

# **IAGT Symposium 2011**

Gas Turbine Basics

### Donald MacDonald

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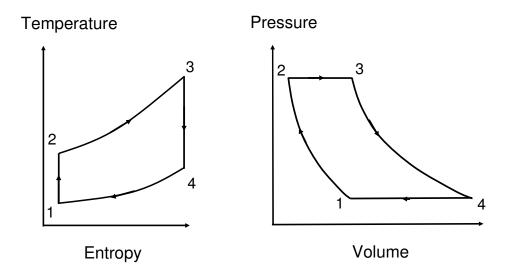
- Performance Theory
- Applications
- Engine Examples
- Performance Specifications
- Water Injection

# **Performance Theory**

## Simple Cycle Gas Turbine

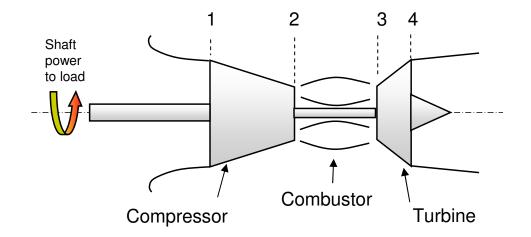
#### •Brayton Cycle:

Compression
Constant Pressure Combustion
Expansion



#### •Characteristics:

High power-to-weight ratio Simple Reliable Low vibration Low oil consumption



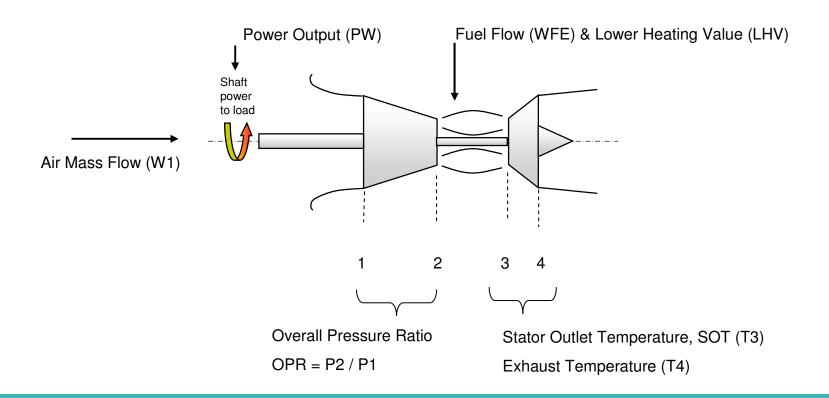
## **Performance Parameters**

#### **Thermal Efficiency**

- = power output / rate of energy input
- = PW / (WFE \* LHV)

#### **Specific Power**

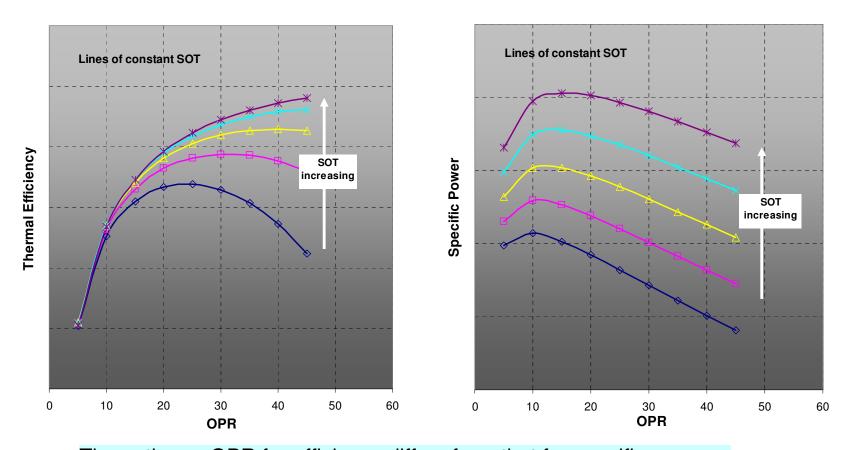
- = power output / air mass flow
- = PW/W1



# **Performance Parameters (2)**

- High Thermal Efficiency
  - Reduces fuel consumption
  - Reduces emissions of carbon dioxide
- High Specific Power
  - Requires a smaller engine
- High Exhaust Temperature
  - Important for combined cycle / combined heat & power
- OPR & SOT
  - Engine performance depends on these

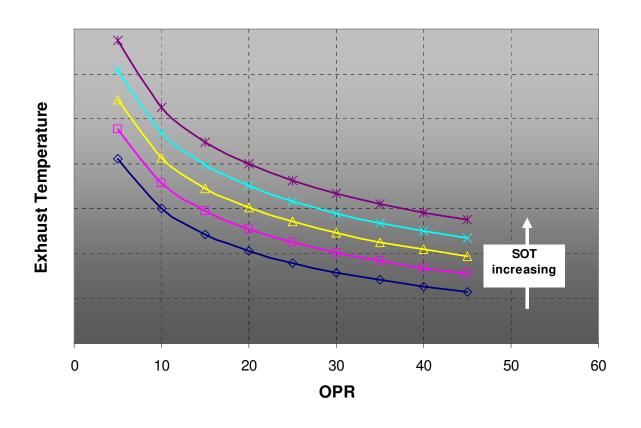
# Effect of OPR and SOT on Thermal Efficiency and Specific Power



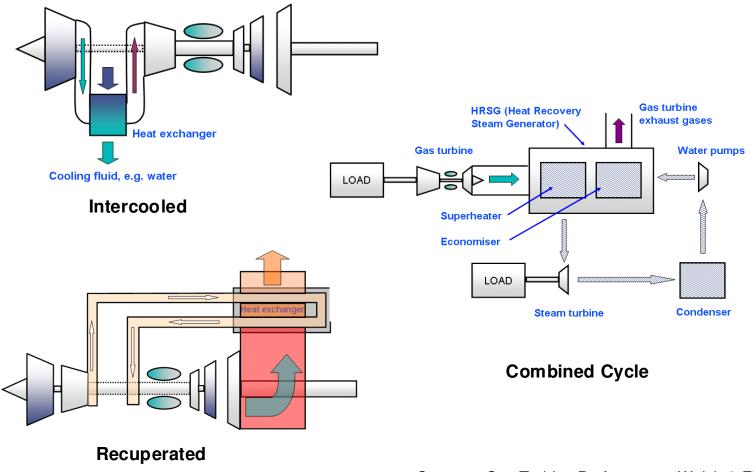
The optimum OPR for efficiency differs from that for specific power - choice of cycle depends on relative importance of each



### **Effect of OPR and SOT on Exhaust Temperature**



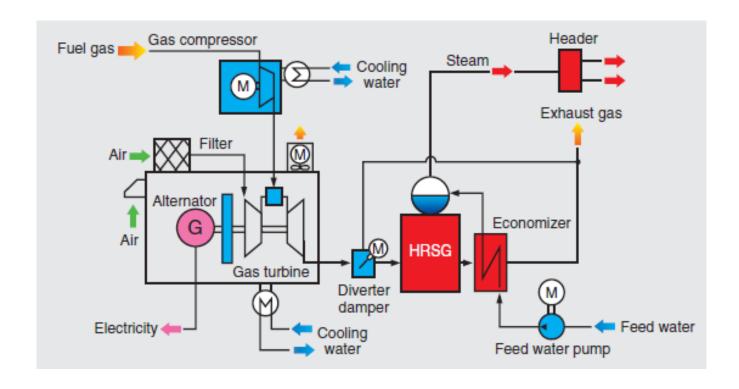
# **Enhanced Cycles**



Source - Gas Turbine Performance, Walsh & Fletcher



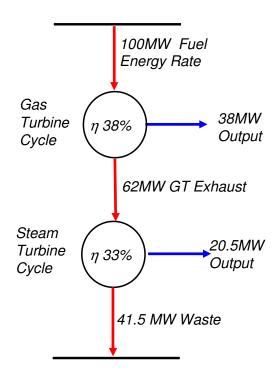
#### **Combined Heat & Power**



Source - Kawasaki, GPB series brochure, Kawasakigasturbines.com



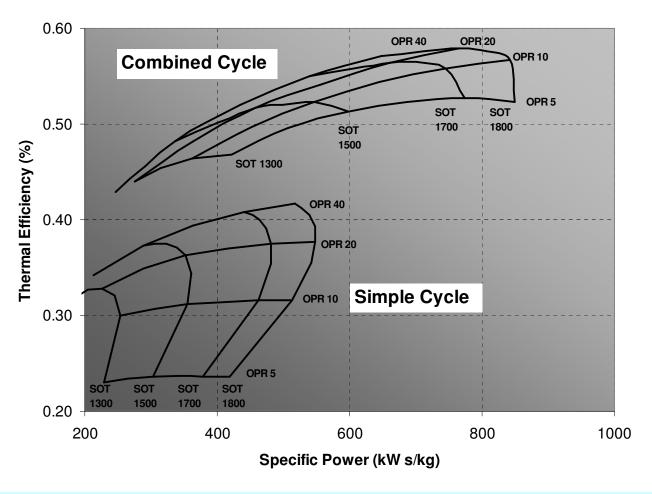
## **Combined Cycle Efficiency**



*Overall*  $\eta = 58.5\%$ 

Adapted from Encyclopaedia of Energy, Volume 6, Turbines, Gas

#### Effect of OPR & SOT on Simple and Combined Cycle Efficiency

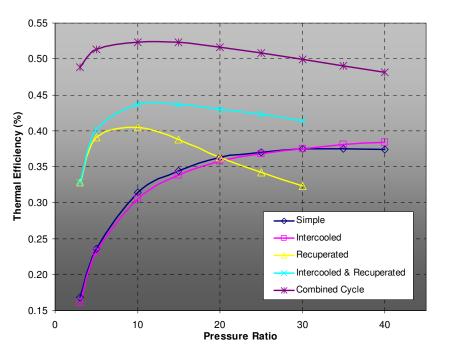


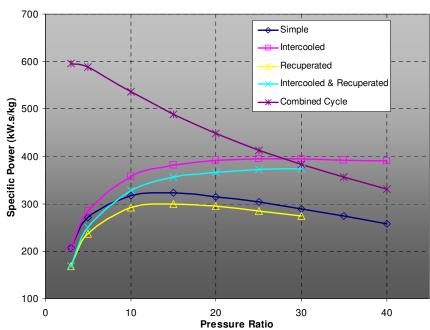
Optimum OPR for combined cycle efficiency is lower than for simple cycle

Adapted from Gas Turbine Performance, Walsh & Fletcher



# **Enhanced Cycle Performance**





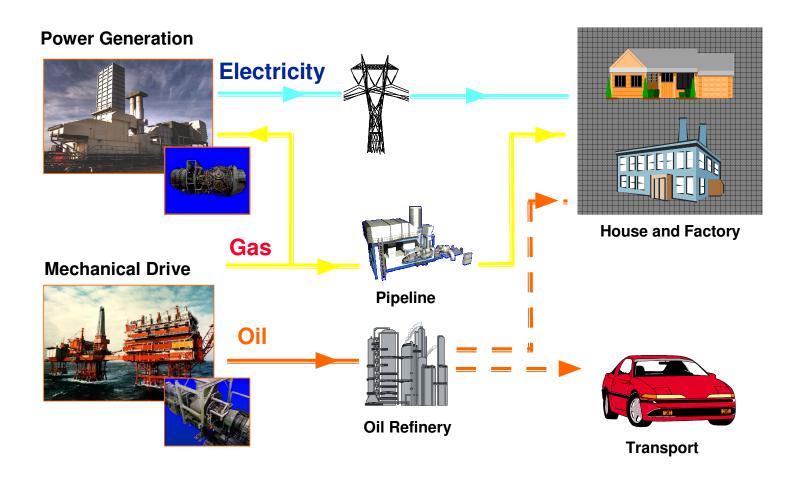
Some significant gains relative to the simple cycle but at expense of complexity and size, especially for combined cycles

Source - Gas Turbine Performance, Walsh & Fletcher



# **Applications**

### **Power Generation & Mechanical Drive**



Adapted from Jet Engines in Energy, Paul Fletcher



# **Power Generation Applications**

Plant type	IF yamnles of annifications	Power per engine (MW)
Standby generator	Emergency power supply for office block, hospital	0.25 - 1.5
small scale CHP	Power & heating for hospital, small process factory	0.5 - 10
large scale CHP	Power & heating for town of up to 25000 people or large process factory, exporting electrical power	10 - 60
Peak lopping	Supply to grid during periods of peak demand	20 - 60
Mid merit power station	Supply to grid during seasonal periods of higher demand	30 - 60
Base load power station	Continuous supply to grid	50 - 500+

Source - Gas Turbine Performance, Walsh & Fletcher



# **Ratings Definitions**

Class	Power Rating	Hours / year	Starts / year
A	Reserve peak	Up to 500	
В	Peak	Up to 2000	
В	ISO standard peak load	2000	500 average
С	Semi-base	Up to 6000	
D	Base	Up to 8760	
D	ISO standard base load	8760	25 average

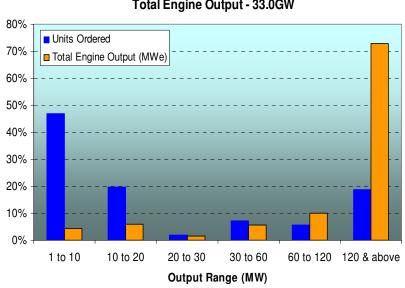
Source - Gas Turbine Performance, Walsh & Fletcher



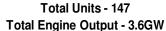
### **Gas Turbine Orders - 2010**

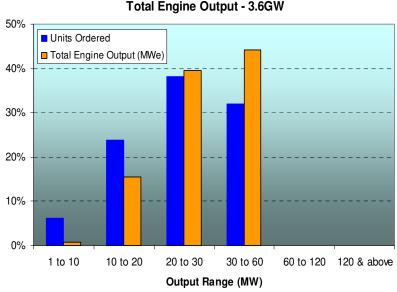
#### Power Generation Market

### Total Units - 687 Total Engine Output - 33.0GW



#### Mechanical Drive Market





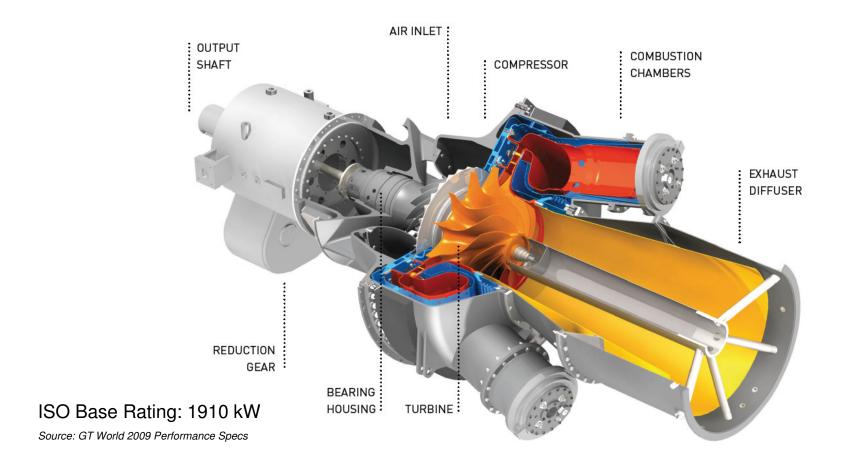
Power generation market is larger and more diverse, in terms of range of power

Source – 2010 Order Surveys, Diesel & Gas Turbine Worldwide



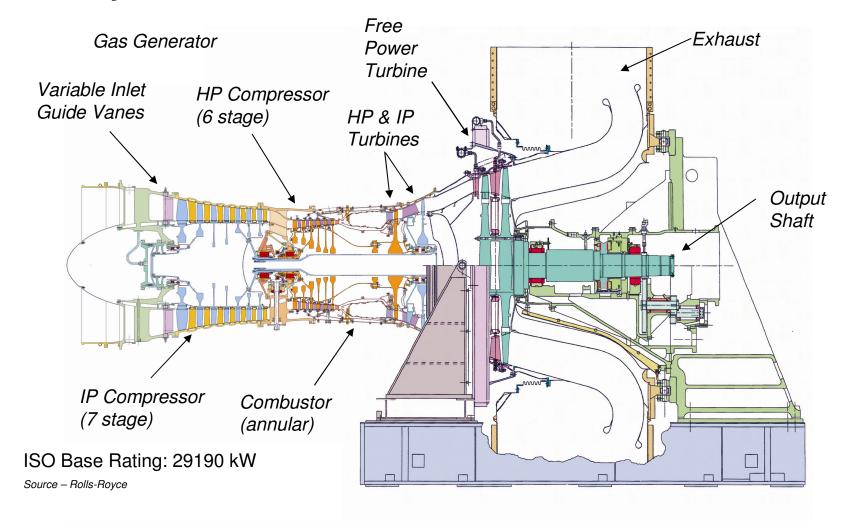
# **Engine Examples**

#### **OPRA OP16 Small Scale CHP Gas Turbine**



Compact, robust machine, low OPR (6.7) results in relatively high exhaust temperature – suitable for combined heat & power applications

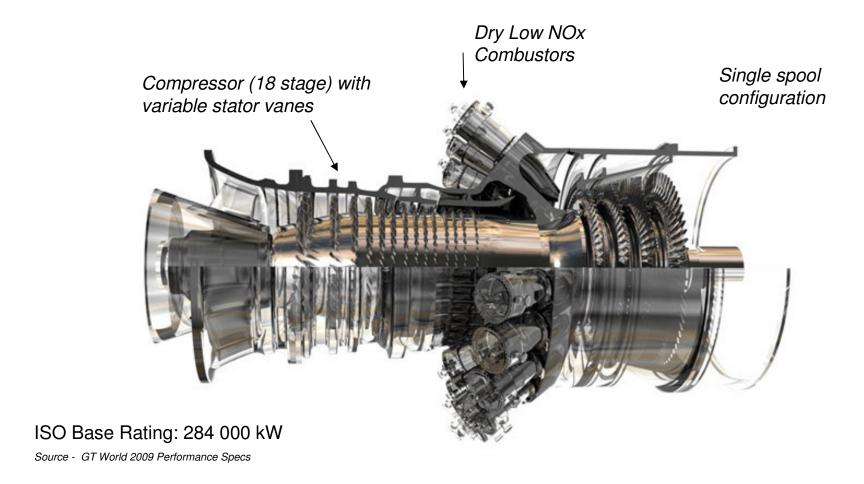
### Rolls-Royce RB211-G62 Aeroderivative Gas Turbine



Flexible operation & high efficiency in simple cycle (OPR ~21), suitable for offshore power generation, peak lopping & mechanical drive applications



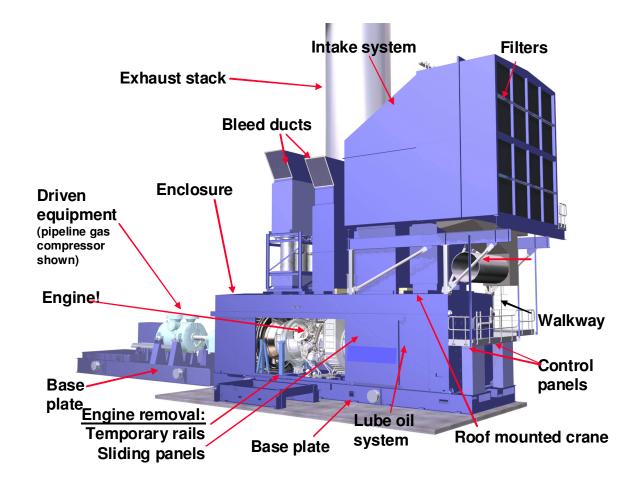
#### **General Electric 9FB Heavy Duty Gas Turbine**



Baseload power generation machine, OPR of 18 - optimised for combined cycle applications



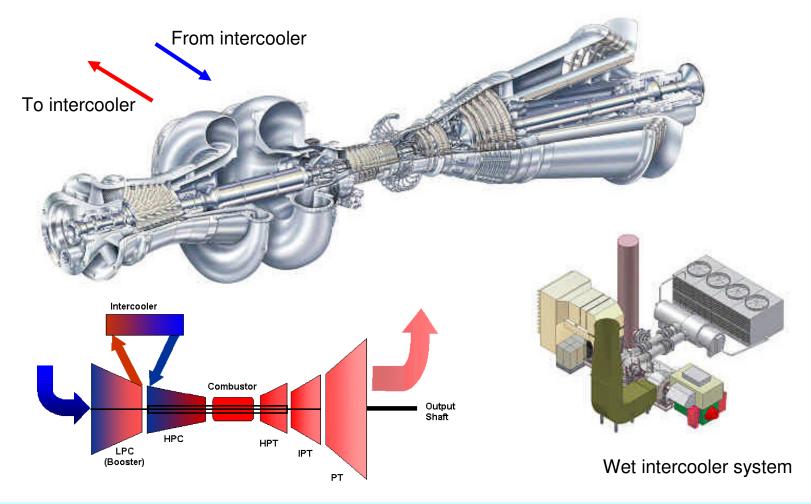
# **Typical Installation**



Inlet & exhaust losses are typically 100 / 125 mm water in simple cycle. Causes about 2-3% loss of power & about 0.5% loss of thermal efficiency.



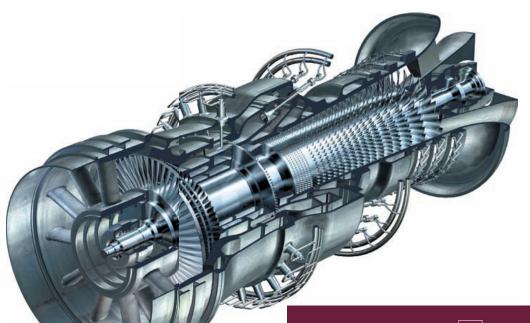
### Inter-cooled Engine, General Electric LMS100

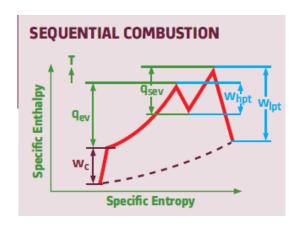


Very high levels of specific power and thermal efficiency (intercooling permits increase in OPR and SOT without increasing HPT blade metal temperature, due to reduced cooling air temp)

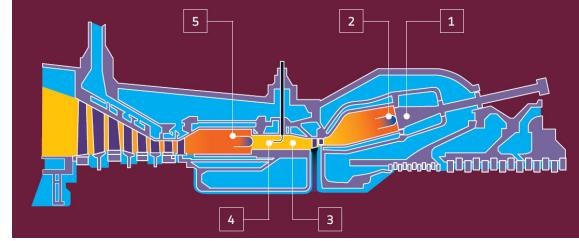


#### Reheated Engine (Sequential Combustion), Alstom GT26

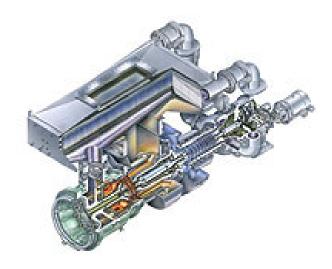




- -Higher specific power at a lower SOT than a conventional configuration.
- High exhaust temperature.
- First combustor is fuelled to maintain constant SOT for better part-load efficiency.



### Recuperated Engine, Solar Mercury 50



Waste heat from the exhaust heats compressor delivery air, reducing the amount of fuel required to achieve a given SOT.

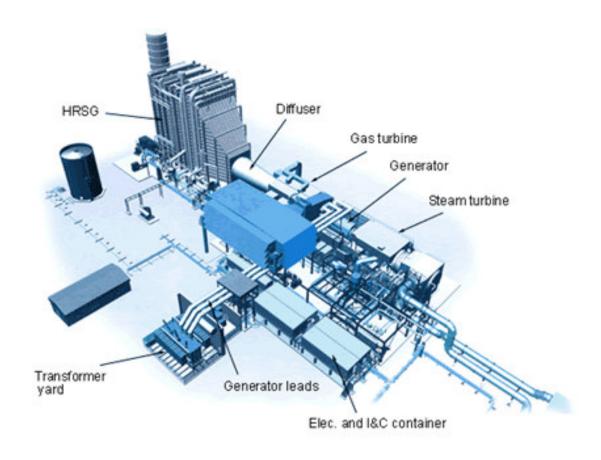
Very high efficiency is achieved at low OPR (~9).

Very good part-load efficiency

Specific power not significantly affected



# **Combined Cycle Plant**



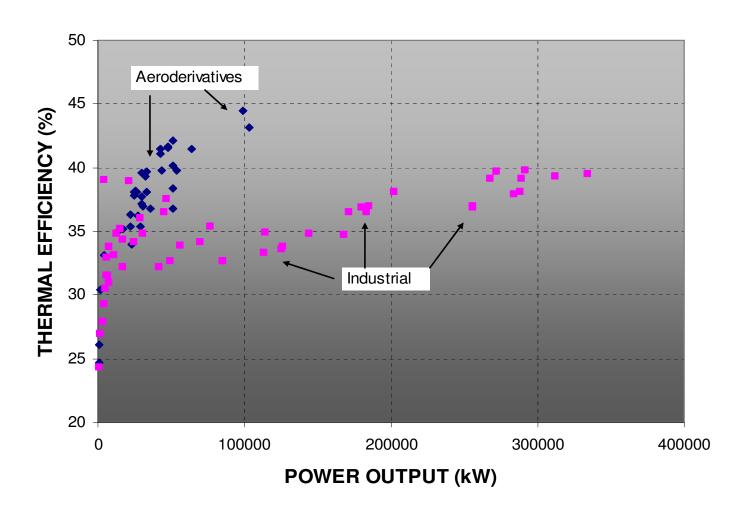
Steam cycle plant is much larger than the gas turbine!

Source - Siemens



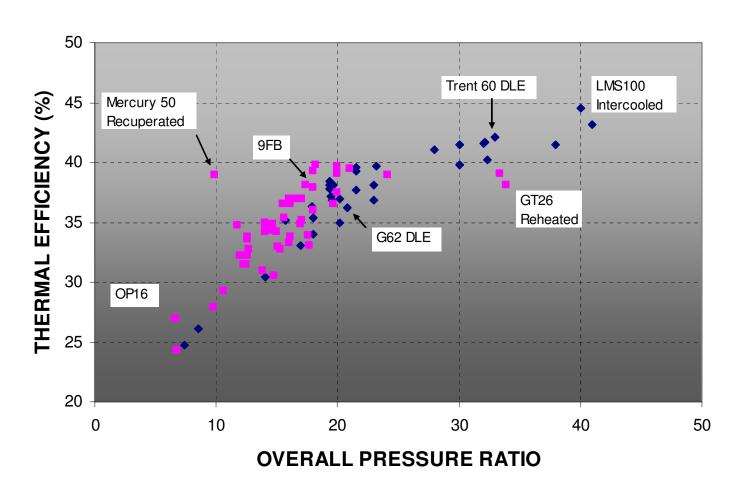
# **Performance Specifications**

## **Thermal Efficiency versus Power**



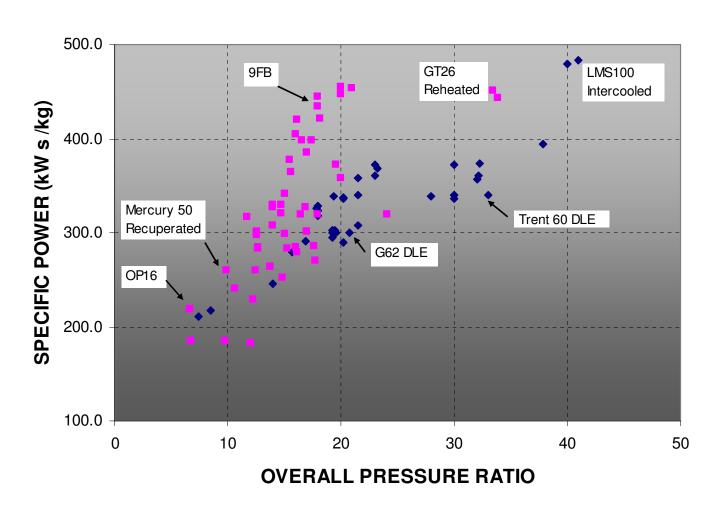


# Thermal Efficiency versus OPR



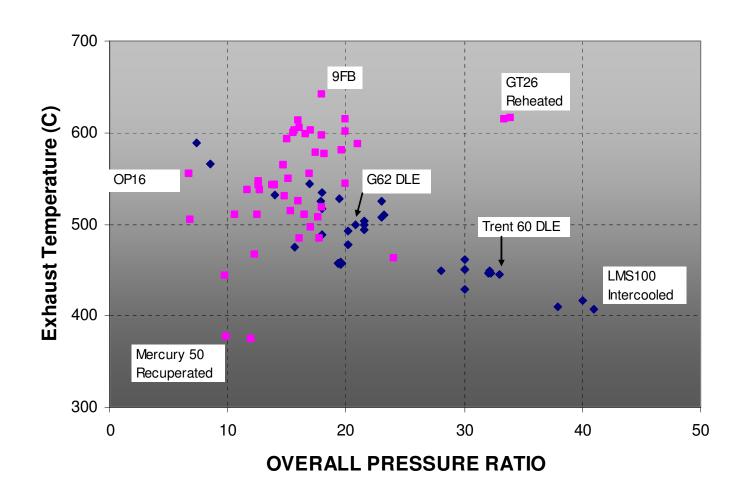


## **Specific Power versus OPR**



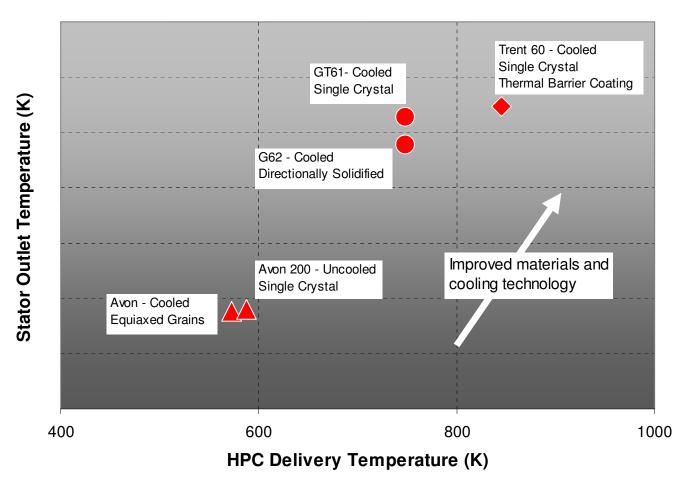


# **Exhaust Temperature versus OPR**





#### **Evolution of HP Turbine Blade Materials**



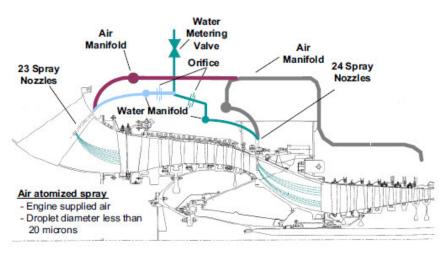
Source - Rolls-Royce Canada



# Water Injection

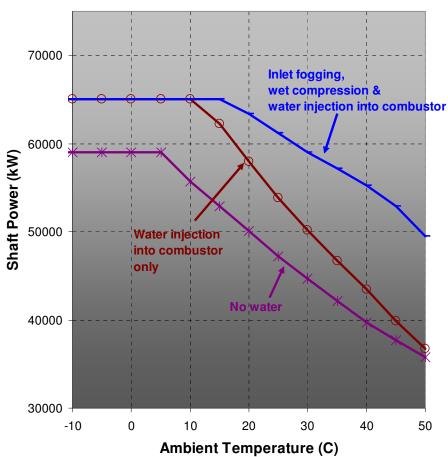
Inlet Chiller
(MEE Industries)





Example of wet compression

(General Electric)



Example of Power Increase (Rolls-Royce)

