Pb# 923:- At a certain point in a stressed body, the principal stresses are 52 = sompa and Sy = - 40MPa. Determine 8 & 2 on the Plane whose mormals are at +30° and + 120' with x-axis. Show your result on a Sketch 95 a differential element. As given that, Sn= Sompe 8y=-40MPe. x Sy = - 40MPa Required data; Sn = 80MPe. Sn' - 7 84 = ? しか=? (4 = ? Solution :-As we know That, 8n'= 8n+8y + 8n-8y con 20 - trysin20 C = 30'. By putting values. 8= = 80-40 + 80+40 (00 2(30) - 0 8x = 50 Mpc 1

$$Sy' = \frac{8n + 8y + 8n - 8y \cos 20 - \log \sin 20}{2}.$$
Here

$$G = 90 + 0$$

$$Sy' = \frac{8 - 40}{2} + \frac{80 + 40}{2} (\cos 2(120)) = 0.$$

$$Sy' = \frac{8x - 8y}{8} \sin 20 + \frac{1}{2} \sin 20 = 0.$$

$$Tn' = \frac{8x - 8y}{8} \sin 20 + \frac{1}{2} \sin 20 = 0.$$

$$Tn' = \frac{8x - 8y}{8} \sin 2(30)$$

$$Ty' = \frac{8x - 8y}{8} \sin 2(30)$$

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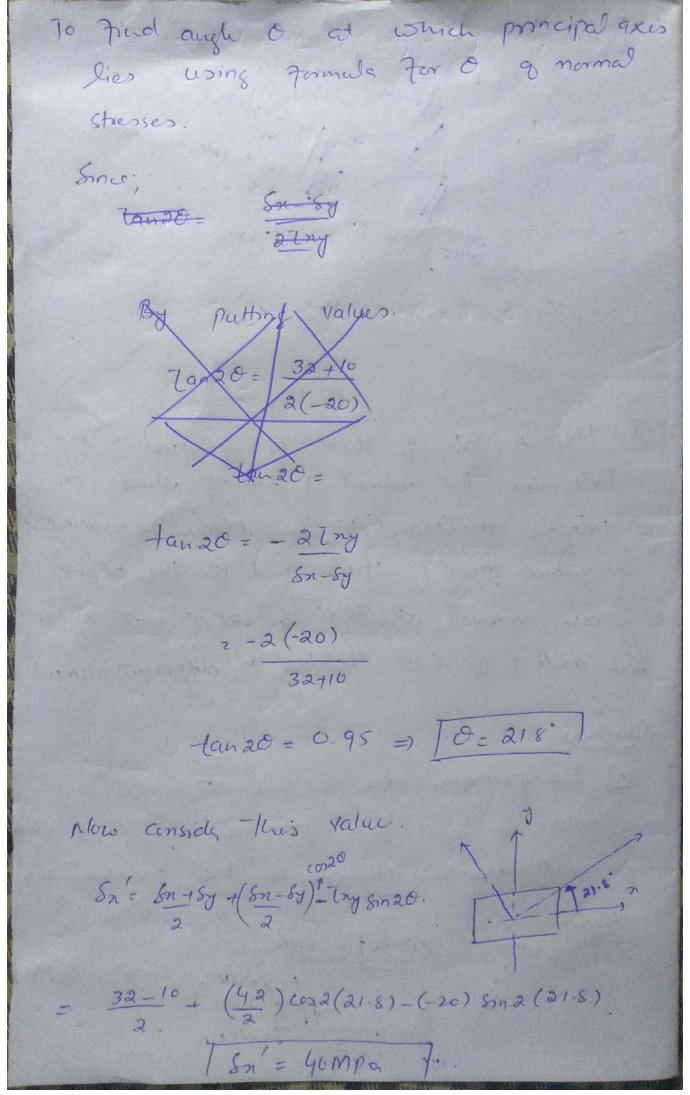
$$Ty' = \frac{8x - 8y}{8} \sin 2(120).$$

$$Ty' = \frac{8x + 40}{8} \sin 2(120).$$

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figure: 1 =-10MPE | SY=51.9M Sy= - lompe SX - BOMPR Pb# 924:- A state of stress is specified in big. Determine the normal & shearing stress on @ Principal axis (plane), (b) the plane of maximum in plane shearing stress and @ the plane whose normal at 36.8° & 126.8° with x-axis show result of @ & @ sketches of different element. Solution :-@ For principal axis. First we will find principal axis and than we will resolve stresses on that principal axis. Tyx = 20 Ma 1 Sy = 10 MPC Figure:



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Sy = 8x+ Sy + Sn- Sy cos 20 - Try 5 n 20. 4 1. Here put 0= 90+0 10 12 90 + 21.8 8= 111.8°. 3.5. 90=) Sy'= 32-10 + 32+10 cos 2 (111.8) - (-20) sin 2 (111.8) /8/=-18MP2 As on principal axis no shearing stress occur so shearing stresses are zero. Figure: Sx = 40MPa (3) The plane of max. Shearing Stress: First Find angle for that plane Using egs of taile for shearing stresser -(al20 = 821 - 84 +2 Try

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Jan 20 =
$$\frac{32+10}{3(-20)}$$
 $1 \text{ an } 20 = -1.05$
 $0 = -23.19$
 $1 \text{ an } 20 = 23.19$
 $1 \text{ an } 20 = 23.19$

Now use This value for stresses.

Normal stress:

 $1 \text{ for } 3 \text{ for$

For shearing stresses: In' = Sx - Sy sin 20 + Try cos 20. 2 6 6 6 8 7n'= 32+10 sin 2 (668) + (-20) as 2 (668) [n= 29MPa Ty'= 5n-8y Sin20 + (nycos20 Here used 6 = 90 + 66.8. 8 2. 156.8 7y'= 32+10 sin 2 (156.8) + (-20) (0) 2556:8) Ty= -29MPa Figure 1-

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Now to find stresses at 0= 36.8 & 126.8" Sx = Sn+by + Sn-Sy cos20 - Try snco. -> @ PW 0= 36.8" = 32-10 + 32+10 (002 (368) - (-20) 802 (36.8) 1 82 = 36 1 MPa For Sy'= Use same formula Put 0= 126.8" 90-1 Sy'= 32-10 + 32+10 (052 (126.8) - (-20) Sin2 (126.8) 1 Sy'= - 14. IMPC shearing stress: 1= 5n-5y sin 20 + iny sin cos 20. Put 0 = 126-8 7'= 32+10 sin 2(36.8) + (-20) cos 2(36.8) Tix'= 14.49MPE] For Ty' Put 0 = 126.8°. [y'= 32+10 Sin 2 (126.8) - 20 (cos(2)(126.8)) Ty'= - 14.49MPa Sn = 30 1 1 14.1 = 80 Sn = 30 1 1 14.1 = 80 Sn = 30 1 1 14.1 = 5n. Figure :-

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15 # 925:- Two wooden joist 50mm x 100mm are glued togething along the joint AB as Show in 08. Determine the normal & shear stresses in glue in P= 200KN. Figure: Solution: First Find Sn & Sy $Sn = \frac{P}{A} = \frac{2\cos x \cos^2 N}{50 \times 100} = 40 MPa$ Sy=0 as no force acting along J-axis In = 52 + 8y + 52 - 8y cos 20 - by sinio. 050 146 300 20 Put 0= 130, Sy=0 & lny=0 8n'= 40 + 40 cos2(138). Figure: Sn'= 16.53 Shear stress: [= 5x-8y sin 20 + lny cosaco i = 90 50 2 (130) = Tty= -19.7 MPE

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Pb# 926:- 96. The element is subjected to to state of stress shown in Dis. Find the principal stresses. Also compute the stress Component on plate at 30. Country clockwise from the x-face. Figure: I swom psi Given dala; Sx = 4000 ps 0 = 30 C.C. W from n-axis 8y = -8000 ps. 78my = - 6000 psi Solution: Stresses at Principal axis. First Find O at which principal lies. Using 99. lan 20 = - 2 lny 8n-84 Jan 20 = -2(-600) - 10 = 22507 So principal exis lies et To= 22.5%

Normal Shess:-

$$Sx + Gy + Sx - Sy(coab - Sny Snn ab.)$$

Put $C = aa. S$.

$$= \frac{4uv - Suv}{2} + \frac{4uv + Suv}{2} (coa(aas) + 6uv Sma(aa. S))$$

$$= -aoro + 8485.3$$

$$= -aoro + 8485.3$$

For $Sy' = ?$ Put $C = 90' + aa. S'$

$$= -112.5.$$
 $Sy' = \frac{6}{2} 4uv - Suv} + \frac{4uv + 5uv}{2} (coa(112.5) + 6uv Snu (112.5))$

$$= -aoro + (-8485.28)$$

Shearing Shess:-

 $7 = \frac{Sx - 5y}{2} 3in 2a + 1ny coab.$

About $x' = \frac{3}{2} 2a + \frac{3}{2} 3in 2a + 1ny coab.$

$$= \frac{3}{2} (4uv + 3uv + 3uv$$

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For Ty'=? Put 0 = 112.5 7/= 400+800 Sin2 (112.5) - 6000 8002 (112.5) 179/20 Diagram: Sue Gust 3ri Gr. Man stresses en aplane: Since; 0=30' so using en. bn= 5n+8y + 8n-8y cos20 - Tryson LO = 400-8000 + 400+800 cm2 (30) +6000m2 (36) Sn'= 6196 psi Put 0 = 90 + 30 = 120. Sy = 8 m + Sy cose (120) - Try sin (120) 2 400-8000 + 40008800 8052(120') + 6000800(120) 18/ = -10196.15 Ps.

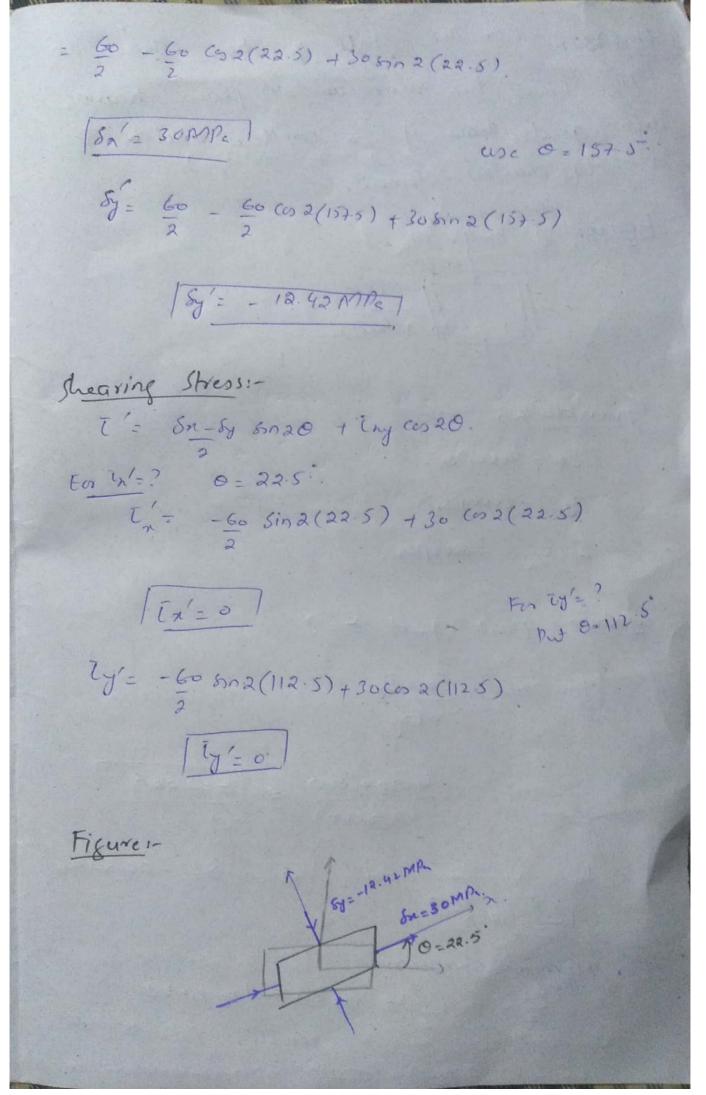
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Shearing stress: Along n'-axis (0=30) [2'= 82- by sin2(0) + Trycos20 = (4cro+sero) sn²(30) + (-6cro)(cos(2)(30) Tin = 2196 Psi ly'= 8n- by sin20 + Try cos 20 Here 0 = 120° .; 30+90. = (400+8000) sin 2 (120) - 6000 (002 (120) 1 y = -12196 psi Diagram: 8x=6196Psi ~ EJ 2 - 2496 \ 84.

Pb # 927: For the state of stress shown in Dr? ademine The principal stresses & max in plane Shearing stress. Show all results on complete Sketches of a differential element. Figure: \$ 60 MPR 1 zompa 20Mpe Save as belon Given data; Sn= 0 Sy = GOMPa TEny= - 30MPa @ stresses on principal axis 6 Max shearing stresses in plane. Sol:- @ For principal axis:tan 20 = - 2 lng Sn Sy 2 - 2 (-30) 1 8 2-22.5°

Since the Oz-ve so add +90° with 0=-22.5+90° > [0=67.5°] So normal stresses will be; 5 = 8n+8y + 8n-8y cos20 - Iny sin 10. → € Put 0 = 67.5° For 8n. Sn' = 60 + (-60) cos2 (675) + 30 sin 2 (+675) Sn= 72 4MP2 1 For Sy'= Put 0= 67.5+90'= 157.5°. (0) Sy'= 60 + (60)(co) 2 (157.5) + 30 8in 2 (157.5) 18y'= -12.42 MPa For shearing stress: Since; 7= 81-84 8120 + ing cos20. For -61 = ? ase 0= 67.5 In = -60 sina (67.5) + 30 con 2 (67.5) 12/20]

For by=? Put 0= 157.5" (y'= -60 sin2(157.5) + 30 (0,2 (15+5) | eg = 42.42 MPg] Figure: (Fer P.A). 1 50 = 72.4 MP-. 8x 42.42 6) For Max. Shearing stress in plane. By cesing q: $tan20 = \frac{Sn-Sy}{2lny} = \frac{-60}{2(-30)}$ [O= 22.5°] Now use this angle to Find Stresses Normal strisses: Si = 8n+8y + 8n-8y cos20 - lay sinzo.



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Pb#1928:- An element is subjected to the principal stresses 81 = 82 = 40 MPZ & 62 = 84 = -30 mg Compute the stresses components on plane whose mormals are at +30° & +120° with x-axis. Show a complete skelch. Figure: 187 = -30MPe try=0 Sx = 40MPa Sy = - 30MPa Solution .-As we know for normal Stresses. 8 = Sn+Sy + Sn-Sy cos20 - Inysin20. for 82' put 0= 30° Sn = 40-30 + 40+30 (52(30) [Sn= 22.5 MPc. For 8/3? Put 0=90+30= 120. 469 59': $40-30 + 40+30 \cos 2(120)$. Sy'= -125 MPa

Shearing shess: (Fer new axis) 1= 52-54 sinac + inglosoc. -0. For In Ped 6=30'. 9 = 1/2 = 40+30 Sin 2(30) + 0 [In'= 30 3 MPa] (y'= ? Put 8= 120. 90=) Ly'= 40+30 (120) Ty:= -17.6 MPC Figure: y - 17.5 MD.

Pb +1929:- For the state of pure shear shown in tis Find the stress components on plane Whose normals are at 30° & 120° with x-axis as shown. Draw a complete sketch. Figure 1-- lay 8000per Given dala As pure shear shess. 82 = 0. Sy = 0. Try= -8000psi 8 = 30°. Solution: Since we know Mat; Sn = Sn+ Sy + Sn- Sy cos 20 - Try sin 20. Putting values & 0=30. Sx'= 0 + 0 - (- seco) sin2 (30') / Sx'= 6928,2 psi 1

Sy = Sni+ Sy + Sni- Sy cordo - Try sin 20. Put 0 = 90+30 0 = 120 Sy'= 0+0- (5000) sind (120) Sy' = -6928.275 Now for shear stress: ['= 5x-84 ginzo + Try cos20_, B) for (n'=? Put 0=30 [n' = (-8000) (002(30)) [[n=-4000psi] For if=> Put 0=120. (H'= (-8000) 805 2 (120°) [] = 4000ps: / Figure:

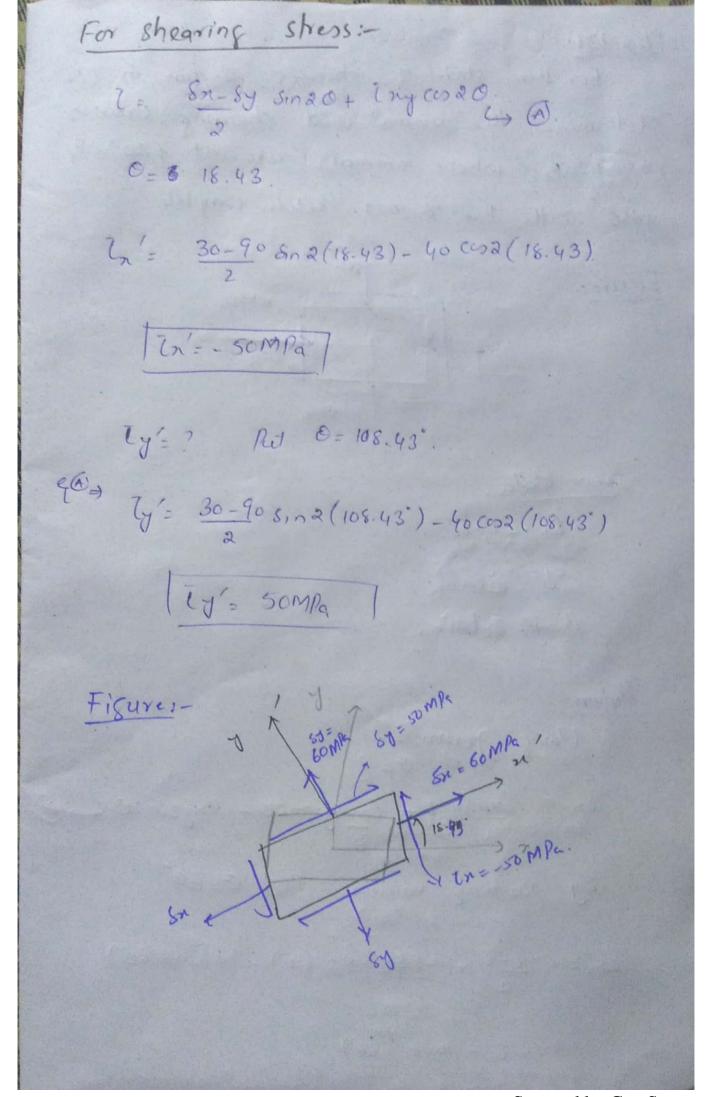
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Pb#19301- 95 a point is subjected to state & stress shown in Dig. Determine the principal stresses and the maximum in plane shearing stress. Draw sketches Complete. 190 MPa Eigure:-IN YOMPE Solution: Given data. 415 Sy = 90 MPa. Sn = 30 MPa 2 mg = - 40 MPa. For Principal axis: First gird &=? at which principal axis lies cont x-axis Since we know Lan 20 = - 2 [ny Sz1 - Sy tan 20 = -2 (-40) 30 - 90 tan 20= -4/3

E: - 26.56° so add with 90. Sina -ve value 0 = 90 - 26.56 e = 63.43° For normal stress 1-Put 8 = 30° € 120° 8'= Sn+8y + Sn-8y cos 20 - Try sm 20. $S_1 = \frac{90+30}{3} + \frac{30-90(0)2(30)}{3} + 40 \sin 2(30).$ 1 52 = 79.64 Bi MPa For 8y 5? Put 0 = 120. Sy'= 90+30 + 30-90 (G2 (120) +40 sin (120) 18y=40.4 MPa1 Note: Sh = 79 ENMPL on principal plane we Figure: have no shear stresses

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Now for Maximum In plane shearing stress: Filst Find 0=? Since De Kerow Rati tan20 = 8x-6y tan 20 - 30-90 2(-40) C = 18.43° Now use this angle and good normal Normal stresses: Sina; 5 = Sn+ Sy + Sn - Sy cos 20 - Tryson 20. By putting values & 8= 18.43. Sx = 30+90 + 30-90 cos 2(18.43) +40 Gin 2(18.43) In/= 60 MPa 8y'= > Put 0=90+18-42 = 108.42° 8y = 30+90 + 30-90 (0) 2 (108.43)+40 51 n 2 (108.43) 1871- 60 MPa. 1



16# 9312-For The state of stresses shown in 18. determine the normal and shearing stresses on plakes whose mormals are at +60° & + 150' with the x-axis. Sketch complile. Figure:-16ksi Given data; Sc = 8K8 · 8y= -8 Ksi Try = 10 Ksi. Solution: Using Fermula 1-c 8n = 8n+8y 8+ 8n-8y cos20 - Tryson20 - A By putting values. Put 0=60 Sn'= -8-8 + -8+8 (cs 2(60°) - 10 sina(60°) 2 -8 - 10 (0. 866) TSn = - 16.66 Ksi

Sy'=? Put 0= 150. Sy'= -8-8 -8+8 cos 20 - 10 sin 2 (150) 189'= 0.66 K81 For shearing stress: 1'= 5x- by sin 20 + Try con 20. -> B Put 8 = 60. $\ln t' = -8 + 8 \sin 20 - 10 \cos 2(60)$ [7n' = 5 Ksi] For 1y=? Put 0= 150. 9(8) =) ty'= -8+8 5in 20 - 10 (5)2 (150) ly'= - 5K8; Eigure: Va & Traster

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Pb # 933:- It an element is subjected to state of stress shown in tis, find the principal stresses and the max in plane Shearing stress. Also determine the stress Components on plane whose normals are at 45° & 135° with x-axis. Draw Sketch? Figure : Given data; 87 = 60MPc. Si = - 6MPa Try = 40MPa. Solution: For Principal axi: First find 0=? tan 20 = - 2 [ny 2 - 2 (40) -60-60 10 = 16.84 7 (Principal axis angle)

Now Normal stress on PA. 8'= 8n+8y + 8n-8y 80320 - lny 8m20. Put 0 = 16.84 = $\frac{-60 + 60}{2}$ + $\frac{(-60) - 60}{2}$ (05 2 (16.84) - 40 8in 2 (16.84) 18n = - 72.11MPa] For &y'=? Put 0=90+ 16.84 = 106.89 900 8g'= -60+60 -60-60 cos(2(106 84)) - 40 sin 2(106 84) T 8y = 72.11 of on P.A no shearing stresses are produce. So Ficure: Su: -72.11MPc 1 57884

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Shearing Stresses:-Since. ('= 5x-8y sin20+ Try cos20. ->0 Put 0 = 618 / cor En. In' = -60-60 6ina (61-8) +40 cona (61-8) [In = - 72.11.MPa [y=? Put 0= 151.8" 9 0=) [y'= . - 60 - 60 6in 2 (151.8) +40 cos 2 (151.8) Ey'= 72:11MPa Figure:

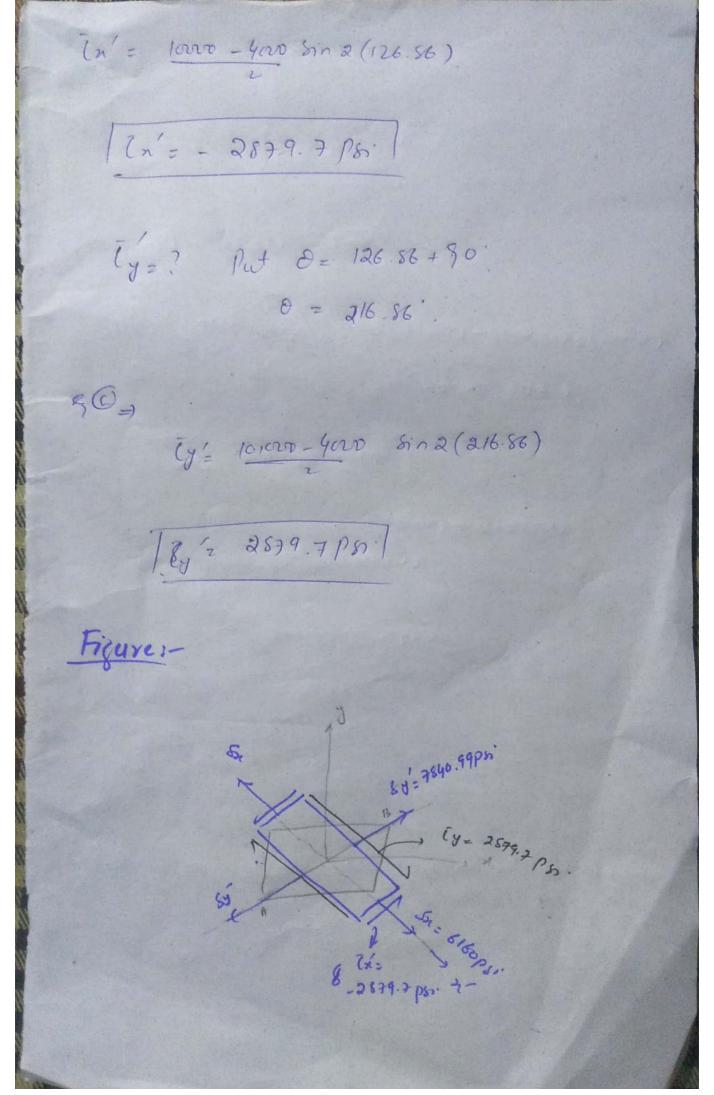
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Now 700 0= 45 & 135 Normal stresses 1coins & B & put 0=45 8n = -60+60 -60-60 cos2 (45°) - 40 sin 2 (45°) 1 Sn3 - 40 MPe For Sy'=? Put 0= 135. Sy'= -60 160 - 60-60 (5) 2 (135) - 40 8in 2 (135') 18y = 40MPe Shearing stresses: Using 9 @ Put 0 = 45°. [n = -60-60 sina (45°) + 40 cosa (45°) Ten'= -60 MPc 1 For ly'=? Put 0= 135° [y'= -60-60 sina (135') + 40 (52(135°) 124'= 60 MPa 67=40MP 21 8 = 40MP 21 Figure : BIER IND A SA

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16 # 934 - A small block is 1.6 in long , 1.2 m, high and 0.2 in thick. It is subjected to uniformly distributed load (tensile forces) having resultant values shown in 5'8. Compute the Stress components develop along diagonal AB. Figure: Solution: $S_{2} = \frac{P}{A} = \frac{2400}{1.2 \times 0.2}$ 1 8x = 10,000 PSi $Sy = \frac{P}{A} = \frac{1280}{10.6 \times 0.2}$ 1 Sy = 4000psi 1 Now the find 8=? tanco = 1.2 = 16 = 36.867 But we measured angle with years to

conck counter clock wise So add 90. 0 = 36.86 + 90 18= 126.86 7 Mormal stresses: As; 8'= 5n+8y+ 8n-8y cos20 - Engsin20 For Sx': Put @ = 126.86 δη' = 1000°+4000 + 10000-4000 (002 (126.86) |Sn' = 6160 . P8: 8y'= ? Put 0 = 126.86+90 = 216.86 8y'= 1000+4000 + 1000-4000 (2(216:86)) - Foro + 3000 (0.2803). 1 8y'= 7840:99 Psi Shearing stressi-As shearing stress is: 1 = 8x - 8y sin 20 + (ny cos 28 - +0) For in'=? put 0= 126.86.



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Pb #1932:- For the State of Stress shown in 58. Find the maximum in plans shearing Stress and show it on complete sketch 90 a differential element. Figure: 1 souppsi Given dala; 8n = 4000,78. Sy = - 8000 psi. · & Try = - 6000ps. Solution - First Find 0 = ? tan 20 = Sn - Sy alny = 4000+8000 = 10=-22.5° 1 2(6000) As -ve so add 90 = 0 = -22.5+90. 10= 67.50 1 For Normal stresses:-8 = 8n+8y + 8n-8y con 20 - ing sin 20. By patting values for sn' pid 0:67.5°. 8/1 = (400-8000) + (4000+8000) cos 2(67-5) + 6000 sin 2 (67-6)

18x = - 2000psi 8y'= ? Put 0= 90'+69.5 = 157.5° Sy'= 4000-8000 + 4000+8000 \$ cos 2 (157.5) + 6000 / 8y'= -2000ps:7 For streaming stoess: 7'= 5n-8y sin 20 + Iny cos 20. for la's P.0 0 = 67 6 Zn' = (4000+8000) 8in 2 (67.5) + (-6000) 84 (002 (67.5) Tn'= 8485.28 psi y=? Pet 0= 90+67.5= 157.5°. 7'= (4000+8000) sin2(157.5)-6000 cos2(157.5) 12y'= -8485.28ps:) 5/2-2000psi Sx = -2000 psi F18:-

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