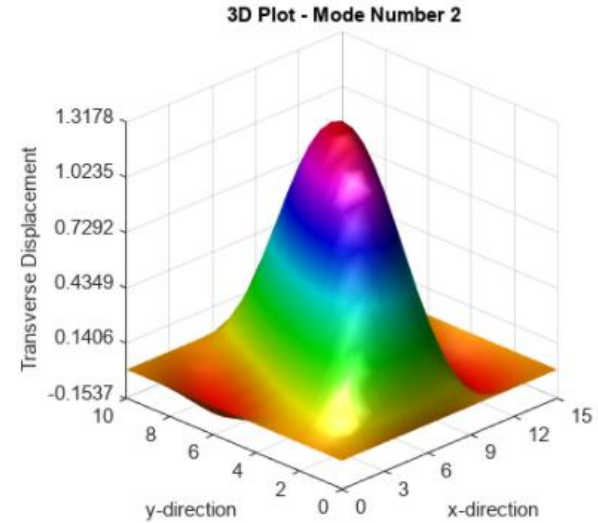


# Case V: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

First Buckling Factor: 1.84

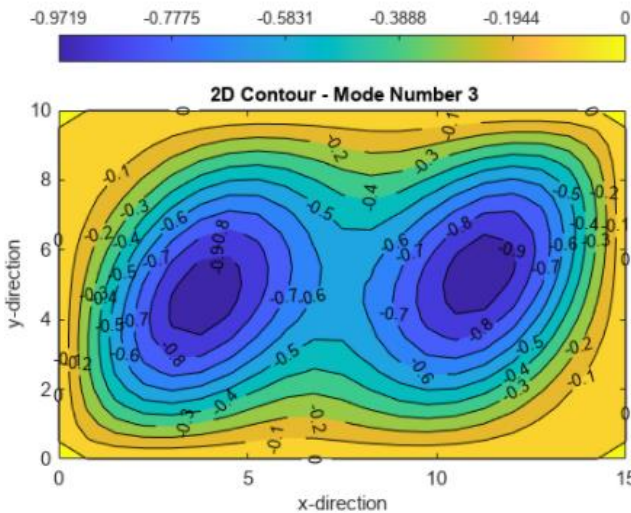
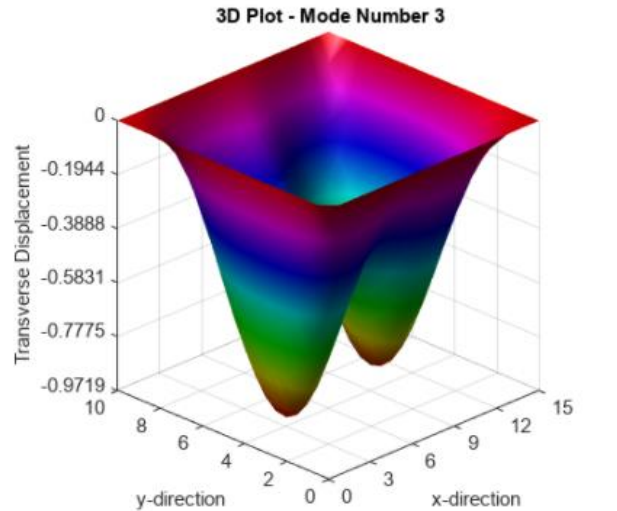


Second Buckling Factor: 1.999

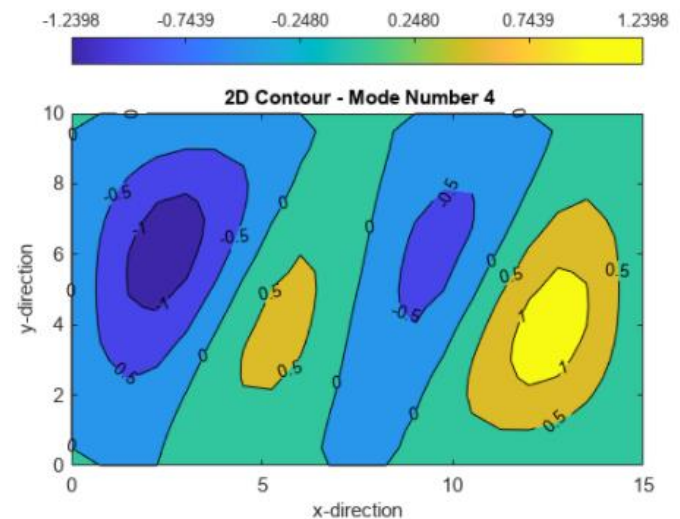
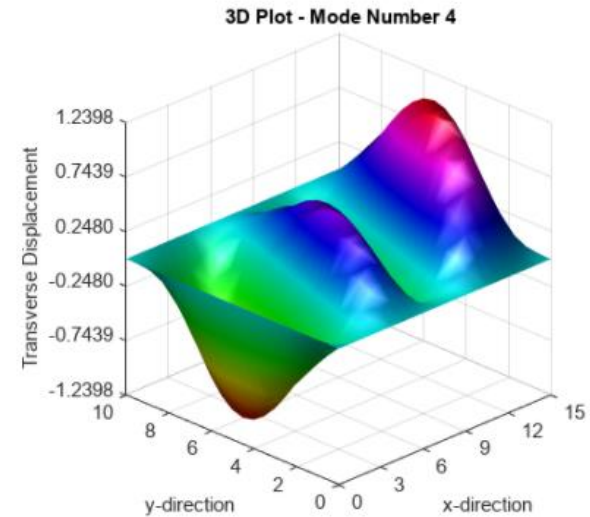


# Case V: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

Third Buckling Factor: 2.649

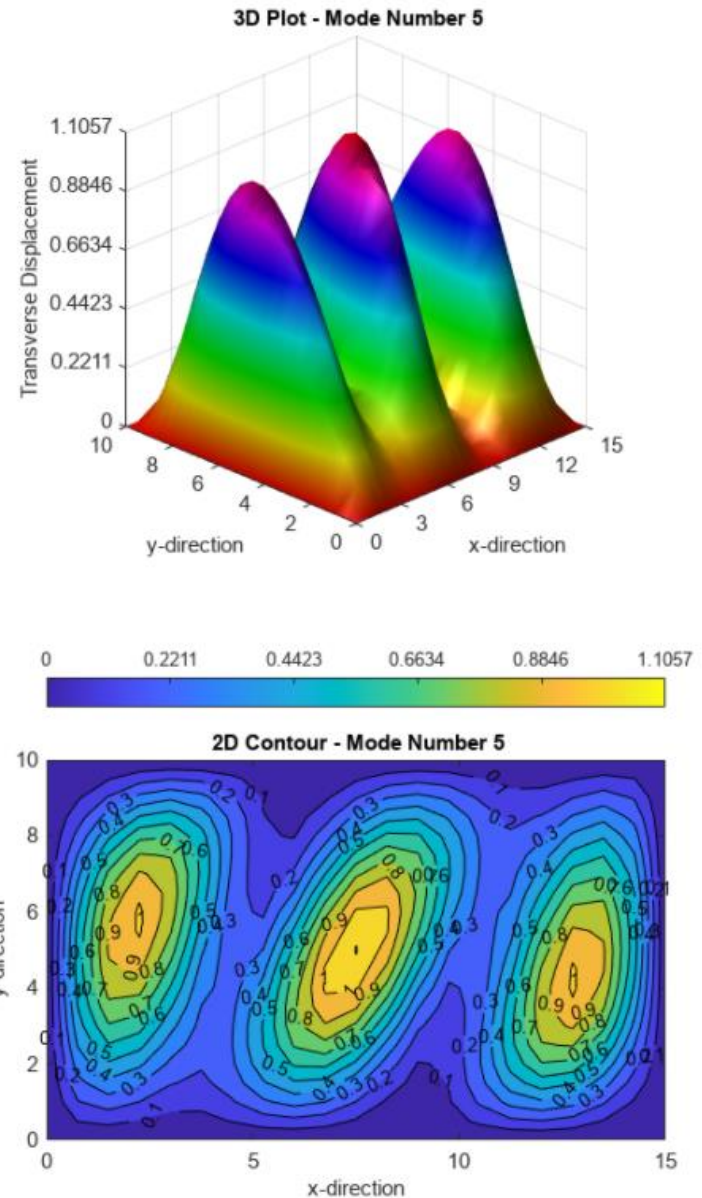


Fourth Buckling Factor: 2.978



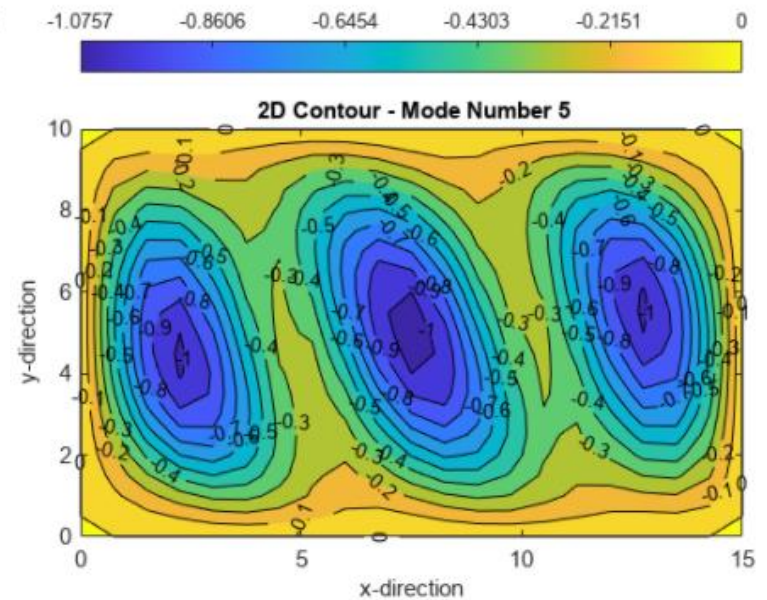
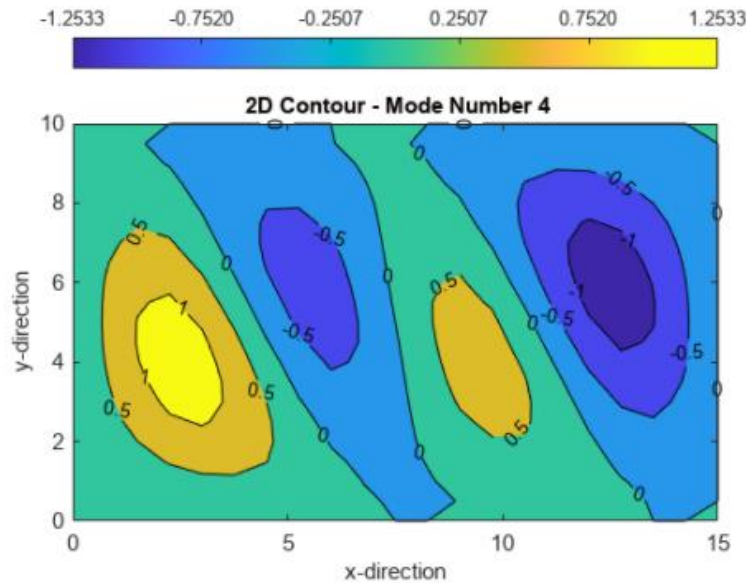
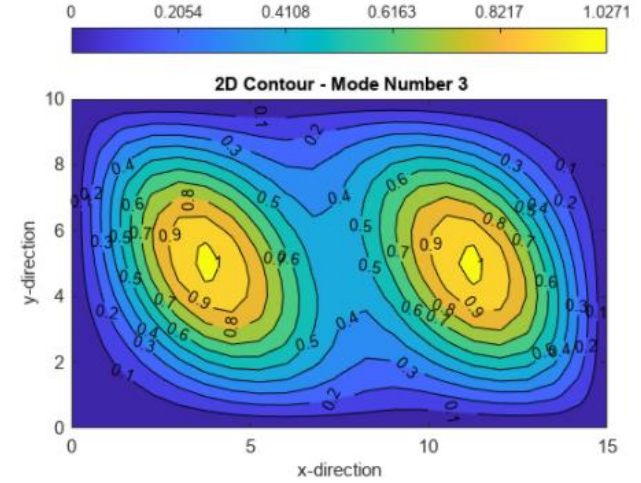
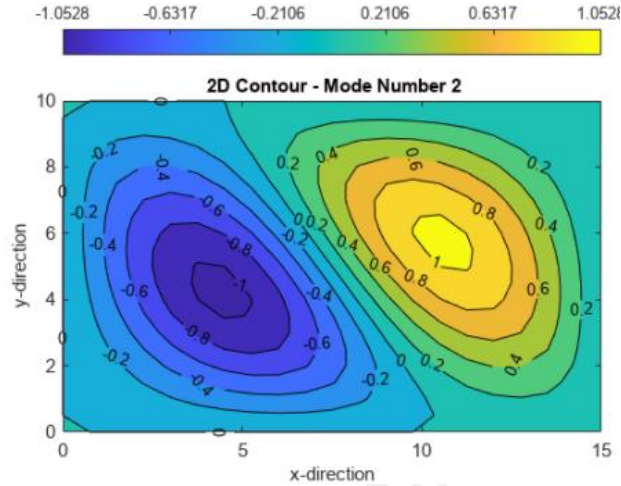
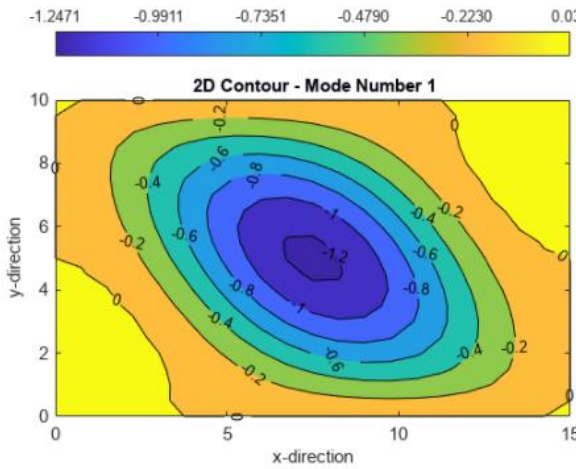
# Case V: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_{2s}$

Fifth Buckling Factor: 4.099



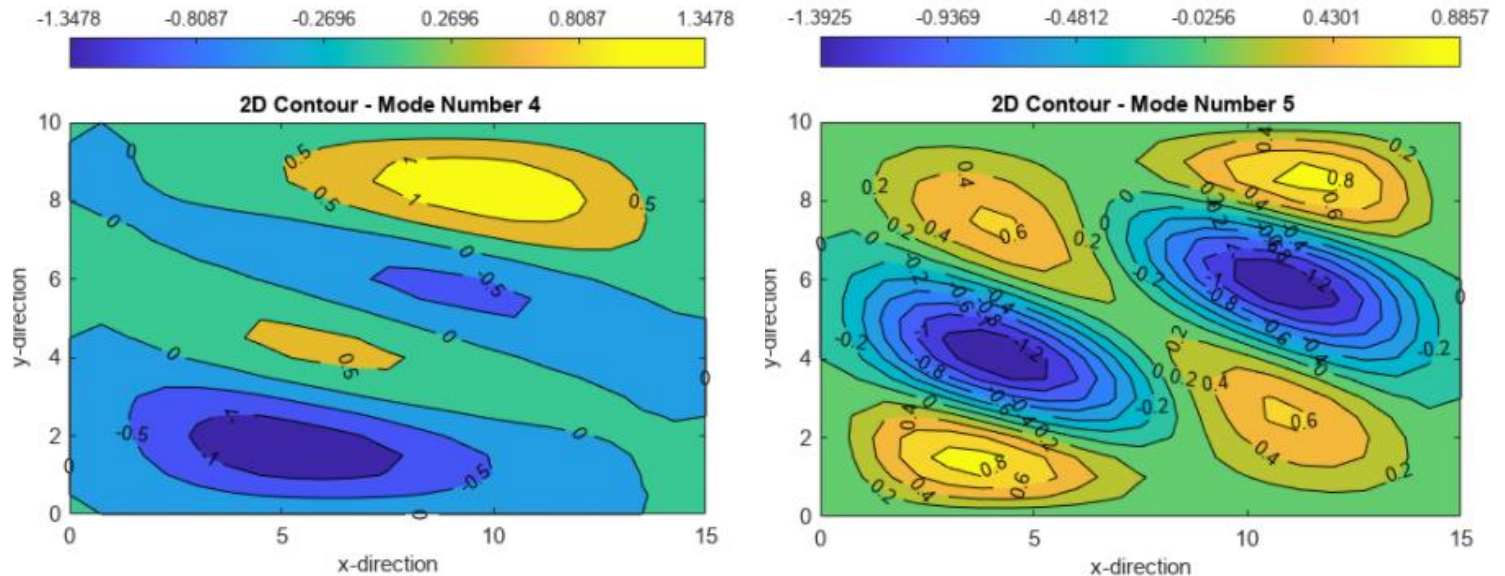
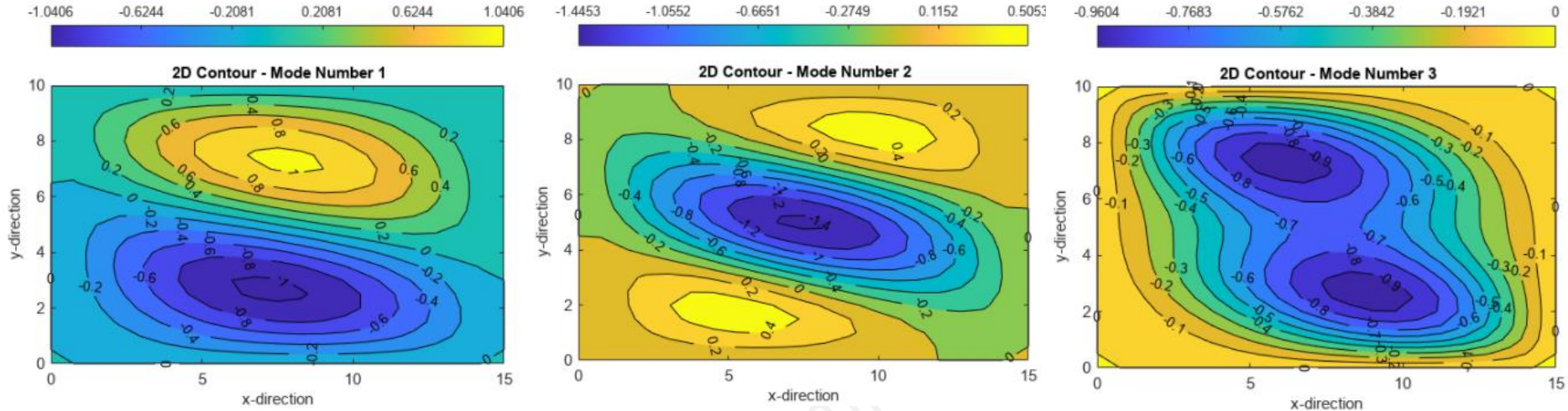


# Case I: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_T$



Buckling Load Factor
0.196
0.201
0.293
0.322
0.441

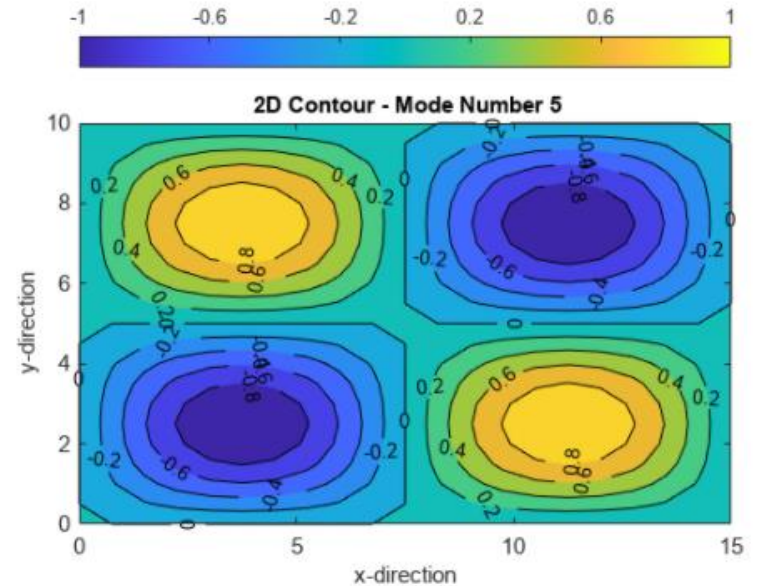
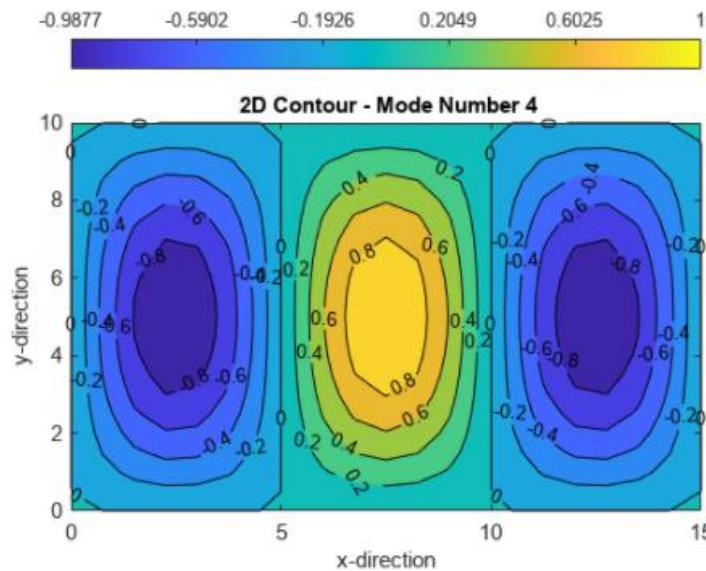
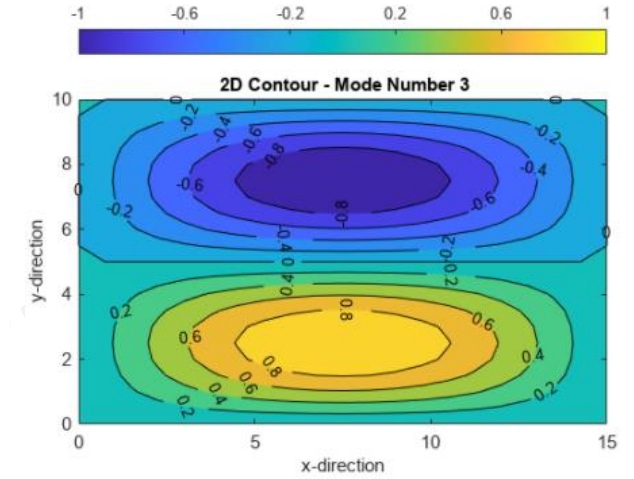
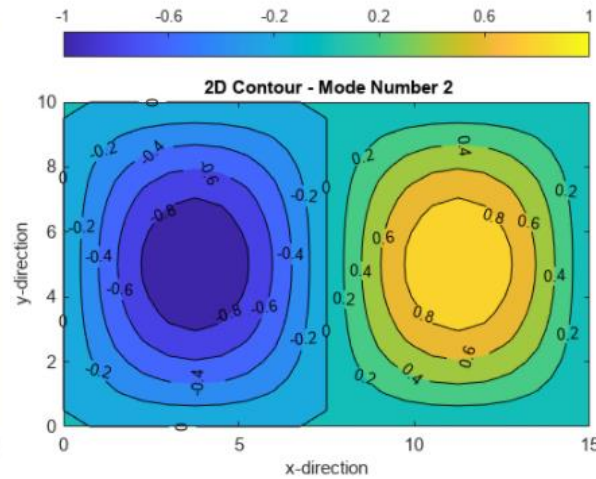
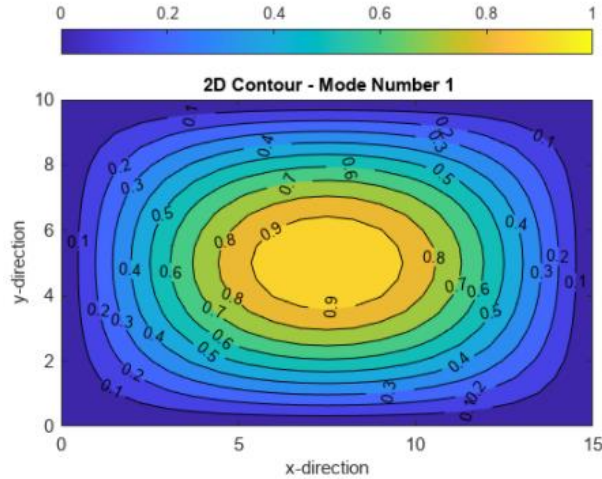
# Case II: Critical Buckling Load Factors And Mode Shapes $[\pm 45]_T$



Buckling Load Factor
0.298
0.361
0.476
0.518
0.665

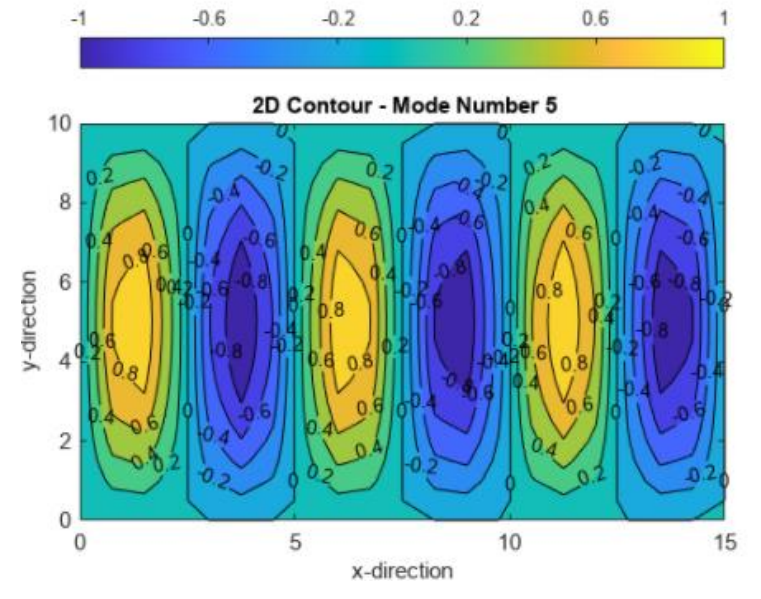
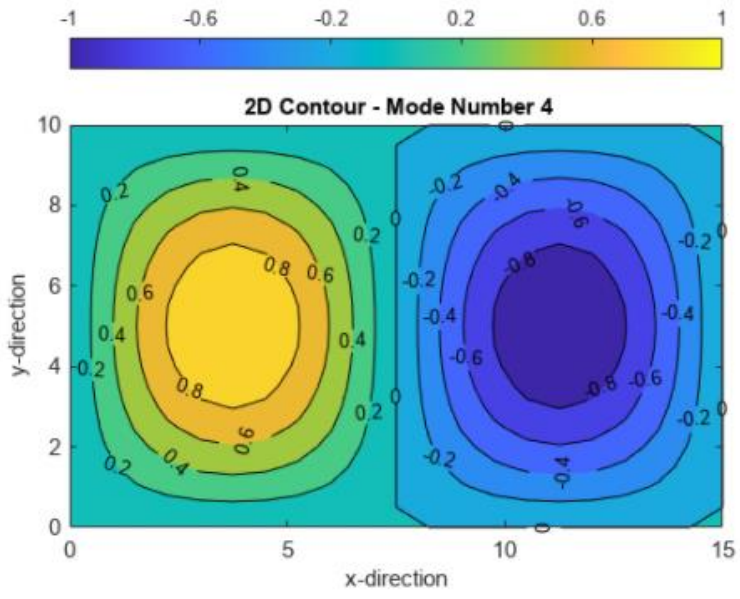
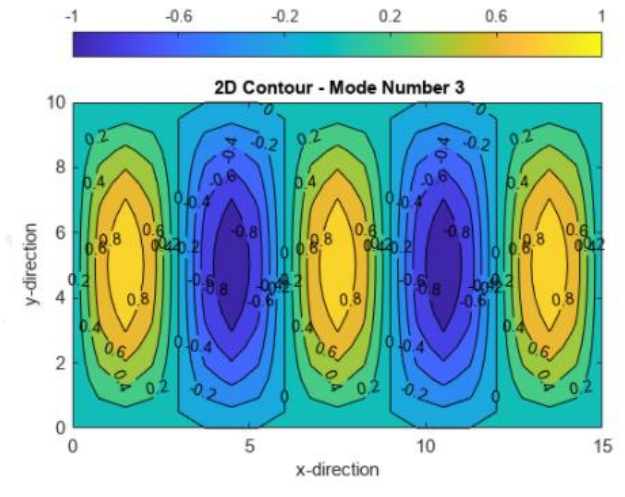
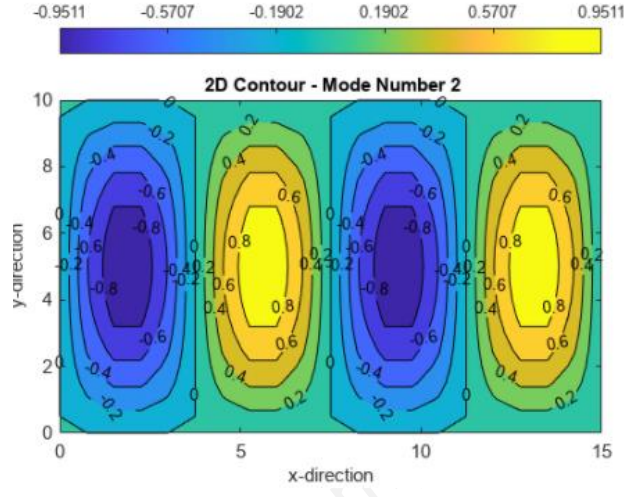
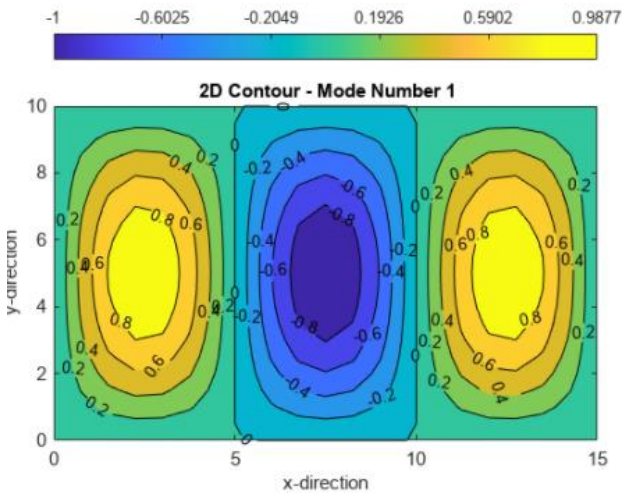


# Case I: Critical Buckling Load Factors And Mode Shapes $[0/90]_T$



Buckling Load Factor
0.0485
0.0907
0.1775
0.1816
0.1939

# Case II: Critical Buckling Load Factors And Mode Shapes $[0/90]_T$



Buckling Load Factor
0.477
0.502
0.627
0.704
0.817