

Case Study: RCA on Fryer Bearing Failures.

life cv

misina

Executive Summary

Repetitive bearing failures on Fryer 1 – particularly Stage A Tail End – have been a concern for some time at the Food Manufacturer. They have caused recurring, unplanned stoppages and lost production which increase life cycle costs for the asset. Bearing failures increased significantly from early 2016 to present day (2019). The principal failure mode of bearing failures is seizure – occurring always during normal production runs. A cause and effect study showed the principal physical and systemic causes to be an inter-connected combination of grease lubrication functional failure, corrosion, heat overload, water ingress, chemical and physical damage to seals, and a lack of a power wash / cleaning SOP. The solutions recommended were a move to AMI bearings, bearing caps, temporary bearing covers to be used when cleaning after a boil-out, and a new SOP with focus on cleaning mechanical, electrical and instrumentation assets and components. A further set of recommendations were included to address maintenance and asset management deficiencies found during the investigation. These included spares holding, maintenance strategies, CMMS functional locations, failure recording and Fryer cleaning, among others.

The Problem Statement

Table 1: The Problem Statement

| The Problem | Frye | r Stage A conveyor bearing(s) seized (chronic failures) |
|----------------|-----------------------|--|
| | Date | Recurring, approximate MTBF 1.1 months |
| When | Time | Cumulative bearing failures accelerated at the end of 2015 and have maintained the high failure rate ever since. Increasing again in January 2018. |
| | Unique | Anytime during normal production. |
| | Facility | Food Manufacturer |
| Where | System | Line 1 Fryer Tail End Conveyor Stage A |
| | Component | DE & NDE Bearings |
| | Safety | N/A |
| | Environmental | N/A |
| | Revenue | N/A |
| Actual | Cost | Based on an assumed average MTTR of 1 hour, Lost production: £1300, New bearing: £250, Labour: £60, Admin/SAP/MRO: £50. |
| Impact | | Total: £1660 per failure. |
| | Frequency | Chronic. Current MTBF 1.1 months. |
| | Company Reputation | N/A |
| Potential | Safety | Failed bearings increase the risk of Fryer fires due to increased heat loading on bearings in the presence of carbon and oil deposits. |
| Impact | Cost | Bearing failures may cause secondary damage to shafts, and drive motors. |

Optimising life cycle costs toget

ner

Cause & Effect Summary

The Main Causal Paths & Themes

The cause and effect chart determined that the principal physical causes of the failure were bearing grease functional failure, corrosion of bearing surfaces, and subsequent heat overload in the bearing raceways. Refer to Appendix 1 for the full chart. The reason for functional failure was determined to be contamination of the base oil, thickener, and additives. The three driving forces that co-existed to cause the grease functional failure were the failure of the bearing seals, water contamination of the grease, and chemical contamination of the grease. The latter, however, was not thought to be a strong contributor to grease functional failure. However, chemical attack of the 'rubberised' seals in conjunction with close contact pressure washing was thought to contribute more strongly – again as physical root causes. Doubts – unconfirmed - were also raised regarding the temperature resistance of the bearings seals. From a systemic cause perspective, the cause effect chart revealed that there is no Standard Operating Procedure (SOP) available for guidance on power washing Fryer 1 - especially in terms of how to deal with mechanical, electrical and instrumentation assets and components. Evidence collection also revealed that Fryer 1 is being run at 33% above asset nameplate capacity where the Food Manufacturer line rate is 20,000 Kgs/Hour as opposed to OEM stated capacity of 15,000 Kgs/Hour.

mising life cvc

Recommended Solutions

The following Table 2 matches root causes to solutions with costs, comments, responsibility, and due dates.

| Cause | Solution | Costs | Comments | Responsible | Completion Date Agreed | Date Completed |
|---|---|---|--|--|------------------------------|-------------------|
| Bearing Seals Failed | AMI Bearings (with high temp. inserts) and Silicone Seals as per Fryer 2 | Approx. £240 per bearing + delivery costs from USA | Performance trial to be run on | Assetivo (quote) & Company Eng. (order & fit) | To be agreed | |
| Water Forced into Bearing Raceways | Bearing Caps (twist/screw type) – where there is | Awaiting costs from same supplier – | Not on original quote – but has been requested | Assetivo (quote) & Company | To be agreed | |

Table 2: Recommended Solutions

| During Cleaning | sufficient space to fit them. | likely minimal | from USA supplier. | Eng. (order & fit) | 1 | |
|---|---|--|---|--|-----------------|--|
| Water Forced into Bearing Raceways During Cleaning | Temporary Bearing Covers (applied when Jet Washing) & mechanical scraping of bearings | Low – Can be an in-house designed solution. | This solution would require an engineering solution and collaboration between Engineering and Operations | Company Operations & Engineering | To be agreed | |
| No SOP Available for Cleaning with Power Wash Lances | SOP – specific guidance on cleaning electrical, mechanical and instrumentation components. New method to be included for cleaning bearings. | Low – SOP is to be developed in-house. | Further investigation and agreement is needed on how to clean bearings of scale and grease. All Fryer operators require training on the SOP. | Company Operations | To be agreed | |

mising life cvc

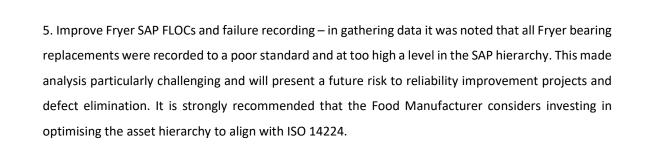
Additional Recommendations

1. Increased bearing failures on Fryer 1 since late 2015/2016 to present day - Internal company investigation required to determine trigger point for the dramatic, step change – see Appendix 5. This is deemed a high priority by Assetivo.

2. Fryer 1 running at 33% above nameplate capacity – it is recommended that an FMEA-driven maintenance strategy is developed for this asset. A strong focus on reliability-centred maintenance principles is needed due to the increased risks to asset performance and life cycle costs. This strategy can be developed via Assetivo or in-house. If the latter is preferred, then it is recommended to engage with Assetivo during the strategy development.

3. Spares holding for Fryer 2 AMI Bearings – it has been noted that the Food Manufacturer do not hold all spares for all Fryer 2 bearings and have no policy in place for re-ordering (i.e. no supplier in place). This is presently a business and operations risk. Please investigate.

4. Remove time-based replacements on Fryer 1 Tail End 4 bolt flange bearings - see comments in Appendix 2.



mising life cvc

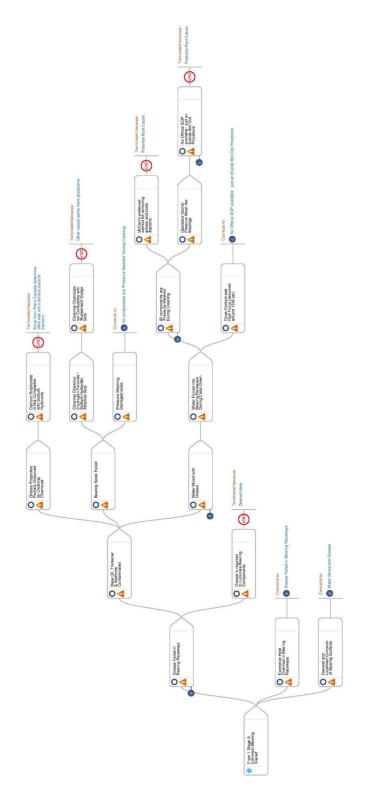
6. Mechanical scrapers – the OEM manual for Fryer 1 mentions scrapers can be used for cleaning the fryer. Investigate if these are a feasible alternative as part of the new SOP to be developed.

Appendices

Supporting documentation on the following pages.



Appendix 1: RCA Cause & Effect Chart





Appendix 2: Selection of Bearing Failure Photographs (corrosion & contamination)



It must be noted that the investigation and analysis indicated that the failed bearings <u>are not</u> genuine wear out failures. They are not reaching their natural wear out phase or anywhere near their L_{10} life (the statistical time at which 10% of these bearings would have failed under the same operating conditions). The bearings appear to be a mixture of infant mortality and random failures. <u>This means</u> <u>that time-based replacements will have no effect on reliability and is not a valid strategy in this</u> <u>instance</u>.

Appendix 3 Bearing Failure Analysis

3. Bearing Investigation

| Bearing type: | Insert Bearing |
|-------------------------------|--|
| Bearing designation: | DODGE SC-55-MM 211 |
| Housing designation: | DODGE F4BC |
| Bearing reference name: | BRG A |
| Manufactured: | USA |
| Visual investigation summary: | The bearing suffered mostly from moisture contamination leading to corrosion found on the housing and bearing components. Photos below are a visual summary of the findings. |
| Disposition | To be Scrapped 21-05-2019 |

Optimising life cycle costs/together





IS SICF is a neglatered trademark of the SICF Group. © SKF Group 2018 Page 5

4. Application assessment

On first reception of the bearing, it was unable to rotate due to the level of corrosion on the internal components. From the inspection, it is clear that the bearing was subject to high levels of moisture contamination, evident in most of the images in section 3 of this report. Figures 7 and 8 highlight the polished rolling contact path that is slightly biased to one side of the outer ring, this indicates that the bearing is subject to an axial load component. The "orange-brown" discolouration found on the inner ring, outer ring and rolling elements is indicative of a high temperature condition.

ptimising life cycle costs/t

All the lubricant found in the bearing was hard, solid and brittle, a combination of moisture contamination and high temperature being the most likely cause. Lubricant in this condition is unable to function correctly. Lubrication is a critical component in the proper functioning of a bearing. If there is insufficient lubrication or if the lubricant has lost its lubricating properties, an oil film cannot form between the bearing internal surfaces and metal-to-metal contact occurs leading to accelerated bearing damage.

Additional Assetivo comments: The grease thickener acts like a sponge, it holds the oil in suspension, as the bearings rotates the thickener is compressed, it releases the oil which is what creates the lubricant film to support the load between the rotating and stationary elements, along with coating the surfaces to prevent corrosion. The hard deposits are what is left of the thickener with no oil present. If the seals become damaged, the oil will not remain within the bearing, so will not be recaptured by the thickener, this will then lead to lubrication failure.

5. Conclusion & Recommendations

5.1. Damage according to ISO 15243:2004

| Bearing A | | | |
|---|--------------------|----------------|--|
| Failure Mode/most distinctive damage | Moisture corrosion | resulting from | Presence of water Standstill corrosion/etching causing: Pitting Poor lubrication conditions |

- Prevent corrosion by improving protection of bearing from moisture contamination
- Ensure the bearing solution can cope with temperatures around 190°C
- Ensure correct lubrication type, quantity and relubrication intervals are employed

Appendix 4: Two-Stage Fryer Design Capacity

C

| Capacity: 15,0 (33, | 000 Kg/Hr ,000 #/Hr) | | |
|----------------------------|---|------------------------------|----------------------------|
| | | | |
| | Stage On | e | |
| Cook Length: | 5.5 m (18') | Flow: 817 | 7.2 Cu Mtr/Hr (3600 Gpm) |
| Design Oil Dep | pth (above belt): 90 mm (3.5") | No. of Zo | mes: 2 |
| Belt: | Marcon EQ-72-18-16 Carbon S ¹ / ₂ " brite steel rods with carbon | teel drags on 3" ce | enters |
| Pump: | Cornell 8H-FISK 125 HP- 415V / 50Hz / 3Ph 13.5" Impeller 3600 Gpm @ 100 Ft Dth | | |
| Heater: 19.3 Bar (280 p | Harris Thermal 12 Mlbtu/Hr St osig) at the vessel 199°C (390°F) out put with an .31m/s (12 Ft/s) tube speed | | |
| Filtration: | Chip Chain Filter- Belt mesh E Lakos CSX-2300-HL counter f Sweco Separator- XS48S666 f Paper filter- 48" wide paper (P | flow @ 6 o'cl for hot oil | lock inlet |
| | Stag | <u>e Two</u> | |
| Cook Length: | 4m (13 Ft) | Flow: | 817.2 Cu Mtr/hr (3300 Gpm) |
| Design oil depth | a (above belt): 90mm (3.5") | No. of | Zones: 3 |
| Belt: | Marcon EQ-72-18-16 Carbon 1/2 bright steel rods | Steel | |
| Pump: | Cornell 8H-FISK 125 HP- 415V / 50HZ / 3PH 13.5" Impeller 3300 GPM @ 110 FT DTH | | |

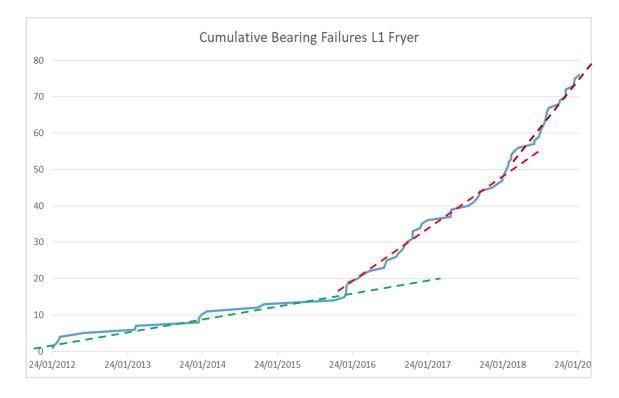
Optimising life cycle costs together

Appendix 5: Bearing Failure Trends

timising

life cvc

Analysis of historical work orders on Fryer 1, clearly show a significant, step-change increase in failures around late 2015/early 2016. The rate of failure was maintained from that date. It was then seen to increase once again around January 2018. Assetivo has been unable to retrieve any robust data/information from the Food Manufacturer regarding potential reasons for this significant change (i.e. product changes, process changes, cleaning changes, projects, upgrades, etc.). In terms of asset maintenance, the Dodge bearings used have been used since initial installation in 2000, and the lubricant type was changed in early 2013 approximately – two years prior to the step change. As a point of note, historically, Dodge bearings have arrived on site with no lubricating grease. It is believed that many have been installed on Fryer 1 with no grease and therefore have failed rapidly. There is now a process in place to stop this occurring. <u>However, the larger, unresolved step change issue seen below is considered critical by Assetivo and needs further investigation by the Food Manufacturer as soon as possible.</u>



Optimising life cycle costs/together

Appendix 6: Current Maintenance Strategy for Fryer 1 Tail End Bearings

| Maintenance Task | Frequency | Other | Comments |
|---------------------|------------------|---|--|
| Lubrication | 7 days / 2 grams | Rolling Element Bearings | |
| РМ | 18 weeks | | Replace both Dodge 4 hole flange bearings on stage A at fryer in-feed end (non drive end - drive side/non drive side) (((TIME - BASED))) |
| VI | 7 days | 'Bearing Inspection'. No issues noted from 19 th Dec.2018 to 11 th April 2019. | Visually inspect all 8 DS & NDS bearings (not including the outfeed bearings) for evidence of dark, cakey or milky grease around the bearings. Look for any signs of corrosion or other deterioration of the surface, seals or obvious physical dimensional characteristics. |

Appendix 7: OEM Lubrication Recommendation

| Hours | Suggested Lubrication Period in Weeks | | | | | | | |
|----------------|---------------------------------------|-------------------|-------------------|-----------------------|------------------------|------------------------|------------------------|------------------------|
| Run Per Day | 1 to 250 RPM | 251 to 500 RPM | 501 to 750 RPM | 751 to 1000 RPM | 1001 to 1500 RPM | 1501 to 2000 RPM | 2001 to 2500 RPM | 2501 to 3000 RPM |
| 8 | 12 | 12 | 10 | 7 | 5 | 4 | 3 | 2 |
| 16 | 12 | 7 | 5 | 4 | 2 | 2 | 2 | 1 |
| 24 | 10 | 5 | 3 | 2 | 1 | 1 | 1 | 1 |

Appendix 8: RCA Solutions Chart

Optimising life cycl

e cos

S

ode

el

| Cause | Solution | Effectiveness | Ease of Implementation | Return on Investment | Potential Negative Impacts | Totals |
|---|--|---------------|---------------------------|-------------------------|----------------------------------|--------|
| Bearing Seals Failed | AMI Bearings (high temp. inserts) with Silicone Seals as per Fryer 2 | 7 | 7 | 7 | 10 | 31 |
| Bearing Seals Failed | <u>OR</u> SKF Bespoke Bearings (new, proprietary sleeves and seals) | 7 | 7 | 3 | 10 | 27 |
| Water Forced into Bearing Raceways During Cleaning | Bearing Caps (twist/screw type) | 3 | 7 | 7 | 10 | 27 |
| Water Forced into Bearing Raceways During Cleaning | Temporary Bearing Covers (applied when Jet Washing) & mechanical scraping of bearings | 7 | 3 | 3 | 7 | 20 |
| No SOP Available for Cleaning with Power Wash Lances | SOP – specific guidance on cleaning electrical, mechanical and instrumentation components. New method to be included for cleaning bearings. | 3 | 7 | 10 | 7 | 27 |

Scoring Range 1 – 10, using values of 1, 3, 7, 10 only.