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Needle Roller Bearing Handbook NTN Needle Roll<mark>er B</mark>earing Handbook



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1. Needle Roller Bearings

1.1 Classification of needle roller bearings

Needle roller bearings are categorized into the following two groups:

- Needle roller and cage assembly bearings
- Full-complement needle roller bearings

1.2 Comparison with general bearings

(1) Greater load-bearing capacity in a more compact design

Needle roller bearings are smaller than general bearings but have a greater loadcarrying capacity. This advantage allows for more compact designs for bearings as well as for bearing housings. Use of these bearings lowers costs by allowing for lighter-weight machinery and more compact structures.

Fig. 1.1 compares bearings with (30) mm bore diameters. Needle roller bearings have 2 to 8 times the load-carrying capacity per given mass than other bearing types.



Fig. 1.1

Fig. 1.2 shows a comparison of bearing sizes of standard dimensions and similar load-carrying capacity. Needle roller bearings have one-half the outside diameter and one-fifth the mass of deep-groove ball bearings.

Needle roller and cage assembly bearings differ from general bearings in that, if the shaft and bearing housing can be made to a specific hardness, accuracy, and surface roughness, the needle roller and cage assemblies can function as independent bearings without inner and outer rings. Bearings used in this way are smaller and can be used in the same space as a metal bearing.



Fig. 1.2

(2) Low inertial force

Because needle roller and cage assemblies have a smaller mass, these bearings are particularly useful for applications in which a smaller inertial force is required, such as in an engine crankshaft. (**Fig. 1.3**)

(3) High rigidity

Fig. 1.4 shows the elastic displacement when the radial load is applied to various bearings with similar basic load ratings.

- 1. Needle roller bearings have the following characteristics in comparison to cylindrical roller bearings.
 - More rollers are included.

• l /d is greater.

- 2. Needle roller bearings have greater rigidity because the load carried per unit area is smaller than ball bearings.





PK Series (for crank pins)

KBK Series (for piston pins)



Fig. 1.3



Fig. 1.4

(4) Needle roller bearings are suitable for applications with oscillatory movement.

If the oscillatory movement does not overlap the adjacent rollers, the lubricant between the rollers and bearing ring is pushed out and becomes inadequate. Accordingly, this condition shortens the life of the bearing. To prevent this condition, the bearing must be designed with an oscillatory angle that, at minimum, overlaps the position of the adjacent roller.

This design ensures good lubrication of the adjacent rollers, because the lubricant is forced out of the rollers and prevents them from wearing. **Fig. 1.5** shows the minimum oscillatory angle of the inner and outer rings. This is the most effective means of increasing the number of rollers in one row and ensuring the minimum oscillatory angle. The needle roller bearing is therefore the most suitable for the purpose.



1.3 Comparison of needle roller and cage assembly bearings and fullcomplement needle roller bearings

Table 1.1

| Type Item | Needle roller and cage assembly bearing | Full-complement needle roller bearing | |
|---------------------------|--|--|--|
| Roller skew | Low incidence | High incidence | |
| Friction coefficient | Small | Large | |
| Temperature rise | Low | High | |
| Permissible speed | High | Low | |
| Load-carrying capacity | bad-carrying pacity Less than that of full-complement needle roller bearings | | |

Needle roller and cage assembly bearings are used in a wide range of diverse applications, but full-complement needle roller bearings are more suitable for high-load, low-speed, and oscillating applications, as they have a greater load-carrying capacity.

(1) Roller skew

Needle roller and cage assembly bearings feature rotational accuracy due to the ample strength and rigidity of the shaft and bearing housing, which guide the case assembly. This feature ensures the correct rotation of the needle rollers. In contrast, the full-complement needle roller bearing has no structure for precisely guiding the needle rollers, and its instability may cause the needle rollers to skew. (**Fig. 1.6**)



Fig. 1.6

(2) Friction coefficient

Needle roller and cage assembly bearing



The slipping speed between the cage assembly and needle rollers is only the circumference speed of the needle rollers.

Full-complement needle roller bearing



The relative slippage speed of adjacent needle rollers is twice the circumference speed of one needle roller; in addition, skew makes the needle roller rotate while slipping.

Accordingly, the friction coefficient of the full-complement needle roller bearing is about twice that of the needle roller and cage assembly bearing.

 Table 1.2 shows the friction coefficient of various types of bearings.

Table 1.2 Friction coefficients of various bearings

| Bearing classification | Friction coefficient μ |
|--|----------------------------|
| Deep-groove ball bearings | 0.0010~0.0015 |
| Angular contact ball bearings | 0.0012~0.0018 |
| Self-aligning ball bearings | 0.0008~0.0012 |
| Cylindrical roller bearings | 0.0010~0.0015 |
| Tapered roller bearings | 0.0017~0.0025 |
| Self-aligning roller bearings | 0.0020~0.0025 |
| Thrust ball bearings | 0.0010~0.0015 |
| Needle roller and cage assembly bearings | 0.0020~0.0030 |
| Full-complement needle roller bearings | 0.0040~0.0050 |
| Thrust needle roller bearings | 0.0030~0.0040 |

(3) Temperature rise

The approximate temperature rise of a running bearing is usually calculated with the following formula:

where,

$$T_{\rm m} - T_0 = 0.00514 \frac{\mu \cdot F_{\rm r} \cdot d \cdot n}{W_{\rm S}}$$

- $T_{\rm m}$: Temperature when the bearing reaches equilibrium °C
- To : Ambient air temperature °C
- μ : Friction coefficient
- Fr : Radial load kgf
- d : Single bore diameter of bearing mm
- n : Revolutions per minute rpm
- Ws : Heat dissipated to the surroundings per 1°C temperature difference. Watt / °C

(The temperature rise is less with needle roller and cage assembly bearings, as it is proportional to μ .)

Fig. 1.7 shows the effect of changing speed and load on needle roller and cage assembly bearings and full-complement needle roller bearings having the same boundary dimensions for bore diameter (32 mm), outside diameter (47 mm), and width (20 mm).

At 750 kgf and 7500 rpm, as shown in the figure, the full-complement needle roller bearing exceeds 170 °C and ceases to rotate, while the needle roller and cage assembly bearing remains below 100 °C, thus permitting higher speeds.





(4) High permissible speeds

According to the characteristics indicated above, needle roller and cage assembly bearings are suitable for high-speed operation. "Permissible speed" is the speed limit that a bearing can safely withstand over a long period. This speed varies with the type and dimensions of the bearing, type of cage assembly, shaft carrying load, lubrication method, peripheral structure around the bearing, and cooling conditions.

The permissible speeds listed in **NTN** catalogs represent the values that can be applied when the bearing is correctly mounted and maintained with the most appropriate lubricant.

(5) Load capacity

The number of rollers in a full-complement needle roller bearing can be increased because there is no cage assembly; therefore, the bearing can be designed with a greater load-carrying capacity.

1.4 Production methods for needle roller bearings

Needle roller bearings (drawn-cup type) are manufactured with the following processes.

The cage shown is a welded type.



Fig. 1.8 Processing method for drawn-cup needle roller bearings



2. Characteristics of Various Series of Needle Roller Bearings

2.1 Series numbers and bearing names

Table 2.1 Needle roller and cage assemblies

| Cate | gory | Series code | Bearing name | Appearance | Characteristics |
|----------------------------|-----------------|-------------|--|-------------------------------|--|
| | | GH | Cage assembly without roller locating (for crank pins) | $\vdash \vdash \vdash$ | H Series, split type |
| | | GK | Needle roller and split cage assembly | | K Series, split type |
| | ned-ring type | GPK | Needle roller and split cage assembly (for crank pins) | | RK Series, split type Cage assembly with M shaped structure |
| | | Н | Cage assembly without roller locating (for crank pins) | $\vdash \vdash \vdash \vdash$ | High-rigidity cage assembly Capable of higher speeds than the PK Series |
| | Machii | к | Needle roller and cage assembly | | Basic type (with high-rigidity cage assembly) |
| | | KBK | Needle roller and cage assembly for piston pins | ſ | Bore diameter guide for cage assembly (high-rigidity cage assembly) |
| roller and cage assemblies | | PK | Needle roller and cage assembly for crank pins | | Outside diameter guide for cage assembly (high-rigidity cage assembly) Cage assembly with M shaped structure |
| | d type | KMJ | Needle roller and cage assembly | | Steel-plate cage assembly Cage assembly with M shaped structure |
| | Presse | PCJ | Needle roller and cage assembly | t | Steel-plate cage assembly Cage assembly with M shaped structure Inch series |
| | | GK∙∙S | Needle roller and split cage assembly | | Thick steel-plate cage assembly High-rigidity cage assembly |
| | | GKV∙∙S | Needle roller and split cage assembly | | Thick steel-plate cage assembly |
| leedle | | K∙∙S | Needle roller and cage assembly | | Thick steel-plate cage assembly High-rigidity cage assembly |
| 2 | ype | KJ∙∙S | Needle roller and cage assembly | | Steel-plate cage assembly |
| | elded t | KMJ∙∙S | Needle roller and cage assembly | | Steel-plate cage assembly Cage assembly with M shaped structure |
| | We | KV∙∙S | Needle roller and cage assembly | | Thick steel-plate cage assembly High-rigidity cage assembly Cage and assembly with V-shaped structure |
| | | SK∙∙S | Needle roller and single-split cage assembly | ل | K-S Series, single-split type |
| | | SKJ∙∙S | Needle roller and single-split cage assembly | | KJ-S Series, single-split type |
| | | SKV··S | Needle roller and single-split cage assembly | | KV-S Series, single-split type |
| | Plastic type | K∙∙T2 | Needle roller and cage assembly | | Molded cage assembly of polyamide plastic Max. permissible temperature: 120°C Max. continuous operating temperature: 100°C |

Table 2.2 Machined-ring needle roller bearings

| Category | | Series code | Bearing name | Appearance | Characteristics |
|---------------|-----------|-------------|--|------------|---|
| | | MR+MI | Machined-ring needle roller bearings | | Inch series Assembly of MR Series and M Series inner rings |
| | | NA 48 | Machined-ring needle roller bearing (Dimension series 48) | t | Assembly of RNA48 Series and IR Series inner rings |
| | | NA 49 | Machined-ring needle roller bearing (Dimension series 49) | t | Assembly of RNA49 Series and IR Series inner rings. Sealed series (L, LL) also available |
| | ring | NA49··S | Adjustable-clearance needle roller bearing (Dimension series 49) | | Assembly of RNA49-S Series and IR Series inner rings |
| | inner | NA 59 | Machined-ring needle roller bearing (Dimension series 59) | | Assembly of RNA59 Series and IR Series inner rings |
| | With | NA 69 | Machined-ring needle roller bearing (Dimension series 69) | t | Assembly of RNA69 Series and IR Series inner rings |
| gs | | NAO | Machined-ring needle roller bearings Separable type | | Assembly of RNAO Series and IR Series inner rings |
| bearin | | NK+IR | Machined-ring needle roller bearings | | Assembly of NK Series and IR Series inner rings |
| needle roller | | NKI | Machined-ring needle roller bearings | | With inner ring, special |
| | | MR | Machined-ring needle roller bearings | | Inch series High-rigidity outer ring High accuracy, Single-row structure |
| d-ring | | NK | Machined-ring needle roller bearings | t | High-rigidity outer ring High accuracy, Single-row structure |
| achine | | NKS | Machined-ring needle roller bearings | t | For heavy loads |
| Ŵ | er ring | RNA 48 | Machined-ring needle roller bearing (Dimension series 48) | t | High-rigidity outer ring High accuracy, Single-row structure |
| | out inne | RNA 49 | Machined-ring needle roller bearing (Dimension series 49) | t | High-rigidity outer ring High accuracy, Single-row structure Sealed series (L, LL) also available |
| | Witho | RNA49 ···S | Adjustable-clearance needle roller bearing (Dimension series 49) | | Radial clearance is adjustable Single-row structure |
| | | RNA 59 | Machined-ring needle roller bearing (Dimension series 59) | | High-rigidity outer ring High accuracy, Single-row structure |
| | | RNA 69 | Machined-ring needle roller bearing (Dimension series 69) | t | High-rigidity outer ring, High accuracy Single row ($Fw \leq 35$), Double row ($Fw \geq 40$) |
| | | RNAO | Machined-ring needle roller bearings Separable type | i ا | High-rigidity outer ring, High accuracy Single-row and double-row series (with ZW) are available |
| Oti iter | ner ms | NKZ | Machined-ring needle roller bearings (with different shape) | | Special specifications |

| Table 2.3 | Drawn-cup | needle roller | bearings-1 |
|-----------|-----------|---------------|------------|
|-----------|-----------|---------------|------------|

| Cate | gory | Series code | Bearing name | Appearance | Characteristics |
|------------|---------|--------------------------|---|------------|--|
| | | DCH | Drawn-cup needle roller bearing | | Inch series With cage assembly, For heavy loading |
| | | DCL | Drawn-cup needle roller bearing | 1 | Inch series, With cage assembly |
| | | нк | Drawn-cup needle roller bearing | | Double-row type (with ZW) bearings with cage assembly are available Small-diameter plastic bearings (with T2) are available |
| | | HK⋯L | Drawn-cup needle roller bearing Sealing type | | With cage assembly Prelubricated with standard grease (3A) |
| | | HKS | Drawn-cup needle roller bearing | X. | With cage assembly, Special |
| | type | НМК | Drawn-cup needle roller bearing | t | With cage assembly, For heavy loading |
| | in-end | нмк∙∙ ^L LL | Drawn-cup needle roller bearing Sealing type | | With cage assembly Prelubricated with standard grease (3A) |
| ers | Ope | HMV | Drawn-cup needle roller bearing | | Full-complement needle roller type |
| edle rolle | | HR | Drawn-cup cylindrical roller bearing | | With cage assembly, For heavy loading |
| p need | | ΗV | Drawn-cup needle roller bearing | | Full-complement needle roller (with C end face rollers) |
| awn-cup | | HVS | Drawn-cup needle roller bearing | as well | Full-complement needle roller type Special Prelubricated with standard grease (3A) |
| Dra | | VS | Drawn-cup needle roller bearing | | Full-complement needle roller (with C end face rollers) |
| | | VSH | Drawn-cup needle roller bearing | t | Full-complement needle roller (with C end face rollers) For heavy loading |
| | | BK | Drawn-cup needle roller bearing | P P | Double-row type (with ZW) bearings with cage assembly are available Small-diameter plastic bearings (with T2) are available |
| | e | BK∙∙L | Drawn-cup needle roller bearing Sealing type | | With cage assembly Prelubricated with standard grease (3A) |
| | end typ | BKS | Drawn-cup needle roller bearing | | With cage assembly |
| | osed-6 | BMK | Drawn-cup needle roller bearing | t | With cage assembly |
| | Ū | BV | Drawn-cup needle roller bearing | | Full-complement needle roller type |
| | | BVS | Drawn-cup needle roller bearing | as well | Full-complement needle roller type Prelubricated with standard grease (3A) |

| Cate | gory | Series code | Bearing name | Appearance | Characteristics |
|--------------------------------------|---------|-------------|---------------------------------|------------|--|
| ollers | e | DBH | Drawn-cup needle roller bearing | | Inch series With cage assembly, For heavy loading |
| Drawn-cup needle r Closed-end typ | and typ | DBL | Drawn-cup needle roller bearing | t | Inch series, With cage assembly |
| | osed-e | VB | Drawn-cup needle roller bearing | | Inch series Full-complement needle roller (with C end face rollers) |
| | C | VBH | Drawn-cup needle roller bearing | t | Inch series, For heavy loading Full-complement needle roller (with C end face rollers) |

Table 2.3 Drawn-cup needle roller bearings-2

Table 2.4 Compound bearings

(Needle roller bearing with thrust ball bearing/Needle roller bearing with thrust cylindrical roller bearing)

(Needle roller bearing with angular contact ball bearing / Needle roller bearing with three-point contact ball bearing)

| Category | Series code Bearing name | | Appearance | Characteristics |
|----------|--------------------------|---|------------|---|
| | ARN | Needle roller bearing with double-direction thrust cylindrical roller bearing | | Loading with radial load and double-direction axial load For high axial loading |
| | AXN | Needle roller bearing with double-direction thrust needle roller bearing | | Loading with radial load and double-direction axial load For high axial loading |
| | NKIA 59 | Needle roller bearing with angular contact ball bearing (Dimension series 59) | | Loading with single-direction axial load |
| S | NKIB 59 | Needle roller bearing with three-point contact ball bearing (Dimension series 59) | | Loading with double-direction axial load |
| earing | NKIT | Compound bearings | | With inner ring, Special |
| nnd b | NKT | Compound bearings | | Without inner ring, Special |
| Compc | NKX | Needle roller bearing with thrust ball bearing | | Open type Loading with single-direction axial load |
| Ū | NKX∙∙Z | Needle roller bearing with thrust ball bearing | | With cover |
| | NKXR | Needle roller bearing with thrust cylindrical roller bearing | | Open type Loading with single-direction axial load |
| | NKXR··Z | Needle roller bearing with thrust cylindrical roller bearing | | With cover |
| | NX | Needle roller bearing with full-complement thrust roller bearing | | Special product |

Table 2.5 Thrust roller bearings-1

| Category | | Series code | Bearing name | Appearance | Characteristics |
|---------------------|-----------------|-------------|--|------------|--|
| | rings | AK | Needle roller and thrust cage assembly | | Flat cage, Special |
| | dle roller beai | AKJ | Needle roller and thrust cage assembly | | Stainless steel pressed cage Series W/Box type, special type |
| tst roller bearings | | ARXJ | Integrated thrust needle roller bearing | | Assembly (Bearing + thrust washer) Available as separable and non-separable types |
| | st nee(| AXK 11 | Needle roller and thrust cage assembly | ı | Stainless steel pressed cage |
| | Thru | NTC | Needle roller and thrust cage assembly | t | Inch series Stainless steel pressed cage |
| | st washers | AS 11 | Thrust washers (Dimension series 11) | | Made from steel plate |
| | | NWA | Thrust washers | t | Inch series, Made from steel plate |
| | Thru | NWB | Thrust washers | t | t |
| | | 811 | Thrust cylindrical roller bearings (Dimension series 11) | | Assembly of K811 Series and WS / GS Series bearing washers |
| | | 812 | Thrust cylindrical roller bearings (Dimension series 12) | t | Assembly of K812 Series and WS / GS Series bearing washers |
| Thru | sĝu | 874 | Thrust cylindrical roller bearings (Dimension series 74) | | Assembly of K874 Series and WS / GS Series bearing washers |
| | - bearir | 893 | Thrust cylindrical roller bearings (Dimension series 93) | | Assembly of K893 Series and WS / GS Series bearing washers |
| | al rolle | СТС | Cylindrical roller and thrust cage assembly | | Inch series |
| | lindrica | СТСА | Thrust cylindrical roller bearings | | Inch series CTC Series and CWS Series in inch series / Assembly of CGS Series bearing washers |
| | rust cy | K811 | Cylindrical roller and thrust cage assembly (Dimension series 11) | | Cage assemblies of aluminum alloy, steel plate (J, JW), brass (L1), and plastics (T2) are also available |
| | ЧT | K812 | Cylindrical roller and thrust cage assembly (Dimension series 12) | t | t |
| | | K874 | Cylindrical roller and thrust cage assembly (Dimension series 74) | | Cage assemblies of aluminum alloy for heavy loading |
| | | K893 | Cylindrical roller and thrust cage assembly (Dimension series 93) | | t |

| Table 2.5 Thrust roller bearings-2 | able 2.5 | Thrust r | oller | bearings-2 | |
|------------------------------------|----------|----------|-------|------------|--|
|------------------------------------|----------|----------|-------|------------|--|

| Category | | Series code | Bearing name | Appearance | Characteristics |
|--|-----------|-------------|--|------------|---|
| | | CGS | Thrust bearing washer (outer ring) | | Inch series |
| | | CWS | Thrust bearing washer (inner ring) | t | Inch series |
| | | GS 811 | Thrust bearing washer (outer ring) (Dimension series 11) | | Machined-ring type |
| | | GS 812 | Thrust bearing washer (outer ring) (Dimension series 12) | t | Machined-ring type |
| | | GS 874 | Thrust bearing washer (outer ring) (Dimension series 74) | t | Machined-ring type |
| | earings | GS 893 | Thrust bearing washer (outer ring) (Dimension series 93) | t | Machined-ring type |
| | oller be | NWC | Thrust bearing washer | | Machined-ring type |
| | Irical ro | NWD | Thrust bearing washer | t | Machined-ring type |
| hrust roller bearings Thrust cylind | t cylind | NWE | Thrust bearing washer | t | Machined-ring type |
| | Thrust | NWF | Thrust bearing washer | t | Machined-ring type |
| | | WS 811 | Thrust bearing washer (inner ring) (Dimension series 11) | t | Machined-ring type |
| | | WS 812 | Thrust bearing washer (inner ring) (Dimension series 12) | t | Machined-ring type |
| Г | | WS 874 | Thrust bearing washer (inner ring) (Dimension series 74) | t | Machined-ring type |
| | | WS 893 | Thrust bearing washer (inner ring) (Dimension series 93) | t | Machined-ring type |
| | | xs | Thrust bearing washer | t | Machined-ring type, Special |
| | | ARA821 | Double-direction thrust cylindrical roller bearing (Dimension series 21) | | Assembly of K811 Series + Central ring (single-bore diameter side is stationary) of WS and GS Series bearing washers |
| | | ARB821 | Double-direction thrust cylindrical roller bearing (Dimension series 21) | | Assembly of K811 Series + Central ring (single outside diameter side is stationary) of WS and GS Series bearing washers |
| | items | ARX | Thrust roller bearings | | Special product (needle roller or cylindrical roller) Assembly of bearing, inner ring, outer ring and spacer |
| | Other | AXA 21 | Double-direction thrust needle roller bearing (Dimension series 21) | | Assembly of AXK811,WS, and GS Series bearing washers and ZS Series central ring (Single-bore diameter side is stationary.) |
| | | AXB 21 | Double-direction thrust needle roller bearing (Dimension series 21) | | Assembly of AXK811,WS, and GS Series bearing washers and ZS Series central ring (Single outside diameter side is stationary.) |
| | | ZS | Thrust bearing washer (central ring) | | Machined-ring type |

Table 2.6 Roller followers

| Category | Series code | Bearing name | Appearance | Characteristics |
|----------|-------------|---|------------|---|
| | NA22…LL | Roller followers (Sealed type) (Dimension series 22) | | With inner ring/cage assembly Prelubricated with standard grease (3A) |
| | NAB2 | Roller followers (Sealed type) (Dimension series 2) | | With inner ring/cage assembly (Open type) |
| | NABR | Roller followers | | With inner ring/cage assembly (Equivalent to IKO, NAST) |
| | NACV···X | Roller followers | | Inch series full-complement rollers/non-separable (Equivalent to McGill, CYR) Outside surface of outer ring is cylindrical Prelubricated with standard grease (3A) |
| | NACV…XLL | Roller followers (Sealed type) | | Inch series full-complement rollers/non-separable Outside surface of outer ring is cylindrical Prelubricated with standard grease (3A) |
| ollowers | NATR | Roller followers | | With cage and assemblies/non-separable |
| | NATR··LL | Roller followers (Sealed type) | | With cage and assemblies/non-separable Prelubricated with standard grease (3A) |
| Roller f | NATV | Roller followers | | Full-complement rollers/non-separable Prelubricated with standard grease (3A) |
| | NATV··LL | ATV··LL Roller followers (Sealed type) | | Full-complement rollers/non-separable Prelubricated with standard grease (3A) |
| | NUTR2 | JTR2 Roller followers (Dimension series 2) | | With double-row cylindrical rollers (Full-complement rollers) / non-separable shield plate Prelubricated with standard grease (3A) |
| | NUTR3 | Roller followers (Dimension series 3) | t | With double-row cylindrical rollers (Full-complement rollers) / non-separable shield plate Prelubricated with standard grease (3A) |
| | NUTW2 | Roller follower with outer ring and center rib (Dimension series 2) | | With double-row cylindrical rollers (Full-complement rollers) / non-separable shield plate Prelubricated with standard grease (3A) |
| | RNA22…LL | Roller followers (Sealed type) (Dimension series 22) | | Without inner ring, with cage and assembly Prelubricated with standard grease (3A) |
| | RNAB2 | Roller followers (Dimension series 2) | | Without inner ring (open type), with cage and assembly |

Remarks: 1. The external surface of the outer ring is spherical as standard. If a cylindrical outside surface is required, add "X" to the bearing series code. (Allowance of outside diameter *D*; Manufactured to JIS Class 0.)

Example: NA2203XLL · · · · ·

2. The outer surface of the outer ring of the NACV--X(LL) type is cylindrical as standard. If a spherical outside surface is required, delete the "X" from the bearing series code. Example: NACV12·····

Table 2.7 Cam follower

| Category | Series code | Bearing name | Appearance | Characteristics |
|----------|-------------|--|------------|--|
| | CR | Cam follower | | Inch series, With cage assembly |
| | CRV··X | Cam follower | | Inch series full-complement rollers Outside surface of outer ring is cylindrical Prelubricated with standard grease (3A) |
| | CRV··XLL | Cam follower (Sealed type) | | t |
| | KR | Cam follower | | With cage assembly Small-diameter plastic type (with T2) is available |
| | KR∙∙H | Cam follower (with hexagonal hole) | • | ↑ Shaft end (at head side) with hexagonal hole |
| | KR∙∙LL | Cam follower (Sealed type) | | With cage assembly Prelubricated with standard grease (3A) |
| | KRT | Cam follower (with tap hole) | | With cage assembly |
| llower | KRU | Cam follower (Eccentric-shaft type) | | With cage assembly Eccentricity: 0.25 to 1.0 mm |
| Cam fe | KRV | Cam follower | | Full-complement needle roller type Prelubricated with standard grease (3A) |
| | KRV··LL | Cam follower (Sealed type) | | t |
| | KRVT | Cam follower (with tap hole) | | t |
| | KRVU | Cam follower (Eccentric-shaft type) | | t Eccentricity: 0.25 to 1.0 mm |
| | KRX | Cam follower | | Special |
| | NUKR | Cam follower | | Double-row cylindrical rollers (Full-complement rollers) Prelubricated with standard grease (3A) |
| | NUKRT | Cam follower (with tap hole) | | t |
| | NUKRU | Cam follower (Eccentric-shaft type) | | t Eccentricity: 0.4 to 2.5 mm |

Remarks: 1. The external surface of the outer ring is spherical as standard. If a cylindrical outside surface is required, add "X" to the bearing series code. (Allowance of outside diameter *D*; Manufactured to JIS Class 0.) Example: KR22XLL.....

 If a spherical outside surface is required for the CRV-X (LL) type, which has a cylindrical outside surface of the outer ring as standard, delete the "X" from the bearing series code. Example: CRV96......

3. If a hexagonal hole is required at the shaft end, add "H" to the bearing series code. Example: KRV12H······ 19

| Table 2.8 | Bearings | for | linear | motion |
|-----------|----------|-----|--------|--------|
|-----------|----------|-----|--------|--------|

| Category | | Series code | Bearing name | Appearance | Characteristics |
|--------------|---------------------------|-------------|---|---|--|
| | | KD | Linear ball bearings Stroke type (open type) | | High-rigidity, high-accuracy outer ring Limited reciprocating motion |
| | | KD…LL | Linear ball bearings Stroke type (sealed type) | | ↑ Prelubricated with standard grease (3A) |
| | sbu | KDX | Linear ball bearings | | Special |
| | l bearii | кн | Linear ball bearings Drawn-cup type | | Outer ring of steel plate; lightweight and compact design Ball rows rotate, with unlimited linear motion |
| - | ear bal | KLM | Linear ball bearings Machined-ring type | j annan | High-rigidity, high-accuracy outer ring Ball rows rotate, with unlimited linear motion |
| inear motion | Line | KLM··LL | Linear ball bearings Machined-ring type (Sealed type) | 1 constant | ↑ Prelubricated with standard grease (3A) |
| | | KLM∙∙S | Linear ball bearings Machined-ring type (Adjustable-clearance type) | | Radial clearance is adjustable |
| igs for | | KLM⋯P | Linear ball bearings Machined-ring type (open type) | KLIM-P | A fan-shaped part of the bearing is removed (in the axial direction) Obstacles such as the shaft support can pass through the bearing |
| Bearir | ស | BF | Linear flat rollers | 00000000 | Cage and assembly of pressed steel Unit length: 1000 mm |
| | at rolle | FF | Linear flat rollers | { <u>00000000</u> ; | Molded cage assemblies of polyamide plastic |
| | near fla | FF…ZW | Linear flat rollers (Double-row type) | <u>{000000000</u> }; { <u>000000000</u> }; | tage assemblies can be mounted on the bent-V-shaped surface |
| | Li | RF | Linear flat rollers | 00000000 | ↑ Unit length: 705 mm |
| | Linear roller bearings | RLM | Linear roller bearings | | The row of cylindrical rollers rotates in unlimited linear motion. To mount the bearing, secure the bearing by using the threaded holes provided on the reference surface. |

Remarks: The standard lengths of BF and RF Series linear flat rollers are specified.

If a special length is required, add the length after the bearing series code. Example: If the total length L_1 of BF3020 must be 500 mm, indicate it as BF3020/500.

| Table 2.9 | Other | items, | Components-1 |
|-----------|-------|--------|---------------------|
|-----------|-------|--------|---------------------|

| Category | | Series code | Bearing name | Appearance | Characteristics |
|----------|-------------|-------------|--------------------------------------|------------|--|
| | | IR | Inner ring | | Machined-ring type provides high rigidity and high accuracy (Outside surface is used as raceway surface) |
| | | IRJ | Inner ring (Drawn-cup type) | | Pressed steel plate |
| | ngs | IRZ | Inner ring | | Special specifications (Flange, etc) |
| | aring ri | МІ | Inner ring | | Inch series |
| | Bea | OR | Outer ring | | Machined-ring type provides high rigidity and high accuracy (Bore surface is used as raceway surface) |
| | | ORJ | Outer ring (Drawn-cup type) | | Pressed steel plate |
| 0 | | ORZ | Outer ring | | Special specifications (Key groove, etc) |
| onents | dle rollers | А | Needle roller (A-end face roller) | | End face is round |
| Comp | | С | Needle roller (C-end face roller) | | End face is pointed |
| items, | | F | Needle roller (F-end face roller) | | End face is flat (the most common roller) |
| Other | | М | Needle roller (M-end face roller) | | End face is stepped |
| | Nee | R | Needle roller (R-end face roller) | | End face is spherical |
| | | т | Needle roller (T-end face roller) | | End section is conical, End face is flat |
| | | TR | Needle roller (TR-end face roller) | | End section is conical, End face is round |
| | | KP | Shaft for bearings for linear motion | | For linear ball bearings |
| | afts | NP | Pin | | Shafts and various pins with diameters not exceeding 12 mm |
| | Sh | VP | Precision shaft | | For use with audio components |
| | | ZP | Pin | | Shafts and various pins with diameters exceeding 12 mm |

Table 2.9 Other items, Components-2

| Category | | Series code | Bearing name | Appearance | Characteristics |
|------------|--------|-------------|---|------------|--|
| | | BR | Snap rings (for housing) | \bigcirc | Mounted on locating snap ring groove in housing |
| | rings | BRX | Snap rings (for housing) | t | Special |
| | Snap | WR | Snap rings (for shaft) | \bigcirc | Mounted on locating snap ring groove in shaft |
| | | WRX | Snap rings (for shaft) | t | Special |
| | | G | Seals | | 1-sheet lip Standard rubber material is nitrile rubber (NBR) |
| | als | GD | Seals | | 2-sheet lip |
| Components | Sea | GSC | Seals | | Inch series |
| | | GX | Seals | | Special |
| | | HF | One-way clutches (Drawn-cup type) | | Pressed steel plate, torque transmission in one direction Prelubricated with standard grease (L313) |
| items, | | HFU | One-way clutch unit | | Unit with built-in HF Series |
| Other | | HFL | One-way clutches (Drawn-cup type, with radial bearing) | | Bearings are mounted in both sides of clutch This bearing can withstand a constant radial load |
| | sei | HFLU | One-way clutch unit | | Unit with built-in HFL Series |
| | dutch | HFZ | One-way clutches | | Inch series |
| | ne-way | HFZU | One-way clutch unit | | Unit with built-in HFZ Series |
| | ō | NCU | One-way clutch unit | | Unit with plastic gears and pulleys mounted on the circumference of the clutch |
| | | NHF | One-way clutches (Machined-ring type) | | BEARFITE oil-retaining bearings are mounted on both sides. This bearing can withstand a constant radial load |
| | | NHFU | One-way clutch unit | | Unit with built-in NHF Series |
| | | NHS | Sprag one-way clutch (Machined-ring type) | | Assembly of multiple sprags (Torque is transmitted by the inclination of sprags) |

| Table 2.9 | Other | items, | Components-3 |
|-----------|-------|--------|---------------------|
|-----------|-------|--------|---------------------|

| Category | | Series code | Bearing name | Appearance | Characteristics |
|----------|-------------------------|-----------------|--|------------|---|
| | inery | FR | Bottom roller bearing | F | For draw frames Prelubricated with standard grease (3A) Assembled drawn-cup needle roller bearing |
| | ile mach | FRIS | Bottom roller bearing (A-series) | | For spinning frames and roving frames Applicable to international standards Prelubricated with standard grease (L113) |
| | s for text | FRIS··SA | Bottom roller bearing (A-series) | | t Bearing with saddle for securing bearings, separable type |
| | bearings | FRIS··SB | Bottom roller bearing (A-series) | t | t Bearing with saddle for securing bearings, non-separable type |
| | m roller | FRIS··SB | Bottom roller bearing (B-series) | | For spinning frames and roving frames, JIS-compatible |
| | Botto | FRIS··NP | Bottom roller bearing (B-series) | | ↑ Bearing with grease nipple and knock pin |
| (0 | chinery | JF∙∙S | Holder | | Special holder for JPU · · S Series |
| onents | tile mac | JPB | Shaft bearing | ₽ | Ball bearing unit for JPU · · S Series Prelubricated with standard grease (3A) |
| Comp | Tension pulleys for tex | JPU∙∙S | Tension Pulley | | For spinning, roving and false twisting frames Prelubricated with standard grease (3A) |
| items, | | JPU ↔S +JF↔S | Tension Pulley (with holder) | | f Mounts on machine roller carrier with holder bolts |
| Other | | JPP | Pulley (unit) | | Pressed steel plate Press-fit mounting to the outside surface of outer ring for JPU··S Series |
| | extile ninery | HKW | Bearing for spindles (Drawn-cup) | | For spinning machines |
| | For t mach | TEXZ | Bearing for textile machinery | | Bearing for textile machinery meets the series standard |
| | | CJ | Machined-ring bearing assembly | | Assembly of bearing (CK) and shaft (CL) |
| | | СК | Machined-ring bearing | | Machined outer ring With cage assemblies or full-complement rollers |
| | joints | CL | Cross shaft (spider or cross pin) | | Used as a set with CK or HCK Series bearings |
| | Cross | GU | Seal for drawn cup | | Used as a set with HCK Series bearings |
| | | нск | Drawn cup (Closed-end drawn-cup needle roller bearing) | | Full-complement rollers, outer ring of steel plate Prelubricated with standard grease (2S) |
| | | HCK+CL +GU | Drawn-cup assembly | | For motor vehicles (Steering and propeller shaft mechanisms) |

| Table 2.9 | Other | items, | Com | ponents-4 |
|-----------|-------|--------|-----|-----------|
|-----------|-------|--------|-----|-----------|

| Category | | Series code | Bearing name | Appearance | Characteristics |
|------------|-------------|-------------|---|------------|--|
| | ings | RAB | Rocker arm bearings | | Full-complement roller-type for motor vehicle engine valve mechanisms |
| | n bear | RJ | Shafts for rocker arm bearings | | Shaft for assembly |
| | ker arr | RO | Outer ring for rocker arm bearing | | Outer ring for assembly |
| | Roc | RS | Side washer for rocker arm bearing | | Side washer for assembly |
| | ings | CRG | Guide roller | | Bearing for CRZ Series assembly (outside) Prelubricated with standard grease (8A) |
| | er bear | CRP | Stud | | Shaft for Series CRZ assembly |
| | Cross rolle | CRS | Side roller unit | | Bearing for Series CRZ assembly (inside) Prelubricated with standard grease (8A) |
| Components | | CRZ | Cross roller | | For truck lifts Assembly, prelubricated with standard grease (8A) |
| | | BU | Bushing | | Special specifications (mainly for spacers) |
| items, | | нкг | Drawn-cup needle roller bearing of different shape | | Special product using drawn-cup needle roller bearings |
| Other | | HSF | Bearing for steering | | With inner and outer bearing rings of steel plate and steel balls Prelubricated with standard grease (2S) |
| | ns | HSL | Drawn-cup double-row cylindrical roller bearing | | Full-complement roller, for heavy loading, outer ring of steel plate, machined inner ring, prelubricated with standard grease (3A) |
| | her iter | NIP | Grease nipple | | For cam followers (Press fitting, mounted with screws) |
| | Ō | SEN | Plug | | Grease nipples provided on unlubricated side |
| | | TKBN | Components | | Components not meeting standards of needle bearing series |
| | | PNA | Self-aligning needle roller and cage assembly bearing | | With inner ring. Self-aligning model available |
| | | RPNA | Self-aligning needle roller and cage assembly bearing | | Without inner ring. Self-aligning model available |

🗊 Tips & Hints

Thrust roller bearings

Two categories of thrust roller bearings exist: assembled needle roller bearings and assembled cylindrical roller bearings.

They are assembled with bearing washers before use. The cylindrical roller bearing group comprises of a cylindrical roller and thrust cage assembly (K811, K812 and K893 Series), inner ring (WS Series), and outer ring (GS Series); bearings that combine these components are available (811, 812 and 893 Series).

Customers can either purchase an assembled bearing or purchase the

components separately and assemble them as required.



The shaft aligns with the single bore diameter for WS Series bearing rings and the housing aligns with the outside surface for GS Series bearing rings; therefore, they feature high rigidity and accurate running.

Fig. 1 Assembly example

| Bearing assembly | Assembly drawing | Assembly of thrust cage assemblies | Bearing ring (inner ring) | Bearing ring (outer ring) |
|---------------------|------------------|--|---------------------------|---------------------------|
| | | AXK11 | AS811 | AS811 |
| | | AXK11 | WS811 | GS811 |
| 811 | | K811 | WS811 | GS811 |
| 812 | | K812 | WS812 | GS812 |
| 874 | | K874 | WS874 | GS874 |
| 893 | | K893 | WS893 | GS893 |

Machined-ring needle roller bearings

Machined-ring needle roller bearings are available in two categories: RNA Series without an inner ring and NA Series with an inner ring. All components are common except the inner ring, as follows.

Example:

RNA4905R+IR25×30×17…/NA4905R

Accordingly, it is possible to purchase the RNA and IR separately and assemble them as needed. Note that the separable RNAO Series without an inner ring and NAO Series with an inner ring have common model numbers, except for the inner ring.



Fig. 2 Assembly example

3. Bearing Selection

Table 3.1 shows the types and characteristics of needle roller bearings.

Needle roller bearings are available in a wide range of varying dimensions. Selecting a particular bearing requires a thorough knowledge of the structure and characteristics of different bearings and their correct applications. No bearing, no matter how well made, will achieve its full potential if not carefully selected and correctly used. When selecting bearings, consider the following points.

| | Category | Needle roller and cage assemblies | Needle roller and cage assembly bearing | Separable needle roller and cage assembly bearing | Adjustable -clearance needle roller bearing | Drawn -cup needle roller bearing | Needle roller bearing with thrust ball bearing | Needle roller bearing with thrust cylindrical roller bearing | Needle roller bearing with angular contact ball bearing | |
|----------------------|--|--|--|--|---|---|---|--|--|--|
| | Illustration | | | E F | | | | | | |
| Series | | K K∙∙ZW KMJ PK KBK | NK··+IR NK··R NA48 NA49R NA59 NA69 MR··+MI | NAO NAO• •ZW RNAO RNAO• •ZW | RNA49∙ •S NA49• •S | $\begin{array}{l} HK \boldsymbol{\cdot} \boldsymbol{\cdot} (+IR) \\ BK \boldsymbol{\cdot} \boldsymbol{\cdot} (+IR) \\ HMK \boldsymbol{\cdot} \boldsymbol{\cdot} (+IR) \\ DCL \boldsymbol{\cdot} \boldsymbol{\cdot} (+MI) \end{array}$ | NKX NKX··+IR NKX··Z NKX··Z+IR | NKXR NKXR··+IR NKXR··Z NKXR··Z+IR | NKIA59 | |
| Load | Axial | ł | Î | ł | ł | f | f | | | |
| Speed | (High speed) Suitable for high-speed use Adequate for high-speed use Unsuitable for high-speed use × | 0 | \bigcirc | \bigcirc | \bigcirc | | | \bigtriangleup | \bigcirc | |
| Accuracy | (High accuracy) Suitable for high-speed use ○ Adequate for high-speed use △ Unsuitable for high-speed use × | 0 | \bigcirc | \bigcirc | \bigcirc | × | | \bigtriangleup | \bigtriangleup | |
| Mounting practice | Simple ○ Fairly simple △ Difficult X | | \bigcirc | 0 | | | | \bigtriangleup | \bigtriangleup | |

Table 3.1 Classifications and characteristics of bearings

Trans-All Construction Construction General Construction machinery, Construction machinery, mission machinery machinery, machinery, production Gear change machinery, Main application machinery etc. engine Printing etc. General machinery, etc. machinery, etc.

- (1) The magnitude size, direction, and type of the load that will be applied to the bearing
- (2) The speed and type (rotating inner ring, rotating outer ring) of rotation, and application to vertical or horizontal shafts
- (3) Required bearing life and maximum permissible load
- (4) Ambient temperature conditions around the bearing

- (5) Required accuracy
- (6) Friction and noise levels
- (7) Lubrication and sealing devices
- (8) Bearing mounting and removal
- (9) Materials and finish accuracy of the shaft and housing
- (10) Space available for bearing installation

| Needle roller bearing with three-point contact ball bearing | Needle roller bearing with double- direction thrust roller bearing | Thrust roller bearing | Roller follower (Separable) | Roller follower (Non- separable) | Cam follower | Linear flat roller | Machined -ring linear ball bearing (KD Series) | Drawn -cup linear ball bearing | Machined- ring linear ball bearing (KLM Series) | Linear roller bearing |
|--|---|---|-----------------------------------|---|-------------------------------------|---|---|---|--|-----------------------------|
| | | | | | | £00000000 } | | | | |
| NKIB59 | AXN ARN | 811 812 893 AXK11 AS WS GS | NA22· ·LL RNA22· ·LL | NATR NATV NUTR | KR••(LL) KRV••(LL) NUKR CR | FF FF••ZW BF RF | KD KD••LL | КН | KLM KLM··S KLM··P KLM··(LL) | RLM |
| | | | A | A | Å | ł | f | ł | Î | I |
| \bigcirc | \bigcirc | | \bigtriangleup | \bigtriangleup | \bigtriangleup | \bigtriangleup | 0 | \bigtriangleup | \bigtriangleup | |
| \bigtriangleup | \bigcirc | | \bigtriangleup | \bigtriangleup | \bigtriangleup | \bigcirc | | \bigtriangleup | \bigcirc | \bigcirc |
| \bigtriangleup | \bigtriangleup | | \bigcirc | \bigcirc | \bigcirc | \bigtriangleup | | \bigtriangleup | \bigtriangleup | \bigcirc |
| Constr mach et | ruction inery, c. | Pump, Construction machinery, General machinery | General machinery | General n Guide | nachinery rollers | General production machinery, Construction machinery, etc. | Printing machinery, etc. | General p Const | production m ruction mach Robots, etc. | achinery, ninery, |

4. Load Rating and Life

4.1 Bearing life

Even in bearings operating under normal conditions, the surfaces of the raceway and rolling elements are constantly subjected to repeated compressive stresses that cause flaking of these surfaces. This flaking is due to material fatigue and eventually causes bearing failure. The effective life of a bearing is normally defined as the total number of revolutions a bearing can undergo before flaking occurs on either the raceway surface or the rolling element surfaces.

4. 2 Basic rated life and basic dynamic load rating

If a group of seemingly identical bearings is subjected to identical load and operating conditions, they will exhibit a wide diversity in their durability. This can be attributed to the difference in the fatigue of the bearing material itself. This disparity is statistically monitored, and the basic rated life is expressed as "the total number of revolutions that 90% ('90% reliability') of the bearings in an identical group of bearings subjected to identical operating conditions will attain or surpass before flaking due to material fatigue occurs." For bearings operating at constant fixed speeds, it is expressed as the total number of hours of operation.

The basic dynamic load rating is an expression of the load capacity of a rolling bearing and can be described as the constant load under which a bearing can sustain a basic rated life of one million revolutions. For radial bearings it refers to pure radial loads; for thrust bearings it refers to pure axial loads.

The relationship between the basic rated life, basic dynamic load rating and bearing load is given in formula (4.1).

where,

 $\begin{array}{cccc} p=10/3 & \cdots & \cdots & \text{For roller bearings} \\ p=3 & \cdots & \cdots & \text{For ball bearings} \\ L_{10}: \text{ Basic rated life of } 10^6 \text{ revolutions} \\ C & : \text{ Basic dynamic rated load } N \left\{ \text{kgf} \right\} \\ & \text{ Radial bearings: } C_{\text{r}} \\ & \text{ Thrust bearings: } C_{\text{a}} \\ P & : \text{ Bearing load } N \left\{ \text{kgf} \right\} \\ & \text{ Radial bearings: } P_{\text{r}} \\ & \text{ Thrust bearings: } P_{\text{a}} \end{array}$

The basic rated life can also be expressed in terms of hours of revolution, and is calculated as shown in formula (4.2).

$$L_{10h} = \frac{10^6}{60n} \left(\frac{C}{P}\right)^p \dots (4.2)$$

where,

L10h: Basic rated life h

n : Rotational speed, rpm

4.3 Factors affecting bearing life

Apart from bearing load and revolution speed, the factors affecting bearing life include lubricating conditions, internal clearance, roughness of raceway surface, hardness, heat treatment (structure), and installation errors (misalignment). Consider all these factors when using bearings. For details, refer to the relevant catalog.

| Table 4.1 C | Dutline of | bearing | operating | conditions |
|-------------|------------|---------|-----------|------------|
|-------------|------------|---------|-----------|------------|

| Permissible revolutions (N) | Refer to catalog values Note: When lubricating, $F_{\rm W} \cdot n \leq 40 \times 10^4$ rpm $F_{\rm W}$: Roller set bore diameter |
|-----------------------------------|---|
| Roughness of raceway surface | Within 0.4a (Rmax 1.6 μ m) |
| Hardness of raceway surface | HRC58-64 Note: Refer to Section 6.3 for materials and heat treatment hardness. |
| Installation error | Less than 1/2 000 |
| Radial internal clearance | Normal (C2, C3, C4) |

4.4 Installation error and crowning

It is well known that bearing life can be dramatically reduced due to stress concentration at the roller ends (edge load) caused by installation error. "Roller crowning" is employed as a countermeasure to this problem. However, this may reduce the effective contact length of the rollers and reduce the bearing life unless a proper design is made. Installation error and loading conditions may require calculation of the proper crowning value. **Figs. 4.1** to **4.3** show examples of analyses of the contact face pressure according to the computer calculation for reference.

Figs. 4.1 to **4.3** (examples of analyses of contact face pressure) show that the rollers without crowning provide higher edge face pressure, but the rollers with crowning are limited to lower edge face pressure in the range of constant installation error. **Fig. 4.4** shows the relationship (example of computer analysis) between installation error and bearing life. The figure shows the influence of installation errors on the bearing life.



Fig. 4.1



Fig. 4.3



Fig. 4.2



Fig. 4.4 Relationship between installation error and bearing life

4.5 Radial internal clearance, surface roughness, surface hardness and bearing life

Fig. 4.5 shows the relationship between radial internal clearance and bearing life.

Fig. 4.6 shows the relationship between surface roughness and bearing life.

Fig. 4.7 shows the relationship between surface hardness and bearing life.

The following figures show factors affecting bearing life.

4.6 Tips for longer bearing life

To increase the life of needle roller bearings, special heat treatment (AS treatment) has been applied to double or even triple the life compared to the standard product, as determined by bench tests. Moreover, a high oil-film formation capability has been achieved through a recently developed surface-processing technology known as the HL process. As a result, a longer life effect is observed in the areas where surface damage at starting points easily occurs. This development has been well received in the market. For details, refer to catalogs and references.

4. 7 Basic static load rating

When bearings are subjected to loads, they suffer from partial permanent deformation of the contact surface between the rolling elements and the bearing ring. The deformity increases as the load increases, and if this increase in load exceeds certain limits, smooth operation of the bearings is subsequently impaired. Experience has revealed that a total permanent deformity of 0.0001 times the diameter of the rolling element -- occurring at the center of the most



Fig. 4.5 Relationship between radial interna clearance and bearing life



Fig. 4.6 Relationship between surface roughness and life



Fig. 4.7 Influence of hardness on rolling fatigue life

heavily stressed contact point between the raceway and the rolling elements -- can be tolerated without any impairment in running efficiency. The basic static load rating refers to the fixed static load limit at which a specified amount of permanent deformation occurs. It represents pure radial loads for radial bearings and pure axial loads for thrust bearing. These values are listed in the bearing dimensions table. They are listed in the C_{or} column for radial bearings. The contact stresses occurring at the center of the rolling element and raceway contact points under the maximum load are given below.

Roller bearings …4000 MPa (408 kgf/mm²) Ball bearings …4200 MPa (428 kgf/mm²)

Generally, the permissible static equivalent load is limited by the basic static load rating. However, depending on the requirements for smooth operation and friction of revolution, these limits may be greater or lesser than the basic static rated load.

In the following formula (4.3) and **Table 4.2**, the maximum static equivalent load can be determined with consideration for the safety factor S_{0} .

Table 4.2 Minimum safety factor So

| Operating conditions | Roller bearing | Ball bearing |
|---|-------------------|-----------------|
| Demand for high rotational accuracy | 3 | 2 |
| Demand for normal rotating accuracy (Universal application) | 1.5 | 1 |

Remarks 1. For thrust roller bearings using the drawn-cup needle roller bearing and pressed thrust washer, the minimum So value should be 3.

 When vibration and/or shock loads are present, a load factor based on the shock load must be included in the Po max value.

where,

So : Safety factor

Co : Basic static rated load N {kgf} Radial bearings: Cor Thrust bearings: Coa

 P_0 max:

Maximum static equivalent load N {kgf} Radial bearings: *P*or max Thrust bearings: *P*oa max

5. Fitting Needle Roller Bearings

5.1 Fitting machined-ring radial needle roller bearings

The inner and outer rings of needle roller bearings are separable. Both inner and outer rings can be mounted with a given interference, but in the case of a tight fit, the bearing ring subjected to static loads can be provided with a loose fit, considering the ease of mounting and removal of the bearing.

Table 5.1 shows the bearing fit according toa given load.

Table 5.2 shows the recommended fits for radial needle roller bearings (machined ring, with inner rings). However, the interference may be reduced due to temperature increase or surface roughness of fitting surfaces. In this case, refer to the relevant catalogs for details.



Table 5.1 Characteristics and fittings of radial loads

| Table 5.2 | Fitting of needle roller bearings |
|-----------|-----------------------------------|
| Table (a) | Axial allowance |

| | Tolerance | | | |
|----------------|---|------------------------|-------------|--|
| Load type | Load size | Shaft diameter d mm | range class | |
| | Light loads | \sim 50 | j5 | |
| Rotating inner | | ~ 50 | k5 | |
| ring load or | Normal loads | 50~150 | m5 | |
| indeterminate | | 150~ | m6 | |
| | Heavy loads | ~150 | m6 | |
| | loads | 150~ | n6 | |
| Stationary | Medium/low revolutions, light loads | | g6 | |
| inner ring | General applications | All shaft | h6 | |
| load | When high accuracy is required | diameters | h5 | |

Table (b) Housing allowance

| | Tolerance range class | | |
|------------------------|--|----|--|
| Stationary | Normal and heavy loads | J7 | |
| outer ring load | Normal load with double split housing | H7 | |
| | Light loads | M7 | |
| Rotating | Normal loads | N7 | |
| outer ning load | Heavy loads and shock loads | P7 | |
| la data mala ata | Light loads | J7 | |
| Indeterminate loads | Normal loads | K7 | |
| | Heavy loads and shock loads | M7 | |
| When high rotatio | When high rotational accuracy is required with light loads | | |

Remarks: For classifications of light loads, normal loads, and heavy loads, refer to the following values: Light load $Pr \leq 0.06Cr$ Normal load $0.06Cr \le Pr \le 0.12Cr$ Heavy load Pr > 0.12Cr

5.2 Fitting of drawn-cup needle roller bearings

The dimensional accuracy of drawn-cup needle bearings is guaranteed when the standard ring is used because they are designed to correct the deformation by press fitting in the housing hole and their specified dimensional accuracy is assured. Refer to the relevant catalog for the dimensional allowances of roller set bore diameter when press-fitting the standard ring.

Table 5.3 shows the recommended fits of bearings.

during fitting of drawn-cup needle roller bearings is described on the next page for reference. If the housing material is a light alloy, care should be taken in consideration of reduced interference due to temperature increase.

| Table 5 | 2 Ei++ | ing of | bousing | and | chaft |
|----------|--------|--------|---------|-----|-------|
| raple 5. | э гш | ing oi | nousing | anu | Shan |

| Bearing | Hou | ising | Shaft | | |
|----------|--------------|-------------|-----------------------|--------------------|--|
| series | Ferric group | Light alloy | Without inner ring | With inner ring | |
| НК, ВК | N6(N7) | R6(R7) | hE(hC) | k5 (ic) | |
| HMK, DCL | J6 (J7) | M6(M7) | (00) CI | KO (jo) | |
| HCK | F7 | _ | k6 | - | |

The method for examining internal clearance

5.3 Examination of contraction ratio and clearance after mounting

Recommended fits of standard products are listed in catalogs. The calculation method is also provided in detailed examination to be carried out.

1) Calculation of bearing contraction ratio

If a drawn-cup bearing is used, the contraction ratio is calculated as shown in the formula.



 λ : Outer ring contraction ratio

DH : Housing single outside diameter mm d_{nom} : Nominal diameter of fitting section mm

- *de* : Diameter of rolling contact surface of the outer ring mm
- E1 : Young's modulus of housing kgf/mm²
- *E*₂ : Young's modulus of outer ring (21 200 kgf/mm²)

$$S = \frac{d_{\text{nom}}}{D_H}$$
 $t = \frac{d_e}{d_{\text{nom}}}$

2) Setting bore diameter after fitting the actual housing

① Press fitting of standard ring



- H: Housing bore diameter mm
- T : Roller diameter + plate thickness mm
- ${\it D}$: Drawn-cup bearing single outside diameter mm
- Li: Roller set bore diameter after press fitting mm

When the standard ring is press-fit, the dimensions of the roller diameter plus the plate thickness do not change. Accordingly, the roller set bore diameter, L_{i_1} is as follows:

 $L_i = D - 2T - \lambda (D - H) = (1 - \lambda) D - 2T + \lambda H \cdots (2)$

The mean value and standard deviation of the "roller diameter + plate thickness" (=T) can be calculated by the formula (2). The mean value given by formula (2) is:

 $m_{Li} = (1 - \lambda) m_D - m_{2T} + \lambda m_H \dots (3)$

Standard deviation of formula (2) is:

 $\sigma_{Li}^2 = (1 - \lambda)^2 \cdot \sigma_D^2 + \sigma_{2T}^2 + \lambda^2 \sigma_{H^2}$ (4) In the case of the standard ring, because $\sigma_{H^2}^2 = 0$, formula (4) is:

Unknown values in formula (3) and (5) are only m_{2T} and σ_{2T}^2 . Accordingly, substitute known numerical values for (3) and (5) to obtain m_{2T} and σ_{2T}^2 .

② For press fitting of the actual housing, consider the same method as the standard ring press fitting.

Here, terms for press fitting of the actual housing are identified with an apostrophe (') for classification/clarification.

$$m_{Li'} = (1 - \lambda') m_D - m_{2T} + \lambda' m_{H'} \dots (6)$$

$$\sigma_{Li'}^2 = (1 - \lambda')^2 \cdot \sigma_D^2 + \sigma_{2T}^2 + \lambda'^2 \sigma_{H'}^2 \dots (7)$$

- ③ Previously calculated values are substituted for m2T and σ 2T² in formulas (6) and (7).
- ④ The roller set bore diameter, *Li*', for press fitting the actual housing can be calculated by the following formula.

 $Li'=mLi'\pm 3 \sigma Li'$ (8)

- (5) To calculate the radial internal clearance, consider the mean value and standard deviation of the shaft in formulas (6) and (7).
- (6) The target value of radial internal clearance is usually set so that the normal clearance can be obtained. However when used for motor vehicles, recommended values are indicated for the respective components. For details, consult NTN.

6. Shaft and Housing Design

6.1 Shaft and housing accuracy

Because the bearing rings of needle roller bearings are so thin, the accuracy of the surfaces of the shaft and housing in which the bearing is mounted has a considerable effect on the accuracy of the raceways.

Under normal conditions of use, a turned finish is adequate for mating surfaces. However, if a heavy load rating, noise reduction, or high precision are required, a ground finish is necessary.

Table 6.1 shows the dimensional accuracy of the fitting face between the shaft and housing, the shape accuracy, the surface roughness, and the shoulder perpendicularity against the fitting face under normal operating conditions.

With split-type housings, if a grinding undercut is made on the inside diameter of the mating surface, distortion of the outer ring is prevented when the housing is assembled.

| Table 6.1 | Shaft | and | housing | accuracy |
|-----------|-------|-----|---------|----------|
|-----------|-------|-----|---------|----------|

| Characteristic | Shaft | Housing |
|-------------------------------------|-----------|-----------|
| Dimensional accuracy | IT6 {IT5} | IT7 {IT6} |
| Circular deviation (Maximum) | | |
| Cylindrical deviation | 110 (112) | 114 (113) |
| Shoulder perpendicularity (Maximum) | IT5 {IT4} | IT5 {IT4} |
| Roughness of fitting surface | 0.8a | 1.6a |

Note: Bearings with an accuracy exceeding Class 5 appear within brackets.

6.2 Raceway accuracy

A shaft and housing are often used directly in place of needle roller bearings with raceways. To regulate the radial internal clearance to the specified allowance and ensure high revolution accuracy, the dimensional accuracy, shape accuracy, and surface roughness of the raceway must be equivalent to that of the raceway surface of the bearing. **Table 6.2** shows the accuracy and surface roughness of the raceway surface.

Table 6.2 Accuracy of raceway surface

| Characteristic | Shaft | Housing |
|-------------------------------------|-------------------------------|-----------|
| Dimensional accuracy | IT5 {IT4} | IT6 {IT5} |
| Circular deviation | IT3 {IT2} | IT4 {IT3} |
| Cylindrical deviation | | |
| Shoulder perpendicularity (Maximum) | IT3 {IT2} | IT3 {IT2} |
| Axial runout (Maximum) | IT5 {IT4} | |
| Thrust bearing | | |
| Surface roughness | Within 0.4a (Rmax1.6 μ m) | |

Remarks: Components with high rotational accuracy appear in brackets.

6.3 Raceway materials and their hardness

When the single outside diameter or single bore diameter of a shaft or housing is used in place of a raceway, the surface hardness should be HRC58 to HRC64 in order to maintain sufficient load-bearing capacity. In this case, the types of materials shown in **Table 6.3** should be used following suitable heat treatment.

When raceways have been hardened by carburization or induction, the hardness penetration is defined as the adequate depth of a hardened surface layer with a hardness of HV550 when measured from the surface. The minimum depth of hardness can be calculated according to Formula 6.1.

 $E_{ht} \min \ge 0.8D \text{W} (0.1 \pm 0.002D \text{W}) \cdots (6.1)$ where,

Eht min : Minimum hardness adequate depth mm *D*w : Roller diameter mm

Table 6.3 Raceway materials and their hardness

| Steel type | Code | Standard |
|------------------------------------|---------|------------|
| High carbon chromium bearing steel | SUJ2 | JIS G 4805 |
| Carbon tool steel | SK3 | JIS G 4401 |
| Nickel chromium molybdenum steel | SNCM420 | JIS G 4103 |
| Chromium steel | SCr420 | JIS G 4104 |
| Chromium molybdenum steel | SCM420 | JIS G 4105 |
| Nickel chromium steel | SNC420 | JIS G 4102 |
7. Tips on Bearing Use

Needle roller and cage assembly bearings have various advantages, yet they also have limitations. Before using these bearings, follow the precautions listed in this section in order to ensure the optimal service life.

(1) Minimal space for lubricant

Although a small cross-sectional bearing height is an advantage, this space includes the cage assembly, which leaves very little room available for the lubricant.

In addition, as the cage assembly is guided by either the inner ring or outer ring, the lubrication of these guiding surfaces must be considered.

Accordingly, except in the case of extremely low speed operation, a lubrication hole must be provided for oil lubrication, or, in the case of grease lubrication, adequate maintenance must be ensured through rigid scheduling of lubrication intervals.

(2) Axial loads

Except for a partial grouping, needle roller bearings cannot carry axial loads. Axial loads caused by pure radial loads are generally about 3% to 5% of radial loads, and paying attention to the lubricating surface is normally sufficient. However, if the axial loads act expressly on a bearing, consider the use of thrust bearings.



🗊 Tips & Hints

Guide surface in the axial direction

When a needle roller and cage assembly is used alone and is guided in an axial direction at the shaft shoulder (**Fig. 7.1**), the shaft shoulder surface should have a burr-free, improved finish where it contacts the side face of the cage assembly. In high-speed applications, the contact faces should be hardened and finish-ground. Thrust rings stamped from spring plate with a press are also suitable as axial guides for the cage assembly.

When the cage assembly is guided in an axial direction through the use of a locating snap ring (**Fig. 7.1**), it is recommended that a thrust ring be used between the cage assembly and locating snap ring so that the cut ends of the locating snap ring do not directly contact the cage assembly.

Generally, when a shaft with a radial needle roller bearing is to be positioned in the axial direction, ball bearings or thrust bearings are used to determine the axial direction. However, when the axial direction load is small and the revolution speed is not high (for example, an idler gear in a gear box), mount the thrust ring on the shaft and position the thrust ring by striking it against the outer ring or housing shoulder, as shown in **Fig. 7.2**. In this design, lubrication on the guide surface requires care. **Fig. 7.3** shows an example of a thrust ring with an oil groove processed on the guide surface. This oil groove and the flat part of the shaft must be chamfered and smoothed.



(3) Shape errors

Needle roller and cage assembly bearings have good accuracy and can be rotated at high speed with small internal clearance. The accuracy of the bearing box of the single outside diameter of the shaft or single bore diameter requires special care, because the shaft or housing shapes directly appear on the raceway surface due to the thin materials used for the inner ring or outer ring.

Regarding general housing accuracy, inferior accuracy is found where differences exist in ribs and material thicknesses.

Care should be taken because needle roller bearings used under such conditions contribute to negative clearances and generate large axial loads or result in heating and seizure.



4) Misalignment

Always be careful to keep the axial deflection to less than 1/2 000, as twisting can have a serious effect due to the high rigidity. When the internal clearance is small, or a wide needle roller bearing is used, a particular effort is required to ensure minimal misalignment.



(5) Materials, hardness, and accuracy of the raceway surface

For general bearings, an appropriate bearing steel is used for inner rings, outer rings, and rollers, and a precision finishing process is applied following careful heat treatment. On the other hand, needle roller bearings are also used without inner rings only, or without either inner or outer rings (in other words, the shaft or housing is used in place of raceway surface), to achieve certain advantages.

The basic rated static load listed in catalogs refers to the bearings mentioned above. If the shaft or housing is used in place of a bearing ring, it should comply with the inner and outer rings of the bearing. Consequently, it is necessary to select a flawless steel containing no nonmetals and to produce a surface hardness of HRC 58-64 through careful heat treatment. If only the surface is hardened by carburization, it is recommended that careful attention be paid to ensure that the hardened surface layer has adequate depth and surface hardness.

The roughness of the raceway surface should be in the range of 0.4a ($R_{max} \times 16 \mu m$), and there should be no winding along the circumference. The roundness and cylindricity should be less than 50% of the dimensional allowance.



8. Engineering Data

8.1 Track loading capacity of cam followers and roller followers

The track loading capacity is calculated by the relationship between the stress and Hertzian stress by establishing a standard hardness (standard tension stress) with the relationship between the hardness and pure tension stress of the materials. The methods for setting the standard hardness (tension stress) vary slightly by manufacturer, but the accompanying table from the JIS Iron and Steel Handbook (showing approximate values based on the revised conversion table of JIS Z8413) is used to determine the relationship between hardness and tension stress. HRC40 = σ =127 kgf/mm² is adopted as the standard hardness (tension stress).

(1) Track capacity factor

The tension stress of a material increases as the hardness increases; at the same time, the track loading capacity also increases. In this case, the actual track loading capacity can be calculated by multiplying the track loading capacity with the track capacity factors listed in **Table 8.1**.

Note: The calculated track loading capacity shown above is based on the pure tension stress, but not the permissible Hertzian stress. In general, the stress (corresponding force) causing material creep is greater than the tension stress. With static loads in particular, this track loading capacity is considered a safe value.

Example: If the track capacity with a specific hardness is calculated with the track capacity factor:

When the track loading capacity is C_t , the track capacity factor is A at the subject hardness, and the track loading capacity can be found as follows:

 $Ct' = A \cdot Ct$

For NATR15X, if the hardness is HRC 50: $C_t = 1\ 220\ \text{kgf},\ A=1\ 987$ $\therefore C_t'=1\ 987 \times 1\ 220=2\ 424\ \text{kgf}$

σ max= 60.9 $\sqrt{\frac{F_{ m r} \Sigma \rho}{B_{ m eff}}}$

• If the outer ring is spherical with R:

(2) References (Calculation of track capacity)

$$\sigma \max = \frac{187}{\mu \nu} \sqrt[a]{(\Sigma \rho)^2 F_r}$$

If the outer ring is cylindrical:

where,

σ max=127kgf/mm²

 $F_{\rm r}$: Track loading capacity (kgf)

 $\Sigma \rho$: Sum of curvatures

 ${\it B}_{\rm eff}$: Effective contact length (mm) where (outer ring width – 2×chamfer)

Table 8.1 Track capacity factor

| Hardness HRC | Tension stress kgf/mm ² | Cylindrical outside surface | Spherical outside surface |
|-----------------|---------------------------------------|--------------------------------|------------------------------|
| 20 | 77 | 0.368 | 0.223 |
| 21 | 79 | 0.387 | 0.241 |
| 22 | 80 | 0.397 | 0.250 |
| 23 | 82 | 0.417 | 0.269 |
| 24 | 84 | 0.437 | 0.289 |
| 25 | 86 | 0.459 | 0.311 |
| 26 | 88 | 0.480 | 0.333 |
| 27 | 90 | 0.502 | 0.356 |
| 28 | 93 | 0.536 | 0.393 |
| 29 | 95 | 0.560 | 0.419 |
| 30 | 97 | 0.583 | 0.446 |
| 31 | 100 | 0.620 | 0.488 |
| 32 | 102 | 0.645 | 0.518 |
| 33 | 105 | 0.684 | 0.565 |
| 34 | 108 | 0.723 | 0.615 |
| 35 | 110 | 0.750 | 0.650 |
| 36 | 114 | 0.806 | 0.723 |
| 37 | 118 | 0.863 | 0.802 |
| 38 | 120 | 0.893 | 0.844 |
| 39 | 124 | 0.953 | 0.931 |
| 40 | 127 | 1.0 | 1.0 |
| 41 | 132 | 1.080 | 1.123 |
| 42 | 136 | 1.147 | 1.228 |
| 43 | 141 | 1.233 | 1.369 |
| 44 | 146 | 1.322 | 1.519 |
| 45 | 151 | 1.414 | 1.681 |
| 46 | 156 | 1.509 | 1.853 |
| 47 | 161 | 1.607 | 2.037 |
| 48 | 167 | 1.729 | 2.274 |
| 49 | 172 | 1.834 | 2.484 |
| 50 | 179 | 1.987 | 2.800 |
| 51 | 186 | 2.145 | 3.141 |
| 52 | 192 | 2.286 | 3.455 |
| 53 | 199 | 2.455 | 3.847 |
| 54 | 205 | 2.606 | 4.206 |
| 55 | 212 | 2.787 | 4.652 |

8.2 Outer ring strength

Generally, outer rings do not break under normal operating loads, but the following is calculated as a checkpoint for operations under shock loads and heavy loads.



Fig. 8.1 shows outer ring shapes. They are calculated with the following formula. Outer ring breaking strength means the fracture strength of rollers when in a bridge formation.

To determine the fracture strength, generally 180 kgf/mm² can be applied to bearing steel, but we use 120 kgf/mm² as max here in keeping with our own experiences and in consideration of applied concentration. For normal use, the stress is recommended to be less than 20 kgf/mm² and the strength in this case is calculated as follows.

$$P = \frac{4\pi}{1+f(\alpha)} \times \frac{D-2h}{h(D-2h_2)^2} \times I \times \alpha \text{ (kgf)}$$
$$f(\alpha) = \frac{(\pi-\alpha)\sin\alpha - (1+\cos\alpha)}{2\cos\alpha} \text{ (rad.)}$$
$$\alpha = \frac{180}{Z} \text{ (rad.)}$$

where,

- I : Outer ring cross-section secondary moment (mm⁴)
- Z : Number of rollers
- σ : Breaking stress (kgf/mm²)
- P: Breaking load (kgf)

Use special care if a spring washer is used to lock the screws, as this may reduce the stud strength.

8. 3 Stud strength of cam followers





In **Fig. 8.2**, if the load Fr acts at the center of the width of the outer ring, the stud receives the bending moment equivalent to $F_{\rm T} \times \ell$. Tension stress created on the stud surface by this moment is established as σ 1. Further, the stud itself is tightened with a nut or machine unit, generating tension stress σ 2.

The sum of the tension stresses σ 1+ σ 2 must not exceed the material allowance. The recommended tightening torque listed in the catalogs is σ 2=10 kg/mm² for most types.

The stud strength is calculated for all types in this section. However, the spring washer is not used in this case*.

- As shown in the figure, loads should act at the center of the outer ring.
- The shaft cross-section should follow the drawing.
- The permissible stress in each case refers to the following items. The tension stress due to tightening is set as σ 2÷10 kg/mm².

Static bending stress $\sigma = 140 \text{kgf/mm}^2$ Repeating bending stress (pulsating)

 $\sigma = 100 \text{kgf/mm}^2$ Repeating bending stress (alternating) $= 40 \text{kgf/mm}^2$

$$\sigma_1 = \frac{M}{Z} = \frac{Fr \cdot \ell}{Z}$$

where,

 $\sigma_1 + \sigma_2 = \sigma$

Z : Shaft section modulus (mm³)

Fr: Load (Stud strength in kgf)

| | Calc | ulation/Ro | ller follow | er NATR, | NATV, N | UTR | |
|----------------|---------|----------------|-----------------|-----------------|---------------------|-----------------------|-------------|
| Outside | 0 · | Rated loa | id {kgf} | Track loading c | apacity {kgf} | Outer ring str | ength {kgf} |
| diameter mm | Series | Cr | Cor | Cylinder | Spherical surface R | Static $\sigma = 120$ | σ=20 |
| | NATR5 | 395 | 400 | | | 1 210 | 202 |
| 16 | NATV5 | 640 | 910 | 350 | 110 | 1 940 | 325 |
| | NATR6 | 460 | 520 | | | 1 840 | 305 |
| 19 | NATV6 | 735 | 1 140 | 415 | 141 | 2 850 | 470 |
| 24 | NATR8 | 675 | 745 | 690 | 102 | 2 600 | 430 |
| 24 | NATV8 | 1 050 | 1 580 | 000 | 193 | 4 200 | 700 |
| 30 | NATR10 | 765 | 930 | 785 | 267 | 4 900 | 820 |
| | NATV10 | 1 190 | 1 980 | 100 | 201 | 7 600 | 1 270 |
| 32 | NATR12 | 865 | 1 130 | 835 | 291 | 5 200 | 860 |
| | NATV12 | 1 280 | 2 250 | | 201 | 8 000 | 1 330 |
| 35 | NATR15 | 1 320 | 2 050 | 1 220 | 325 | 7 600 | 1 270 |
| | NATV15 | 1 830 | 3 750 | | | 11 800 | 1 970 |
| 40 | NATR17 | 1 390 | 2 250 | 1 480 | 390 | 11 800 | 1 970 |
| | NATV17 | 1 930 | 4 150 | | | 18 300 | 3 050 |
| 47 | NATR20 | 2 070 | 3 350 | 2 150 | 480 | 17 200 | 2 850 |
| | NATV20 | 2 900 | 6 100 | | | 26 000 | 4 350 |
| 52 | NATR25 | 2 280 | 4 000 | 2 370 | 565 | 17 800 | 2 950 |
| | NATE 25 | 3 150 | 7 350 | | | 28 500 | 4 750 |
| 62 | NATR30 | 3 600 | 6 550 | 3 350 | 710 | 21 000 | 3 500 |
| | NATD25 | 4 750 | 7000 | | | 31 500 | 5 200 |
| 72 | NATV25 | 3 850 | 7600 | 3 750 | 820 | 30 000 | 5 000 |
| | NATR40 | 5 200 | 0.150 | | | 45 000 | 7 500 |
| 80 | NATV40 | 4 950 6 850 | 9 150 17 100 | 4 500 | 1 000 | 27 500 | 4 600 |
| 95 | NATR45 | 5 150 | 9 950 | 4 800 | 1.060 | 28 000 | 1 400 |
| 00 | NATR50 | 5 300 | 10 700 | 4 000 | 1 000 | 28 500 | 4 700 |
| 90 | NATV50 | 7 600 | 20 400 | 5 100 | 1 160 | 46 500 | 7 700 |
| 35 | NUTR202 | 2 280 | 2 620 | 1 220 | 325 | 4 350 | 730 |
| 40 | NUTR203 | 2 450 | 2 970 | 1 480 | 390 | 8 000 | 1 340 |
| 42 | NUTR302 | 2 280 | 2 620 | 1 460 | 415 | 10 900 | 1 820 |
| | NUTR303 | 2 450 | 2 970 | 1 740 | | 16 700 | 2 800 |
| 47 | NUTR204 | 3 950 | 4 900 | 2 150 | 480 | 9 100 | 1 520 |
| 50 | NUTR304 | 3 950 | 4 900 | 2 370 | 505 | 15 700 | 2 600 |
| 52 | NUTR205 | 4 350 | 5 850 | 2 370 | COC | 10 100 | 1 680 |
| 62 | NUTR305 | 4 350 | 5 850 | 2 830 | 710 | 25 500 | 4 250 |
| | NUTR206 | 5 750 | 7 400 | 3 350 | 710 | 12 700 | 2 120 |
| 72 | NUTR306 | 5 750 | 7 400 | 3 900 | 820 | 29 000 | 4 800 |
| | NUTR207 | 6 350 | 8 700 | 3 750 | | 19 400 | 3 250 |
| 80 | NUTR307 | 6 350 | 8 700 | 4 150 | 1 000 | 34 500 | 5 700 |
| | NUTR208 | 8 850 | 12 700 | 4 500 | | 16 900 | 2 800 |
| 85 | NUTR209 | 9 350 | 14 000 | 4 800 | 1 060 | 17 800 | 2 950 |
| 90 | NUTR308 | 8 850 | 12 700 | 5 100 | 1 160 | 35 500 | 5 900 |
| | NUTR210 | 9 800 | 15 300 | 5 100 | | 18 600 | 3 100 |
| 100 | NUTR309 | 9 350 | 14 000 | 5 650 | 1 330 | 49 000 | 8 100 |
| 110 | NUTR310 | 9 800 | 15 300 | 6 200 | 1 500 | 64 000 | 10 700 |

| Calculation/Cam follower KR, KRV, NUKR | | | | | | | | | | | |
|--|---------|----------|----------|---------------|---------------------|-----------------------|--------------------------|---------------------------|-----------------------|--------------|--|
| Outside | Carias | Rated lo | ad {kgf} | Track loading | capacity {kgf} | Stud | strength | [kgf] | Outer ring st | rength {kgf} | |
| diameter mm | Series | Cr | Cor | Cylinder | Spherical surface R | Static $\sigma = 140$ | Pulsating $\sigma = 100$ | Alternating $\sigma = 40$ | Static $\sigma = 120$ | σ=20 | |
| 16 | KR16 | 395 | 400 | 350 | 110 | 450 | 315 | 104 | 1 210 | 202 | |
| | KRV16 | 640 | 910 | 550 | 110 | 430 | 515 | 104 | 1 940 | 325 | |
| 19 | KR19 | 460 | 520 | 415 | 141 | 1 070 | 740 | 247 | 1 840 | 305 | |
| | KRV19 | 735 | 1 140 | | | | | | 2 850 | 470 | |
| 22 | KR22 | 515 | 635 | 525 | 172 | 1 920 | 1 330 | 445 | 2 600 | 435 | |
| | KRV22 | 810 | 1 360 | | | | | | 3 950 | 660 | |
| 26 | KR20 | 515 | 1 260 | 620 | 216 | 1 920 | 1 330 | 445 | 5 400 | 910 | |
| | | 765 | 030 | | | | | | 8 200 | 820 | |
| 30 | KRV30 | 1 1 9 0 | 1 080 | 785 | 267 | 2 850 | 1 990 | 660 | 7 600 | 1 270 | |
| | KR32 | 765 | 930 | | | | | | 6 4 0 0 | 1 070 | |
| 32 | KRV32 | 1 1 9 0 | 1 980 | 835 | 291 | 2 850 | 1 990 | 660 | 9 900 | 1 650 | |
| | KR35 | 1 210 | 1 760 | | | | | | 9 300 | 1 560 | |
| 35 | KRV35 | 1 750 | 3 400 | 1 220 | 325 | 5 300 | 3 650 | 1 220 | 14 100 | 2 360 | |
| | NUKR35 | 2 280 | 2 620 | | | | | | 4 350 | 730 | |
| | KR40 | 1 390 | 2 250 | | | | | | 11 800 | 1 970 | |
| 40 | KRV40 | 1 930 | 4 150 | 1 480 | 390 | 6 900 | 4 750 | 1 580 | 18 300 | 3 050 | |
| | NUKR40 | 2 450 | 2 970 | | | | | | 8 000 | 1 340 | |
| | KR47 | 2 070 | 3 350 | | | | | | 17 200 | 2 850 | |
| 47 | KRV47 | 2 890 | 6 100 | 2 150 | 480 | 7 900 | 5 500 | 1 830 | 26 000 | 4 350 | |
| | NUKR47 | 3 950 | 4 900 | | | | | | 9 100 | 1 520 | |
| | KR52 | 2 070 | 3 350 | | | | | | 25 500 | 4 250 | |
| 52 | KRV52 | 2 890 | 6 100 | 2 370 | 565 | 7 900 | 5 500 | 1 830 | 38 500 | 6 500 | |
| | NUKR52 | 4 350 | 5 850 | | | | | | 10 100 | 1 680 | |
| ~~ | KR62 | 2 960 | 5 650 | 0.500 | 740 | 44 500 | 7 000 | 0.050 | 48 000 | 8 000 | |
| 62 | | 3 950 | 9 850 | 3 500 | 710 | 11 500 | 7 900 | 2 650 | 12 400 | 12 400 | |
| | KD72 | 2 060 | 7 400 | | | | | | 75 000 | 2 240 | |
| 72 | KRV72 | 3 950 | 9 850 | 3 900 | 820 | 11 500 | 7 900 | 2 650 | 116 000 | 19 400 | |
| | NUKR72 | 6 350 | 8 700 | 0.000 | 020 | | 1 000 | 2 000 | 20 500 | 3 400 | |
| | KR80 | 4 500 | 8 800 | | | | | | 8 100 | 13 500 | |
| 80 | KRV80 | 5 800 | 14 700 | 5 400 | 1 000 | 18 600 | 12 900 | 4 300 | 121 000 | 20 200 | |
| | NUKR80 | 10 300 | 15 400 | | | | | | 19 600 | 3 250 | |
| 85 | KR85 | 4 500 | 8 800 | 5 750 | 1 060 | 18 600 | 12 900 | 4 300 | 97 000 | 16 200 | |
| | KR90 | 4 500 | 8 800 | | | | | | 11 500 | 19 200 | |
| 90 | KRV90 | 5 800 | 14 700 | 6 100 | 1 160 | 18 600 | 12 900 | 4 300 | 17 100 | 28 500 | |
| | NUKR90 | 10 300 | 15 400 | | | | | | 41 500 | 6 880 | |
| 100 | NUKR100 | 12 100 | 17 000 | 8 050 | 1 300 | 26 000 | 17 900 | 6 000 | 62 000 | 10 300 | |
| 120 | NUKR120 | 17 600 | 27 100 | 11 500 | 1 670 | 35 000 | 24 700 | 8 200 | 90 000 | 15 000 | |
| 140 | NUKR140 | 20 500 | 30 000 | 15 500 | 2 040 | 47 000 | 32 500 | 10 900 | 140 000 | 22 700 | |
| 150 | NUKR150 | 26 300 | 39 000 | 17 600 | 2 250 | 57 000 | 39 500 | 13 100 | 112 000 | 18 700 | |
| 160 | NUKR160 | 27 900 | 41 000 | 19 800 | 2 450 | 68 000 | 47 000 | 15 700 | 127 000 | 21 100 | |
| 170 | NUKR170 | 32 500 | 48 500 | 22 200 | 2 650 | 79 000 | 55 000 | 18 200 | 136 000 | 22 600 | |
| 180 | NUKR180 | 37 500 | 56 500 | 25 800 | 2 840 | 88 000 | 61 000 | 20 000 | 155 000 | 26 000 | |
| | | | | | | | | 1 | | | |

8.4 Calculating tightening torque of cam follower

The following relationship exists between the screw tightening torque and tightening force.

$$T = \frac{F}{2} \left\{ (1.15 \,\mu + \tan \beta) \, d_2 + \,\mu \, w d_w \right\}$$

where,

- T : Tightening torque
- F : Tightening force
- μ : Friction coefficient
- μw : Nut bearing surface coefficient
- β : Lead angle of thread
- d2 : Effective diameter of thread
- dw : Effective diameter between the nut bearing surface and mounting hole (with hexagon nut)

Here, $\mu = \mu w = 0.15$ is assumed. $d_2 = 0.92d$

 $\tan\beta = \mathbf{P}/(\pi \cdot d\mathbf{2})$

$$d_{\rm W} = \frac{0.608B^3 - 0.524Di^3}{0.866B^2 - 0.785Di^2}$$

where,

d: Nominal diameter of thread

P : Thread pitch

 ${\it B}~$: Width across flat of nut

Di : Mounting hole bore

 $Di \doteq d$ is established.

On the other hand, the relationship between the tightening force and tension stress is expressed as follows:

$$\sigma_2 = \frac{F}{S}$$

 σ <code>2</code> : Tension stress

S : Shaft cross-section area

where, σ_2 refers to the setting according to the cam follower stud strength calculation in Section 8.3.

 $\sigma_2 = 10 \text{kgf/mm}^2$ $F = S \cdot \sigma_2 = S \cdot 10 \text{ kgf}$ The tightening torque is as follows:

$$F = \frac{F}{2} \left\{ (0.1725 + \tan \beta) \times 0.92d + 0.15dw \right\} \times 10^3$$

$$F = \frac{S \cdot 10}{2} \{ (0.1725 + \frac{P}{0.92d \cdot \pi}) \times 0.92d \}$$

$$+0.15 \times \frac{0.608B^3 - 0.524d^3}{0.866B^2 - 0.785d^2} \times 10^3 \text{ {kgf-m}}$$

Table. 8.2 Calculation of standard cam follower

| Calculation | Tighteni T {kg | ng torque f—m} | Tightening force F {kgf} | | |
|-----------------------|---|---|---|---|--|
| Series KRV NUKR | Shaft lubrication hole is taken into account | Shaft lubrication hole is not taken into account | Shaft lubrication hole is taken into account | Shaft lubrication hole is not taken into account | |
| #16 | — | 0.36 | _ | 283 | |
| #19 | — | 0.84 | — | 503 | |
| #22 | 1.36 | 1.62 | 660 | 785 | |
| #26 | 1.36 | 1.62 | 660 | 785 | |
| # 30 | 2.05 | 2.73 | 848 | 1 131 | |
| # 32 | 2.05 | 2.73 | 848 | 1 131 | |
| # 35 | 5.33 | 6.20 | 1 728 | 2 011 | |
| # 40 | 7.78 | 8.75 | 2 262 | 2 545 | |
| # 47 | 10.0 | 11.9 | 2 639 | 3 142 | |
| # 52 | 10.0 | 11.9 | 2 639 | 3 142 | |
| #62 | 18.1 | 20.4 | 4 021 | 4 524 | |
| #72 | 18.1 | 20.4 | 4 021 | 4 524 | |
| #80 | 36.9 | 39.7 | 6 566 | 7 069 | |
| # 85 | 36.9 | 39.7 | 6 566 | 7 069 | |
| #90 | 36.9 | 39.7 | 6 566 | 7 069 | |
| #100 | 64.7 | 68.1 | 9 676 | 10 179 | |
| #120 | 104 | 108 | 13 352 | 13 854 | |
| # 140 | 157 | 161 | 17 593 | 18 096 | |
| # 150 | 199 | 203 | 20 735 | 21 237 | |
| #160 | 253 | 258 | 24 127 | 24 630 | |
| #170 | 310 | 315 | 27 772 | 28 274 | |
| # 180 | 374 | 380 | 31 667 | 32 170 | |

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Needle roller and welded cage assemblies with large diameters



We developed needle bearings for idlers in the transmissions of large vehicles. Conventional welded cage assemblies have been difficult to use in this application.

Features

- The use of a plate thicker than the former welded cage assembly improves the strength of the cage assembly.
- 2) Ensures a stable roller guide surface and roller retention.
- For excellent value, bearings are available with a roller set bore diameter of up to 120 mm dia.







Bearing series and characteristics

| KJ······S | Outside diameter guide cage assembly Made from thin band steel. Accommodates medium-size shaft diameters ranging from 20 to 40 mm dia. Used mainly for car-class transmissions. |
|------------|---|
| KVS | Outside diameter guide cage assembly. Made from relatively thick steel plate. Cage assembly has high-rigidity ring. Accommodates large diameters. Used mainly for truck-class transmissions. Conventional type does not have rollers installed in the welded pocket. |
| SKV······S | KV······S is temporarily welded, then the weld is separated to create a single-split cage assembly. Has anti-fretting feature. (See following chart.) |
| GKV·····S | KV······S is cut apart at almost half its length, and is then bent to form a double-split (half-split) type cage assembly. Has anti-fretting feature. (See following chart.) |



Measured data on the relative rotation speeds according to the difference in cage assembly

Full-complement needle roller bearings for sliding seats



A rolling mechanism is employed to improve the sliding motion of rear seats in minivans.

Structure

Features

Economical full-complement roller bearings with thick pressed outer ring

- Thick steel plate is applied directly to the outside surface of the pressed outer ring as the rolling surface for the roller follower.
- A new pressing process has been introduced for mating components, so less chamfering of the outside surface of the pressed outer ring is required.
- 3) All press processes are designed for economy.



Rail cross section



Bearings for Rocker Arms



Features

A rocker arm bearing rotates in contact with the engine camshaft. The surface roughness of shaft L is usually R_{max} 2 to 4 μ m (Ra 0.5 to 1 μ m), and may cause peeling of the surface of the rocker arm bearing. To prevent this, the HL treatment is standard for **NTN** rocker arm bearings.



Design comparison

Automotive manufacturers, both domestic and international, place the highest priority on the technical issue of improving fuel consumption. One aspect of this commitment has been the aggressive pursuit of low-friction engine design. It has been determined that the friction loss of the valve system is high in the low-speed and middle-speed ranges, which are the most frequently used.

Usually, a sliding mechanism known as the "slipper system" is employed, but the needle roller type has been adopted to improve fuel consumption and performance.

Structure

Both ends of the shaft are caulked and fixed with a yoke.





Difference in drive torques

Needle roller bearings with solid grease



The bearing is prelubricated with thermosetting grease to ensure maximum advantage.

Application



Features

1) Less lubricant leakage

When thermosetting grease is used, oil is gradually supplied to the rolling contact surface through the generation of heat and centrifugal force during operation of the bearing. As a result, there is less oil leakage.

2) Proper lubrication characteristics The lubricant is unlikely to leak, even if strong vibrations or a large centrifugal force acts on the bearing. The lubricant will not become emulsified or flow out even if water enters.

Lubrication characteristics are superior to the general grease. However, it may get rust if moisture remains in the shaft, or if the bearings are used under the condition to be forced to get water splash on. Accordingly, we recommend you to take sealing measures such as using the additional seal.

Notices for use

Operate the bearing only in an ambient temperature range of –20 to 80°C. For continuous operation, the ambient temperature should not exceed 60°C.

Care should be taken since these bearings cannot be used under the condition that the organic solvents (such as acetone, pertoleumbenzine and white kerosene) may splash on.

Table of operation results of solid grease bearings (Extracts)

| Reason for selection | Application | Actual bearing (Type) | |
|-----------------------------------|--|---|-----------------------------|
| No greasing | Sheet printers Flip-over trunk Sheet printers Claw shaft oscillating Sheet printers parts Sheet printers Register device Sheet printers Transport section Packaging machines Link mechanism Mechanical presses Guide roller Bottling machines Rotary Two-wheeled vehicles Suspension | Machined-ring bearings Cam followers, Machined-ring bearings Machined-ring bearings Drawn-cup bearings Machined-ring bearings, cam followers Machined-ring bearings Integrated thrust bearings Full-complement drawn-cup bearings | (1.12 (1.12 (1.12) |
| Protection against grease leakage | Automatic weaving machines Sheet printers Roller with ini Medicine packaging machines Slurry pump Crankshaf Painting machines | Machined-ring bearings Drawn-cup bearings Machined-ring bearings Machined-ring bearings Drawn-cup bearings | |
| Water sealing | Food machines Conveyors Confectionery machines Conveyors Food machinery Cutting roller for Packaging machines Kamaboko boiled fish past Water jet room Traveling section Canning machinery Guide rollers Rollers of window-washer gondola winch | Machined-ring bearings Roller followers Machined-ring bearings Drawn-cup bearings Drawn-cup bearings Cam followers Drawn-cup bearings | يدين ميتنا مستحد |
| Dust protection | Transferring unit Filling machines Bag-forming machines Presses Steel facilities Ceramic machinery (for special equipment | Drawn-cup bearings Cam followers Machined-ring bearings Roller followers Cam followers Cam followers | |

Cam followers/Roller followers with outer ring grooves



Cam followers with outer ring grooves (above) and various roller followers with outer ring grooves (right)



Bearings are provided with grooves on the outside diameter of the outer ring to match the shape of the mating material. These bearings are used as guide rollers. The groove shape can be processed as an R or V shape. Mounting instructions are identical to those of standard bearings.

Features

- The R-groove shape can be mirror-finished according to the requirements of the application. (Generally, turning followed by lapping.)
- 2) The basic design accommodates the cam follower standard.
- 3) Easy mounting.
- 4) Can also be used as an economical guide roller for direct drive machines.

Applications

- Guide rollers for steel wires and steel pipes
- Leveling rollers
- Rope-gripping devices for winches of window-washing gondolas



Example applications of poly-lubricated bearings





Roller follower with outer ring with center rib



A roller follower with an outer ring with a center rib (Fig. 1 NUTW Series) comprises

cylindrical rollers, an outer ring, an inner ring, side plates, and shield plates. The outer ring follows a rolling motion on the raceway (track). The outer ring is designed to be thick in order to withstand shock loads, as it operates in direct contact with the track.

In addition to the NUTR Series (**Fig. 2**), which was introduced in a succession of models, the width of the outer ring has been widened, the center rib has been equipped with an outer ring, and the roller follower has been designed to carry larger axial loads. The outside surface of the outer ring is spherical or cylindrical. An outer ring with a spherical surface is effective at easing edge loads due to mounting error.



Features

- 1) The outer ring is designed to be thick in order to withstand shock loads.
- The outside surface can have a spherical or cylindrical surface.
- The outer ring is equipped with a center rib, making it effective against axial loads and the moment loading effect.
- Increasing the prelubricated grease capacity increases the lubrication effect and contributes to longer service life.

Applications

- 1) Under heavy loads or shock loads
- 2) As a cam mechanism or guide roller for straight and curved lines
- Where moment loads act due to installation errors



NUTW Series

Dimensions

| Outside diameter | Bearing series | | | Dimensions (mm) | | | | | Basic load rating | | Mass |
|-------------------------------|-------------------------|---------------------------|----|-----------------------------------|----|----|-------------|-------------|---------------------|---------------------|-------------|
| mm D ⁻⁰ 0.05 | Spherical outer ring | Cylindrical outer ring | d | В | С | e | $F_{\rm W}$ | ❷ Ƴs min | Cr { | kgf} Cor | (reference) |
| 35 | NUTW202 | NUTW202X | 15 | 22 ⁰ _{-0.210} | 21 | 20 | 19 | 0.3 | 24 100 {2 460} | 28 300 {2 880} | 0.115 |
| 40 | NUTW203 | NUTW203X | 17 | 24 ⁰ _{-0.210} | 23 | 22 | 21.5 | 0.3 | 26 000 {2 650} | 32 000 {3 250} | 0.167 |
| 47 | NUTW204 | NUTW204X | 20 | 29 ⁰ _{-0.210} | 28 | 27 | 25.5 | 0.3 | 40 500 {4 150} | 51 500 {5 250} | 0.280 |
| 52 | NUTW205 | NUTW205X | 25 | 29 ⁰ _{-0.210} | 28 | 31 | 30 | 0.3 | 45 000 {4 600} | 61 500 {6 250} | 0.322 |
| 62 | NUTW206 | NUTW206X | 30 | 35 ⁰ _{-0.210} | 34 | 38 | 35 | 0.3 | 59 500 {6 050} | 77 000 {7 900} | 0.549 |
| 72 | NUTW207 | NUTW207X | 35 | 35 ⁰ _{-0.210} | 34 | 44 | 41.5 | 0.6 | 65 000 {6 650} | 91 000 {9 250} | 0.747 |
| 80 | NUTW208 | NUTW208X | 40 | 38 ⁰ _{-0.250} | 36 | 51 | 47.5 | 0.6 | 90 500 {9 250} | 131 000 {13 400} | 0.953 |
| 85 | NUTW209 | NUTW209X | 45 | 38 ⁰ _{-0.250} | 36 | 55 | 52.5 | 0.6 | 95 500 {9 750} | 144 000 {14 700} | 1.03 |
| 90 | NUTW210 | NUTW210X | 50 | 38 ⁰ _{-0.250} | 36 | 60 | 57 | 0.6 | 100 000 {10 200} | 158 000 {16 100} | 1.11 |

• For bearings with a cylindrical outside surface, add "X" to the bearing series code. In this case, the allowance of the outside diameter "D" of the outer ring is manufactured according to JIS Class O. Example: NUTW203X

2 This is the permissible minimum of chamfering dimension r.



RNA 22 Series NA 22 Series



NATR Series

Standard series of roller followers

NATV Series NACV Series



NUTR Series

Cam followers (with eccentric shaft/tapping hole)



Adoption of an eccentric shaft provides easy adjustment of the mounting position. As a result, highly accurate mounting hole positioning has become unnecessary. Furthermore, the included threaded grease nipple can be installed at the tapping hole on the stud for easy greasing. It can also be used as a mounting thread for centralized pipelines.

Features

- 1) This bearing for outer ring rolling has a thick outer ring and high rigidity.
- Available in three types: needle roller and cage assembly (KRU Series); needle roller without cage assembly (KRVU Series); and double-row cylindrical rollers (NUKRU Series) according to the type of rolling element.
- The outside surface of the outer ring is spherical or cylindrical. The KRU Series and KRVU Series are available with seals. The NUKRU Series is sealed with sealing plates.
- Axial eccentricity is 0.25 to 1.0 mm. This is especially advantageous for level adjustments when many bearings are mounted.
- 5) The driver groove on the flange side and tapping holes at both ends are provided for the stud. Either a screwdriver or hexagon wrench can be used.
- 6) Conventional press-greasing work is

unnecessary because the threaded grease nipple is provided for greasing.

Applications

- Construction machinery, packaging machinery, industrial robots, medical appliances
- Transportation equipment at parking garages and automatic warehouses



Rubber mold cam followers



These are cam followers with an outer ring whose outside surface is vulcanized and coated with urethane rubber. This provides an effective means of preventing noise and absorbing vibration.

Features

- 1) Prevents noise.
- 2) Prevents bearing wear and damage.
- 3) Maintenance-free design
- 4) Easy installation

Applications

Bridle roller guide





Cam followers for indexing



Automatic machinery performs a variety of movements including intermittent, oscillating and continuous rotation with high accuracy and at high speed. Cam followers are used as the roller cam mechanisms of driving devices.

Features

- 1) Provides ample outer ring and stud rigidity in a compact space.
- Full-complement roller design ensures a long service life and larger load-carrying capacity than a roller and cage assembly.
- 3) This cam follower offers precision-class dimensions and rotational accuracy.
- 4) Secured with set screws for easy installation.



Examples of index cam follower shapes



Structure of indexing drive



Small-diameter drawn-cup needle roller bearings

Features

The small diameter drawn-cup needle roller bearing has a low section height, is assembled with a cage assembly and rollers, and has a set bore diameter dimension (F_w) less than 10 mm. The outer ring surface is hardened by carburizing following precision deep drawing of the thin band steel.

Application

These bearings have a wide range of application thanks to their low section height and compact design.

Printer (timing pulley a ssembly).



Application: Printer (timing pulley assembly)





These bearings are used as backup rollers for leveling band steel. They are available in long and narrow models.

Structure and advantages of long bearings

This unit is an assembly comprising a roller, shaft, and bearing. The bearing differs with the intended roll operating conditions, but the design has both a needle roller and cage assembly bearing (when carrying radial loads) and a deep-groove ball bearing (for carrying axial loads).

With the sealed type, the deep groove ball bearing seal, rubber seal, and clearances between cover and roller provide a satisfactory seal.

The starting torque and grease prelubrication, as well as the shape, hardness, roughness and rotational accuracy of the roller, meet all the requirements of a backup roller.



NTN can accommodate specifications other than the above for work rollers or leveler units used as middle rollers. For details, refer to "CAT. No. 2250/J, Large Rolling Bearings."

Structure and advantages of narrow bearings

The outer ring has a thick design in order to provide greater rigidity, as the outside surface of the outer ring is used directly as the raceway surface. These bearings carry larger loads because double-row cylindrical rollers are used as the rolling elements.

The internal rubber seal and external steel seal plates of the sealed type form a labyrinth seal that provides a sufficient sealing effect.

The starting torque and grease prelubrication, as well as the shape, hardness, roughness and rotational accuracy of the roller, meet all the requirements of a backup roller.



Application



Thrust bearing with integrated non-separable raceway



This non-separable thrust bearing is a remodeled separable bearing comprising of a thrust needle roller bearing and bearing ring made of steel plate, with a new structure designed to hold the cage assembly or other bearing ring with the rib of the bearing ring.

Features

The non-separable type of cage assembly and bearing rings ensures easier handling and fewer assembly steps than the standard thrust needle roller bearing.

Applications

These bearings are used in automotive transmissions and air conditioner compressors.



Structure



Application example 1: Automatic transmission (Front-mounted engine, front-wheel drive models)



Application example 2: Car air conditioner compressor

Thrust cylindrical roller and cage assemblies (JW) of pressed steel plate



The cage assembly made of pressed steel plate makes for an economical thrust cylindrical roller and cage assembly.

Conventionally, the cage assembly of thrust cylindrical rollers is made of machined aluminum. However, the mass productivity of machined cage assemblies is inferior and their cost is high.

As a result, integrated cage assemblies made of pressed steel plate have been found to provide satisfactory results for the thrust needle rollers of cylindrical roller thrust bearings.

Previously, it was considered difficult to create the pocket shape for W Series cylindrical rollers because of the low ratio of roller length to roller diameter. Now, however, we have succeeded in producing the appropriate shape so that these rollers can be employed in cylindrical roller bearings.

Features

- 1) Because steel plate is used for the cage assembly and the rollers are manufactured by press forming, mass productivity is excellent.
- 2) The use of steel plate for these rollers allows for a variety of heat treatments and surface treatments.

3) Because it is possible to narrow the width of the pillar as well as the distance between the bore diameter/outside diameter and pocket end, the rated capacity of a given size can be increased by increasing the roller length and the number of rollers.



K811 • • JW

K812 • • JW

Cage assembly of steel plate (Series JW)

Dimensions

| | Bore | Outside | Roller diameter | Basic ra | ting load |
|----------------|---------|---------|-----------------|-------------------------|------------------|
| Bearing series | Dcl E11 | Dc c12 | 0 -0.010 | $C_{a} \in N \{ kgf \}$ | Coa N {kgf} |
| K81100JW | 10 | 24 | 3.5 | 11 000 {1 120} | 21 500 {2 190} |
| K81101JW | 12 | 26 | 3.5 | 11 600 {1 180} | 23 900 {2 430} |
| K81102JW | 15 | 28 | 3.5 | 12 900 {1 320} | 28 600 {2 920} |
| K81103JW | 17 | 30 | 3.5 | 13 500 {1 370} | 31 000 {3 150} |
| K81104JW | 20 | 35 | 4.5 | 20 300 {2 070} | 46 500 {4 700} |
| K81105JW | 25 | 42 | 5 | 27 500 {2 800} | 68 000 {6 900} |
| K81106JW | 30 | 47 | 5 | 28 000 {2 860} | 72 500 {7 400} |
| K81107JW | 35 | 52 | 5 | 31 000 {3 200} | 87 000 {8 900} |
| K81108JW | 40 | 60 | 6 | 43 000 {4 400} | 121 000 {12 400} |
| K81109JW | 45 | 65 | 6 | 45 500 {4 650} | 135 000 {13 800} |
| K81110JW | 50 | 70 | 6 | 48 500 {4 900} | 150 000 {15 300} |
| K81111JW | 55 | 78 | 6 | 64 500 {6 600} | 225 000 {22 900} |
| K81112JW | 60 | 85 | 7.5 | 71 500 {7 300} | 225 000 {23 000} |
| K81113JW | 65 | 90 | 7.5 | 75 500 {7 700} | 247 000 {25 200} |
| K81114JW | 70 | 95 | 7.5 | 79 000 {8 050} | 268 000 {27 300} |
| K81115JW | 75 | 100 | 7.5 | 80 500 {8 200} | 279 000 {27 400} |
| K81116JW | 80 | 105 | 7.5 | 81 500 {8 350} | 290 000 {29 500} |
| K81117JW | 85 | 110 | 7.5 | 85 000 {8 700} | 310 000 {31 500} |
| K81118JW | 90 | 120 | 9 | 118 000 {12 000} | 430 000 {43 500} |
| K81206JW | 30 | 52 | 7.5 | 53 500 {5 450} | 129 000 {13 100} |
| K81207JW | 35 | 62 | 7.5 | 57 500 {5 850} | 150 000 {15 300} |
| K81208JW | 40 | 68 | 9 | 74 500 {7 600} | 190 000 {19 400} |
| K81209JW | 45 | 73 | 9 | 82 000 {8 400} | 222 000 {22 600} |
| K81210JW | 50 | 78 | 9 | 85 000 {8 650} | 238 000 {24 200} |
| K81211JW | 55 | 90 | 11 | 121 000 {12 400} | 340 000 {34 500} |
| K81212JW | 60 | 95 | 11 | 126 000 {12 800} | 365 000 {37 000} |
| K81213JW | 65 | 100 | 11 | 130 000 {13 300} | 385 000 {39 500} |
| K81214JW | 70 | 105 | 11 | 134 000 {13 700} | 410 000 {42 000} |
| K81215JW | 75 | 110 | 11 | 139 000 {14 100} | 435 000 {44 500} |
| K81216JW | 80 | 115 | 11 | 143 000 {14 500} | 460 000 {47 000} |



Application: Riding mower



Bearing assembly structure





Features

If the shaft or housing is used as the raceway surface of a needle roller bearing, in place of an inner ring or outer ring, the bearing will tend to have a short operating life because of a lack of lubricant film caused by the difference in surface roughness between the rollers and mating components (mainly the shaft).

NTN has developed a surface processing method through which a sufficiently thick oil film is maintained under these operating conditions. This method changes the orientation of the surface roughness according to the Micro EHL Theory. These bearings are called HL (High

Lubrication) roller bearings.

A HL surface has no orientation because, as shown in **Fig. 1**, the surface has a large number of minute dimples of more than $10 \,\mu$ m to provide a roughness wave form in the axial and circumference directions. The depth of the dimples is approximately $1 \,\mu$ m.



Fig. 1 Roughness of HL surface

To confirm the capacity of a HL surface to form an oil film, we compared the oil-film formation capacity of the HL surface with that of a super-finish surface with the 2-cylinder test unit. **Fig. 2** shows the results, which confirm that the HL surface has a greater oil-film formation capacity.



Fig. 2 Metal contact ratio of HL surface

Application

The superior oil-film formation capacity enables these roller bearings to achieve optimal performance under severe lubricating conditions, as shown below. Moreover, protection against peeling and an extended service life can be expected. The HL surface can also be applied effectively to bearing rings other than roller bearings.

- Travel speed reducers for construction machinery
- Various types of transmissions
- Engine roller rocker arm
- Hydraulic pumps





These needle roller bearings contribute to high-quality printing and easier maintenance.

Structure

Outer ring

Central ring

(Staggered double-row arrangement)

Needle roller and cage assembly

Needle roller and cage assembly

Inner ring

Seal

Features

These needle roller bearings boast highaccuracy rotation and dimensions. The adoption of the needle roller bearing design ensures a bearing section with a simple structure and compact design. Assembly and adjustment are simplified and stable, longterm rotation is assured.

Applications

- Offset sheet printing machine Blanket cylinder (with rubber trunk)
- Maximum speed: 15,000 sheets/hour



PK Series needle roller and cage assemblies for general production machinery



Features

The cage assembly features an outside diameter guide structure, which has the same structure and rigidity as the cage assembly for the connecting rod. It is designed for a high loading capacity within the limits of the available space. The cage assembly is of carburized steel, and appropriate heat treatment ensures sufficient strength.

Applications

These bearings are widely used in construction machinery, agricultural machinery, hydraulic equipment and the steel industry.

Often, numerous needle roller bearings are used in the planetary gear speedreduction mechanism for construction machinery. In particular, PK Series roller and cage assemblies are in wide use.



Application: Planetary gear speed reduction mechanism for construction machinery





In recent years, the use of an arc-shaped rolling bearing behind the swash plate, instead of the conventional trunnion system and sliding contact system, has been considered for the variable mechanism of the variable capacity plunger pump motor. Application of this system allows the bearing to carry larger loads in a smaller space. It also solves the problem of friction on the sliding

Application



surfaces of the sliding system. In addition, the reduction of oscillating resistance improves response.

Types

The needle roller and cage assembly of these bearings is split (usually into 3 parts) along its circumference. Many bearings have a split outer ring. The single-rib type and double-rib type are shown in the figure.



Bearings for linear motion (KLM, KD, KH)



Features

These rolling bearings use steel balls and exhibit excellent friction and motion characteristics compared to sliding bearings. With their easy handling and compact size, they are widely used in applications where stroke movement is required.

With the standard KLM and KH Series, the row of balls rotates to provide unlimited linear motion along the shaft. The outer ring of the KH Series is the drawn-cup type and offers a smaller cross-section. This series is more economical than the KLM Series.

With the KD Series, the equally distributed ball rows in the axial direction reciprocate in limited linear motion along the shaft without rotating.

Application

These bearings are used for sliding parts of office appliances or construction machinery. Bearings can be selected according to their specifications (shaft diameter and stroke length).



Application: Work transferring slide unit

10. Nominal number

A nominal number of a bearing expresses the type, dimensions, accuracy and internal structure of the bearing and is composed of the basic number and supplemental code.
 Table 10.1 shows the nominal number composition.
 Table 10.2 also shows the arrangement order of nominal numbers.

| Table 10.1-1 Nominal numbe | r composition (P | refix supplemental o | ode, basic number) |
|----------------------------|------------------|----------------------|--------------------|
|----------------------------|------------------|----------------------|--------------------|

| Prefix supplemental code | Basic number | | | | | |
|--|--|--|--|--|--|--|
| Special application, material, heat treatment | Bearing name | Series code | Dimension code or dimension group code+inner diameter number | | | |
| E-Bearings using carbonized steel | Needle roller and cage assemblies bearing | K, KJ, KMJ, PCJ, PK, KBK | $F_{W} \times E_{W} \times B_{C}$ (Bore diameter) | Outside diameter×Width) | | |
| F-Bearings using stainless steel | Drawn-cup needle | HK, HMK, BK | Fw • C (Bore diameter / Width) | | | |
| | roller bearing | DCL | $Fw" \cdot C"$ (Inch se | eries: Bore diameter / Width) | | |
| C-Bearings using carbon steel M-Bearings with plating process | Machined-ring needle roller | RNA, NA | 49, 48, 59, 69 | 5~9 (Inner diameter: ∳ 5~9) 00~03 (Inner diameter: ∲ 10, 12, 15, 17) / 22, 28, 32 (Inner diameter: ∲ 22, 28, 32) 04~88 (Inner diameter: ∲ 20~440) | | |
| HL-Bearings using HL rollers refer to pg. 62 | bearing | NK (+IR), RNAO, NAO | Fw / C (Bore dial Fw (or d)XDXC (Bore | meter / Width) diameter or Inner diameterXOuter diameterXWidth) | | |
| | | MR (+MI) | $Fw" \cdot D" \cdot C"$ (Inch s | eries: Bore diameter / Outer diameter / Width) | | |
| 8Q-Bearings with cage assemblies treated by soft-nitriding | Thrust roller bearing | AXK, AS, WS8, GS8, K8, 8 | 11 11, 12, 93 11, 12, 93 | 00~03 (Inner diameter: ∳ 10, 12, 15, 17) 04~32 (Inner diameter: ∳ 20~160) | | |
| TS2-Bearings for high temperature up to 160°C | | NKX (+IR) NKXR (+IR) | Fw (Bore diamet | er) | | |
| treated by the dimension stabilization. | Compound bearing | NKIA NKIB | 59 | 02, 03, /22 (Inner diameter: ¢ 15, 17, 22) 04~14 (Inner diameter: ¢ 20~70) | | |
| TS3-Bearings for high | | AXN, ARN | d · D (Inner diameter / Outer diameter) | | | |
| temperature up to 200°C treated by the dimension stabilization. | | RNA, NA | 22 | / 6, 8 (Inner diameter: ¢ 6, 8) 00~03 (Inner diameter: ¢ 10, 12, 15, 17) 04~10 (Inner diameter: ¢ 20~50) | | |
| TO 4 De enir ve fan binb | Roller follower | NATR, NATV | d • D (Inner diameter / Outer diameter) | | | |
| temperature up to 250°C | | NACV | d" • D" (Inch series: Inner diameter / Outer diameter) | | | |
| treated by the dimension stabilization. | | NUTR | 2, 3 | 02 \sim 03 (Inner diameter: ϕ 15, 17) 04 \sim 10 (Inner diameter: ϕ 20 \sim 50) | | |
| | Cam follower | KR, KRV, KRU, KRVU, NUKR | D (Outer diameter) | | | |
| | | CRV | D" Inch series: (I | nner diameter / Outer diameter) | | |
| | Components | IR MI A, F WR, BR G, GD | $d \times F \times B$ (Inner diameter × Raceway diameter × Width) $d^* \cdot F^* \cdot B^*$ (Inch series: Inner diameter × Raceway diameter × Width) $Dw \times Lw$ (Diameter × Length) d_1 (Shaft diameter or Hole diameter) $d \times D \times b$ (Inner diameter × Quter diameter × Width) | | | |
| | Bearing for linear motion | KLM KH KD RLM FF RF, BF | Fw (Bore diamete $Fw \cdot C$ (Bore diar $Fw \cdot D \cdot C$ (Bore $H \times L$ (Height x L L $Dw \cdot b$ (or B) (Dia $Dw \cdot b / L_1$ (Diam | er) neter / Width) diameter / Outer diameter / Width) angth) meter×10 / Width) eter×10 / Width/Length) | | |
| | One-way clutches | HF NHF, NCU | Fw • C (Bore diameter / Width) Fw (Bore diameter) | | | |

| Suffix supplemental code | | | | | | | | | | |
|---|--|--|---|---|---|--|---|---|--|--|
| Internal change code | Cage assemblies code | Seal, shield code | Raceway shape code | Combination code | Internal clearance code | Accuracy code | Lubricant code | Special code | | |
| ZW: Double-row cage assemblies A, B, C: Internal structure change R: Outer ring with double ribs X: Outer ring outside surface of the cam follower and roller follower is cylindrical. | L1: High strength brass machined cage F1: Carbon steel machined cage J, JW: Steel plate pressed cage T2: Plastic mold cage L3: Aluminum alloy cage S: Welded cage | L: With the synthetic rubber seal on one side (contact type) LL: With the synthetic rubber seals on both sides (contact type) | N: Locating snap ring with groove NR: With the locating snap ring D: With lubrication holes DO: Without lubrication holes and oil groove H: Cam follower with the hexagonal hole S: Clearance adjusting type | D2, Dn: Combination of same bearings more than two pieces + α : With the spacer (α is expressed by the width dimension of spacer.) | C2: Narrower than the usual clearance (CN) Usual clearance C3: Wider than the usual clearance C4: Wider than the C3 clearance | P6: JIS Class 6 P5: JIS Class 5 P4: JIS Class 4 PX1 to PXn: Special dimension tolerance | /2A: Albania 2 /3A: Albania 3 /8A: Albania EP2 /5K: Martenp SRL /LP03: Heat hardening type grease (solid grease) | V1 to Vn: Special specifications, requirements | | |

Table 10.1-2 Nominal number composition (Suffix supplemental code)

Table 10.2 Arrangement of nominal numbers

| | nominal numbers | Ex.1 Drawn-cup needle roller bearing | | | | | Ex.2 Cam follower | | | | |
|--------------------------------|---|--------------------------------------|---------------|--|---|--|-------------------|--|--|------------------|--|
| Arrangem | nent of nominal numbers | | | | | | | | | | |
| Suffix supplemental code | Special application code Material, Heat treatment | | <u>5045 2</u> | | | | | | | <u>, 12 - 03</u> | |
| Basic number | Series code | - J. | | | | | | | | | |
| | Dimension group code inner diameter number | | | | | | | | | | |
| | Dimension code | 4 | - | | - | | | | | | |
| | Internal change code | - | | | | | | | | | |
| | Cage assemblies code | | | | | | | | | | |
| | Seal, shield code | 4 | | | | | | | | | |
| Suffix | Raceway shape code | 4 | | | - | | | | | | |
| supplemental | Combination code | | | | | | | | | | |
| code | Internal clearance code | | | | | | | | | | |
| | Accuracy code | • | | | | | | | | | |
| | Lubricant code | • | | | | | | | | | |
| | Special code | 4 | | | | | | | | | |

Remarks:

1. Contact NTN about the basic number, prefix or suffix supplemental codes other than the list.

 Soft nitriding treatment is the standard specification for the welded cage assemblies and the prefix supplemental code (8Q-) should be omitted.

3. If the prelubricated grease is standardized per the bearing type and product type, the grease code should be omitted.

4. If the special code (Vn) is used, Vn includes the material, heat treatment and lubrication code, but the internal change code such as prefix supplemental code HL- and ABC, S (welded cage assemblies), seals, locating snap rings, H (with hexagonal hole) combination codes and precision codes are not included in Vn but described.

NTN Needle Roller Bearing Handbook

| Comparison of Bearing Series by Manufacturer Needle roller and cage assemblies/Machined-ring needle roller bearings | | | | | | | | | | | |
|---|------|------------|---------------------|------------|------------|------------|------------|------------|---------|--------------|---------|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | THK | NADELLA | SKF | Mc,GILL |
| Needle roller and cage assemblies | | К | К | FWJ | FWJ | KT | R,RS | | В | К | |
| | | K∙∙ZW | K∵ZW | | FDJ | FTW | WR,WRS | | BB | K∵ZW | |
| | шШ | KMJ | | FWJ | FWJ | кт | R,RS,V,VS | | В | | |
| | - | RK | KZK | | FWF | KT-EG | VS,VS··P | | | KZK | |
| | | КВК | КВК | | FBN | KTVEG | R··P,UR··P | | | КВК | |
| | inch | PCJ | C | WJ | WJ | | | | | | |
| | | NK | NK | | | | | | | NK | |
| | | NK+IR | NKI | | | | | | | NKI | |
| | | NK∵R | NK | | (RLM) | TAF | NQ | TAF | NB | NK | |
| ıgs | | NK…R+IR | NKI | | | TAFI | NQI | TAFI | NBI | NKI | |
| | | RNA49 | | | | | | | | | |
| | | NA49 | | | | | | | | | |
| | | RNA…R | RNA49 | | RNA49 | RNA49 | RNA49 | RNA49 | RNA49 | RNA49 | |
| | | NA49…R | NA49 | | NA49 | NA49 | NA49 | NA49 | NA49 | NA49 | |
| | | RNA49…L,LL | RNA49"RS,2RS | | | RNA49.U,UU | RNA49UU | RNA49.U,UU | | RNA49"RS,2RS | |
| | | NA49…L,LL | NA49"RS,2RS | | | NA49…U,UU | NA49. UU | NA49∙∙U,UU | | NA49"RS,2RS | |
| | | RNA59 | | | RNA59 | | RNA59 | | | | |
| ari | | NA59 | | | NA59 | | NA59 | | | | |
| pe | | RNA69 | RNA69 | | RNA69 | RNA69 | RNA69 | | | RNA69 | |
| Iler | | NA69 | NA69 | | NA69 | NA69 | NA69 | | | NA69 | |
| 2 | шш | | | | RNA69…U,UU | | | | | | |
| sdle | | | | | NA69…U,UU | | | | | | |
| nee | | RNA48 | RNA48 | | RNA48 | RNA48 | | | | RNA48 | |
| Machined-ring | | NA48 | NA48 | | NA48 | NA48 | | | | NA48 | |
| | | RNAO | RNAO | | RNAF | RNAF | | | | RNAO | |
| | | NAO | NAO | | NAF | NAF | | | | NAO | |
| | | RNAO…ZW | RNAO | | RNAFW | RNAFW | | | | RNAO | |
| | | NAO…ZW | NAO | | NAFW | NAFW | | | | NAO | |
| | | NKS | NKS | | | | NQS | | | NKS | |
| | | NKS+IR | NKIS | | | | NQIS | | | NKIS | |
| | | RNA49…S | RNA49 S | | | | | | | | |
| | | NA49…S | NA49S | | | | | | | | |
| | | RPNA…R | RPNA | | | | | | | RPNA | |
| | | PNA∵R | PNA | | | | | | | PPNA | |
| | | IR | IR | | | IRT | IRM | | | LNA | |
| | ç | MK | NCS | HJ | HJ | IRK I | | | | | MR |
| | inc | MR+MI | NCS+PI | HJ+IR | HJ+IR | IRKI | | | | | MR+MI |
| | | IVII | IPI | IIK | IIK | IIKB | IIK | | | | INI |

NTN Needle Roller Bearing Handbook

| Comparison of Bearing Series by Manufacturer ODrawn-cup needle roller bearings | | | | | | | | | | | |
|--|-----|---------|----------|------------|------------|---------|------|-----|---------|---------------|---------|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | THK | NADELLA | SKF | Mc,GILL |
| Iler bearings | | нк | НК | FJ | FJ | TLA∵Z | BTM | | DB | НК | |
| | | HKZWD | НК | | | TLA∵Z | | | | | |
| | | HKL,LL | HKRS,2RS | FJT,FJTT | FJT,FJTT | TLA··UU | | | DB∙E | HK ·· RS, 2RS | |
| | | нмк | | | FJL | TA∵Z | BHTM | | | | |
| | | HMKZWD | | | | TA∵Z | | | | | |
| | | HMKL,LL | | | FJLT,FJLTT | TA…U,UU | | | | | |
| | | ВК | BK | MFJ | MFJ | TLAM | MKM | | DBF | ВК | |
| | | BKZWD | BK | | | TLAM | | | | | |
| | mm | BKL | BKRS | MFJT | MFJT | TLAM∙∙U | | | | BK∵RS | |
| | | ВМК | | | MFJL | TAM | MHKM | | | | |
| | | BMKZWD | | | | TAM | | | | | |
| | | BMKL | | | MFJLT | | | | | | |
| | | ни | | F | F | | BM | | (DL) | | |
| | | нми | | | | | BHM | | | | |
| | | BV | | | MF | | MM | | (DLF) | | |
| 2 S | | | | FY | | YTL | YM | | | | |
| needle | | | | | | YT | YM | | | | |
| | | DCL | SCE | J | J | BA∙·Z | BT | | | | |
| dn | | DCLL,LL | SCEP,PP | JT,JTT | JT,JTT | | | | | | |
| Drawn-c | | DCH | SCH | JH | JH | BHA··Z | BHT | | | | |
| | | DCHL,LL | SCEP,PP | JHT,JHTT | JHT,JHTT | | | | | | |
| | | DBL | BCE | MJ | MJ | BAM | MK | | | | |
| | | DBLL | BCEP | MJT | | | | | | | |
| | | DBH | BCH | MJH | MJH | BHAM | MHK | | | | |
| | nch | DBHL | BCHP | MJHT | | | | | | | |
| | .= | VS | S | В | В | | В | | JL | | |
| | | VSH | SH | BH | BH | | BH | | | | |
| | | VB | CS | M | M | | M | | JLF | | |
| | | VBH | CSH | MH | MH | | MH | | | | |
| | | | SN | Y | Y | YB | Y | | | | |
| | | | SNH | YH | YH | YBH | | | | | |
| | | | CSN | | | | | | | | |
| | | | CSNH | | | | | | | | |
| С | Comparison of Bearing Series by Manufacturer OThrust roller bearings and thrust bearing rings | | | | | | | | | | |
|------|---|--------|------------|------------|-----------|-----|--------|-----|----------|-----|---------|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | ТНК | NADELLA | SKF | Mc,GILL |
| | | AXK11 | AXK | FNTA | FNTA | NTB | TP,TPK | | | AXK | |
| | | K811 | K811 | | | AZK | | | (ARZ) | | |
| | | K812 | K812 | | FNTH | AZK | | | | | |
| | | K893 | K893 | | FNTH | | | | (ARZ) | | |
| | | K874 | K874 | | | | | | | | |
| | | 811 | 811 | | | AZ | | | | | |
| SC | | 812 | 812 | | FNTHA | AZ | | | | | |
| rin | | 893 | 893 | | FNTHA | | | | | | |
| bu | E E | 874 | 874 | | | | | | | | |
| ari | | AS11 | AS | (FTA) | FTRA | AS | W | | (CP) | AS | |
| t be | | GS | GS | | (FTRD··F) | GS | | | (CP,CPR) | GS | |
| ust | | WS | WS | | (FTRD··F) | WS | WS | | (CP,CPR) | WS | |
| thr | | AXA21 | AXK+GS+ZS | | | | | | PM | | |
| pu | | AXB21 | AXK+WS+ZS | | | | | | PM | | |
| Sa | | ARA821 | K811+GS+ZS | | | | | | | | |
| ing | | ARB821 | K811+WS+ZS | | | | | | PMH | | |
| ear | | ZS | ZS | | | | | | | | |
| sr b | | NTC | TC | NTA | NTA | | | | BT | | |
| olle | | CTC | | NTH | NTH | | | | | | |
| str | | CTCA | | NTHA | NTHA | | | | | | |
| Iru | | NWA | TWA | TRA | TRA | | | | TW | | |
| È | _ | NWB | TWB | TRB | TRB | | | | TW | | |
| | inch | NWC | TWC | TRC | TRC | | | | TW | | |
| | | NWD | TWD | TRD | TRD | | | | TW | | |
| | | NWE | | TRE | TRE | | | | | | |
| | | NWF | | TRF | | | | | | | |
| | | CWS | | TRI,TRJ | | | | | | | |
| | | CGS | | TRID,TRJD | | | | | | | |

NTN Needle Roller Bearing Handbook

| Comparison of Bearing Series by Manufacturer ●Compound bearings | | | | | | | | | | | |
|---|---|------------|------------------------|------------|-----|--------|------|-----|---------|------------------------|---------|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | ТНК | NADELLA | SKF | Mc,GILL |
| | | NKX | NKX | | | NAX | | | | NKX | |
| | | NKX∵Z | NKX··Z | | | NAX∵Z | | | | NKX··Z | |
| | | NKX+IR | NKX+IR | | | NAXI | | | | NKX+IR | |
| | | NKX…Z+IR | NKX.Z+IR | | | NAXI. | | | | NKX.Z+IR | |
| s | | NKXR | NKXR | | | NBX | | | | NKXR | |
| ing | | NKXR…Z | NKXR Z | | | NBX··Z | | | RAXZ | NKXR…Z | |
| ear | | NKXR+IR | NKXR+IR | | | NBXI | | | | NKXR+IR | |
| p | ε | NKXR…Z+IR | NKXR ··Z+IR | | | NBXI. | | | | NKXR.Z+IR | |
| nn | E | NKIA59 | NKIA59 | | | NATA59 | | | | NKIA59 | |
| d | | NKIB59 | NKIB59 | | | NATB59 | | | | NKIB59 | |
| mo | | NX∵ZNR | NX∵Z+IR | | | | | | | NX.Z+WR | |
| 0 | | NX…ZNRD | NX+WR | | | | | | | NX+WR | |
| | | NX…ZNR+IR | NX Z+WR+IR | | | | | | | NX Z+WR+IR | |
| | | NX…ZNRD+IR | NX+WR+IR | | | | | | | NX+WR+IR | |
| | | AXN | ZAXN | | | | | | AXNB | | |
| | | ARN | ZARN | | | | | | ARNB | | |

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| С | om | parison o | of Bearin | g Series I | by Manuf | acturer C | Cam foll | owers | | | |
|---------------|------|--|---|--------------------------------|--|--|--|-----------------------------------|--------------|-----------------------------------|--|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | THK | NADELLA | SKF | Mc,GILL |
| Cam followers | mm | NTN KR KR··LL KR··X KR··XLL KR··H KR··LLH KR··XH KRV KRV KRV·LL KRV··LL KRV··X KRV··XLL KRV··XH KRV··LH KRV··XH KRV··LH KRV··LH KRV··LL KRV··LL KRV··LL KRV··LL KRV··LL KRV··LL KRV··XH KRV··XH KRV··XH KRV··XH KRV··XH KRVE KRV KRV KRV KRV KRV KRV KRV KRV | INA KR KR-PP KR-X KRV-PPX KRV-PP KRV-PP KRV-PP KRV-PP KRV-PP KRV-PP KRV-PPX | TORRINGTON FRJC FRJSC | NSK FCJ··R FCJS··R FCJS FCR··R FCR··R FCRS··R FCR FCRS (FCRS) | IKO CFR CFUU CF CFBR CFBUUR CFVR CFVUUR CFVUUR CFVBR CFVUR NUCFR | KOYO KM-RM KM-UURM KM-UMU KM-UMU CM-RM CM-UURM CM-UUM | THK CFR CFUUR CF CFUU | GC GC··EE | SKF KR KR·X KRV KRV·X | MC,GILL MCFR MCFR··S MCFR··SX MCF··SS MCF··S MCF··SX |
| | inch | CR CR··LL CR··X CR··XLL CRV CRV··LL CRV··LL CRV··X CRV··XLL CRV··H CRV··LLH CRV··XH | CFCY CFC.PPY CFC CFC.PP CFY CFPPY CF CFPP | CRC CRCS CR CR CRS | CRC CRCS CR CR CRS | CR-R CR-UUR CR CR CR-UU | CR CR.·UU | | | | CCF CCF··S CF CF··S |

| Comparison of Bearing Series by Manufacturer ●Roller followers | | | | | | | | | | | |
|--|----|-----------|----------------------|------------|---------|------------|-----------|------------|---------|------------------------|---------|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | ТНК | NADELLA | SKF | Mc,GILL |
| | | RNAB2 | RST0 | | | RNAST"R | | RNAST··R | | RST0 | |
| | | NAB2 | ST0 | | | NAST··R | | NAST··R | | ST0 | |
| | | RNAB2…X | RST0 X | | | RNAST | | | | RST0 ^{.,} X | |
| | | NAB2…X | STO-X | | | NAST | | NAST | | RSTX | |
| | | RNA22…LL | RNA222RS | | | | | | | RNA222RS | |
| | | NA22…LL | NA22 2RS | | | | | | | NA22··2RS | |
| | | RNA22…XLL | RNA22··2RSX | | | | | | | RNA22 2RSX | |
| | | NA22··XLL | NA22··2RSX | | | | | | | NA22··2RSX | |
| | | NATR | NATR | FYRJC | FYCJ∵R | NART∵R | CXMRM | NART∵R | | NATR | MCYRR |
| | | NATR··LL | NATR··PP | FYRJSC | FYCJS…R | NART.UUR | CXM··UURM | NART··UUR | | NATR ·· PP | MCYRRS |
| | | NATR··X | NATR∙X | | FYCJ | | CXMW | | | NATR∙·X | MCYRRX |
| ຽ | ε | NATR··XLL | NATR··PPX | | FYCJS | NART…VR | CXMUNW | | | NATR ·· PPX | MCYRRSX |
| wei | Ē | NATV | NATV | | FYCR"R | NART··VUUR | CYM. RM | NART⋯VR | FG | NATV | MCYR |
| Ollo | | NATV··LL | NATV··PP | | FYCRS"R | | CYM··UURM | NART··VUUR | FG··EE | NATV··PP | MCYRS |
| r fo | | NATV··X | NATV··X | | FYCR | | СҮМ…М | | | NATV∙·X | MCYRX |
| olle | | NATV··XLL | NATV ·· PPX | | FYCRS | | CYMUNM | | | NATV··PPX | MCYRSX |
| Ř | | NUTR | NUTR | | | NURT…R | DCZMR | | | NUTR | |
| | | NUTR∵X | NUTR∵X | | | | | | | NUTR··X | |
| | | NUTW | | | | | | | | | |
| | | NUTW∵X | | | | | | | | | |
| | | NABR | | | | NAST··ZZR | CZM∵R | NAST··ZZR | | NAST ··- 2Z | |
| | | | | | | NAST.ZZUUR | | NAST.ZZUUR | | | |
| | | NABR…X | | | | NAST··ZZ | CZM | NAST··ZZ | | NAST··P-2Z | |
| | | | | | | NAST.ZZUU | | NAST. ZZUU | | | |
| | | NACV | RF…Y | YCRC | YCRC | | | | | | CYR |
| | ÷ | NACV…LL | RF··PPY | YCRSC | YCRSC | | | | | | CYRS |
| | Ë. | NACV…X | RF | YCR | YCR | | | | | | CCYR |
| | | NACV…XLL | RF ·· PP | YCRS | YCRS | | | | | | CCYRS |

| С | Comparison of Bearing Series by Manufacturer ●Linear motion bearings/Other items | | | | | | | | | | |
|------|--|------------|----------|------------|-----|------------|---------------|--------|-----------------|---------|---------|
| | | NTN | INA | TORRINGTON | NSK | IKO | KOYO | ТНК | SKF | NADELLA | Mc,GILL |
| | | FF | FF | | | FT··N | | | | | |
| | | FF∙∙ZW | FF…ZW | | | | | | | | |
| | | BF | BF | | | FT | | FT | | | |
| gs | | RF | | | | | | | | | |
| arin | | KD | RLF | | | ST | | ST | | | |
| bea | | KD…LL | | | | ST-UU | STUU | | | | |
| u | ε | КН | кн | | | | | | | | |
| oti | 8 | KLM | КВ | | LB | LM··N | | LM | | | |
| E | | KLM…LL | | | | LM…NUU | | LM··UU | | | |
| lea | | KLM∙∙S | | | | LM··NAJ | | LM··AJ | | | |
| Ľ | | KLM…SLL | | | | LM…NUUAJ | | | | | |
| | | KLM⋯P | | | | LM.NOP | | LMOP | | | |
| | | KLM…PLL | | | | LM.NUUOP | | | | | |
| | | RLM | RUS | | | (SR,GSN) | | LRU | PNC ··EE | | |
| | | HF | HF | FC | FC | | | | | | |
| | | HFL | HFL | FCB | FCB | | | | | | |
| | | WR | WR | | | WR | | | JA | | |
| | | BR | BR | | | AR | | | JB | | |
| ms | ε | G | G | | VC | OS | HM,HMS,MH,MHS | | ET | G | |
| ite | E | GD | SD | | КС | DS | HMSA,MHSA | | | UL | |
| her | | FRIS | UWL,UW21 | | NFB | M…ZZ,MB…ZZ | | | | | |
| đ | | FR | RUW21 | | | | | | | | |
| | | JPUS | BSR | | | | | | | | |
| | | JPU…S+JF…S | BSRF | | | | | | | | |
| | c | HFZ | HFZ | RC | RC | | | | | | |
| | i | GSC | | | | | | | | | |

NTN Needle Roller Bearing Handbook

NAME

| ADDRESS | PHONE |
|---------|-------|
| OFFICE | PHONE |





| 60/1/00 |
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